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Overview

Timeline

- Project start: 2/01/2016
- Project end: 1/31/2019
- Percent complete: 35%

Budget

- Total project funding: \$715,933
 - DOE share: \$643,839
 - Contractor share: \$72,094
- Budget Period 1: \$299,417
- Budget Period 2: \$258,664

Barriers

- Increase the fuel efficiency of passenger cars
- Lack of modeling capability to accurately simulate engine knock
- Inadequate fundamental understanding of turbulencechemistry interactions during engine knock

Partners

- OSU (Lead)
- ORNL
- Convergent Science



Relevance

Project Objective: Improving predictive capability for engine knock by developing a new, physics-based large eddy simulation (LES) combustion model

- Reproducing cycle-to-cycle variations
- Accurately predicting mean heat release/reaction rates during end gas ignition
- Using detailed (reduced) chemical mechanism (~100 species) in engine knock LES

Objectives in the Period of Feb. 2016-Mar. 2017

- Developing a base code
- Generating direct numerical simulation (DNS) data base for model validation
- Acquiring in-cylinder pressure data from laboratory engine experiments under knocking and knock-free conditions



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Approach: Model

CMC-FPF: LES combustion model for knock

- CMC for end-gas ignition
 - Conditional Moment Closure: Klimenko and Bilger (PECS, 1999)
 - Accurate reaction rate prediction
 - Volume Avg. vs. Surface Avg.
 - Surface averaging in CMC preserves small-scale scalar structures and thus can lead to accurate reaction rate estimation

• FPF for SI premixed flame

- Front Propagation Formulation: Kim (JCP, 2015)
- Cycle-to-cycle variations in premixed burning rates
- Reproducing the level set method in the flamelet regime
- Applicable to broader premixed combustion regimes



Approach: Model

End-gas ignition CMC can accurately estimate reaction rates in a computationally efficient way

- Small-scale scalar structures resolved in enthalpy space
 - Total enthalpy or sensible enthalpy
- Thus, the number of grid points in ignition CMC can be much smaller than the number of typical LES grid points
 - To be computationally more efficient than conventional LES where volumeaveraged (filtered) species mass fractions and temperature are used to estimate reaction rates



 $\overline{\omega_T |\zeta}$

Model development supplemented by direct numerical simulation (DNS) and engine experiments **Milestones**

- Developing a CMC solver for end-gas ignition (Y1Q2)
- Developing a base CMC-FPF solver (Y1Q4)
- Experimental campaign I (Y1Q4)
- DNS of end-gas knock (On-going)
 - Detailed chemistry ignition 2-D DNS: Reaction rate estimation
 - Two-step-chemistry ignition 3-D DNS: Scalar dissipation
 - Premixed flame DNS: Premixed flame modeling
- Validation of CMC-FPF with DNS data (On-going)
- Base engine simulations knock-free (Started)
 - Premixed model to be implemented as UDF of Converge CFD



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Developed base knock CMC code

- Transient, 1-D transport in enthalpy space, and reaction
- Time integration
 - Stiff ODE solvers
 - CVODE, DVODE (Lawrence Livermore National Laboratory)
 - Plan to include other solvers
- Transport in enthalpy space
 - Second-order finite difference
- Chemistry
 - CHEMKIN compatible
 - Use of CHEMKIN II (Sandia National Laboratory)
- Adaptive mesh refinement



End-gas ignition in engine-like pressure evolution (Stoichiometric iso-octane/air mixture; 116 species reduced mechanism for PRF, Luong et al. CnF 2013; imposed thermal stratification; three spark timings)



Model Development

Developed initial interface module to couple CMC-FPF solver with an LES solver enthalpy density

- CMC-FPF combining
 - Knock CMC
 - Base solver described before
 - Included transport in physical space: operator splitting, highorder TVB finite difference
 - Premixed Flame FPF
 - Fifth-order WENO scheme (Jiang and Shu JCP 1996) for regularized Delta function
- Coupling with an LES solver
 - Initial interface module developed
 - Coupled with low Ma flow solver, NGA (Desjardins et al. JCP 2008)



Premixed flame propagation and end-gas ignition in constant volume configuration

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Experiments

Single-cylinder engine laboratory used to identify knocking conditions by statistical analysis

- 2007 GM LNF 2.0L Ecotec Single cylinder
 - 9.2:1 compression (stock)
- Conditions
 - Fuel: Iso-octane
 - 2000 r/min
 - CA50 from knock to 50 aTDC_f
 - Intake temp from 35°C to 135°C
 - Constant airflow (4 levels) all positive gauge pressure
 - 0 and 15% EGR
- Knocking cycles identified as 1 bar KI





Experiments

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Experimental conditions "slice" through ignition delay regions for each boost condition using intake temp.

- Experimental conditions varied
 - Intake temperature, boost, and EGR rate (0% and 15%)



DNS: Detailed Chemistry

Performed detailed-chemistry 2-D DNS to generate database for model validation

- Low Ma, second-order kinetic energy conserving scheme (NGA)
 - Combined with CHEMKIN and TRANSPORT package
- 116 species PRF reduced mechanism
- Isotropic turbulence with thermal stratification
- Conditions
 - PRF80/air, 20-30bar, 800-970K, 0.8-1 equivalence ratios
 - $u' \sim 1.2$ m/s, $I_t \sim 1.2$ mm, $T_{rms} \sim 40$ K
- Focusing on reaction rate estimation during ignition



Temperature and H_2O_2 mass fraction fields during end-gas ignition (T_0 =850K, P_0 =30bar, stoich. PRF80/air, 5mm², 1024² grid points). Refined grid DNS (3072²) in progress.

Model Validation

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Assessed accuracy of reaction rate estimation

- Sensible-enthalpy or total-enthalpy can be used as a conditioning variable (for surface averaging)
- Sensible-enthalpy CMC accurately estimates heat release rates, while total-enthalpy CMC leads to substantial errors
- More investigation for a wider range of conditions is necessary



Model Validation

Assessed accuracy of reaction rate estimation

 Conventional LES where volume-averaged (filtered) species mass fractions and temperature are used to estimate reaction rates leads to substantial errors



THE OHIO STATE UNIVERSITY DNS: Mixing Statistics

Two-step chemistry 3-D DNS to extract scalar dissipation statistics under way

- Two-step chemistry
 - Iso-octane oxidation step/CO-CO₂ conversion (Misdariis *et al.* PCI 2015)
- Conditions
 - Stoich. Iso-octane/air
 - P₀: 30bar, T₀: 970K, initial T_{rms}~50K
 - Isotropic, forced turbulence/thermal-energy
 - Taylor-scale Reynolds number~96
 - Turbulence intensity~3.2m/s
 - 3.2mm³, 512³ grid points (planning a case with 1536³ grid points)
- Extracting statistics of scalar dissipation rates and FDF for total enthalpy and sensible enthalpy
 - Models needed for CMC



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DNS: Premixed Flame THE OHIO STATE UNIVERSITY

Performed single-step chemistry 3-D DNS to validate and refine premixed flame modeling

- DNS with varying Reynolds numbers in the thin reaction zones regime
- Extracting flame wrinkling characteristics and validating subfilter flame speed models used in FPF



DNS Conditions

Taylor-scale Reynolds number: 52, 68, 100 (one more planned: ~150) 22M-420M grid points Karlovitz number~4 $u'/s_1 \sim 5 - 7.7$

Reaction progress variable



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Responses to Previous Year Reviewers' Comments

• This is the first annual merit review of the project



- Ohio State University
 - Model development and validation
- Oak Ridge National Laboratory
 - Experiments and assist in engine simulations
- Convergent Science
 - Converge CFD
- Oak Ridge Leadership Computing Facility
 - DD allocation
- Ohio Supercomputer Center



THE OHIO STATE UNIVERSITY Remaining Challenges/Barriers

- Development of a submodel for scalar dissipation rates of sensible enthalpy (reacting scalar)
 - Many studies for conserved scalar (total enthalpy), few for reactive scalars (sensible enthalpy) in the context of LES
- Coupling with commercial CFD software
 - Coupling with Converge CFD (implemented as UDF)

THE OHIO STATE UNIVERSITY Proposed Future Research

- Validating/developing reaction rate estimation and scalar dissipation models using end-gas ignition DNS data (Year 2)
 - Complete larger-scale DNS cases
 - Further assess total-enthalpy-based and sensible-enthalpybased approaches
 - Develop/validate scalar dissipation models
- Engine simulations for knock-free conditions and validating premixed flame modeling (Year 2) – Basis of Year 3 efforts for knocking engine simulations
- Engine experiments (Year 2)
 - Complete acquiring in-cylinder pressure and gas sampling data for a range of knocking and knock-free operations

Any proposed future work is subject to change based on funding levels.

Geometry model of engine from scanned hardware



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- CMC-FPF consistently combines CMC for end-gas ignition with FPF for SI premixed flame propagation
 - LES-based knock prediction employing detailed chemistry
- Base code and validation data made available
 - Developed base CMC-FPF code
 - Generated DNS and experimental data for validation
- In Year 2, continuing model validation
 - Continuing generation of DNS and experimental data
 - Submodel development/validation using DNS data
 - Knock-free engine simulations with premixed flame FPF model - serving as a basis of Year 3 efforts for knocking engine simulations and model validation



Technical Backup Slides



Approach: Model

Knock CMC



Conditional moment

$$\overline{Y_i}|\zeta^* = \frac{\int_V \rho Y_i \delta(h-\zeta) H(c_p^* - c_p) F(\mathbf{x} - \mathbf{x}') d\mathbf{x}'}{\int_V \rho \delta(h-\eta) H(c_p^* - c_p) F(\mathbf{x} - \mathbf{x}') d\mathbf{x}'}$$

samples taken for mixture with a particular value of enthalpy (h) in the end-gas region

Enthalpy~temperature~reaction rate

Mapping of scalar structures in physical space (x) to enthalpy space enabling accurate estimation of reaction rates



Approach: Model

FPF for SI Premixed Flame

Solving the conventional transport equation for the reaction progress variable (end-gas indicator)

$$\frac{\partial \overline{\rho} c_p}{\partial t} + \nabla \cdot (\overline{\rho \mathbf{v} c_p}) = \nabla \cdot (\overline{\rho D \nabla c_p})$$

With reaction rates designed to reproduce a specified propagation speed of the flame regardless of resolution levels



Premixed flames are typically underresolved in LES

$$\overline{\rho}\widetilde{\omega_{c,p}} = \rho_u s_t \delta_\Delta(\mathbf{n} \cdot (\mathbf{x} - \mathbf{x}_s))$$

Sub-filter flame speed Regularized delta function

Kim, J. Comput. Phys. (2015)



Model Development

Premixed flame FPF validation under way



Dynamic fractal model



LES of a premixed jet flame (F3 Flame of Chen et al. CnF 1996)

- Needing models for
 - Sub-filter flame speed
 - Regularized delta function
- Assessment of sub-filter flame speed model in progress



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Experiments

- Coupled experimental measurements of incylinder thermodynamics with simulated constant volume ignition delay
- 2-zone model for experimental data
- Indicate measured bulkgas state at knock
- Determine cycle-by-cycle ignition exposure at knock *ignition exposure* (-)

$$= \left[\frac{1}{encoder \ resolution} \left(\frac{\theta}{CA}\right) * RPM * \frac{360 \ (CA)}{60000 \ (ms)}\right] \\ * \int_{IVC}^{knock} \frac{1}{const. \ vol \ ID \ (ms)} d\theta$$



Experiments

Effects of Low-Temperature bulk as reactions on knock observed in experiments

- Increased low temperature reactions (PSHR) reduced time to knock
- EGR independent
- Reduced variability with PSHR
 - Thermal or chemical effect? ¹/₂ s
 - will be studied in simulation effort

