

Low-Temperature Combustion for High-Efficiency, Ultra-Low Emission Engines

University Consortium

University of Michigan
Massachusetts Institute of Technology
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University of California, Berkeley

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Big Picture

- HCCI university consortium concentrated on
 - developing workable control systems
 - obtaining experimental data
 - developing analytical tools to optimize and assess implementation ideas
- LTC university consortium will focus on
 - extending the practical operating range of LTC engines at both low and high load
 - improve system fuel economy benefits
 - include PPCI engines and alternate fuels



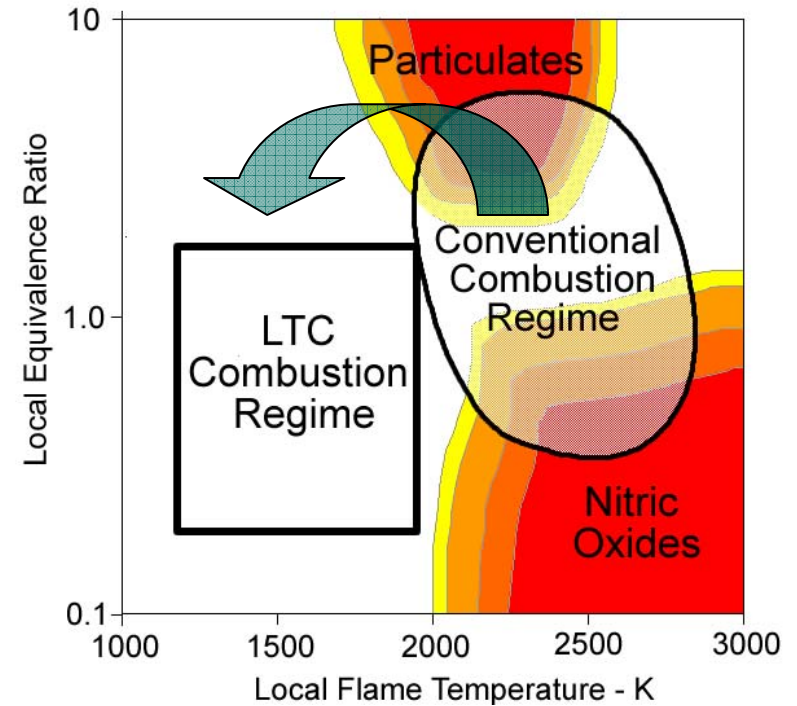
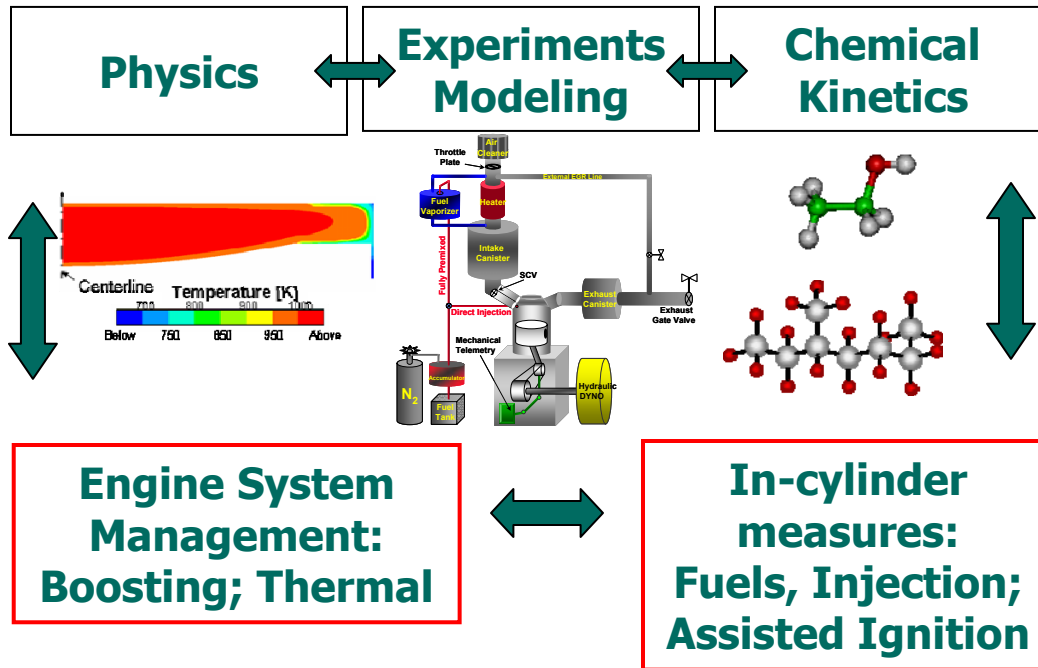
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Low Temperature Combustion

Precisely Control Combustion
to Expand LTC Operating Range

- Maintain High Thermal Efficiency
- Avoid pollution forming regimes

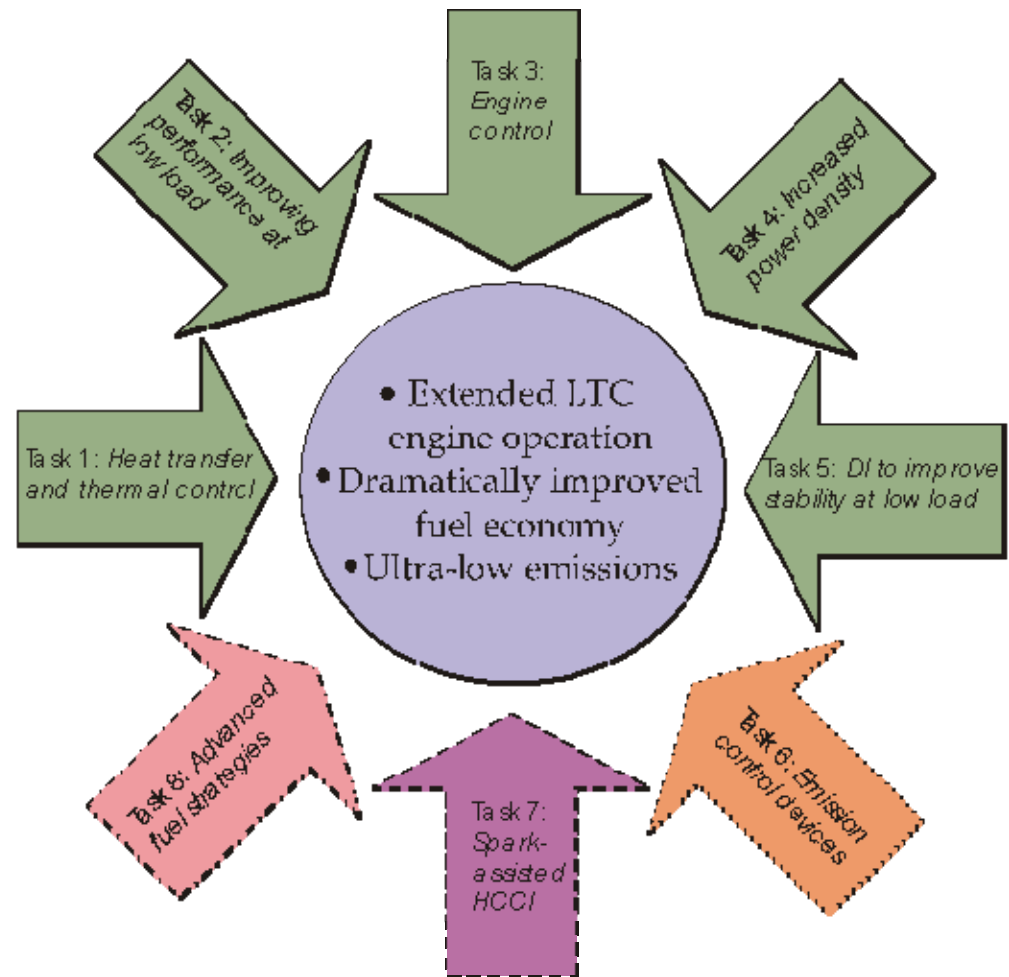


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LTC Consortium Tasks

1. Thermal management (UM)
2. Combustion stability at low load (UM)
3. Engine control for extended operation (MIT)
4. Increased power density (UCB)
5. DI studies for low load (SU)
6. Emission control devices for PPCI (UM)
7. Spark assisted HCCI (UM)
8. Ignition properties of alternative fuels (UM)



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Engine University Consortium Set-Ups

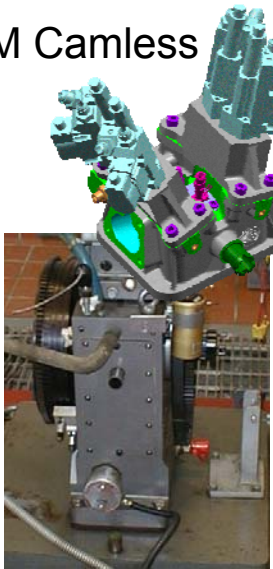
UM Optical Engine



UM Heat Transfer Engine



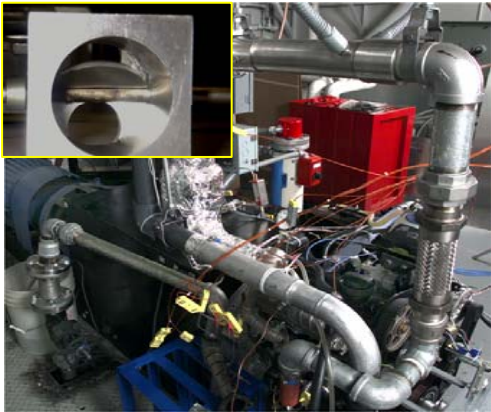
UM Camless Engine



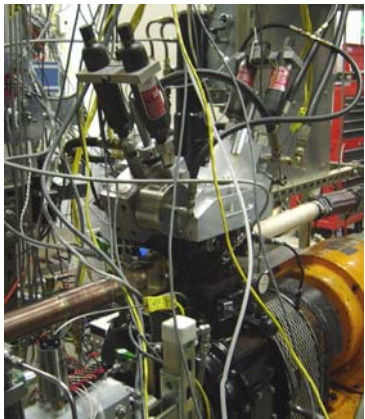
MIT Camless Engine



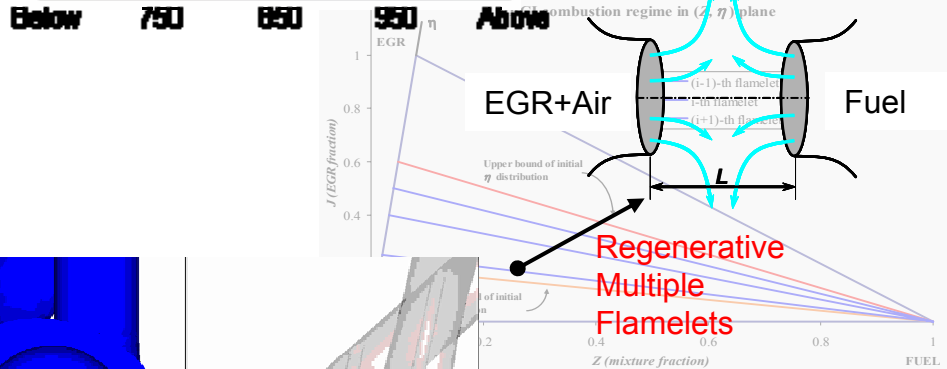
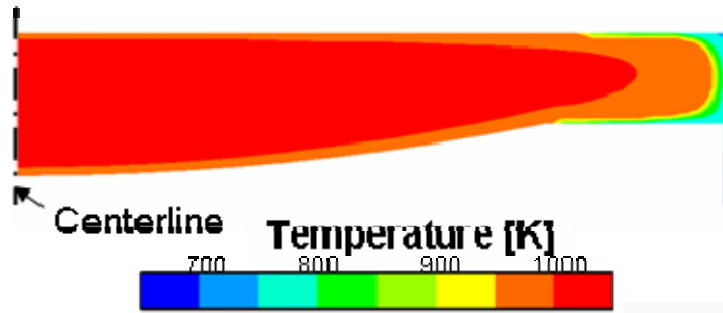
UCB Multi-cylinder engine



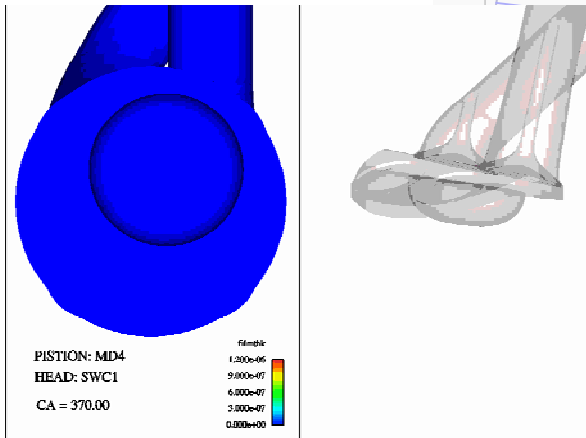
Stanford Camless Engine



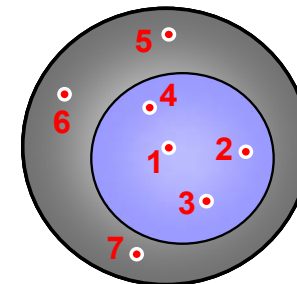
Thermal and Compositional Stratification: Near-wall Conditions and Mixture Preparation



Stanford VVA engine



UM Heat Transfer and VVA Engine

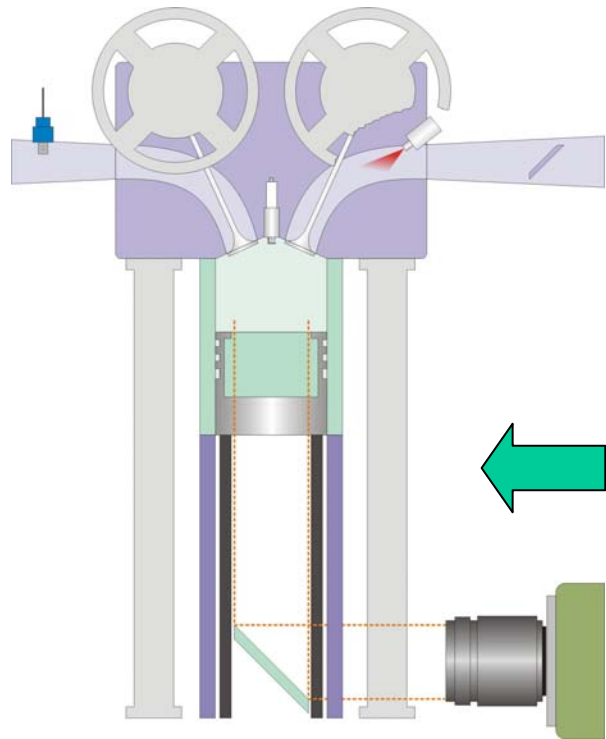


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Spark-Assisted Concepts: Expand the LTC Operating Range

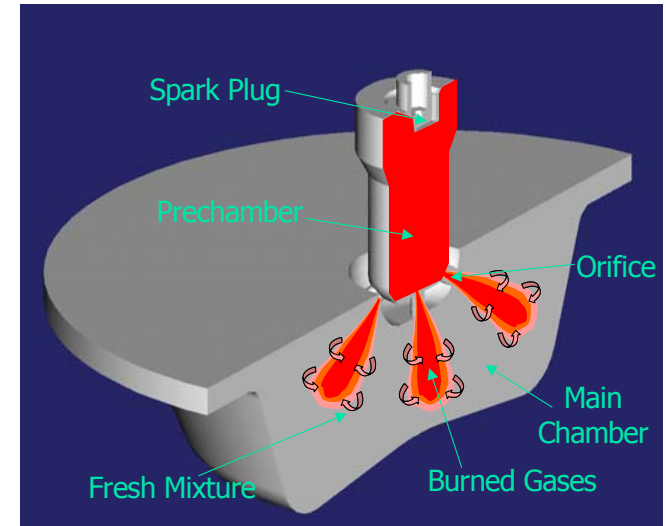
SA-HCCI
Open Chamber



UM Optical Engine



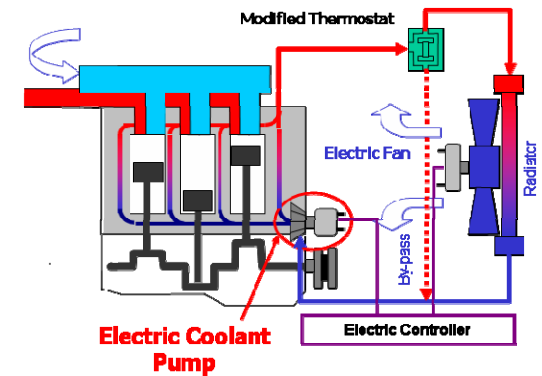
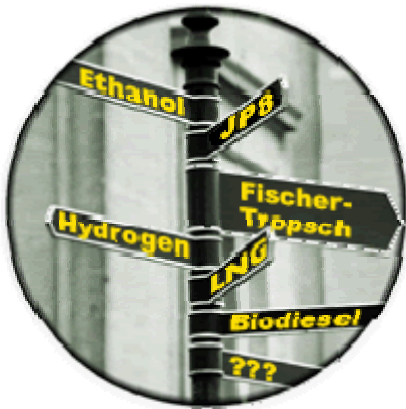
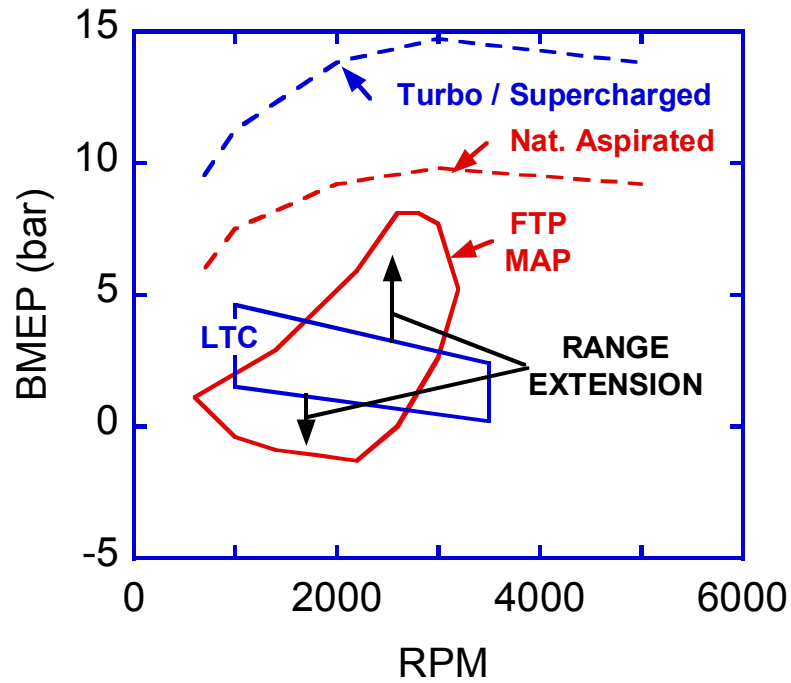
SA-Prechamber
Concept



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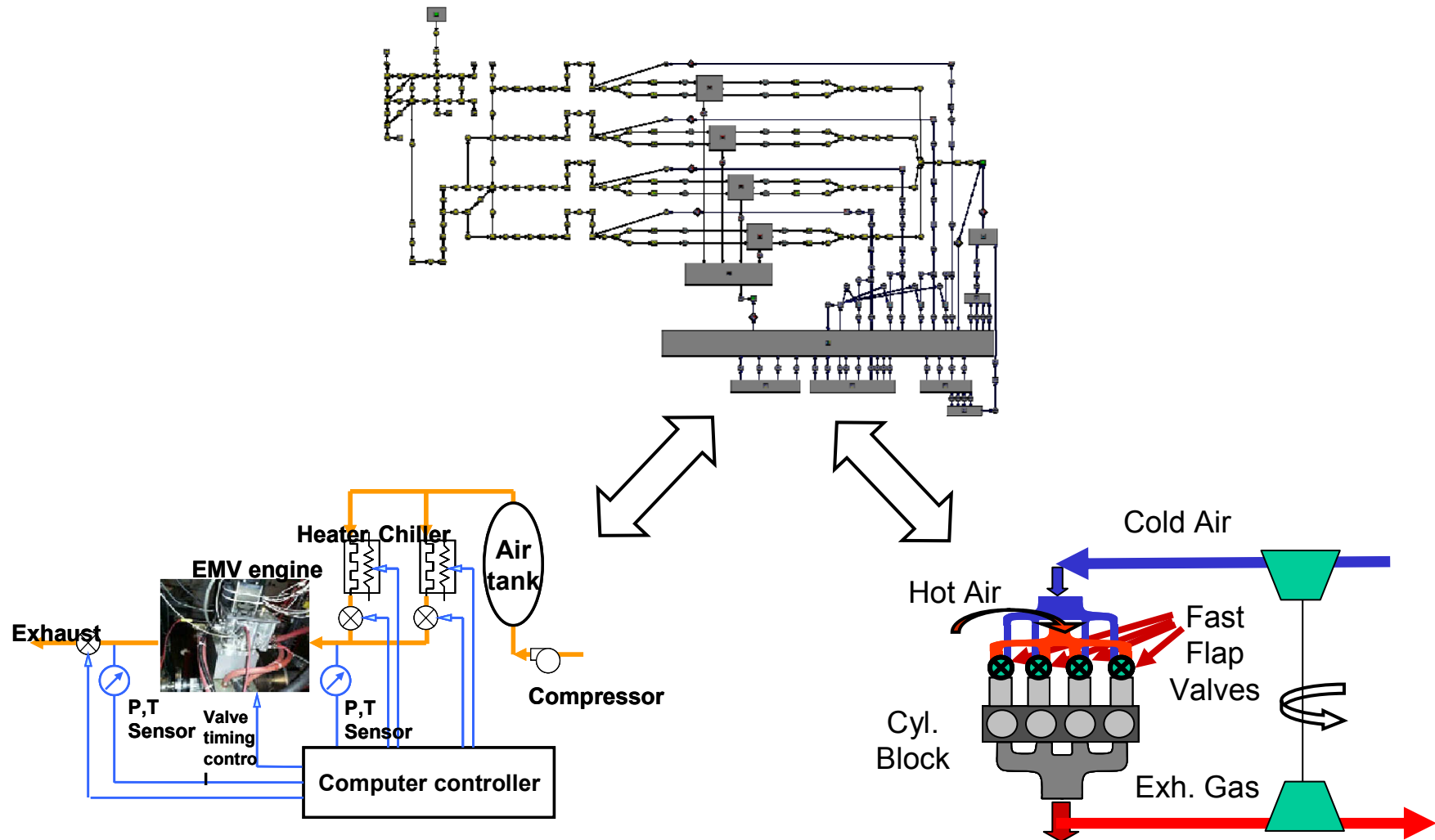
From In-cylinder to Systems



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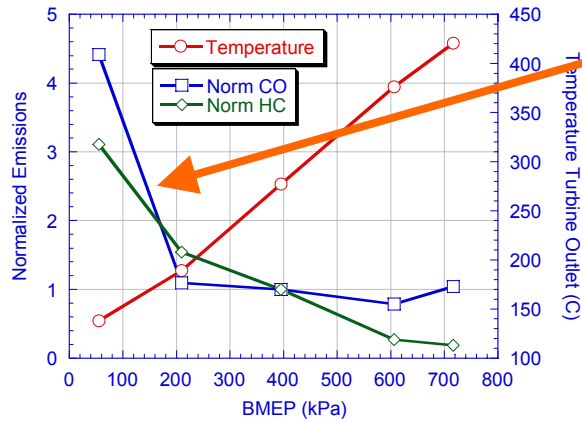
Expanding the LTC Range: Turbo-charging and Exhaust Heat Recuperation



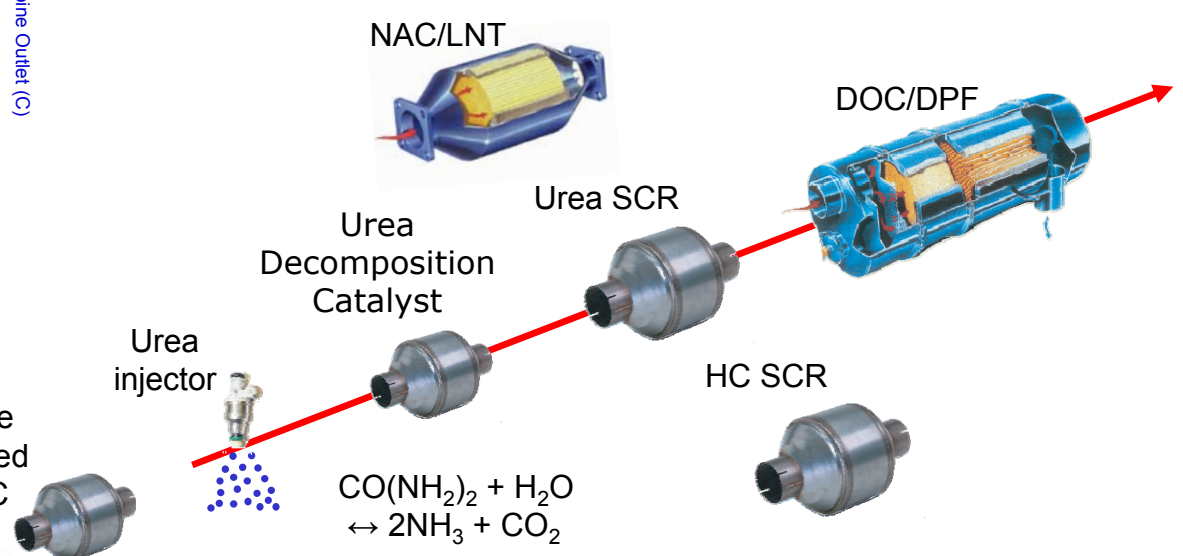
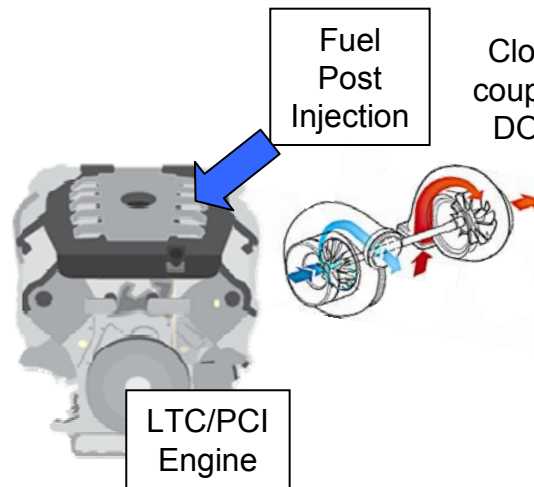
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Aftertreatment Options for LTC Engine



Low Temperature Combustion leads to higher HC, CO



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Thermal Management: Predictive HCCI Engine System Simulation

Auto-ignition
delay expression

$$\int (1/\tau_{ign}) dt = 1.0 \quad \tau_{ign} = 1.3 \cdot 10^{-4} \cdot P^{-1.05} \cdot \phi^{-0.77} \cdot y_{O_2}^{-1.41} \cdot \exp(33700/R \cdot T)$$

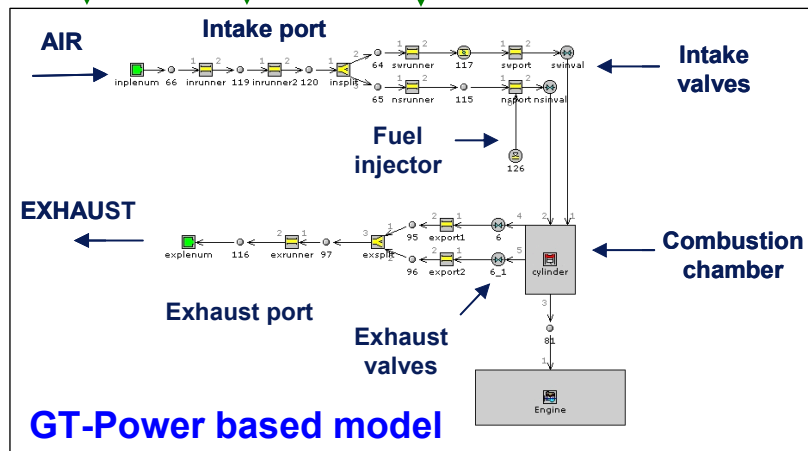
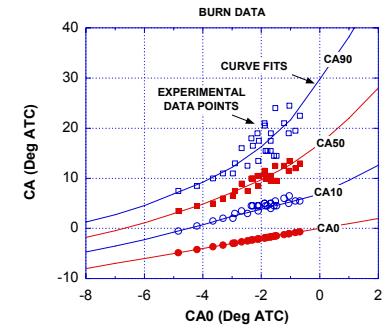
Combustion efficiency
and Burn rate correlation

$$C_{eff} (\%) = \min[95.5, 92.5 - 1.1 \cdot (CA0) - 0.06 \cdot (CA0)^2]$$

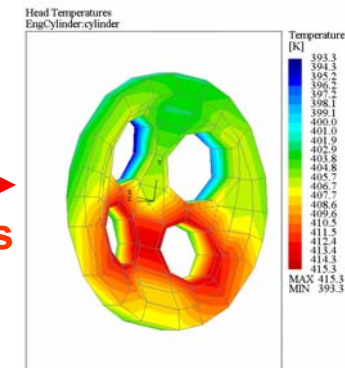
$$x = C_{eff} \cdot [1 - \exp(-(\frac{CA - CA0}{\Delta\theta})^{w+1})]$$

Heat Transfer correlation

$$h(t) = \alpha_{scaling} \cdot L(t)^{-0.2} \cdot P(t)^{0.8} \cdot T(t)^{-0.73} \cdot W(t)^{0.8} \quad W(t) = C_1 S_p + \frac{C_2 V_d T_r}{6 P_r V_r} (P - P_{mot})$$



Finite Element Analysis
of Wall Temperatures

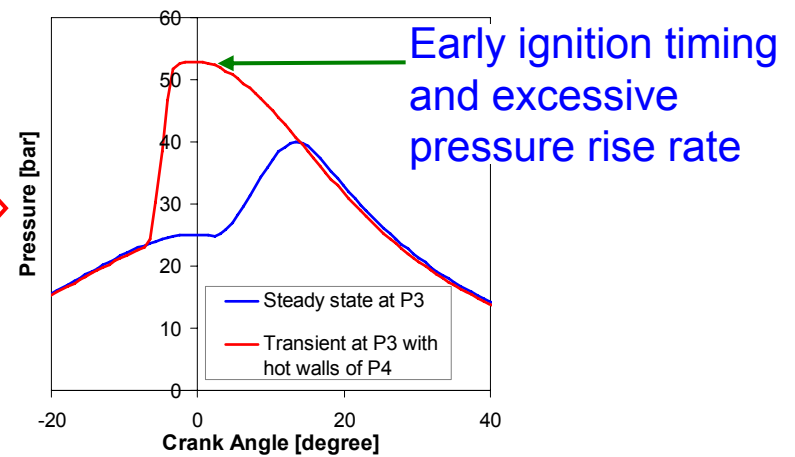
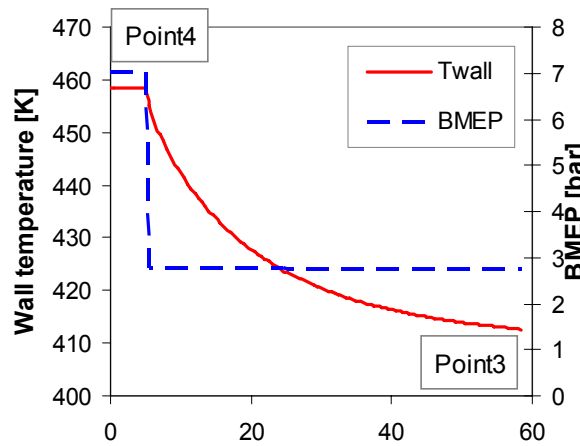


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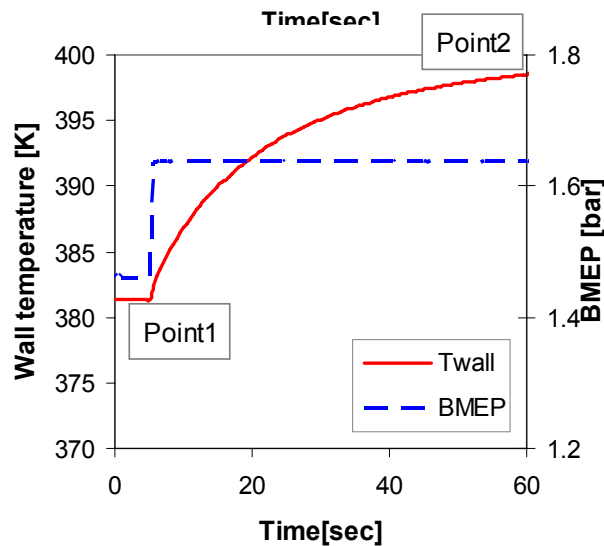


Thermal Management: Wall Temperature Effects

Hot to cold



Cold to hot

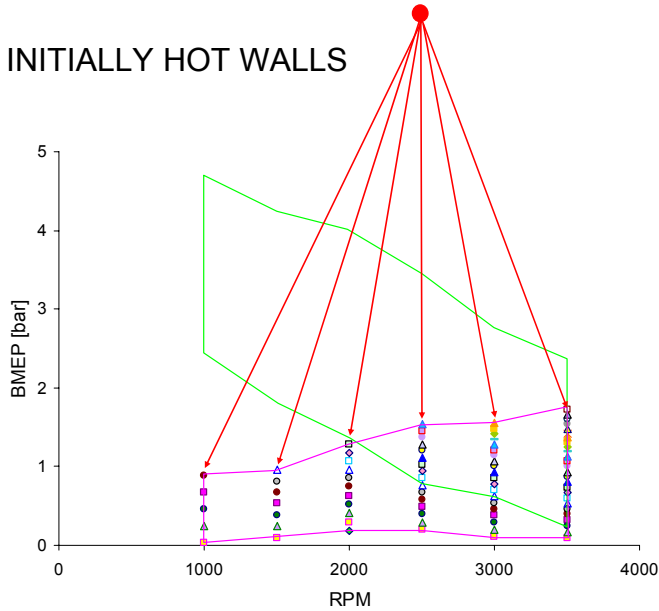


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


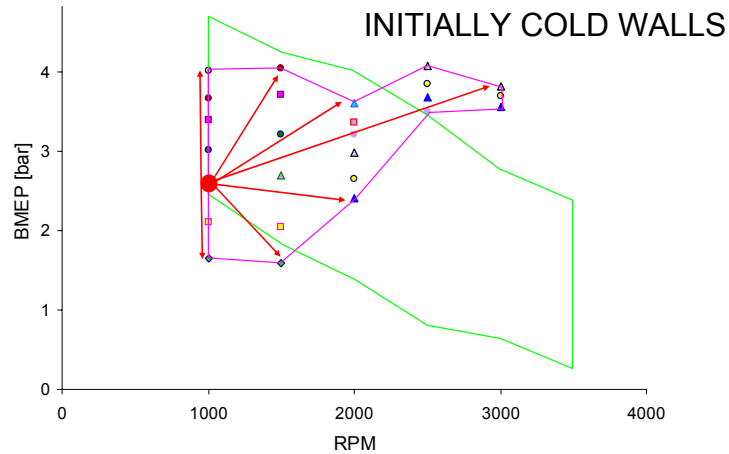
HCCI operation depends on load history: Transient wall temperatures affect the range

INITIALLY HOT WALLS

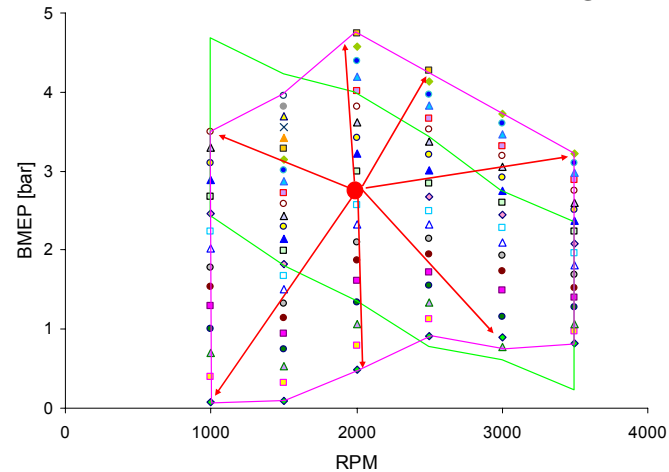


- Engine with rebreathing:
 - Residual fraction adjusted for optimal combustion
- Compression ratio = 12.5

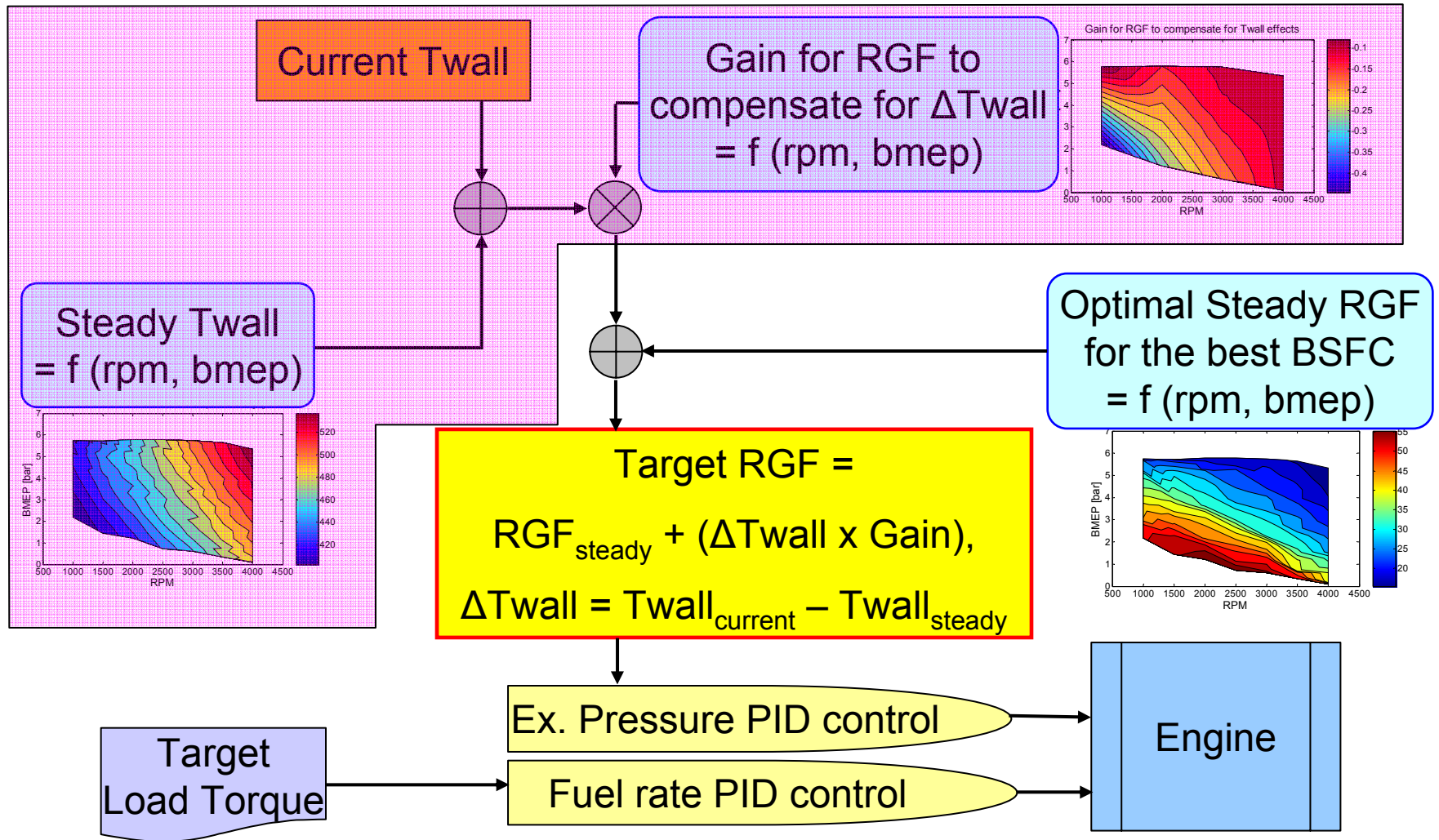
 RANGE WITH STEADY STATE WALL TEMP
 RANGE WITH TRANSIENT WALL TEMP



INITIALLY MID-RANGE WALLS

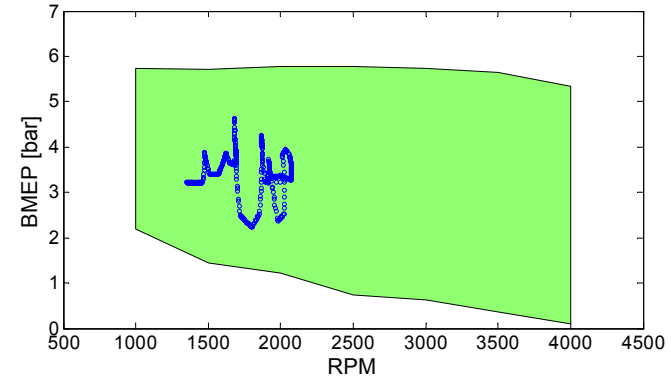


Schematic Diagram Twall Compensation

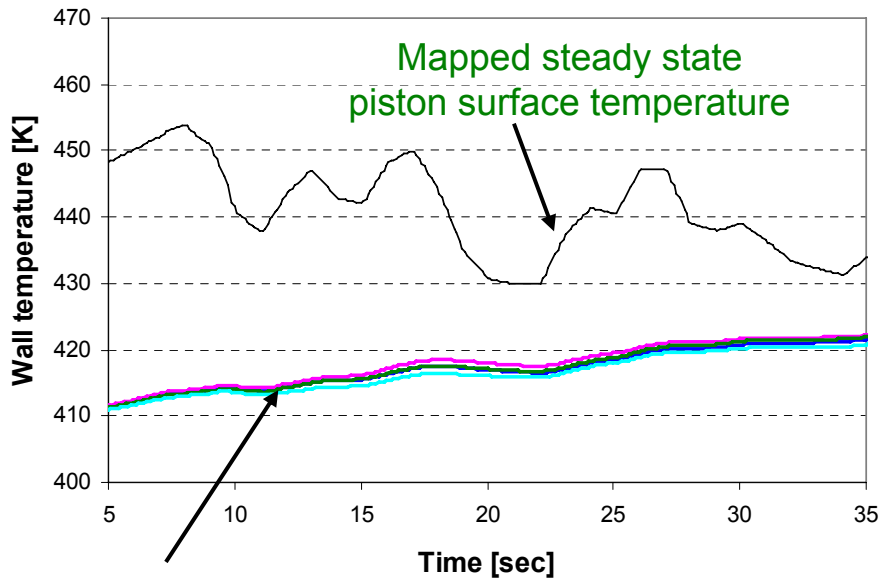


Transient Simulation in HCCI Regime – Cold Walls

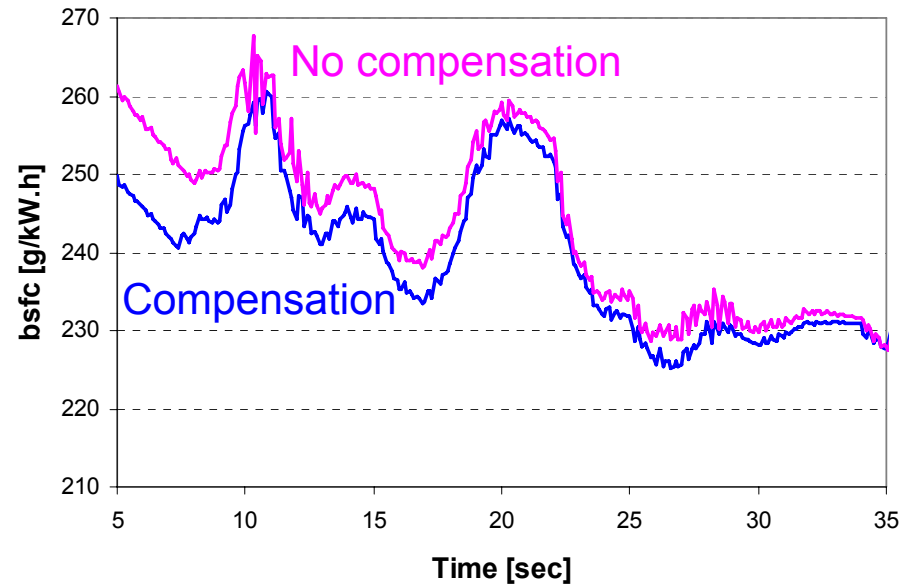
- Brake specific fuel consumption
 - Overall better BSFC with compensation for the cold T_{wall}



Piston surface temperature



Calculated transient piston surface temperature



Acknowledgements

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Thank you!



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