## **Energy Storing Efficient HVAC**

A <u>Technology Commercialization Fund Project</u>



**National Renewable Energy Laboratory** 

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**Blue Frontier** 

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## **Project Summary**

## Objective and outcome

Development of a pre-Pilot HVAC system based on NREL's technology that achieves 40% energy savings over traditional AC and has inherent 6+ hours of energy storage. Evaporative Liquid Desiccant Air Conditioner (eLD-AC) paired with an Electrically Driven Desiccant Regenerator (EDDR).



NREL is the project lead providing technical expertise in materials and system development.

Blue Frontier AC is the commercialization partner and will market the technology through an HVAC-as-a-Service business model.



### **Stats**

Performance Period: Jan 2022 - March 2024

DOE budget: \$600k, Cost Share: \$600k

Milestone 1: Desiccant property characterization and system

model development

Milestone 2: Journal article submission describing the EDDR

and comparing model and prototype performance.

<u>Milestone 3</u>: Summary report describing the outcome of each research task and analysis of prototype performance.

## Why Conventional A/C is a Problem

## Carbon & Energy Intensive<sup>1</sup>

- 20% of global electricity in buildings
- AC contributes 4% of global CO2 emissions 53% attributed to humidity loads

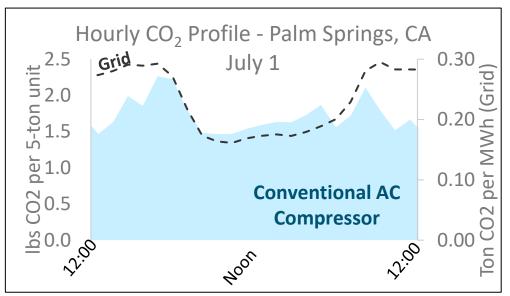
### Not Flexible:

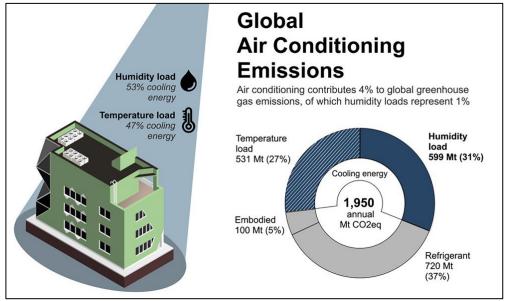
On-demand service without storage

## No incentive to install efficient equipment

- Landlords pay capital costs
- Renters pay electricity bills

1. Woods, Jason, Nelson James, Eric Kozubal, Eric Bonnema, Kristin Brief, Liz Voeller, and Jessy Revest. 2022. "Humidity's impact on greenhouse gas emissions from air conditioning." *Joule* 6(4): 726-741.





## Alignment and Impact: Develop Commercial HVAC (RTU)



### • 80% CO<sub>2</sub> Savings

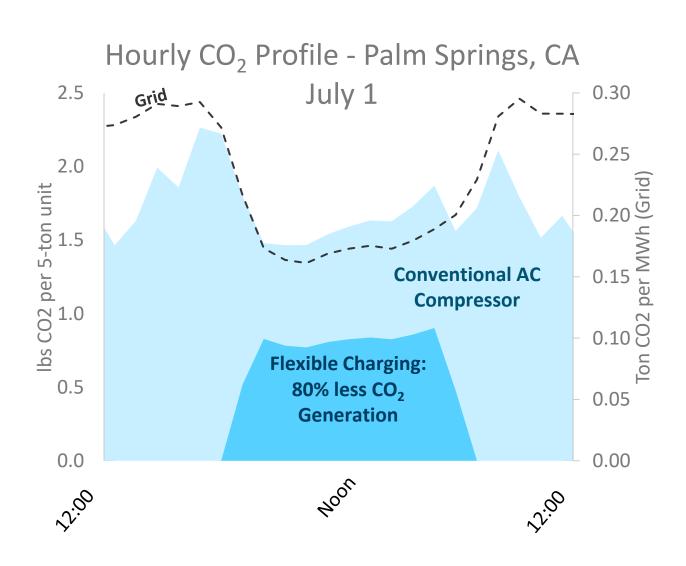
- Charging system draws power during low carbon periods
- Annualized energy savings
  - 40% in humid climates
  - 80% in dry / hot climates

### Flexible and Equitable

- Lifetime energy cost savings >\$8,000 / RT
- Removes capital cost as a barrier
- Reduces building electrification costs
- Low energy cost for all occupants (renters or owners)

### Success Criteria

- Modelled & prototype performance shows 40% annualized energy savings in zone 2A
- Thermal energy storage 6+ hours capable
  - 330 J/g or 123 kWh/m<sup>3</sup>



## Approach – Vapor compression vs liquid desiccant cooling

DX Coil

Reheat Coil

## Liquid desiccant cooling

- Open-Cycle Process
  - Desiccant dehumidifier + indirect evaporative cooling
  - Theoretical efficiency is 2X– 10X higher than ideal vapor compression

## **Vapor compression**

- Overcool → reheat
  - Energy intensive because cooling capacity and lift is higher than necessary

Comparison of theoretical and real system humidity removal efficiency<sup>1</sup> efficiency (kg/kWh) Moisture removal Carnot limi Inlet air humidity ratio (g/kg) ure removal efficiency for existing technologies compared with the minimum separation Graph from Woods et al. 2022, Humidity's impact on greenhouse gas emissions from air conditioning

## How it works – 3 Sub-systems

## 1. Electric Regenerator: EDDR

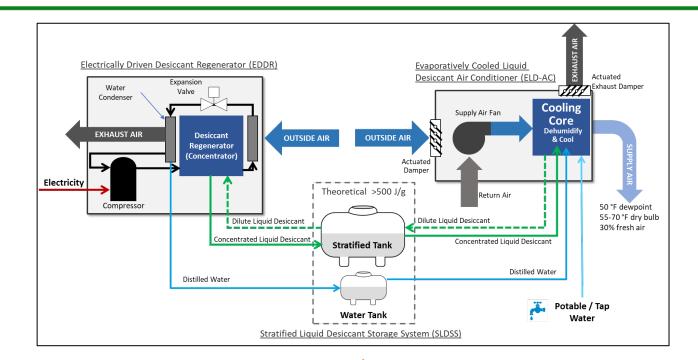
 Regenerates diluted desiccant into strong desiccant + distilled water

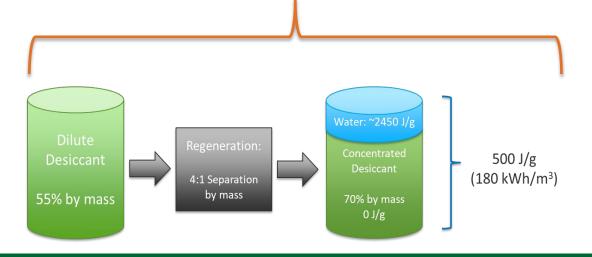
## 2. Thermal Storage: SLDSS

 Stratified solution tank creates halocline of weak and strong desiccant.

### 3. Conditioner: eLD-AC

- Separate sensible and latent cooling (SSLC)
- Open cycle cooling process:
- Uses strong desiccant to remove moisture
- Evaporates water (refrigerant) to remove heat

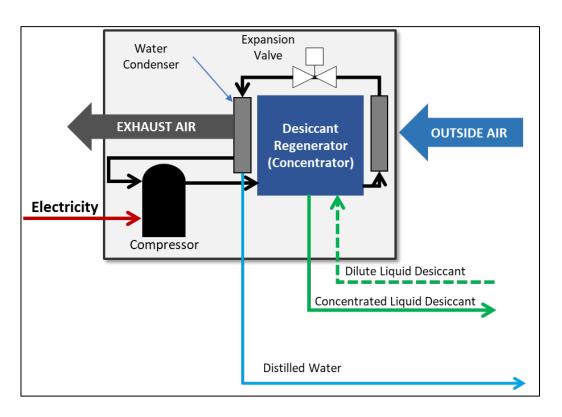




## **How it works – Electrical regeneration / Air Conditioning**

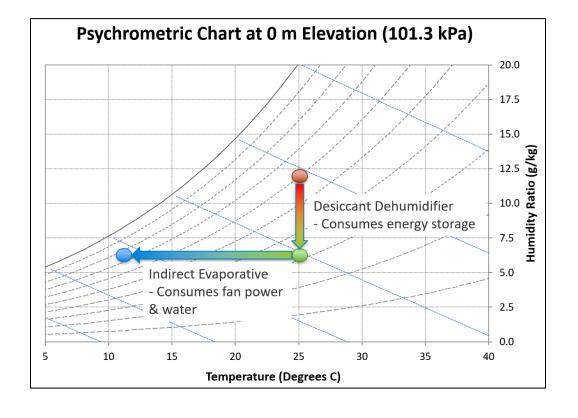
### Electrically Driven Desiccant Regenerator

- Thermal regeneration using heat pump technology
  - Heat pump thermal COP >6
  - Desiccant moisture removal efficiency > 5 kg/kWh
- US Patent<sup>4</sup>



### Separate Sensible & Latent Conditioning

- Technology initially developed in 2010-2012<sup>3</sup>
- US Patent<sup>5</sup>



## **Project Overview**

Wells Fargo  $IN^2 - 2020-2021$ 

## TCF Project: 2022 - 2024

Pilot Stage

### Prior work → TCF project scope:

- Increase regenerator subsystem efficiency
- Increase sensible cooling performance
- Further characterize desiccant properties
- Optimize energy storage





Proof of concept in HVAC Laboratory

Task 1 Thermodynamic Modeling	Component / Subsystems	$\rangle$	Dynamic Model-based design

<u>Task 2</u> HMX Design Prototype HMX Design & Experiments

Physics Validation

Task 3

Desiccant Characterization 1. Research

2. Methods

3. Measure

Formulate Results

## Task 4: Gen 2.0 Engineering Prototype

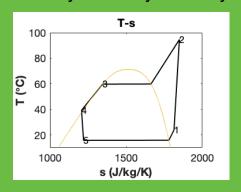
- Manufacture prototype
- Laboratory Evaluation
- Annualized energy simulations
- Publication

## Model-based Design + HMX Experiments → Gen 2.0 Design

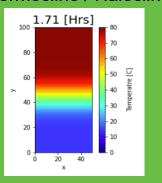
### **Model attributes:**

- Finite difference exchangers<sup>2</sup>
- Dynamic storage tank
- Drive-cycle analysis
  - Charge, Discharge tank
  - Cooling schedule

### Thermodynamic Cycle Analysis

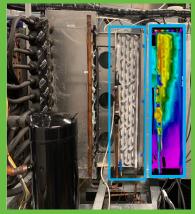


# Tank: Thermocline / Halocline

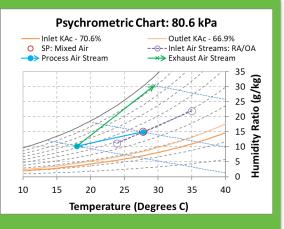


### **Experiments:**

- Model inputs to real-world phenomena
- Model result verification

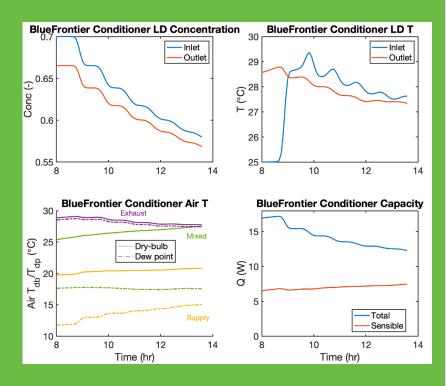


Thermal imagery



Conditioner

# Drive Cycle Results → Annualized Energy



<sup>2.</sup> Woods, Jason, Eric Kozubal. 2013. "A desiccant-enhanced evaporative air conditioner: Numerical model and experiments." *Energy Conversion and Management 6:* 208-220.

## **Desiccant Material Characterization**

Property	Ranges	Instrument
Vapor Pressure vs T, C	T = 0 °C to 120 °C C = 30% to 70%	Eravap
Water diffusivity - D_AB	T = 0 °C to 120 °C C = 30% to 70%	Nuclear Magnetic Resonance (NMR)
Specific Heat - Cp	T = -40 °C to 120 °C C = 0% to 70%	DSC
Thermal Conductivity - k	T = -40 °C to 120 °C C = 0% to 70%	DTC and Flash diffusivity
Enthalpy of dilution	T = 20 °C to 90 °C C = 10% to 70%	Titration calorimeter or DSC + humidity
Density - rho	T = -40 °C to 120 °C C = 0% to 70%	Density Meter
Viscosity	T = -30 °C to 120 °C C = 0% to 70%	Rheometer
Liquid / Hydrate Phase Boundary	T = -40 °C to 120 °C C = dependent variable	DSC
Liquid / Ice Phase Boundary	T = -40 °C to 0 °C C = dependent variable	DSC
refractive index	T = 0 °C to 80 °C C = 0% to 70%	Atago Refractometer

Lack of data on low-cost / low-corrosive desiccants motivates construction of property library using characterization methods available at NREL

## **Vapor Pressure Analyzer (eravap)**

 Piston expansion measures vapor pressure



### **Differential scanning calorimetry (DSC)**

Phase-change enthalpy (heat of fusion) and transition temperature



### **Nuclear Magnetic Resonance (NMR)**

 Determines bonding & local conditions of molecules using magnetic fields



### **Guarded Heat Flow Meter (DTC)**

 Applies one-dimensional heat flow through samples



### **Parallel-Plate Rheometer**

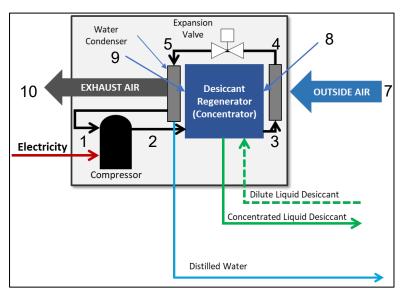
 Rotational force applied to fluids between plates measures viscosity

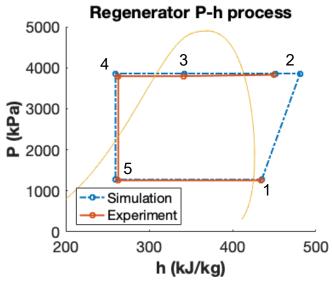


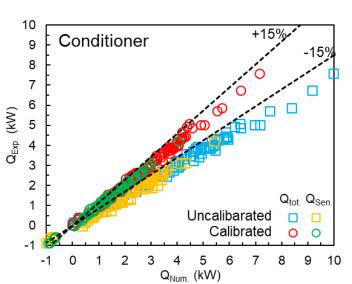
## **Task 1: Completed work**

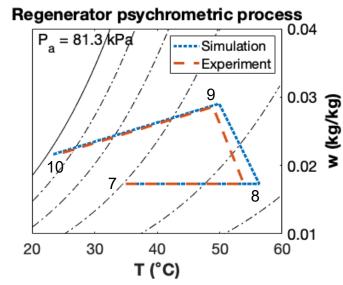
### **Generated models:**

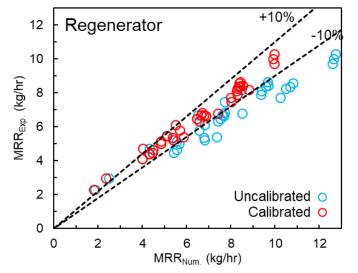
- Finite difference:
  - Conditioner and Regenerator HMXs
  - Dynamic stratified tank
- Dynamic system model:
  - Conditioner mode (cooling)
  - Regenerate mode (tank re-charge)
  - Hybrid mode (cool and regenerate)









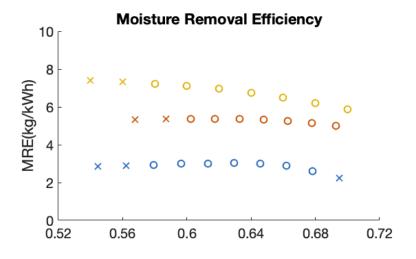


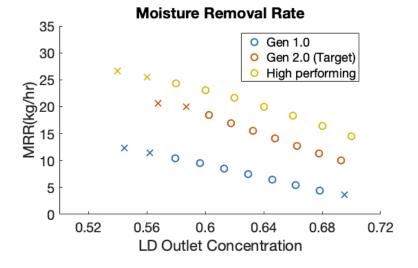
## Task 1: Next Steps & Model based design

- Dynamic Modeling
  - Model annualized performance of target system (Gen 2.0)
  - Create representative cooling loads using EnergyPlus
  - Representative cooling days for each month

- Generate and compare annualized energy use and efficiency
  - Gen 2.0 (Target) engineering prototype design
  - Baseline

## Example: Target EDDR performance





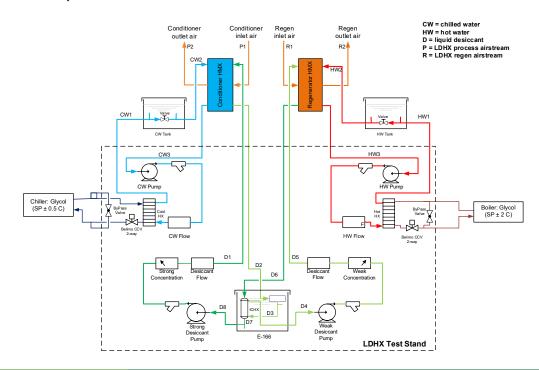
## Task 2: Improvement of regenerator heat and mass exchanger

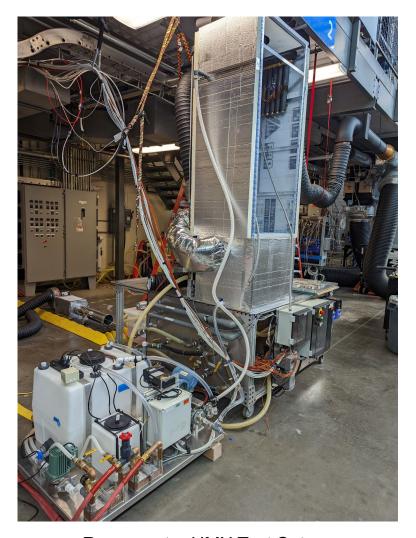
### **Completed Work**

- Generate regenerator prototype samples that achieve counter-flow
- Completed laboratory setup (NREL HVAC Lab)

### **Next steps:**

- Perform experiments
- Feed performance into Gen 2.0 models for model-based design task





Regenerator HMX Test Setup NREL's HVAC Laboratory

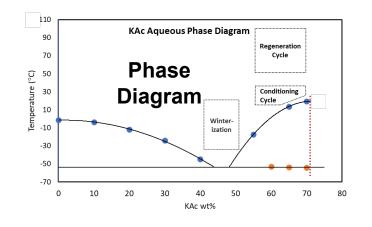
## **Task 3: Desiccant Material Characterization**

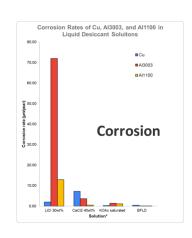
### Many KAc properties have been well characterized

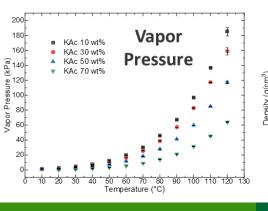
- Data fits well with established theories
- Some require improved resolution at low T/high KAc
  - Limitations of equipment

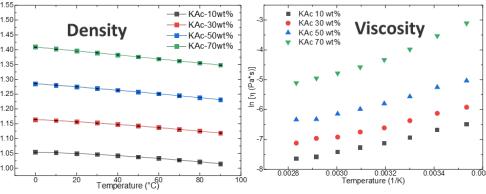
### Future Work

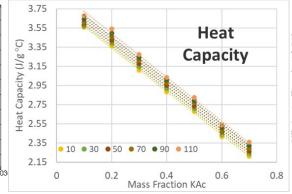
- Refine eutectic properties on phase diagram
- Expand on diffusion coefficient
- Expand on thermal conductivity
- Measure refractive index
- Measure enthalpy of dilution

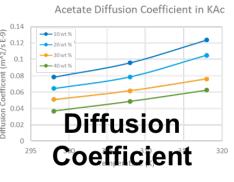












## **Task 3: Potassium Acetate vs. Lithium Chloride**

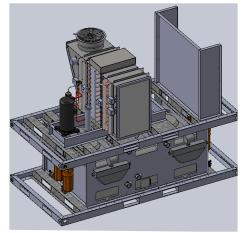
Data at 20 °C	KAc	LiCl					
Salt concentration (wt%)	70%	36%					
Vapor Pressure (kPa)	0.61						
Relative Vapor Pressure (%)	26.1						
Water Diffusivity (m²/s x 10-9)	0.16	1.6					
Corrosion Rate, Al 3003 (μ/yr)	0.06	72					
Density (g/cm³)	1.396	1.222					
Viscosity (mPa*s)	29.20	6.35					
Storage Density (kWh <sub>thermal</sub> /kg)	0.139	0.238					
Storage Density (kWh <sub>electric</sub> /kg)	0.028	0.048					
Storage cost (\$/kWh <sub>thermal</sub> )	\$7.20	\$210					
Storage cost (\$/kWh <sub>electric</sub> )	\$36.00	\$1,050					

## Task 4: Engineering Prototype (Gen 2.0) – Not started

### **Future work:**

- Construct prototype
- Laboratory characterization
- Model calibration
- Analysis of performance
  - Model calibration
  - Re-do energy comparison
- Published results

Below: Generation 1.0 Design and Prototype











## **Thank You**

National Renewable Energy Laboratory

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WBS 3.9.1.60 – TCF – Energy Storing Efficient HVAC

## **REFERENCE SLIDES**

## References

- 1. Woods, Jason, Nelson James, Eric Kozubal, Eric Bonnema, Kristin Brief, Liz Voeller, and Jessy Revest. 2022. *Humidity's impact on greenhouse gas emissions from air conditioning*. Joule 6(4): 726-741.
- 2. Woods, Jason, Eric Kozubal. 2013. *A desiccant-enhanced evaporative air conditioner: Numerical model and experiments.* Energy Conversion and Management 6: 208-220.
- 3. Kozubal, Eric; Woods, Jason; Judkoff, Ron. 2012. *Development and Analysis of Desiccant Enhanced Evaporative Air Conditioner Prototype*. NREL/TP-5500-54755
- 4. Kozubal, Eric; Woods, Jason. 2021. *Heat-Pump Driven Desiccant Regeneration*. U.S. Patent No. 11,029,045 B2.
- 5. Kozubal, Eric. 2017. *Indirect Evaporative Cooler Using Membrane-Contained, Liquid Desiccant for Dehumidification*. U.S. Patent No. 2017/0074530 A1.

## **Project Execution**

		<b>FY2022</b> \$245,000 \$181,477		<b>FY2023</b> \$300,773				<b>FY2024*</b> \$123,602				
Planned Budget												
Spent Budget (\$307k to date)				\$125,746		\$0						
	Q1	Q2	Q3	Q4	Q1	Q2	Q3	Q4	Q1	Q2	Q3	Q4
Past Work												
Q3-2022: Heat and mass exchanger code transioned from EES to Matlab												
Q4-2022: Preliminary EDDR subsystem results and future design path												
Q1-2023a: Desiccant Property Measurements												
Q1-2023b: System model developed showing transient example results												
Q2-2023: Journal article draft: Desiccant material properties												
Current / Future Work												
Q3-2023: Report describing advancements in heat and mass exchanger												
Go / No Go: System efficiency design pathway to >40% over standardard equipment												
Q4-2023: Laboratory scale system construction and test plan												
Q1-2024: Laboratory scale system evaluated in HVAC laboratory												
Q2-2024a: Journal article draft: desiccant regenerator system and HVAC system results											•	
Q2-2024b: Final summary report to DOE summarizing outcomes of each research task												
*FY2024 budget includes carry-over funds not spent in FY2022												

## **Team**





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- CONTROLS & INSTRUMENTATION
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