

U.S. DEPARTMENT OF ENERGY BUILDING TECHNOLOGIES OFFICE

BTO Peer Review:

Grid Interactive Micro-Distributed Refrigerated Display Case

Ramin Faramarzi, Pl ramin.Faramarzi@nrel.gov

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Project Summary

OBJECTIVE, OUTCOME, & IMPACT

Develop a next-generation self-contained, refrigerated open vertical display case (OVDC) that leverages radiation and thermal energy storage (TES) to:

- Improve energy efficiency
- Provide demand flexibility
- Improve human comfort
- Improve environmentally friendliness of refrigeration systems

TEAM & PARTNERS

- The National Renewable Energy Lab
- Copeland Corporation
- Net Energy

Deployment and Commercialization Partners:

- Arneg USA
- Albertsons
- Commonwealth Edison Co
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STATS

Performance Period: 10/1/2021 – 3/31/2025 DOE Budget: \$2,527k Cost Share: \$720k Milestones:

Collect commercial refrigeration data Hybrid radiative and convective cooling modeling Model and design phase change material heat exchanger (PCM HX) Characterize PCM material Case performance and efficiency modeling (24 h) Characterize PCM HX prototype Optimize case design and TES controls for maximum efficiency Characterize hybrid system heat extraction effectiveness Characterize refrigerated case with hybrid cooling and TES (full-scale proof-of-concept) Disseminate project information and findings

Problem Statement: Supermarkets

- Supermarkets have the second highest EUIs in the commercial buildings sector:
 - Refrigeration accounts for roughly 50% of their electric energy.¹
 - OVDCs comprise nearly 50% of total case line-ups.

Major fuels intensity by principal building activity, 2012–2018 thousand British thermal units per square foot



Data source: U.S. Energy Information Administration, *Commercial Buildings Energy Consumption Survey* * Change is statistically significant at the 10% significance level. ** Change is statistically significant at the 5% significance level.

Note: <u>Building Type Definitions</u> on the CBECS web page provides more information about the principal building activities.

Problem Statement: OVDC

- Forced convection used to cool refrigerated products results in large mass exchange with the surrounded space:
 - Air infiltration accounts for **80% of cooling load.**
 - The spilled cold air adversely impacts human comfort.
 - Frost formation on evaporator restricts air flow and hampers heat transfer → degrades energy efficiency.
 - Highly variable and non-uniform product temperature between shelves (up to 10°F).
- **Refrigeration heat** rejected into the sales floor **cannot be reclaimed** by heating systems.
- **Inability** to reliably **participate** in **demand response** (DR) events and load shaving/shifting strategies.





Alignment and Impact

- The successful completion of this project will:
- Accelerate the deployment of a cost-effective distributed energy resource technology.
- Improve energy efficiency and reduce site/source greenhouse gas emissions.
- Pave the path for decarbonizing retail food market segment and enhance resiliency.

Potential Decarbonization Impacts (approximate estimates)

• Annual electric energy of 9.72 TWh (25% of national refrigeration energy consumption) and natural gas savings of 320 million therms.

Potential Grid-Edge Impacts (approximate estimates)

- Permanent electric peak demand reduction of 1.54 GW
- DR and load shaving/shifting to enhance grid integrity (2 h, 1.9 GWh load curtailment).

Integrated Research Approach



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Novel Features:

- 1. Complete removal of air curtain system replaced with hybrid radiant and convective cooling using R290.
- 2. Thermal energy storage (TES).
- 3. Water cooled integrated with building space and water heating.





R&D Targets

Energy Efficiency: 30% Improvement.

- by **reducing infiltration** load, frost formation, defrost cycles, and post-defrost pull-down loads ٠ with radiant cooling and minimizing forced convection by air curtain removal.
- by integrating water-cooled condenser with the building's heating systems to reduce ٠ temperature lift and recover waste heat.

Transform Grid Edge through Demand Flexibility: 80% peak kW reduction of display case over 2 hours.

by adding **TES**

Affordability: 3-year payback period.

by **reducing operational costs** with energy savings and DR programs. ۰

Occupant Comfort: 50% lower infiltration.

by **reducing** cold **air spillage** in shopping aisles.

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Thermal Energy Storage Component

- The thermal energy storage consists of a Phase Change Material (PCM) composite of 10% graphite and 90% PCM.
- The PCM composite is integrated with aluminum microchannels to form a **PCM heat exchanger** with both refrigerant and glycol circuits.
- The PCM heat exchanger was designed using a finite difference model to achieve 2-hour 80% load shifting.



Graphite/PCM composite

Comparison of PCM to ice

Property	Composite PCM	lce/water	Units				
Transition Temperature (\mathbf{T}_{t})	-11	0	С				
Energy density	60	85	kWh/m ³				
Conductivity	8	0.6	W/m-K				
Form in liquid state	Shape-stable	Flows	-				



Composite PCM heat exchanger

Aluminum microchannels

Thermal Energy Storage – Bench-scale Prototype

Numerical model prediction matches experiments for bench-scale prototype (0.2 kWh). This model was used to design the full-scale prototype (\sim 8 kWh_{th})

 25 cycles of bench-scale prototype (0.2 kWh) show stable operation (above graph)

Baseline/Bench Scale Experimentation

- Goal: Increase confidence in product temperature maintenance capability
- Laboratory characterization of a typical 8-ft long, 5-deck, selfcontained OVDC with a water-cooled condenser to calibrate the thermo-fluid model
- **117** channels of data collected every 1s over 24 hrs monitoring:
 - Temperatures: Air, refrigerant, water, product
 - Pressure: Refrigerant Suction & Discharge, Condenser water
 - Power: Total case and components
 - Mass Flow: Condenser water
 - Condensate Mass measured to calculate latent load

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Baseline System

Baseline Display Case Cabinet: 8ft, 5-deck 13 ft³ gross volume

The Baseline refrigerated case is equipped with:

- water-cooled condensing unit
- advanced controls
- variable speed compressor
- electronic expansion valve

Electronic Expansion Valve

Advanced Controls

Liquid-cooled Condensing Unit with Variable Speed Compressor

Experimentation Approach

- ASHRAE 72 and AHRI 1200 Standards were used to maintain consistent conditions between tests
- For the bench-scale validation, a chiller supplied the panels' coils with silicone coolant at controlled temperature and flow rate.

Baseline case was modified with:

- 1. Complete elimination of the air curtain
- 2. Custom-fit radiant panels,
- 3. Variable-speed evaporator fan motors/controller
- 4. Back panel with modified perforations

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Convergence of Resistive Model with CFD and Experimental Data

Average product temperatures from EES, CFD, and experimental measurements:

	EES (model approach 1)	CFD (model approach 2)	Experimental Measures
Baseline system	38.4°F	38.2°F	38.45°F
Hybrid system (bench-scale)	39.2°F	39.1°F	39.01°F

Conclusions / Accomplishments

- Convergence of resistive model with CFD and experimentation data increased confidence in modeling platform, which will be leveraged to design the POC
- The hybrid cooling system:
 - Complied with FDA product temp targets. (39.0°F mean product temp at 900 RPM evaporator fan speed)
 - Reduced infiltration load by 96% (at 900 RPM evaporator fan speed) based on mass of condensate data)

- Use validated thermo-fluid model to identify further strategies for minimizing the cooling load of the case & developing specs (in progress)
- Fabricate a hybrid proof-of-concept prototype fixture (including TES and advanced controls integration) (in progress)
- Ascertain the energy efficiency benefits of the hybrid proof-of-concept in a laboratory setting
- Assess the impacts of load flexibility strategies by leveraging the TES system in a laboratory setting

*Beyond Current Scope:

- Field demo at Albertson's site
- Work with ComEd to include the technology in their incentive portfolio

Thank you

NREL, Copeland, Net Energy, Arneg, Albertsons, and ComEd Ramin Faramarzi, PE ramin.Faramarzi@nrel.gov WBS # 3.2.2.127

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Reference Slides

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Project Execution

Task		Subtask	Lead	Q1 22	Q2 22	Q3 22	Q4 22	Q1 23	Q2 23	Q3 23	Q4 23	Q1 24	Q2 24	Q3 24	Q4 24	Q1 25	Q2 25
			Budget Period		BI	21				В	P2					BF	23
0.1 Proj	iect Manag	ement	Emerson/NREL														
1.2 Lite	rature Surv	ey and TAC Formation															
	1.2.1	Collect data on commercial refrigeration	Emerson		🔺 -												
	1.2.2	TAC formation	Emerson/NREL														
1.3 Hyb	rid Display	/ Case Thermal Analysis															
	1.3.1	Model case heat removal strategies and product temperature maintenance															
		1.3.1.a Radiative cooling	NREL														
		1.3.1.b Low airflow convective cooling and CFD modeling	NREL										🔶 Go/N	o-Go Decis	ion Point		
		1.3.1.c Hybrid radiative and convective cooling	NREL										🔺 Miles	tone			
		1.3.1.d Refrigeration cycle design	NREL														
		1.3.1.e Heat rejection modeling	NREL														
		1.3.1.f Total case performance and energy efficiency modeling over 24 hrs	NREL							•							
	1.3.2	Surface condensation risk mitigation research	NREL														
	1.3.3	Customer discovery and market analysis	NETenergy														
	1.3.4	Technology assessment with building-level analysis	NREL														
1.4 TES	Design and	d System Characterization															
	1.4.1	Model TES															
		1.4.1a Model and design PCM HXs	NREL														
	1.4.2	PCM creation and characterization															
		1.4.2.a Synthesize PCM material	NETenergy														
		1.4.2.b Characterize PCM material	NETenergy/NREL														
	1.4.3	PCM HX creation and characterization															
		1.4.3.a Construct PCM HX prototype (bench-scale)	NETenergy														
		1.4.3.b Characterize PCM HX prototype (bench-scale)	NREL/Copeland														
		1.4.3.c Construct PCM HX prototype (full-scale)	NETenergy														
		1.4.3.d Characterize PCM HX prototype (full-scale)	NREL												_		
1.5 Disp	olay Case D	lesign and Specifications															
	1.5.1	Model and optimize the hybrid refrigerated case with TES															
		1.5.1.a Develop a model of integrated system	NREL														
		1.5.1.b Optimize case design and TES controls for maximum efficiency	NREL														
		1.5.1.c Develop cost estimate for solution based on optimized Design	Copeland														
	1.5.2	Determine mechanical component sizing and specifications	NREL														
1.6 Disp	olay Case P	roof of Concept (POC) Characterization and Fabrication															
	1.6.1	Construct hybrid case prototype (bench-scale)	NREL														
	1.6.2	Mechanical components integration (bench-scale)	NREL														
	1.6.3	Heat rejection component integration (bench-scale)	NREL														
	1.6.4	Characterize hybrid system heat extraction effectiveness (bench-scale)	NREL											•			
	1.6.5	Construct hybrid case prototype (full-scale)	Copeland														
	1.6.6	Mechanical components integration (full-scale)	Copeland														
	1.6.7	Heat rejection component integration (full-scale)	Copeland														
	1.6.8	Characterize hybrid system heat extraction effectiveness (full-scale)	NREL/Copeland											▲			
1.7 Disp	olay Case S	ystem Integration															
	1.7.1	TES Integration	Copeland														
	1.7.2	Controls Integration	Copeland														
1.8 Disp	olay Case P	OC Performance Assessment	Arneg														
	1.8.1	Characterize refrigerated case with hybrid cooling and PCM HX	Arneg													A	
1.9 Info	rmation D	issemination	NREL/Copeland														

COPELAND "

Robert Nash Jr. Senior Lead Engineer

Suresh Shivashankar Director – Solutions and Systems Engineering

Ramin Faramarzi Principal Engineer

Eric Kozubal

Senior Mechanical Engineer

Transformina ENERG

Dr. Jason Woods Sr. Research Engineer

Alex Bulk Research Engineer

Ravi Kishore Research Engineer

Emily Sherburne Project Controller

Khanh Nguyen Cu Research Engineer

Dr. Yana Galazutdinova Postdoctoral Researcher

Dr. Monica Cook VP business technology

Modeling Approach

Modeling Approach Based on Experimental Data

- Thermo-fluid heat extraction 1. Resistive Network modeling of the OVDC
 - Validation using 3-D CFD modeling
 - Experimentation Data
- Refrigeration system modeling 2.
- 3. Thermal energy storage system modeling
- Integrated system model 4.
- Whole-building EnergyPlus[®] 5. hourly simulation based on an actual Albertsons store

Actual Albertsons Store

EnergyPlus Building Model

Modeling Results Informing Hybrid Case Design

- Both CFD and EES models closely aligned with measured data: ٠
 - Informed back panel air flow at 900 RPM and custom perforation for optimally-distributed air flow ٠
 - Informed mean panel surface temperature of 32°F ٠

Mass Flow Distribution at BP & Canopy

Hybrid Radiant + Convective Cooling Experimental Results

• Varied fan speeds from 500 RPM (fan motor minimum) to 1600 RPM (baseline airflow)

• 900 RPM provided optimum BP airflow while maintaining mean product temp & minimizing

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• Hybrid Radiant Cooling • Baseline w Air Curtain

Thermo-Fluid Modeling

- 1st approach (thermal resistive network model) used Engineering Equation Solver (EES): considers conduction, convection, and radiation heat transfer modes and predicts temperatures
- 2nd independent approach leveraged 3D CFD modeling.

*models reflect the geometry of an actual OVDC.

Experimentation Setup: Bench-Scale

Case Inside Environmental Chamber

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Radiant Cooling Loop

- A: port into/out of chamber
- B: condensate drum and scale
- C: flowmeters, pressure gauges, and thermocouples
- D: chiller serving radiant panels
- E: control box for the chiller, valves, and flood switches to detect leaks.