

# Impact of Variable Valve Timing on Low Temperature Combustion

William de Ojeda  
**Navistar**

**Advanced Combustion**  
**DOE DEER CONFERENCE**  
**September 27-30, 2010**  
**Detroit, Michigan**

## **Low Temperature Combustion Roadmap**

Clean and Efficient Combustion

Review of Variable Valve Actuation in LD and HD SCTE engines

## **Navistar MAXXFORCE V8 6.4L**

Levering technologies

An Effective Electro-Hydraulic VVA Device

Thermodynamic Effects of VVA

## **Performance and Combustion Effects of VVA**

Ignition Temperature and Ignition Delay

Soot Chemistry

Combine effects with PCCI

Extension of LTC Operation

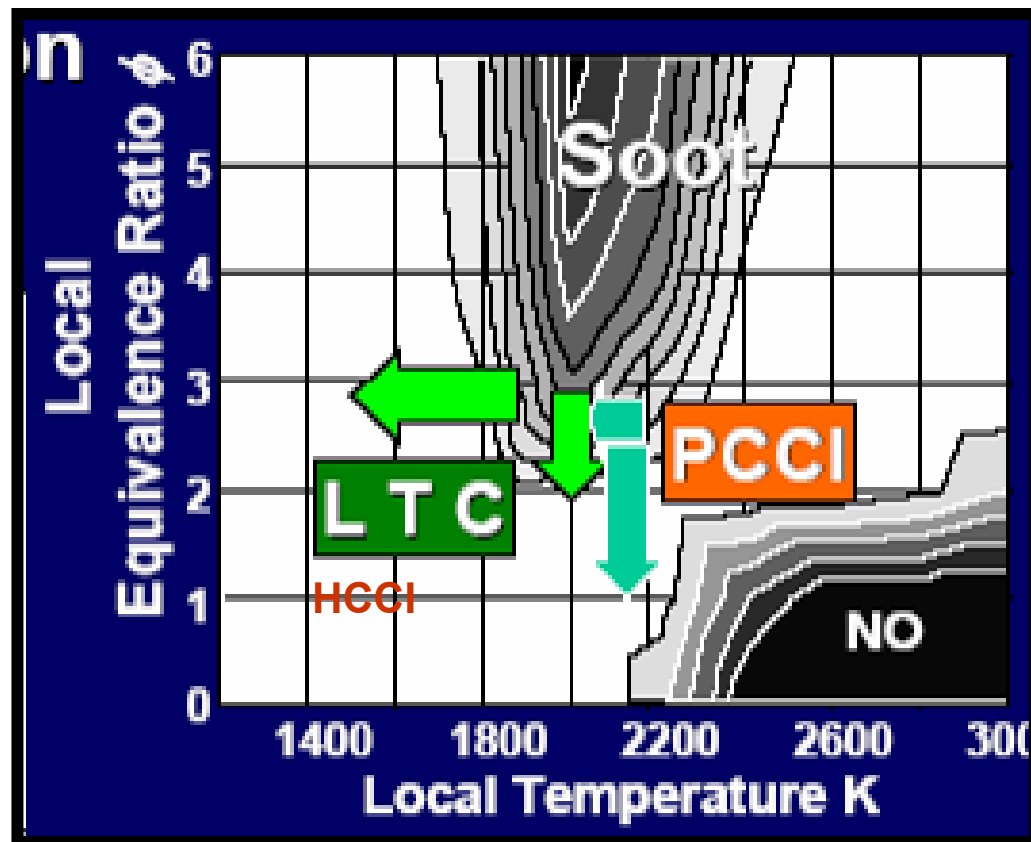
## **Summary**

Acknowledgements and Project Partners

# LTC Road Map

**NOx suppression:** Low flame temps  
 $T < 2200K$

**Soot formation:**  $\phi > 2$   
 $1500 < T < 2200$



[Herzog et. al 1992]  
[Akihama et. al 2001]

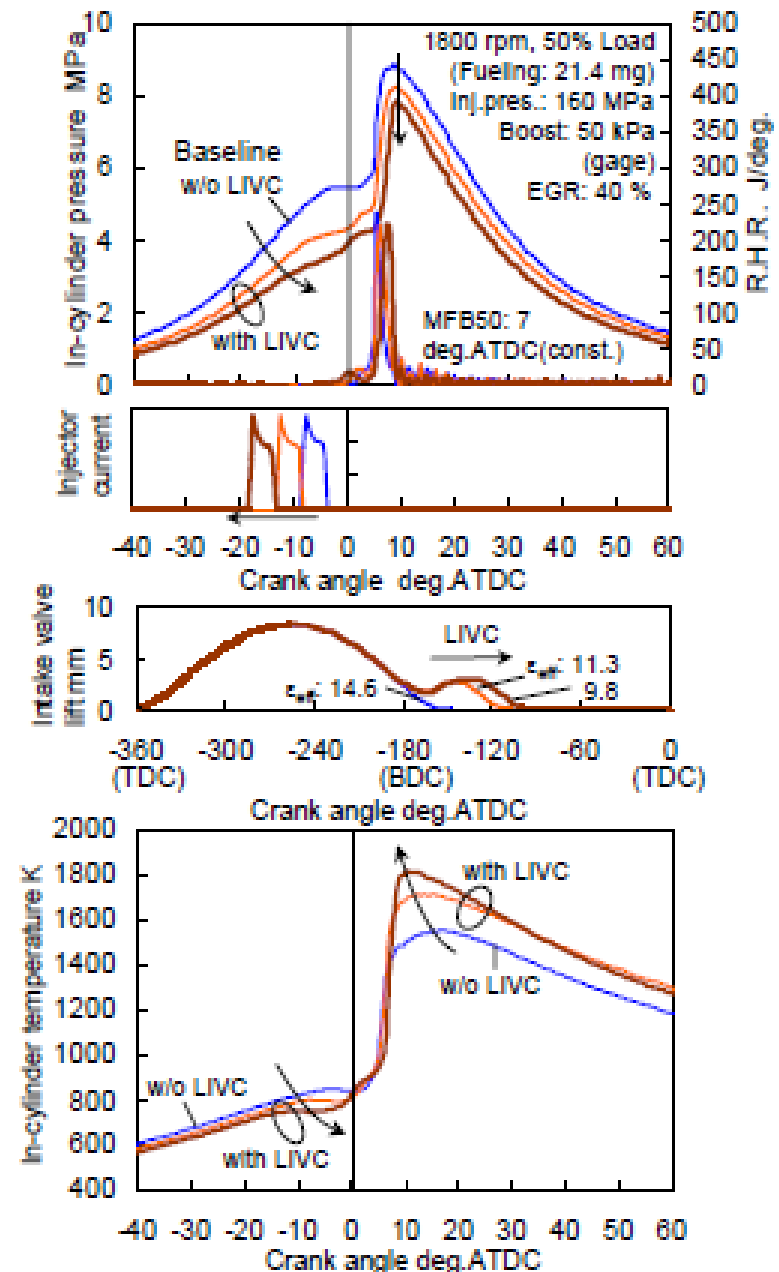
**Efficiency:** Combustion phasing  
Combustion efficiency  
Reduce pumping losses  
Reduce parasitic

➔ Turbocharger design  
EGR / Charge Air system

## Light Duty SCTE (0.5L)

Effects of Late Intake Valve Closing (LIVC) at 25 and 50% loads

- Lowers effective compression ratio
- Lowers in-cylinder temperatures
- Increases ignition delay
- **Reduced smoke** despite lower excess air ratios.
- LIVC, EGR, supercharging and high-pressure fuel injection **simultaneously reduces NOx and smoke.**
- High CO and THC competes with fuel economy.

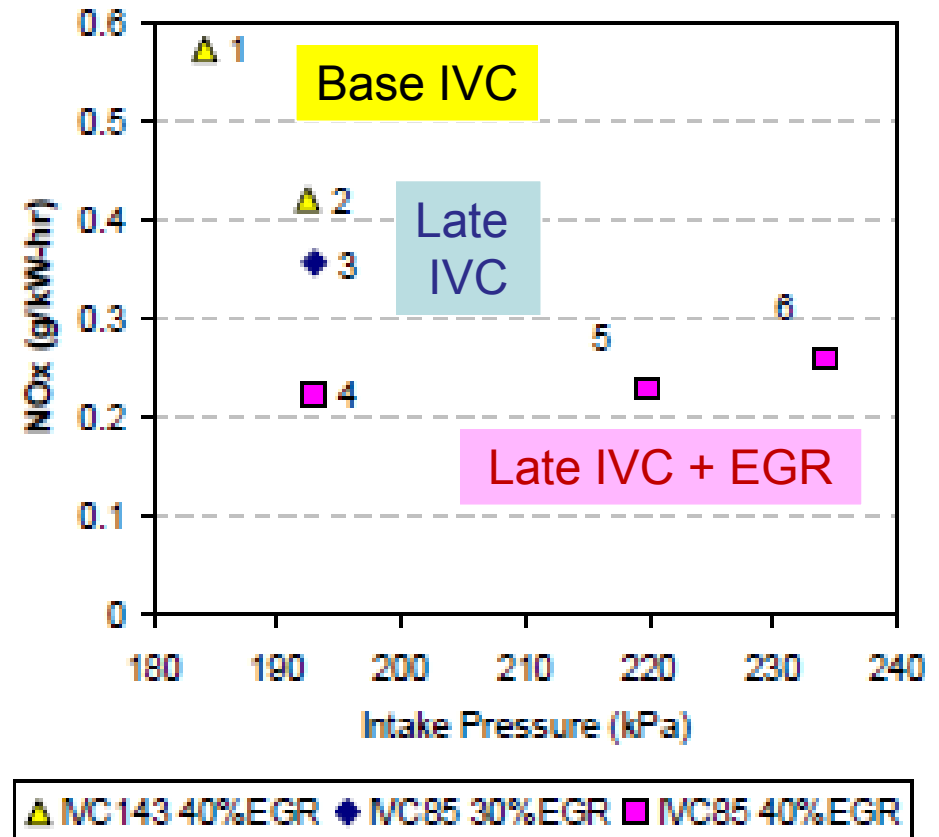


Murata, Y., Kusaka, J., Odaka, M., Daisho, Y., Kawano, D., Suzuki, H., Ishii, H., and Goto, "Achievement of Medium Engine Speed and Load Premixed Diesel Combustion with Variable Valve Timing" SAE Paper 2006-01-0203.

## Heavy Duty SCTE (2.4L)

Effects of LIVC at 50% load

- Effective to **reduce NOX**
- Boost compensated to keep **soot at ~ 0.01g/kW-hr**



Nevin R.M., Sun Y., Gonzalez M.A. and Reitz R.D.

“PCCI Investigation Using Variable Intake Valve Closing in a Heavy Duty Diesel Engine”

SAE Paper 2007-01-0903

# Navistar MCTE V8 Engine

## Engine Configuration

### Multi-Cylinder Test Engine

1. Upgraded turbo
2. Improved HP EGR loop
3. Variable valve actuation  
(early intake valve closing)
4. Combustion feedback

	Base Engine	Test Engine
Displacement	6.4L	6.4L
Bore	98.2mm	98.2mm
Stroke	105mm	105mm
FIE	DI Common Rail	DI Common Rail
CR	16.5	16.2
Turbo Charger	Dual Stage Waste Gate	<b>Dual Stage VNT</b>
EGR system	HP loop Single Cooler	<b>HP loop Dual Cooler</b>
IVC	-130 ATDC	<b>-230 to -130 ATDC</b>
Bowl Geometry	Re-entrant	Re-entrant

1  
2  
3

Data reported here

<b>N</b>	<b>2050</b>
<b>BMEP(bar) tested</b>	<b>5</b> 2 - 7
<b>Combustion phasing CA50</b>	<b>2 , 7.5 , 12.5</b>

# Leveraging Technologies



## Combustion Efficiency

## Enablers

Parasitic Losses      Bottoming Cycles      Aftertreatment Management

Combustion Phasing  
Combustion Duration

Injection strategy (PCCI)  
High Injection Pressure  
Controls

Cylinder Heat Rejection

High Boost / Lean

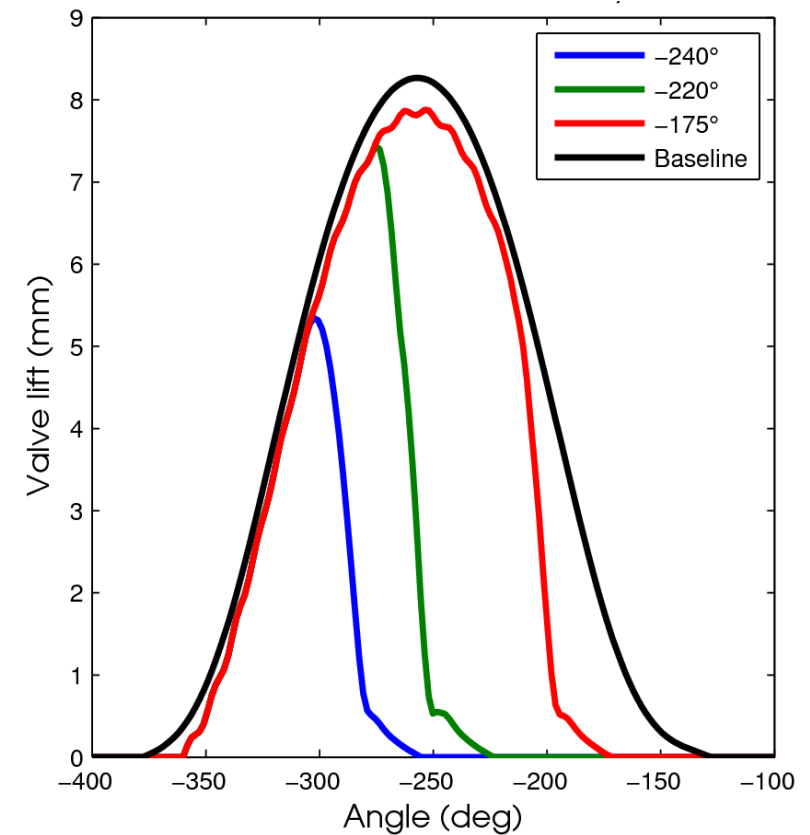
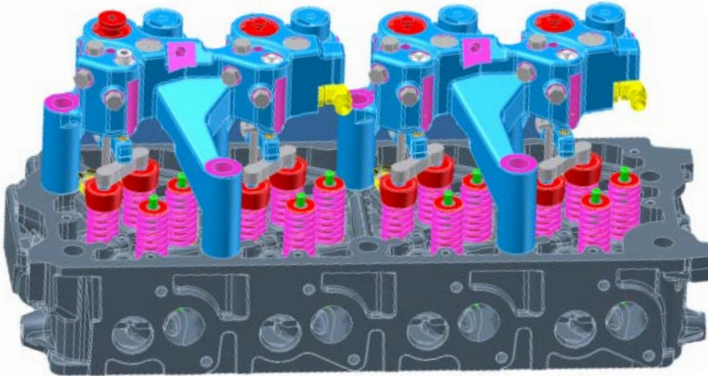
Efficient Turbocharging

Optimum CR



## Advantages of Electro-Hydraulic System:

- Simple and Robust
- Fine resolution for IVC
- Cylinder to cylinder adjustment
- Cycle to cycle adjustments
- Simple package over baseline valve train



## Other Advantages

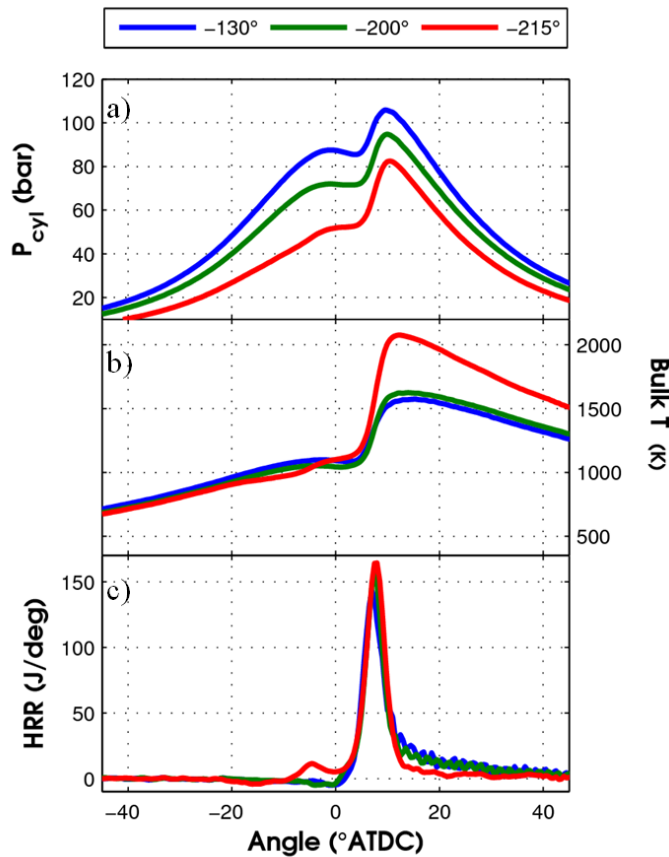
- Real world soot and BSFC reductions



# Thermodynamic Effects of VVA

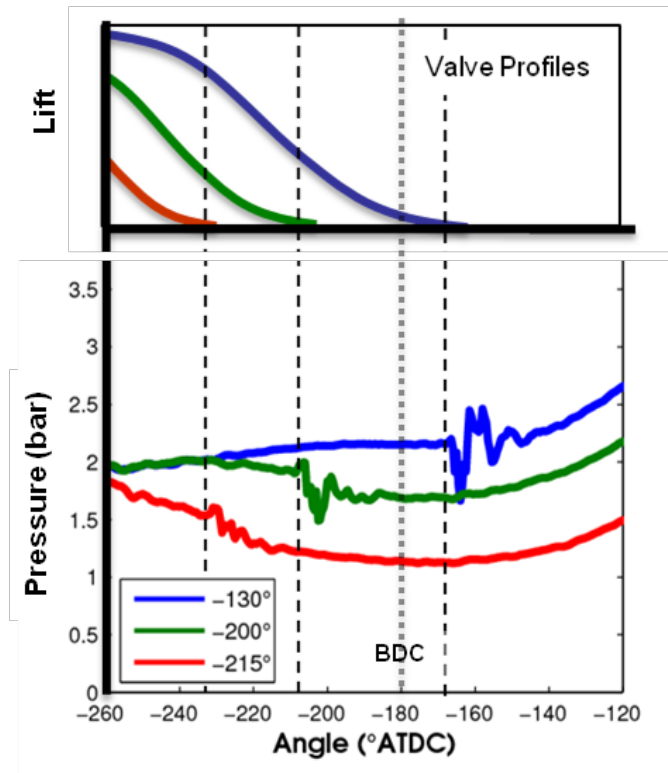
1

lowered effective CR



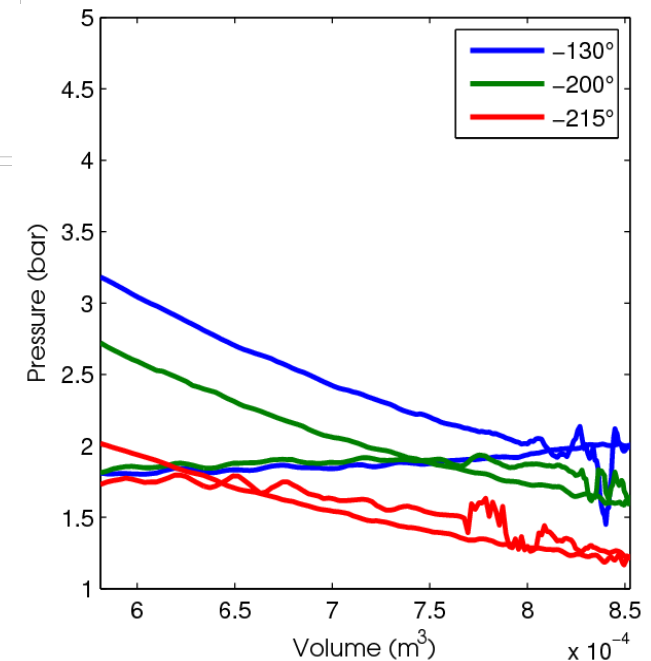
2

Early intake valve closing (EIVC)  
lowered in-cylinder pressure



3

EIVC produces a  
nearly isentropic  
expansion  
(reduced losses)



## Advancing Intake Valve Closing

1 Testing is carried out at a constant NO<sub>x</sub> (0.18g/bhp-hr) and constant phasing

2 BSFC is reduced ~ 5%

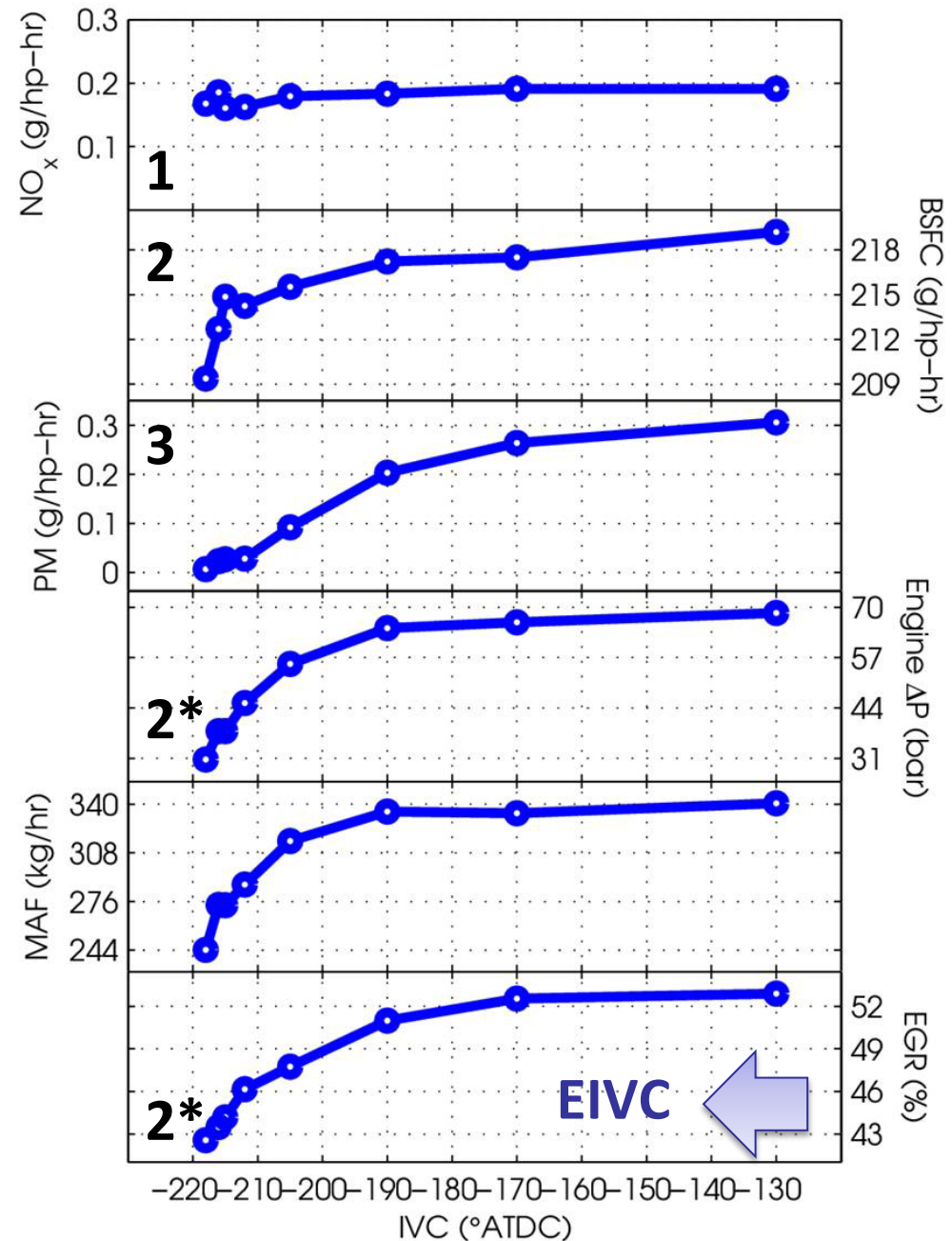
3 Soot is reduced ~95%

2\* { MAP ~ constant  
Boost-Back ~ reduced by 50%  
MAF ~ reduced by 30%  
EGR ~ reduced by 10 percent points

3\* { Increased ignition delay

Ref SAE 2010-01-1124

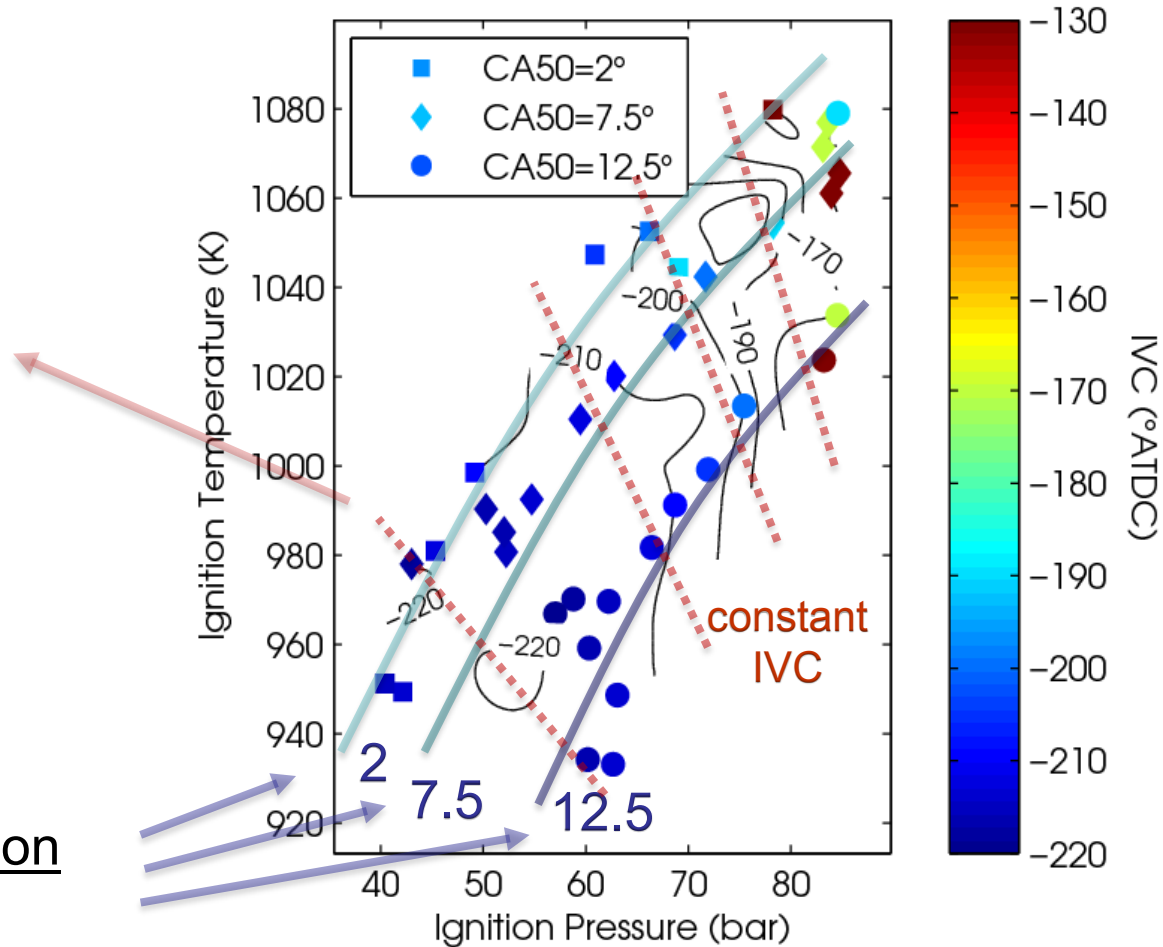
trend holds for CA50 = 2, 7.5, 12 deg



## Advancing Intake Valve Closing

Ignition temperatures drop as IVC lines are advanced

Data at three combustion phasing  
( 2 , 7.5 , 12 )



IVC sweeps  
baseline -130 °  
to -220 °



EIVC

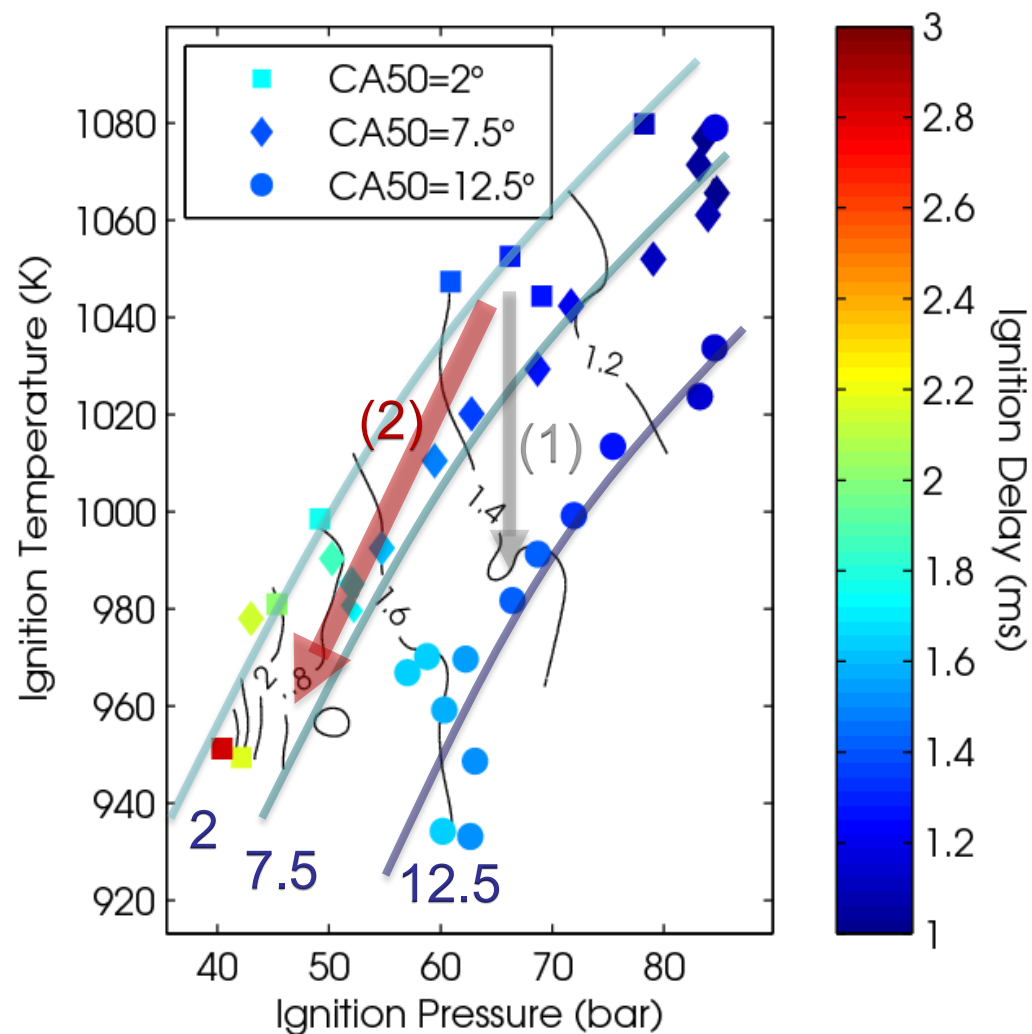
## Ignition Delay

(1) Small effect of combustion phasing

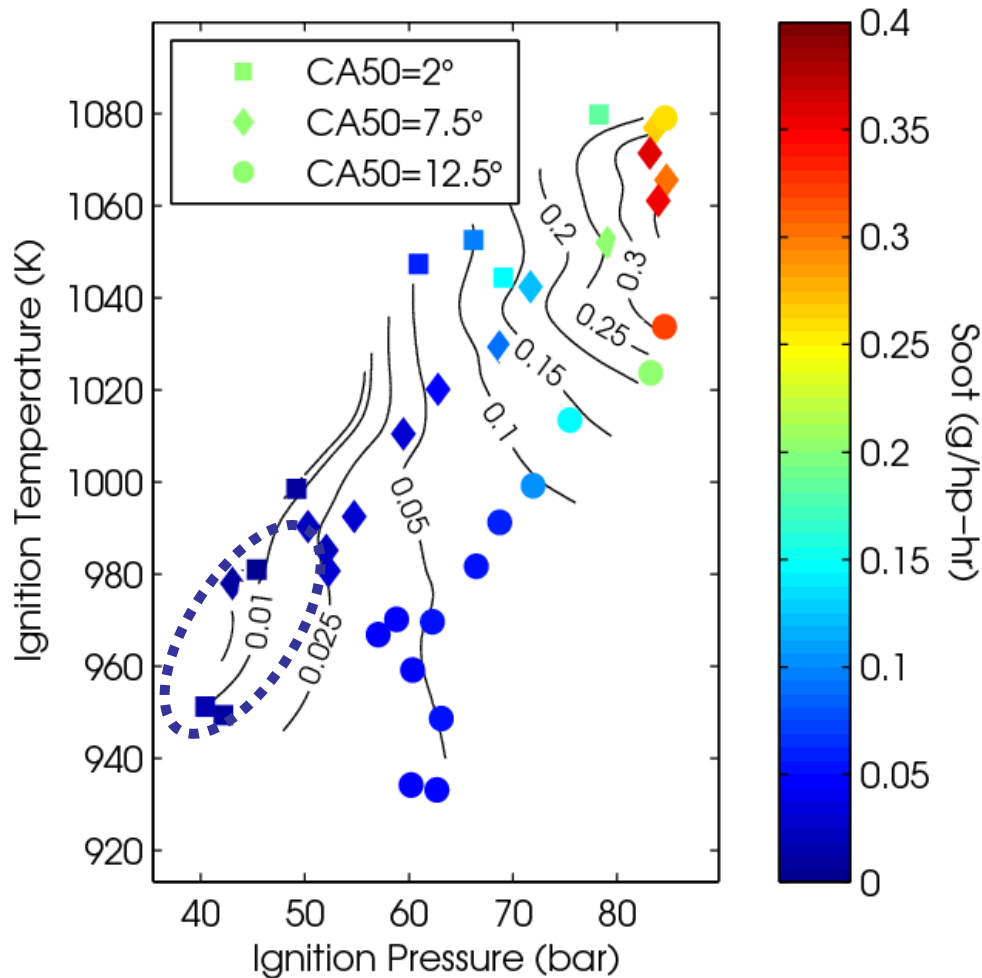
$\tau_{id} \sim 0.1\text{ms}$   
per  $10^\circ$  CA50

(2) Large effect of EIVC

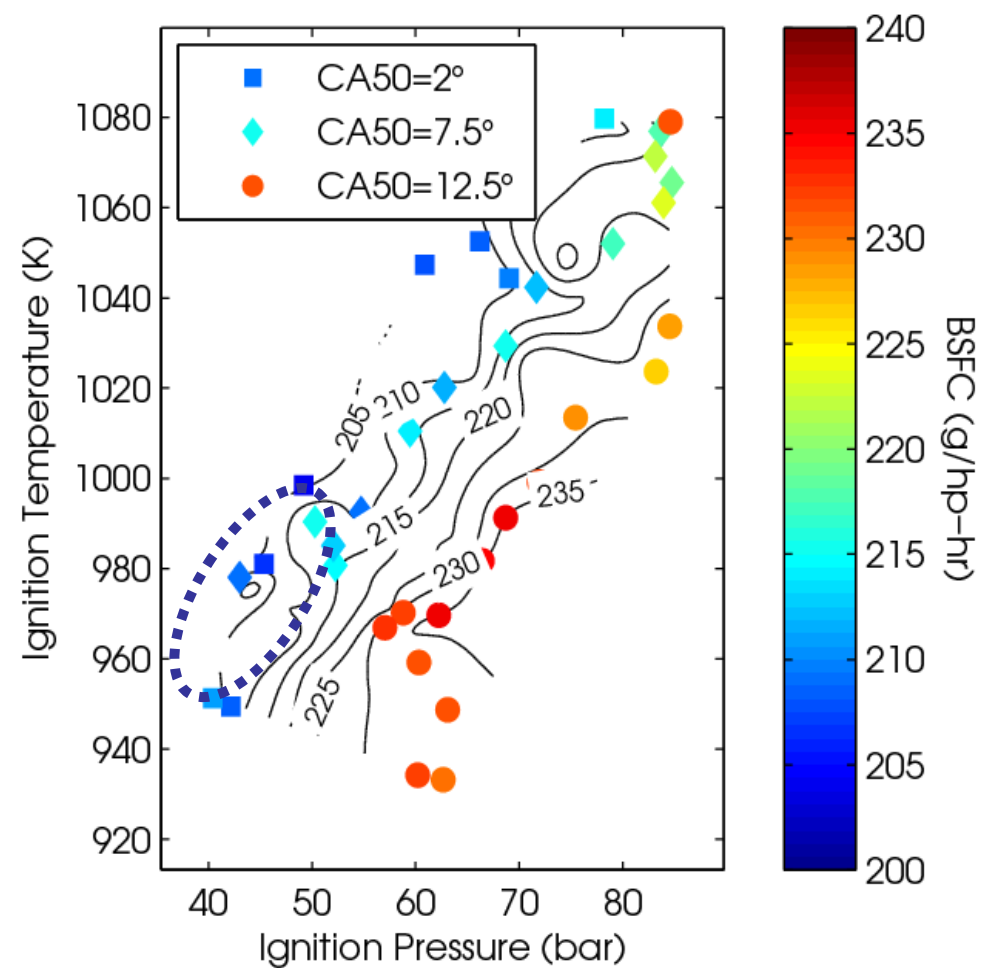
$\tau_{id} \sim 1\text{ms}$   
per  $20^\circ$  IVC



## Reduction in soot and impact on fuel consumption



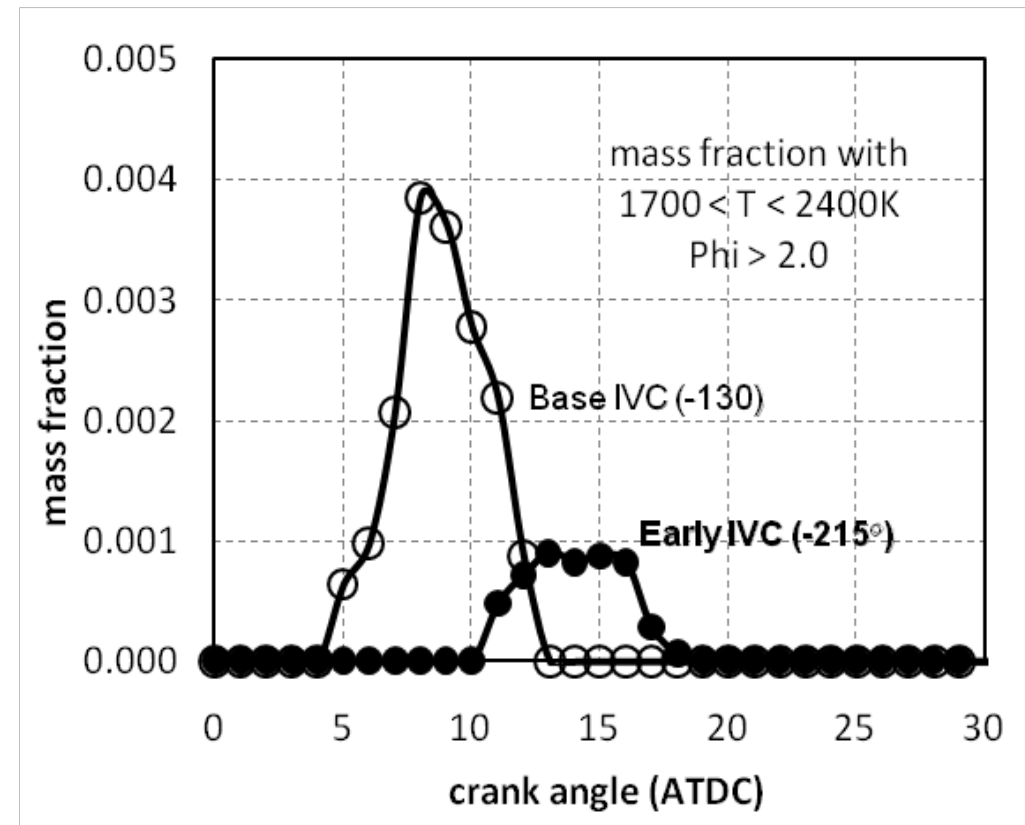
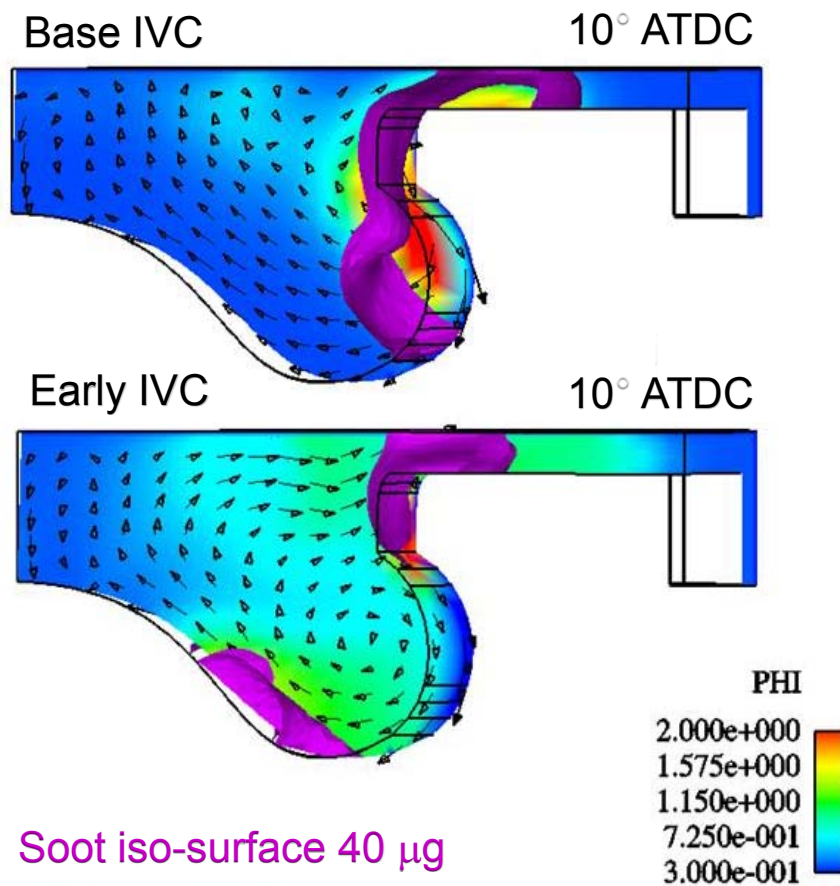
Very low soot

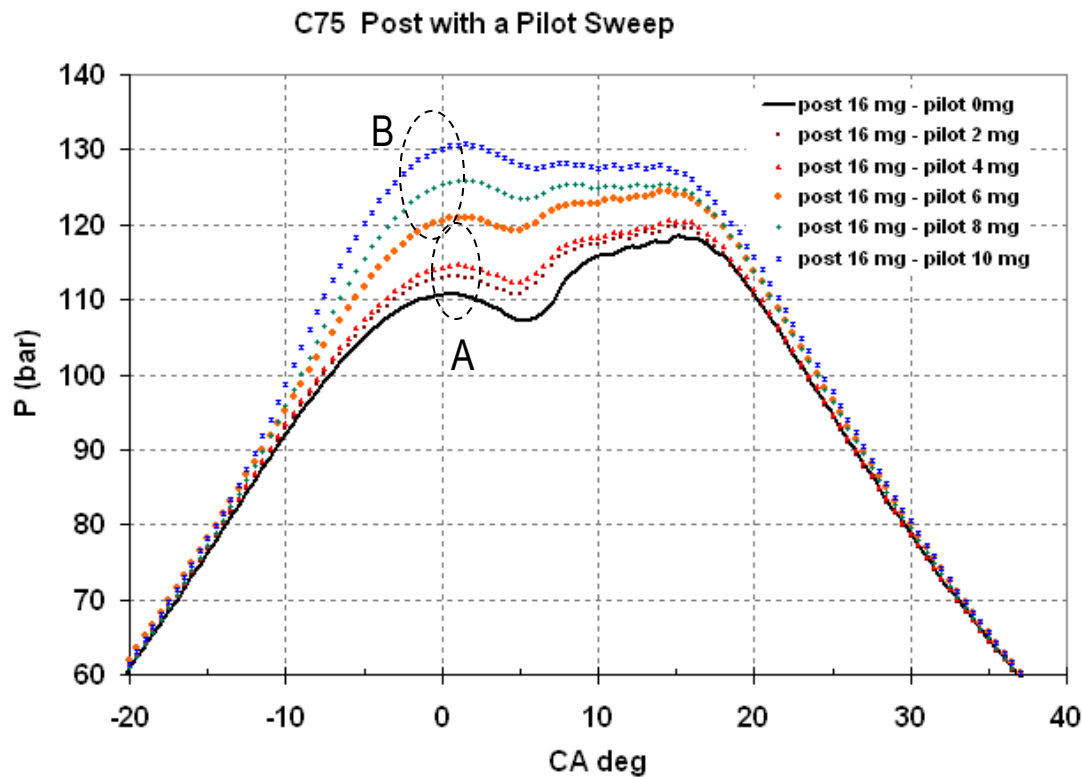


Simultaneous with reduction in BSFC

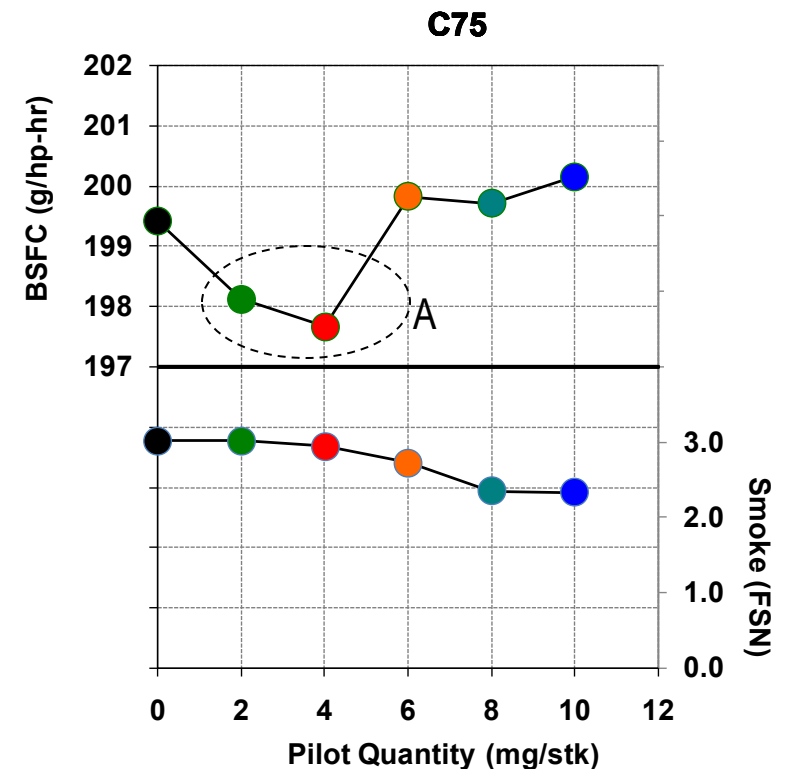
Soot reduction mechanisms with advanced IVC:

- \* better mixture characteristics
- \* Dependency on local equivalent ratio





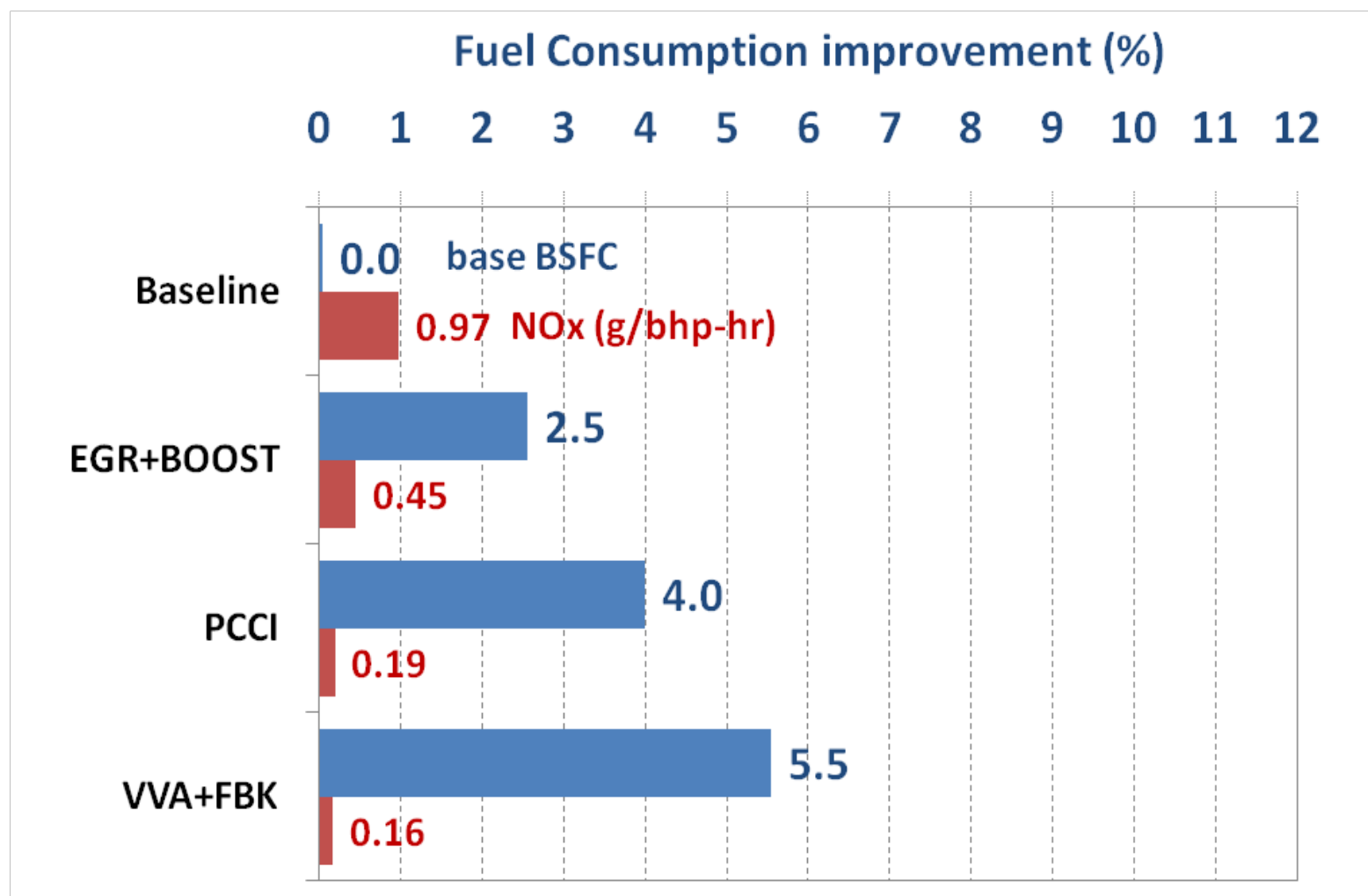
A BSFC improvement of 1-2% can be identified by PCCI or fuel pilot and harvested with accurate cylinder pressure control.



B Excess pilot deteriorates performance due to premature combustion.

1. Boost-EGR Control: optimized combustion phasing
2. PCCI – premixed fuel injection strategies

## 3. Application of Variable Valve Actuation and Combustion Feedback





## **A Variable Valve Actuation device was used on a Medium Duty V8 6.4L Diesel engine. The VVA adjusted:**

- the intake valve closing individually on each cylinder ,
- demonstrated expansion process did not contribute to pumping losses,
- improved uniformity across cylinders provided better soot and efficiency.

## **Tests at a constant engine out 0.2 g/bhp-hr NOx showed:**

- reduced soot emissions by 95% at loads up to 5 bar BMEP,
- lowered fuel consumption by 5%,
- reduced the amount of EGR required to dilute intake to a constant [O2].

## **Improved thermal efficiencies:**

- lower ignition temperatures and pressures enabled long ignition delays,
- reduced the differential pressure across the engine.

# Acknowledgements

Project Partners



CFD

Fuels

Enabling Technologies

Combustion Diagnostics

