

Development of Dual-Fuel Engine for Class 8 Applications (Dept of Energy Supertruck Program)

Yu Zhang, William de Ojeda, Dennis Jadin
Navistar Inc.

Andrew Ickes, Thomas Wallner
Argonne National Laboratory

David Wickman
Wisconsin Engine Research Consultants

Technical Session: High-Efficiency Engine Technologies Part 2

DOE DEER CONFERENCE

**18 October 2012
Dearborn, Michigan**

Background

- High Efficiency Combustion Modes
- Base Engine and Efficiency Target

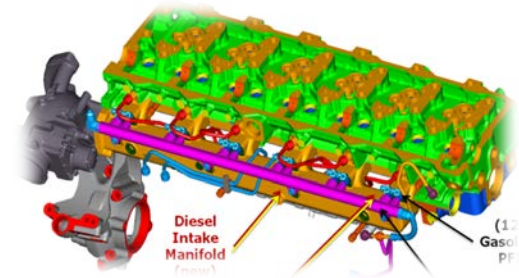


Development Strategy

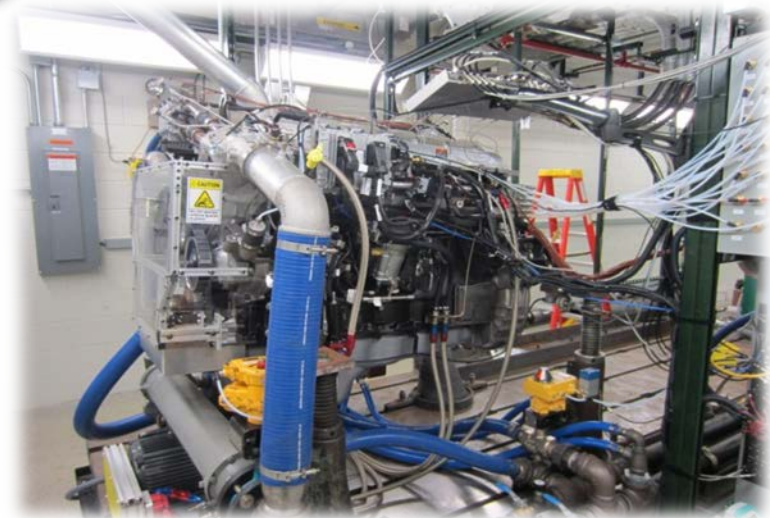
- Fuel Reactivity Options
- Fueling Strategy
- Efficiency Roadmap

Results

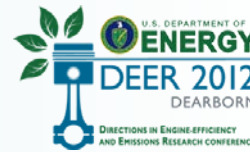
- Gasoline + Diesel
- Alcohol/Gasoline + Diesel



Summary



High Efficiency Combustion Modes

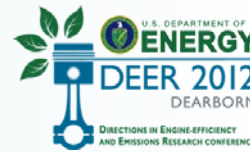


	HCCI	Fuel Reactivity	Diesel Combustion
Charge Preparation	Homogeneous	PFI + DI	DI
Fuels	Flexible, Single	Dual-Fuel	Diesel
Combustion Modes	Premixed	High Reactivity → Low Reactivity	Diffusion
Reactivity Stratification			
Controllability			
Challenges	Controllability	Controllability	High PM & NOx
	Load Limitation	Load Limitation	

Fuel Reactivity

- ✓ Provides flexibility in tailoring combustion process by manipulating PFI/DI ratio
- ✓ Widens load range by introducing reactivity stratification

Base Engine and Efficiency Target



Project Target:

✓ Demonstrate a technical path towards **55%** BTE with **fuel reactivity**

MY 2010 MAXXFORCE 13

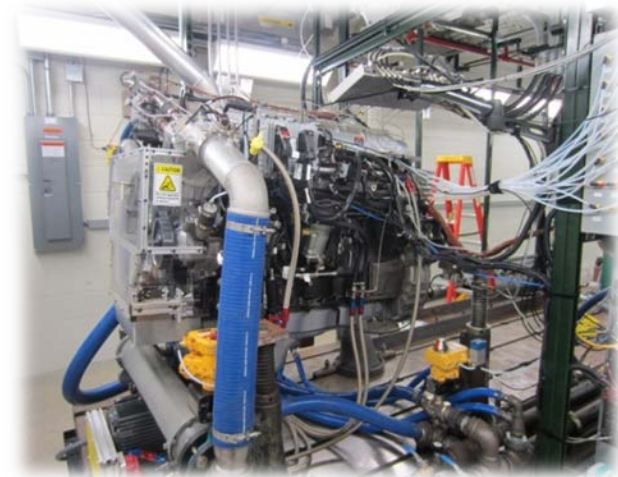
- common rail fuel injection system
- regulated 2-stage turbocharger with intercooler
- 2-stage HP loop EGR cooling

Rated Power 475hp



MAXXFORCE 13 Dual-Fuel

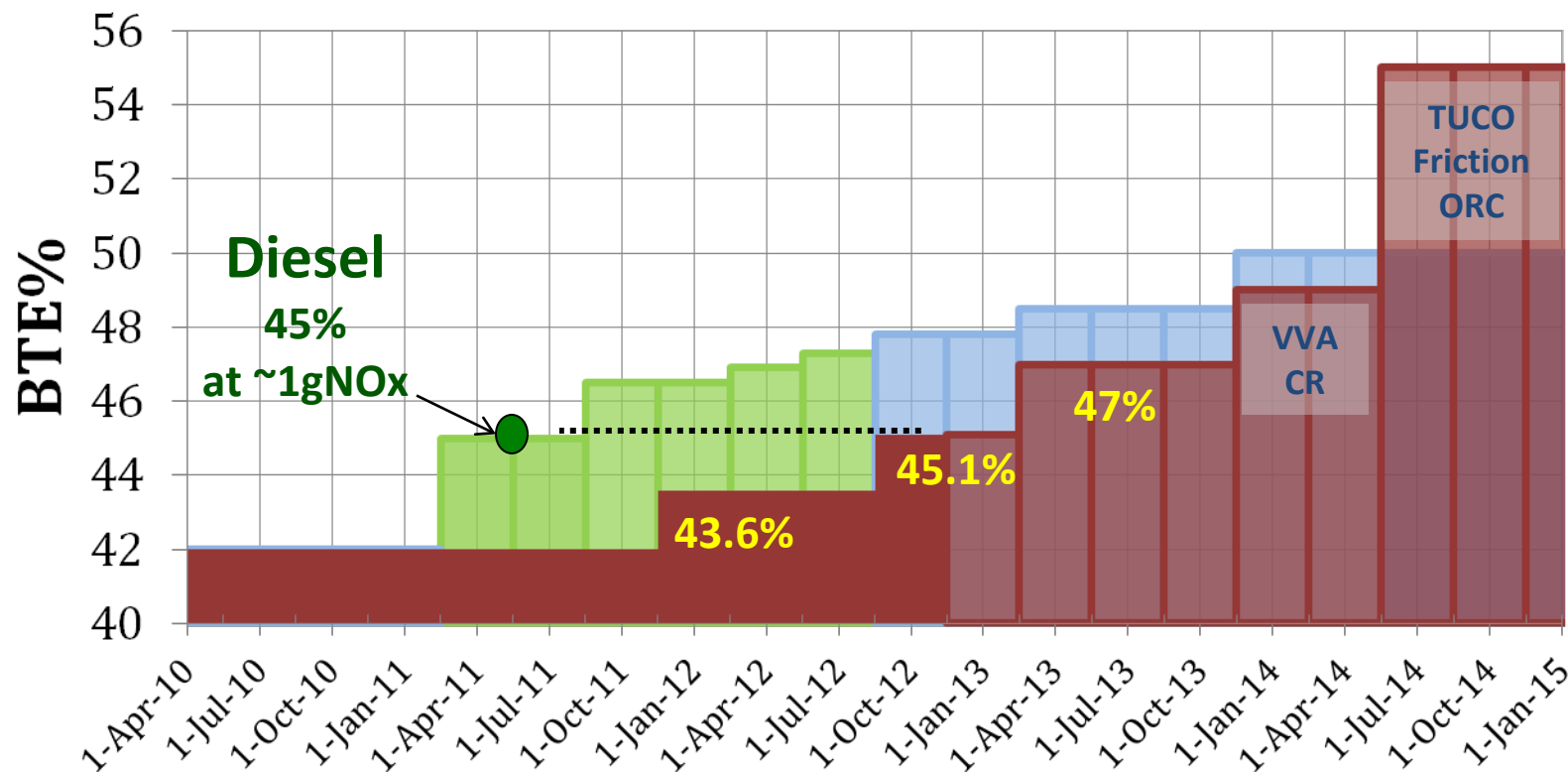
- ✓ multi-point port fuel injection
- ✓ dual-fuel control system
- ✓ variable geometry turbocharger
- ✓ variable valve actuation
- ✓ optimized piston geometry
- ✓ high pressure common rail



Fuel Reactivity Efficiency Roadmap



- ✓ Significant improvement on BTE with fuel reactivity
- ✓ At better controlled engine out emissions



Progress to Date

**Gasoline
+ Diesel
43.6%
NOx ~ 0.1gNOx**

**Alcohol/Gasoline
+ Diesel
45.1%
NOx ~ 0.1gNOx**

Current Target

**Reactivity + CR
BTE > 47%
at NOx < 0.15gNOx**

Technologies

**from the Diesel
platform**

Port Fuel Injection



- ✓ lower reactivity suppresses charge autoignition
- ✓ oxygenates provide additional benefit in soot

Direct Injection

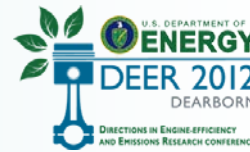


- ✓ longer ignition delay
 - improved mixing
 - less soot

- ✓ robust ignition source
 - enhance combustion stability at late SOIs
 - mitigate PRR

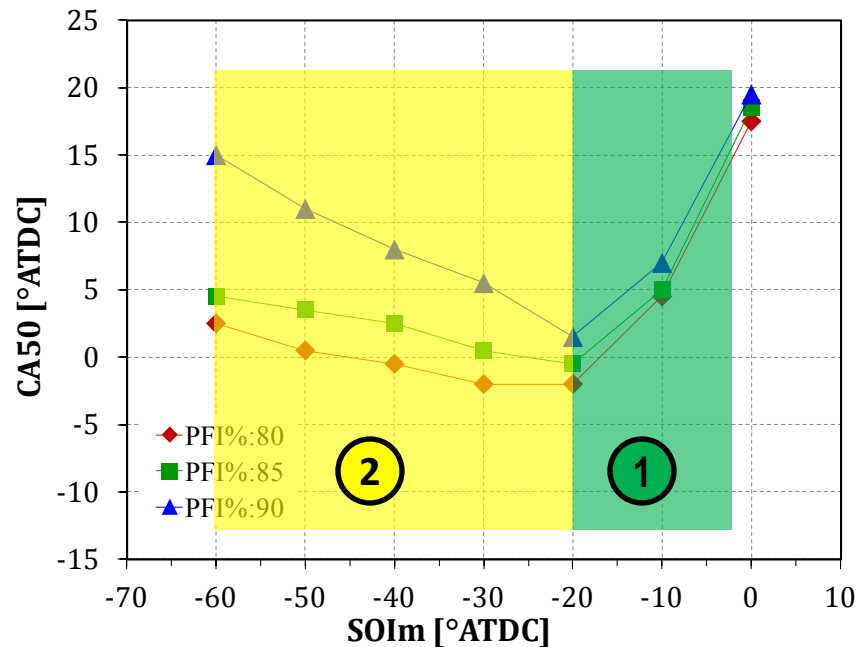
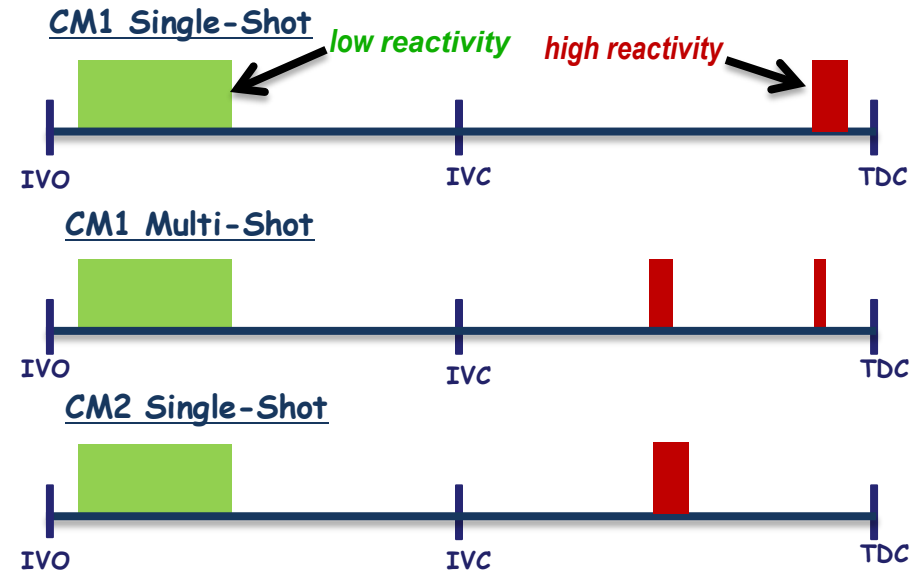
Fuel Reactivity

Fueling Strategy



Fueling Strategy Development

- ✓ PFI%
- ✓ Diesel Injection Strategy
- ✓ Load Range:
 - *low-to-medium: CM1-SS, CM1-MS, CM2-SS*
 - *medium-to-high: CM1-SS*



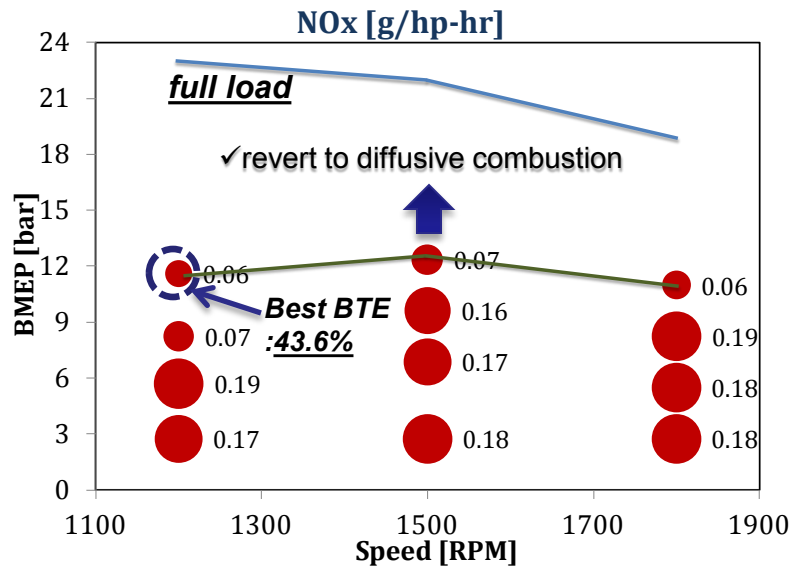
- ① Combustion phasing is robustly coupled to diesel SOI (CM1):
- ② Combustion phasing is largely controlled by charge reactivity (CM2):

Fuel Reactivity

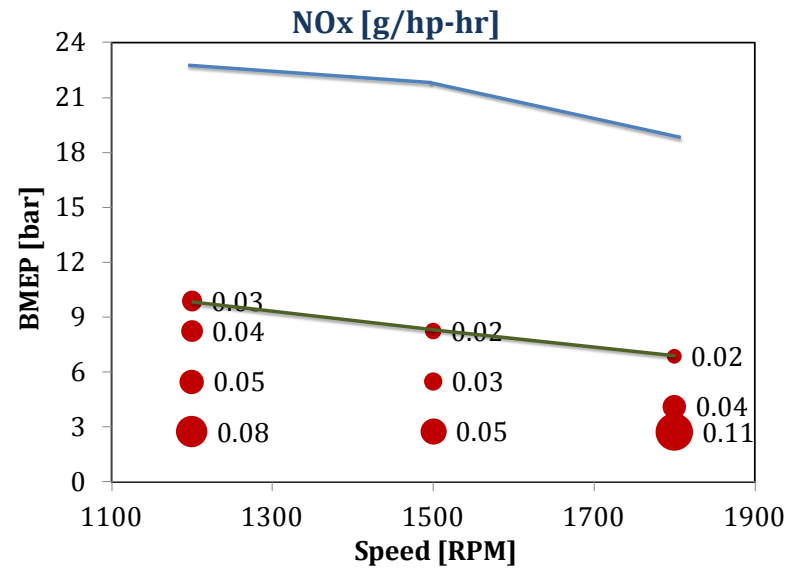
Gasoline + Diesel – Overview

- CM1 led to wider low temperature combustion range than CM2
- CM2 yielded lower NOx and soot than CM1

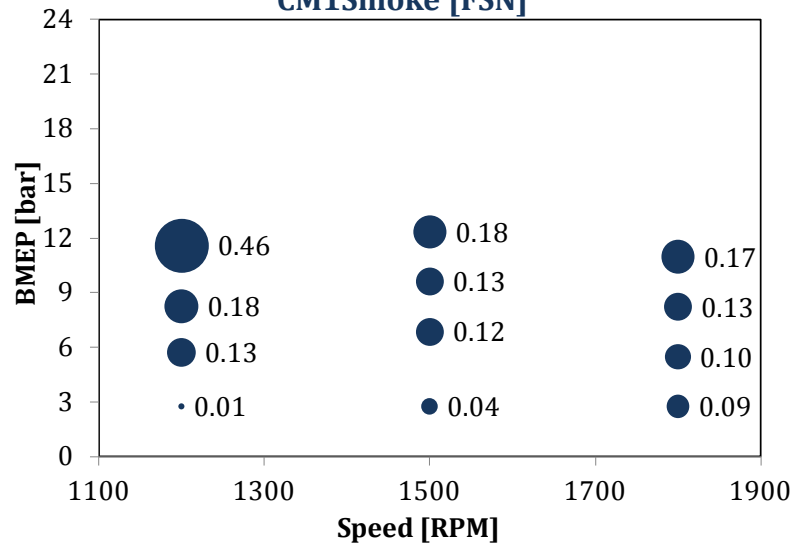
CM1-SS



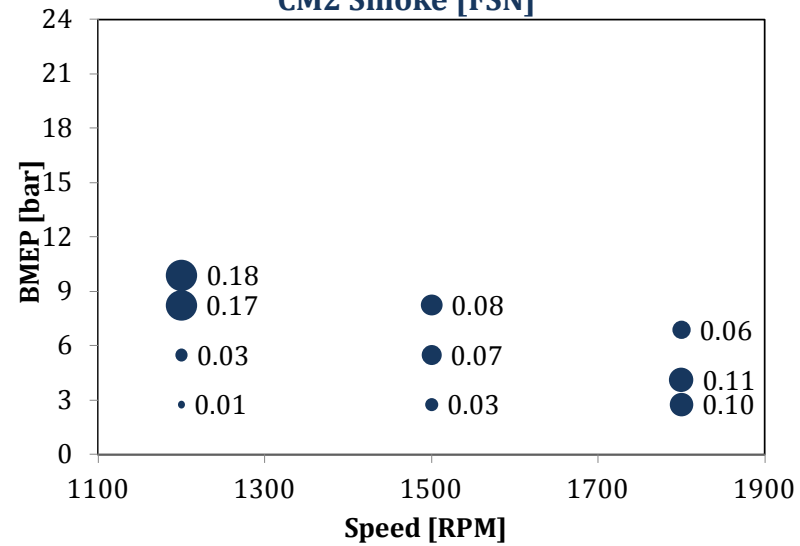
CM2-SS



CM1Smoke [FSN]



CM2 Smoke [FSN]

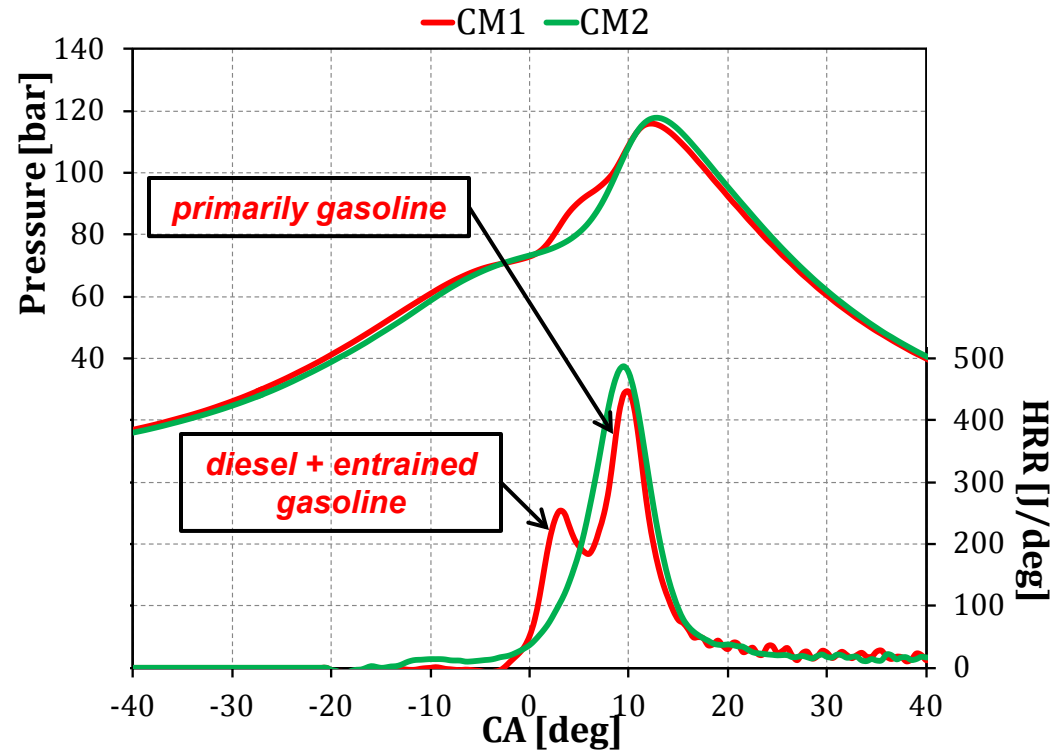


Fuel Reactivity

Gasoline + Diesel – CM1 vs. CM2

1200 rpm; 10 bar BMEP

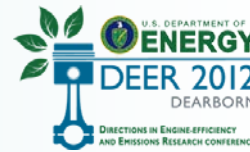
	CM1	CM2
BTE (%)	42	42
Gasoline (%)	84	93
NOx (g/hp-hr)	0.05	0.03
Smoke (FSN)	0.32	0.18
SOI (aTDC)	-12	-58
EGR (%)	increase →	



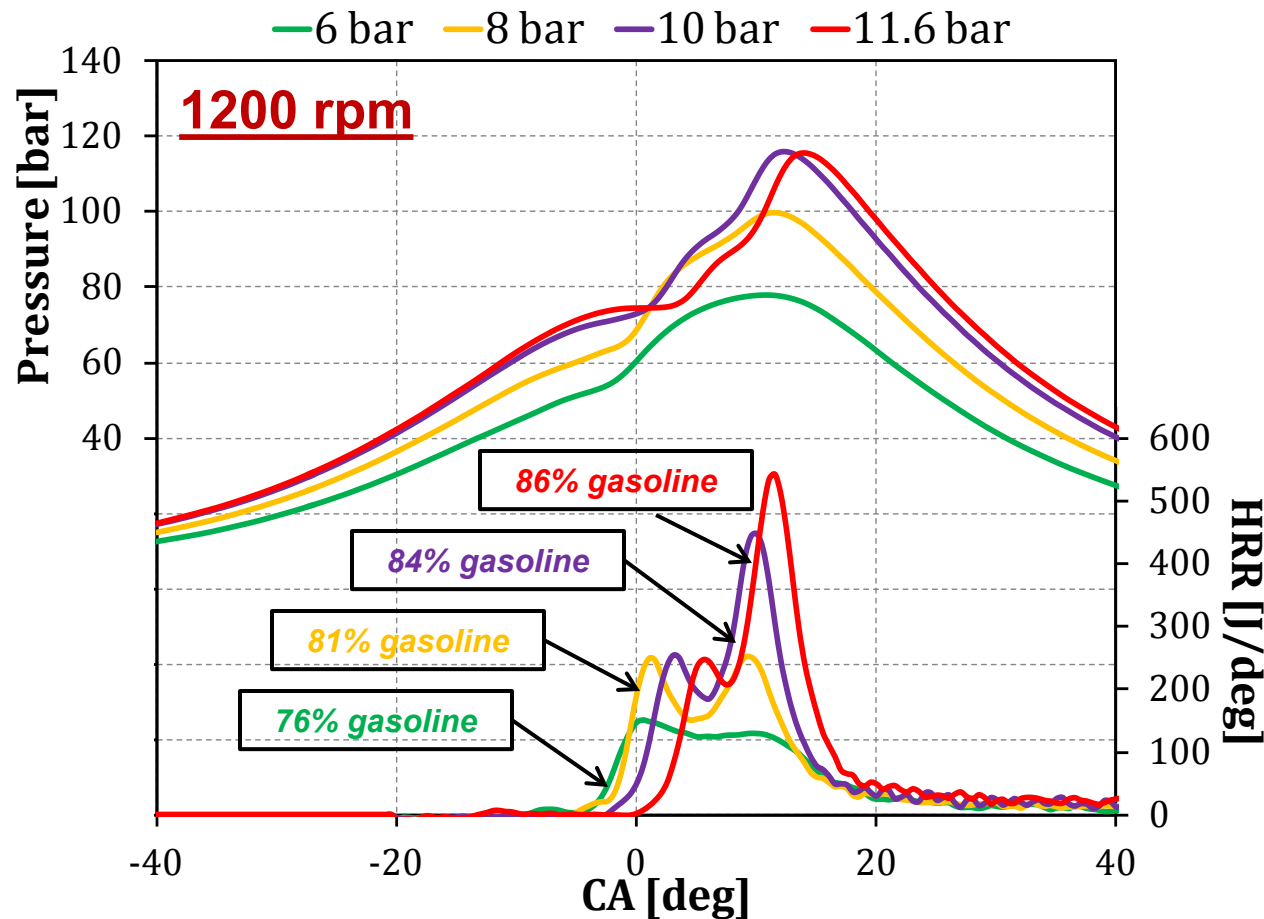
- Comparable fuel efficiency
- CM2 demands more EGR and higher gasoline%
- Combustion characteristics:
 - CM1 combustion proceeds through two-stages
 - CM2 combustion goes through a single-stage heat release

Fuel Reactivity

Gasoline + Diesel – Combustion Characteristics

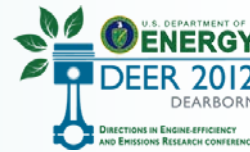


- *CM1 heat release peaks merge with load increase*
- *Towards load-limit, higher gasoline% led to more premixed combustion and higher PRR*

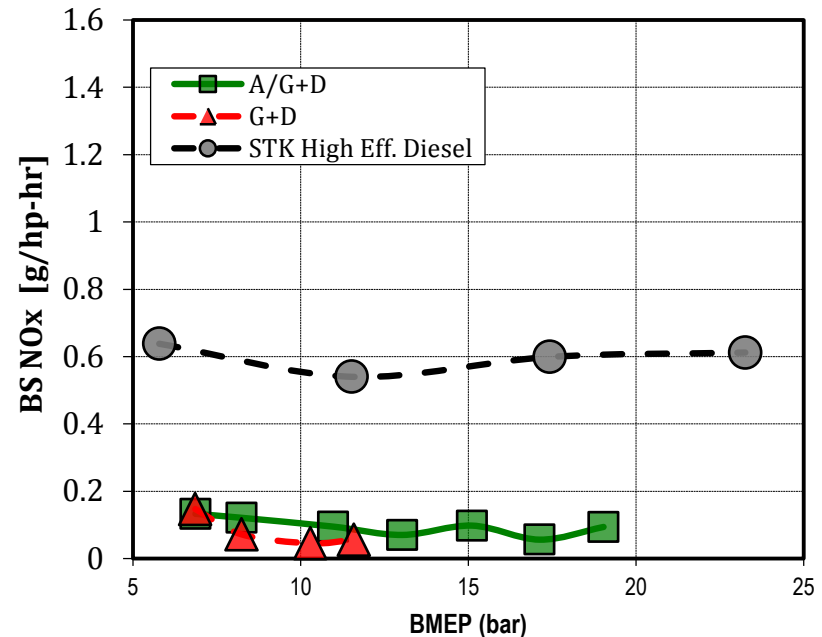
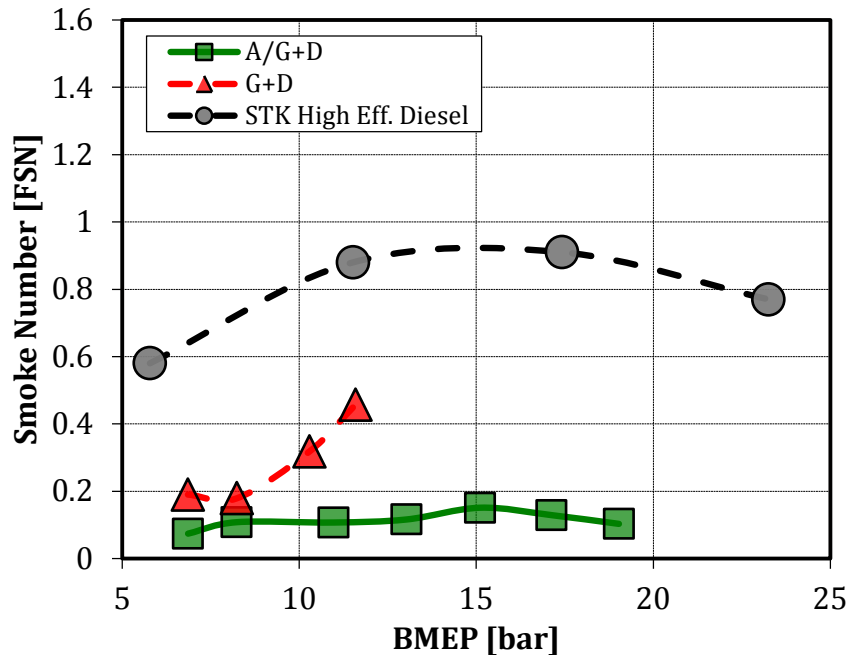
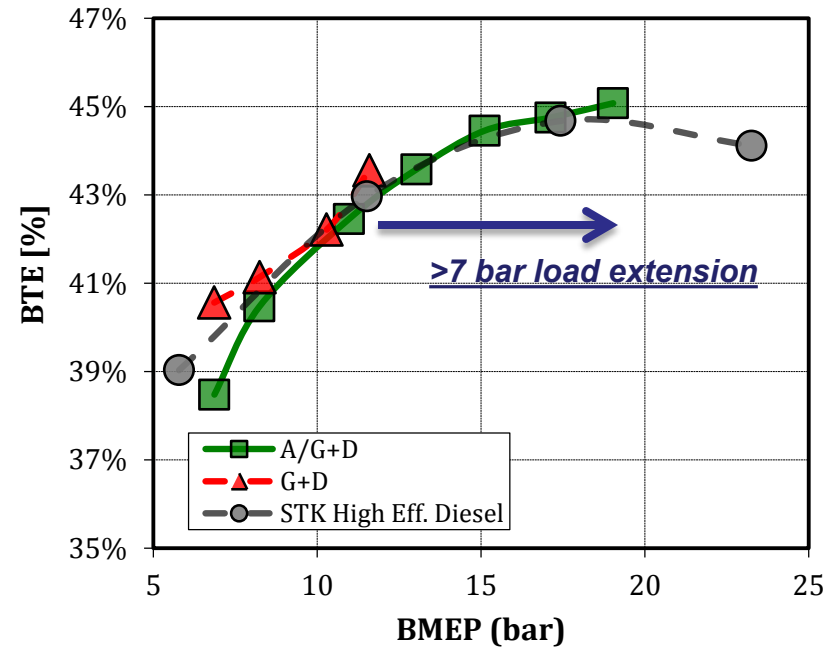


Fuel Reactivity

Alcohol/Gasoline + Diesel – Performance

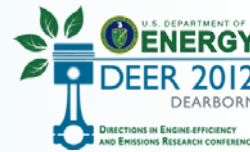


- ✓ Alcohol/gasoline extended LTC load range to **19 bar** BMEP
- ✓ Fuel-bound oxygen led to soot reduction
- ✓ improved fuel efficiency
 - *best BTE: **45.1%***



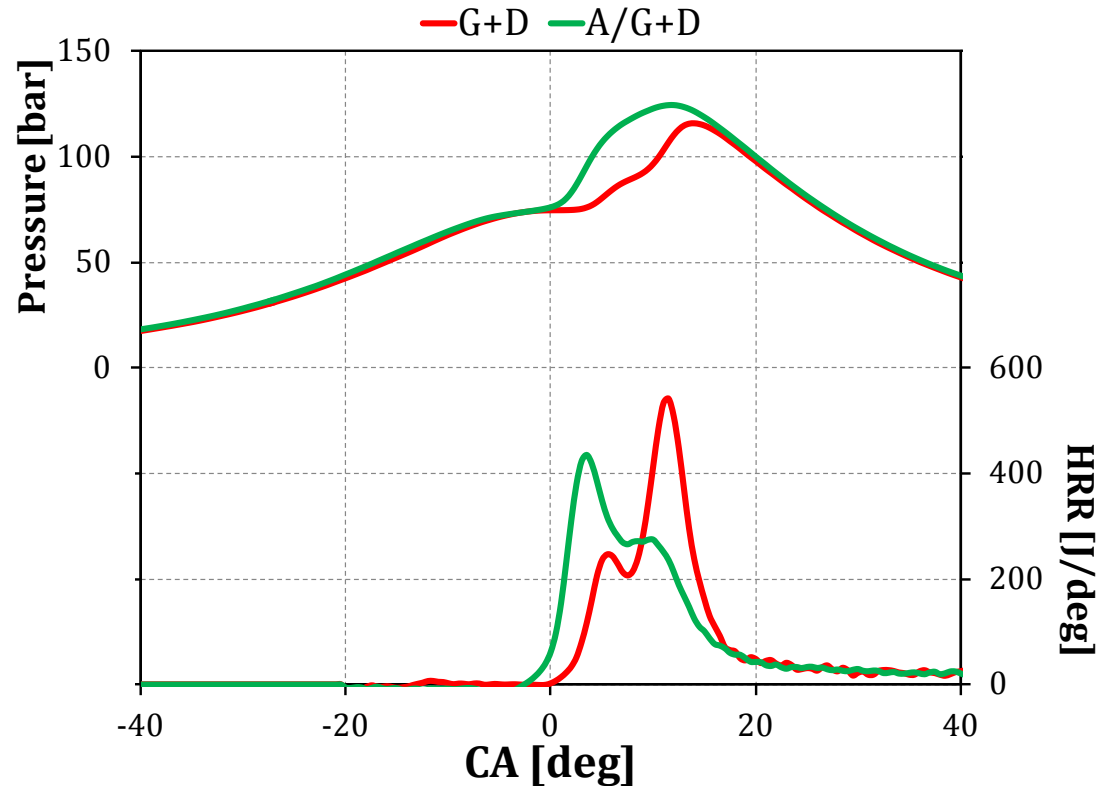
Fuel Reactivity

Alcohol/Gasoline + Diesel vs. Gasoline + Diesel



1200 rpm; 11.6 bar BMEP

	G+D	A/G+D
BTE (%)	43.6	43
PFI (%)	84	78
EGR (%)		reduce →
SOI (aTDC)	-7	-13
NOx (g/hp-hr)	0.06	0.08
Smoke (FSN)	0.46	0.11



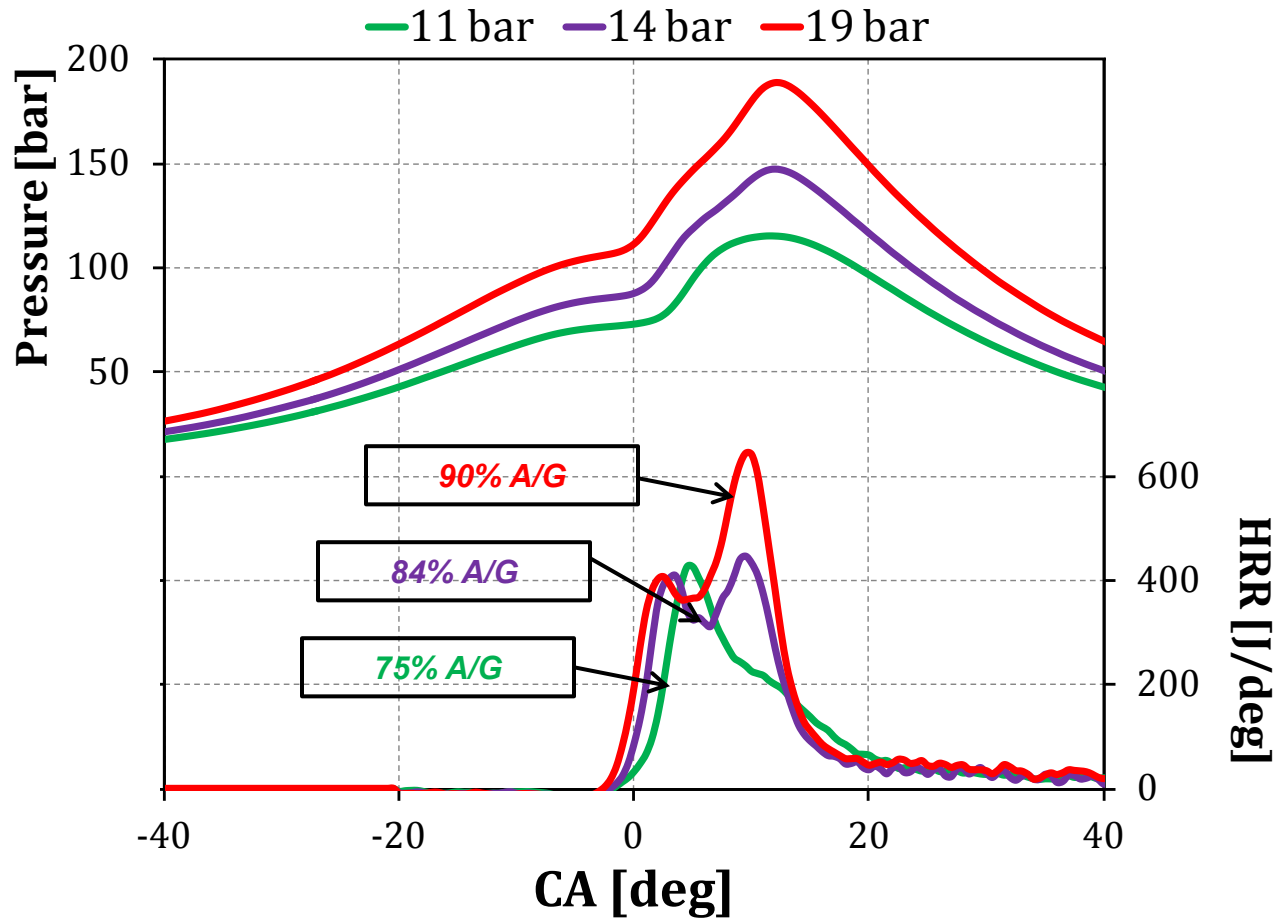
- Alcohol/gasoline showed less rapid second-stage heat release
- Alcohol/gasoline allowed for more advanced SOI and earlier combustion phasing without sacrificing soot and PRR

Fuel Reactivity

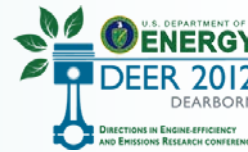
Alcohol/Gasoline + Diesel – Combustion Characteristics



✓ Alcohol/gasoline led to wider heat release spread and better contained PRR



Fuel Reactivity Summary



Project Target:

- ✓ Demonstrate a technical path towards 55% BTE with fuel reactivity

Milestones:

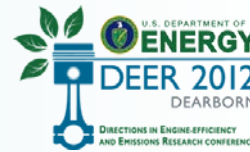
- ✓ Phase I with gasoline + diesel demonstrated LTC operation up to 11.6 bar BMEP with best BTE of 43.6%
- ✓ Phase II with alcohol/gasoline + diesel achieved LTC operation up to 19 bar BMEP with best BTE of 45.1%

Further Improvements

- ✓ Further exploration of fuel reactivity
- ✓ Combustion system optimization (CR, piston geometry)
- ✓ Air system upgrade (VVA+turbo)

Acknowledgements

Engine Project Partners



Combustion
Simulation

Fuels

Enabling
Technologies

WERC

ARGONNE
NATIONAL
LABORATORY

BOSCH

FEDERAL
MOGUL

Thank You for Your Attention

Yu Zhang
Navistar, Inc.
2601 Navistar Drive
Lisle, IL 60532
Yu.Zhang@Navistar.com

