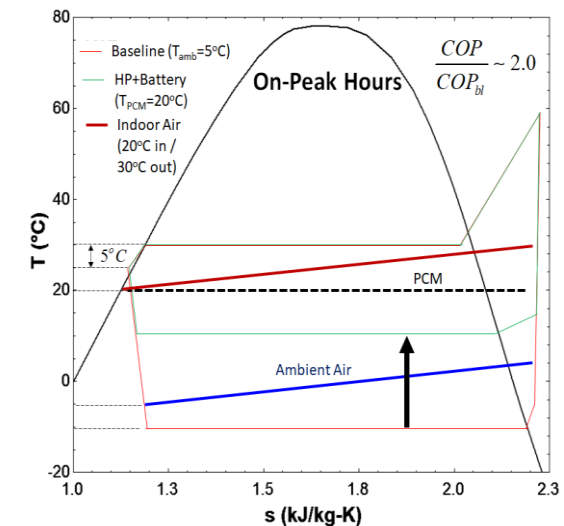
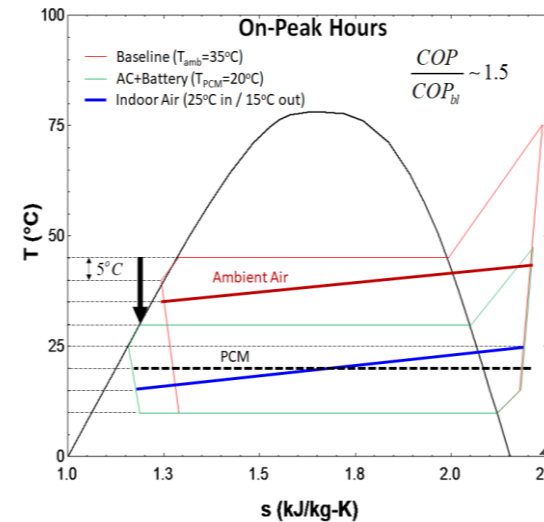
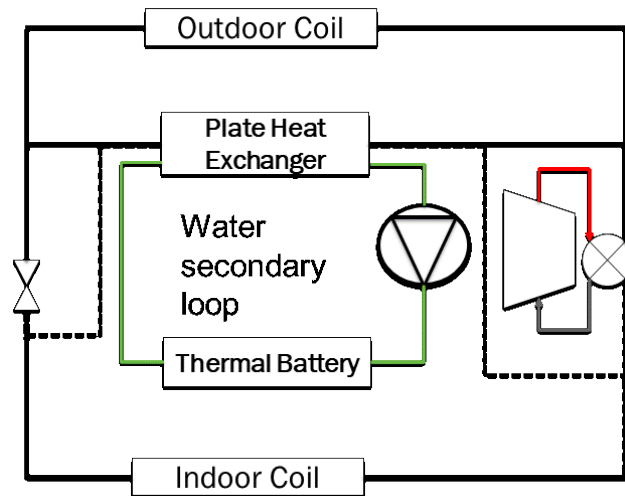


# Dual-Purpose – Heating & Cooling - Thermal Battery to Enable Flexible and Energy Efficient Heat Pump Systems



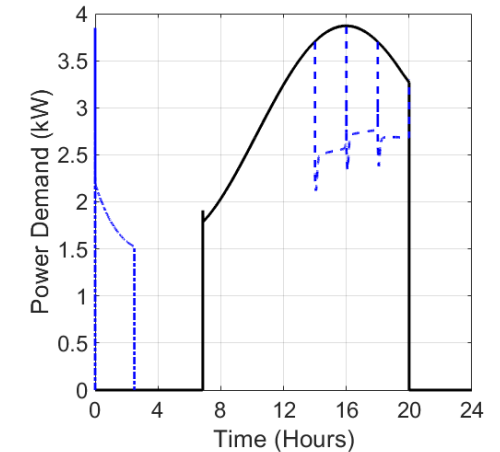
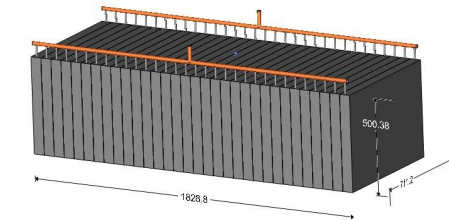
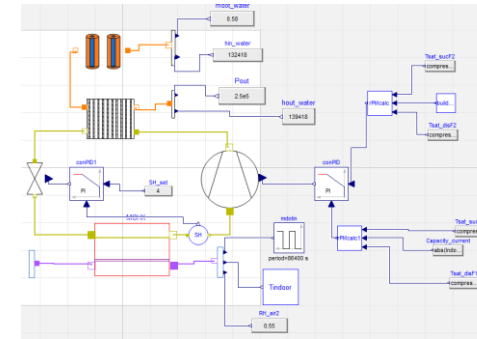
Performing Organizations: University of Maryland, Oak Ridge National Laboratory, Electric Power Research Institute, Rheem Manufacturing, Insolcorp, Heat Transfer Technologies  
Professor Reinhard Radermacher, Dr. Vikrant Aute, Dr. James Tancabel  
vikrant@umd.edu  
DE-EE0009681



# Project Summary

## Objective and outcome

- Develop an integrated heat pump-thermal energy storage (HP-TES) for load shifting in cooling & heating modes
- Assessment of demand reduction potential in all US climate zones
- System model validation, laboratory testing and technology demonstration (field test & commercialization plan)



## Team and Partners



CENTER FOR  
ENVIRONMENTAL  
ENERGY ENGINEERING



## Stats

Performance Period: 10/2021-09/2024

DOE budget: \$3,000K, Cost Share: \$750K

Milestone 1: Component & system design

Milestone 2: Laboratory testing & validation

Milestone 3: Technology demonstration

# Problem

- HVAC&R equipment is responsible for 40-70% of residential & commercial building loads
- They are one of the main drivers of peak loads (30% peak electricity share), particularly in the summer season
- Systems in the range of 3-5 Tons correspond to the largest **light commercial rooftop units** market share (approximately 4 million units shipped in 2020)
- Thermal energy storage can be used in a plethora of different configurations with HVAC&R equipment and are 2-3x less costly than electrochemical batteries

# Project Objectives & Opportunities

- **Objective**

- Develop & validate a single integrated HP-TES operating in both cooling & heating modes

- **Timeline**

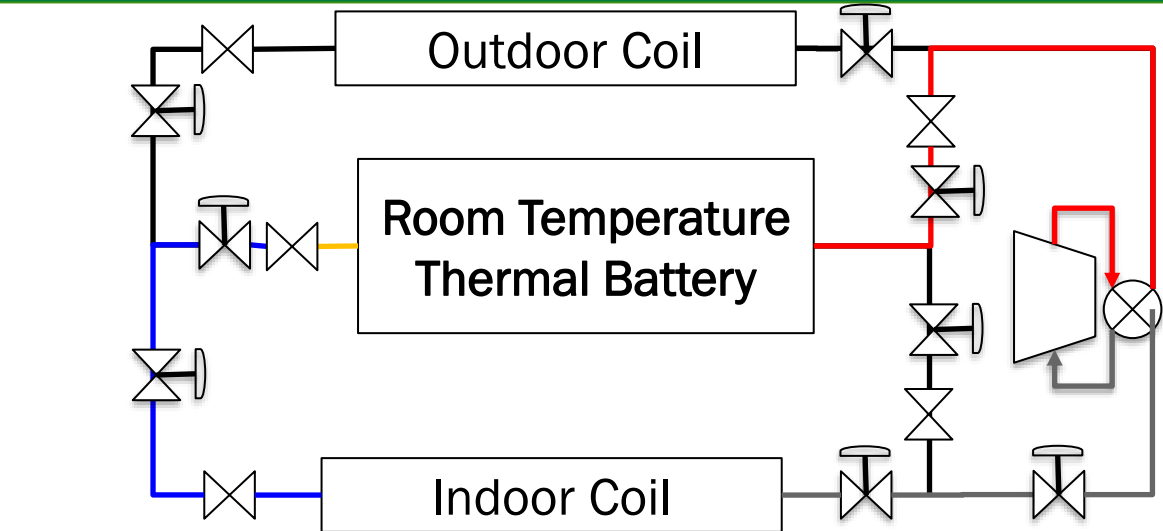
- Year 1
  - Component & system design
  - Prototyping & component testing
- Year 2
  - System construction & lab tests
- Year 3
  - Technology demonstration, field testing, & validation

- **Opportunities**

- Use a single room-temperature TES replacing the outdoor coil suitable for both heat pump modes
  - Reduce system (capacity) size
  - Reduce added cost
  - Possibly competitive with other high-efficiency heat pumps
- Reduce demand during the peak load use, thus operating costs
- Reduce system performance degradation, e.g.,
  - Constant temperature lift (less cycling)
  - Less frost (cold climates)

# Approach & Challenges

- **Original approach**
  - Direct refrigerant/TES system
  - TES operation: 5 tons, 4 hours
- **Challenges**
  - TES weight limits rooftop applications (6 x HP weight)
  - 5 x Refrigerant charge increase
  - TES aluminum HX are susceptible to corrosion from PCM salt hydrates
  - Demand reduction potential is dependent on location

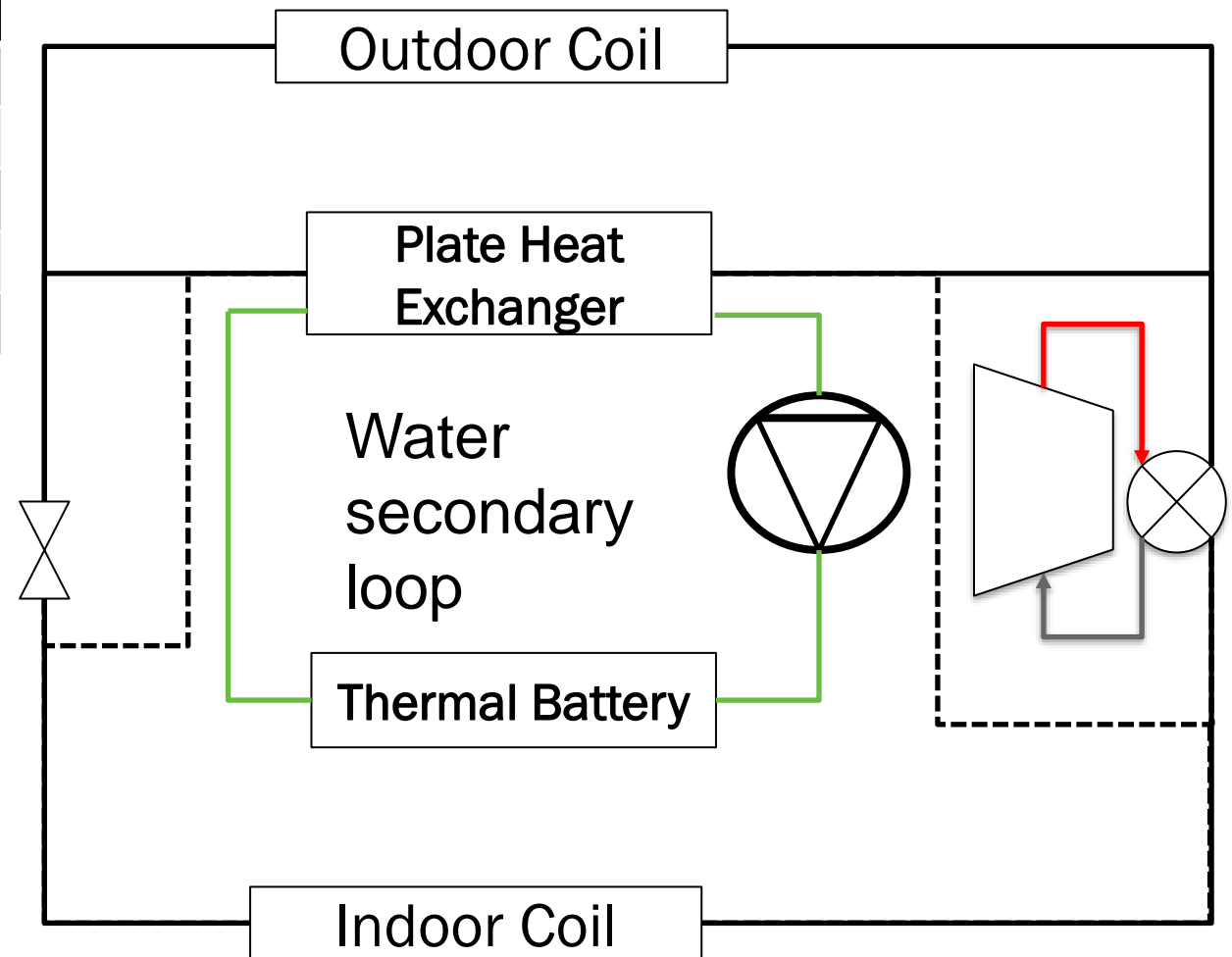


Targets	Metric	Unit	Target Values	Model-predicted performance
SOPO	System Capacity	Ton	5	5
SOPO & DOE	Demand Reduction	%	50+	20-80
	Period	hours	4+	4
	Volumetric Storage	kWh/m <sup>3</sup>	80	72
DOE	Volume Increase	%	10	40
Overlooked metrics	Weight Increase	%	No target set	Critical issue for RTUs
	Refrigerant Charge Increase	%	No target set	500

# Adjusted Targets & Approach

Metric	Unit	Original	New
(Commercial) System Capacity	Ton	5	4*
Demand Reduction	%	50+	50+
Period	hours	4+	2+
Overall COP Improvement	%	20+	20+
TES Cost	\$/kWh	15-	15-

- Current TES weight is 3 x HP
- Lower refrigerant charge increase in PHX compared to direct ref/TES
- Water  $\Delta T$  in PHX degrades the HPTES COP compared to direct ref/PCM



\*Highest selling system by millions of unit shipped (Source: AHRI  
([Statistics | AHRI \(ahrinet.org\)](https://www.ahrinet.org))

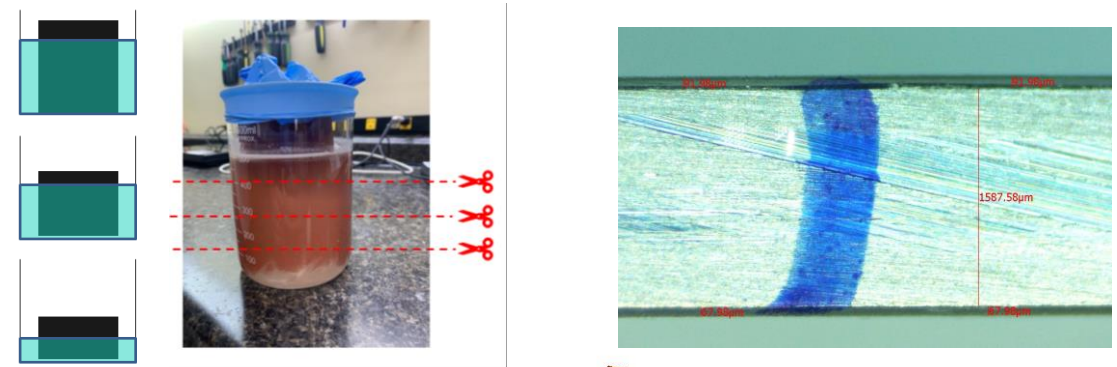
# TES Design, Assembly, & Material Durability

- TES requirements
  - Cooling mode: 16-17 kW
  - Heating mode: 10-11 kW
- Storage requirement:  $V_{PCM} = 0.45 \text{ m}^3$
- Discharge rate requirement:  $\dot{Q} = 17 \text{ kW}$
- HXs types that meet TES requirements
  - Microchannel HX
    - Small internal volume, low-cost and weight, high compactness
    - Difficult to assemble & package
  - ✓ Tube-fin HX (32 HX required)
    - ✓ Adequate compactness, low-cost and weight
    - ✓ Easy to assemble & package

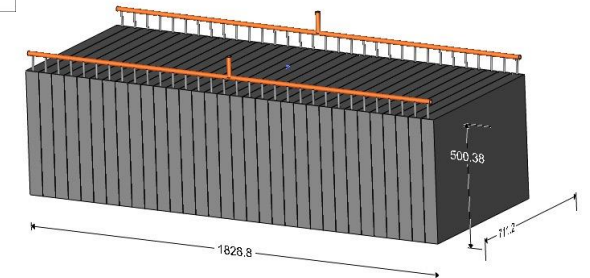
- Corrosion Testing

- All-aluminum tube-fin corrodes in salt hydrates
- Heresite™ immersed coating applied and tested

- Sampling Approach & Coating Results



- Packaged TES



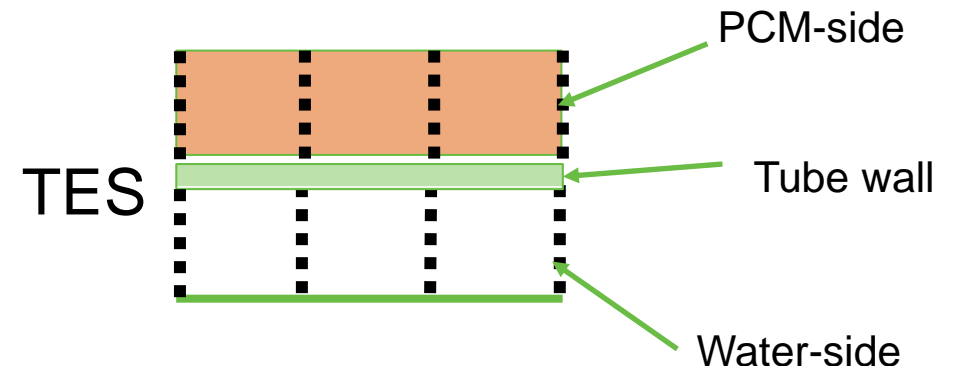
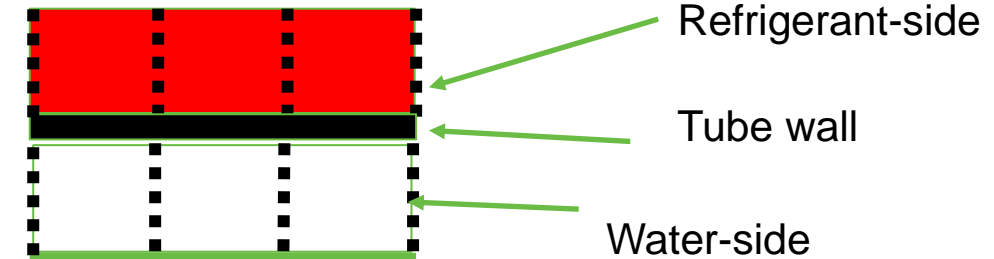


# Transient Model Setup & Validation

- **Baseline HP cycle was modeled & validated in Modelica at AHRI A-test conditions**
- **Secondary loop components in Modelica**
  - Modelica results for selected plate HX in agreement with PHESim
  - Segmented TES model created using simplified resistance-capacitance model
- **System level modeling**
  - HP cycle: variable-speed rotary compressor with EXV
  - HP-TES cycle: created for charging and discharging, same as HP cycle

Metric	Unit	Experiment	Modelica	Abs. Diff.
Superheat	K	4.44	4.89	0.45
Subcooling	K	7.25	5.25	2.00
Suction T	°C	14.00	14.28	0.28
Discharge T	°C	71.11	74.94	3.83
Comp. Power	kW	3.22	3.18	1.24%
Indoor HX Capacity	kW	13.97	13.86	0.79%

PHX



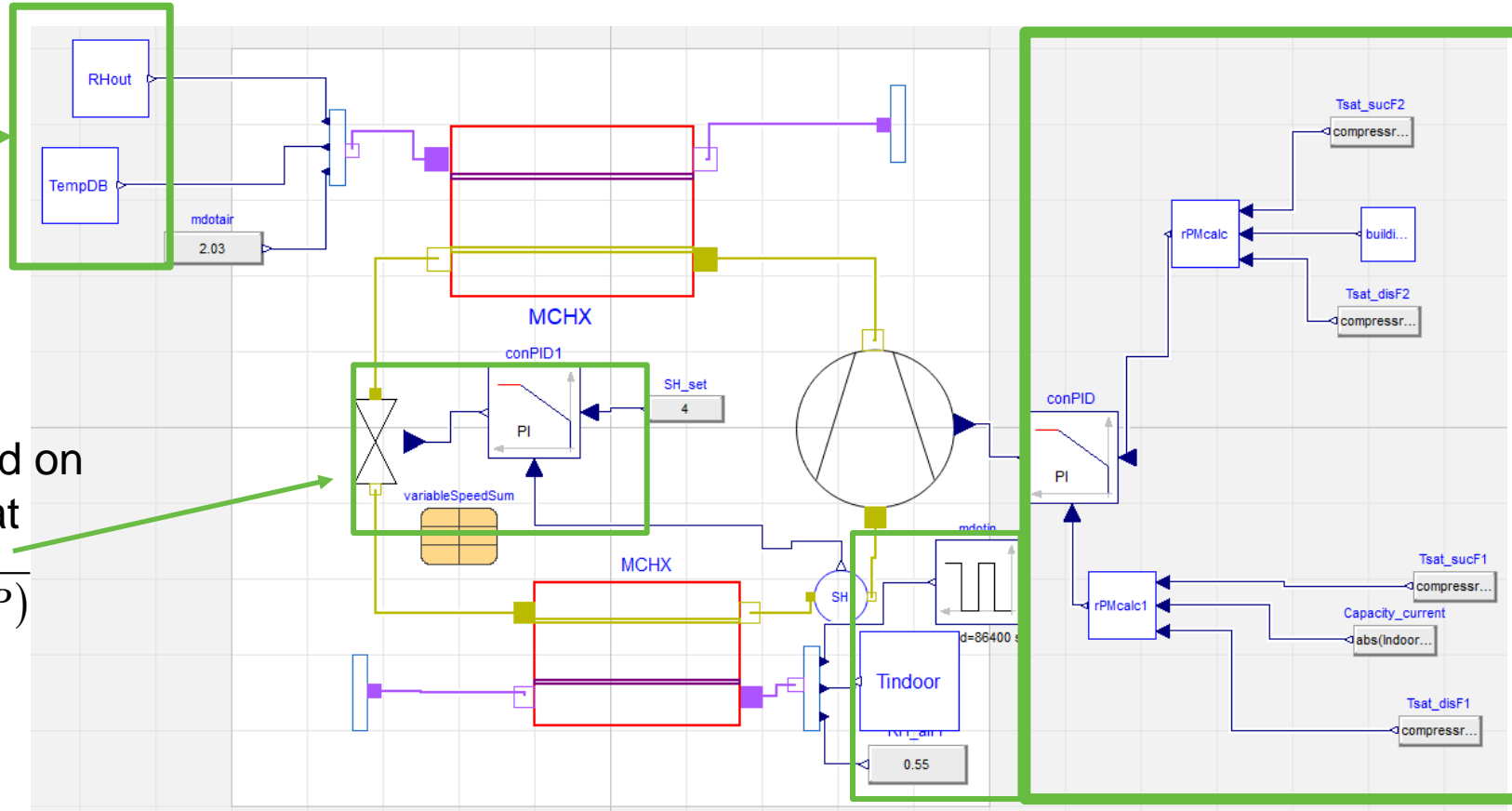


# System Level Model with Weather Data & Building Loads

Outdoor weather TMY3, for summer and winter

EXV control, based on setting a superheat

$$\dot{m} = \frac{u}{u_{\max}} C_v A \sqrt{\rho_{in} (\Delta P)}$$



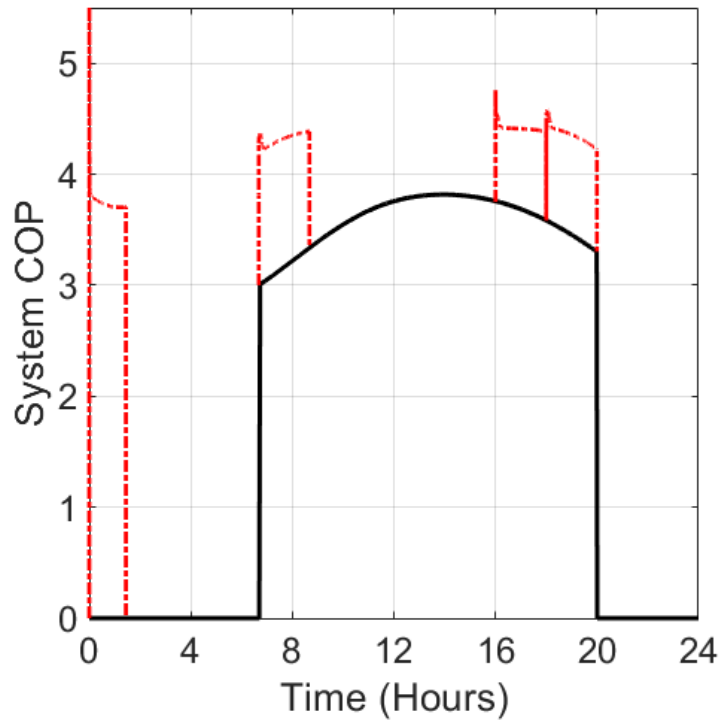
Control of the compressor RPM, to match internal loads with the indoor HX capacity

Indoor boundary conditions from building loads (Roth et al. 2022)

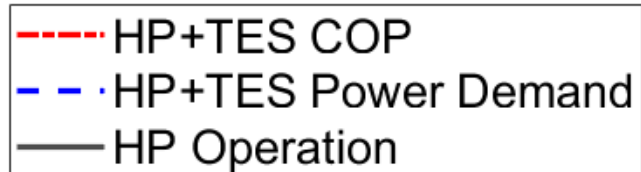
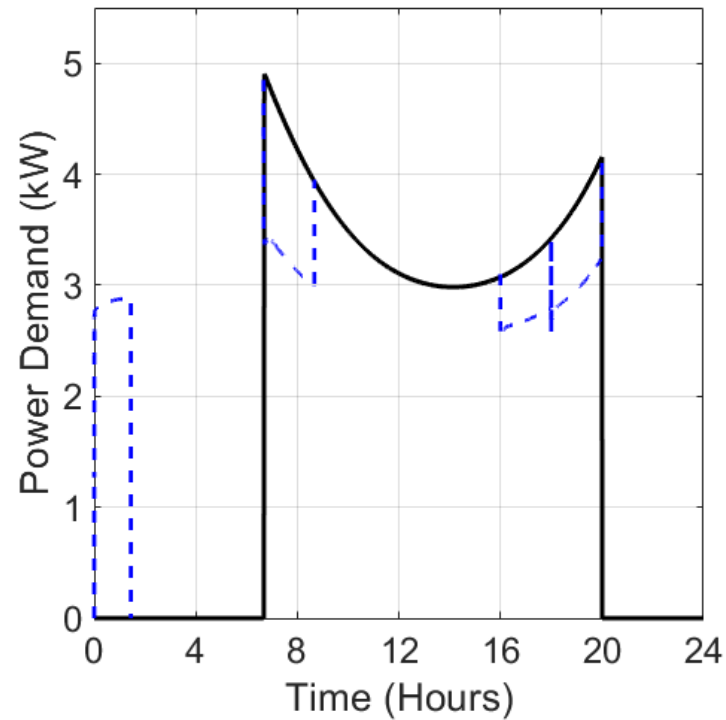
# System Level Model with Weather Data & Building Loads

- **Similar HP system level components are available with the HP-TES except**
  - **Discharging**
    - Outdoor HX replaced with secondary loop
    - Only indoor weather data are used
  - **Charging**
    - Indoor HX replaced with secondary loop
    - Only outdoor weather data are used at nighttime
- **Lowest COP time slots during the day**
  - Cooling: 2:00 pm – 4:00 pm
  - Heating: 6:30 am – 8:30 am
- **Time slots for peak hours in Southern California TOU**
  - 4:00 pm – 6:00 pm
  - 6:00 pm – 8:00 pm
- **Key metrics: COP improvement & demand reduction**
- **Key assumption: HP-TES water pump power is the same as outdoor fan power**

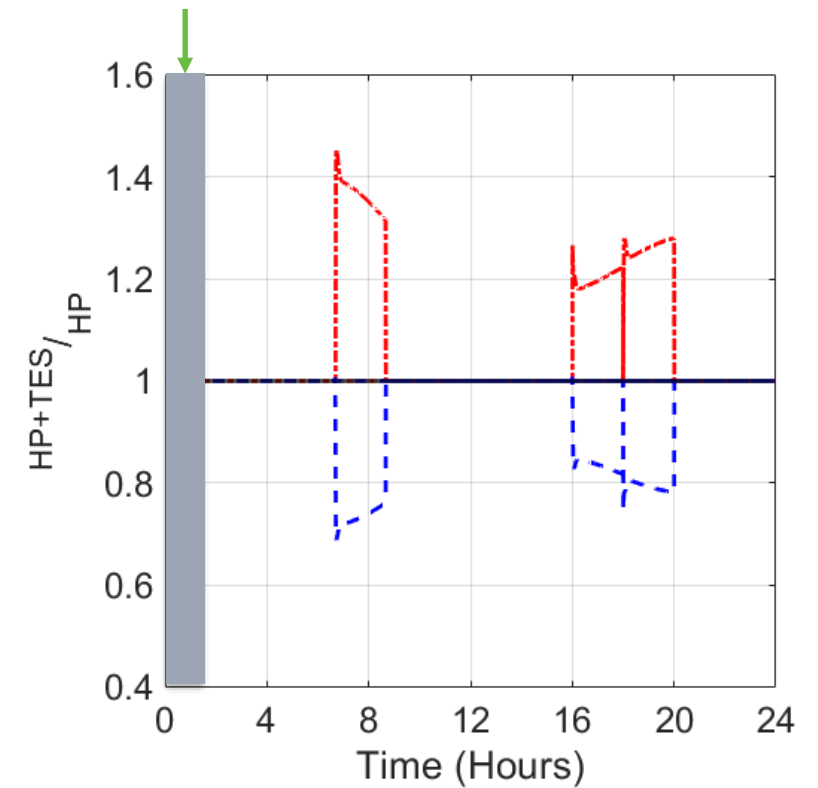
# Heating Mode



HP operation: 51.3 kWh  
HP-TES (all): 51.2 kWh

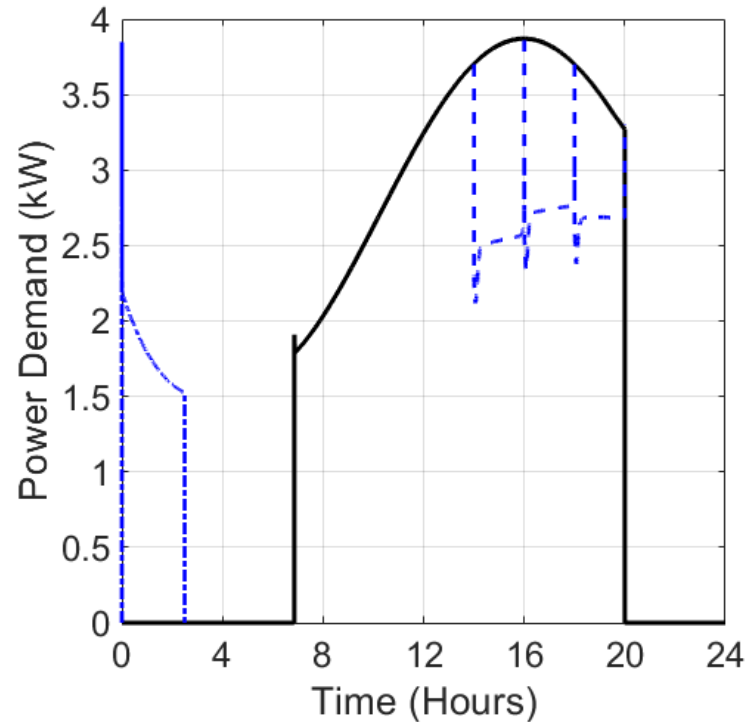
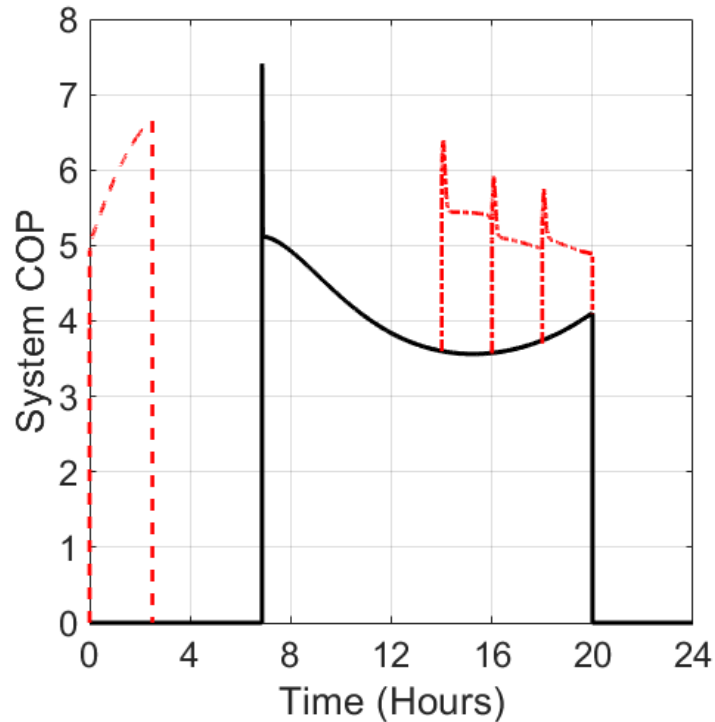


Recharge Time

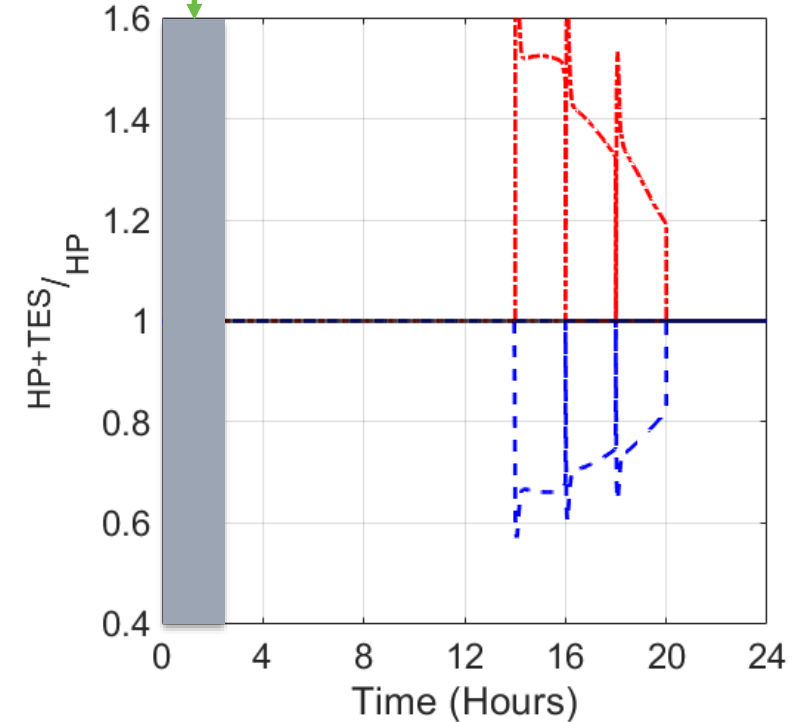


Slot 0: TES recharge time 12:00 – 2:00 am  
Slot 1: TES operation time 6:30 – 8:30 am  
Slot 2: TES operation time 4:00 – 6:00 pm  
Slot 3: TES operation time 6:00 – 8:00 pm

# Cooling Mode



Recharge Time



HP operation: 42.4 kWh  
HP-TES (all): 42.2 kWh

--- HP+TES COP  
- - - HP+TES Power Demand  
— HP Operation

Slot 0: TES recharge time 12:00 – 2:30 am  
Slot 1: TES operation time 2:00 – 4:00 pm  
Slot 2: TES operation time 4:00 – 6:00 pm  
Slot 3: TES operation time 6:00 – 8:00 pm

# Fabrication and Test Timelines

- **Thermal Battery Fabrication**
  - Fabrication & coating of 32 all-aluminum HXs (04-05/2023) (Rheem, Rahn)
  - Manifold construction & overall assembly (05-06/2023) (HTT, Insolcorp, ORNL, EPRI)
- **Baseline system testing**
  - ORNL Laboratory (03-04/2023)
  - EPRI Laboratory (05-06/2023)
- **Integrated system testing**
  - 06-09/2023 (ORNL, EPRI)
- **Field test site selection**
  - 09/2023

# Stakeholder Engagement

- **Project partners (sub-recipients)**

- Rheem Manufacturing

- Baseline heat pump system hardware & test data
    - HP-TES components: all-aluminum tube-fin HXs for thermal battery, plate HX for secondary loop, compressors & expansion devices
    - Commercialization advisor

- Insolcorp

- PCM manufacturer
    - Thermal battery assembly
    - Commercialization advisor

- **Industry involvement**

- Rahn Industries

- All-aluminum HX coating for thermal battery

# Next Steps

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- Model smooth switch between HP & HP-TES operation
- Analyze the HP+TES for summer, winter, & shoulder season for all 7 climate zones
- Conduct full transient seasonal simulations
- Fabricate/assemble battery (Ongoing)
- Test baseline system (Ongoing)
- Test integrated system
- Validate model using laboratory test data



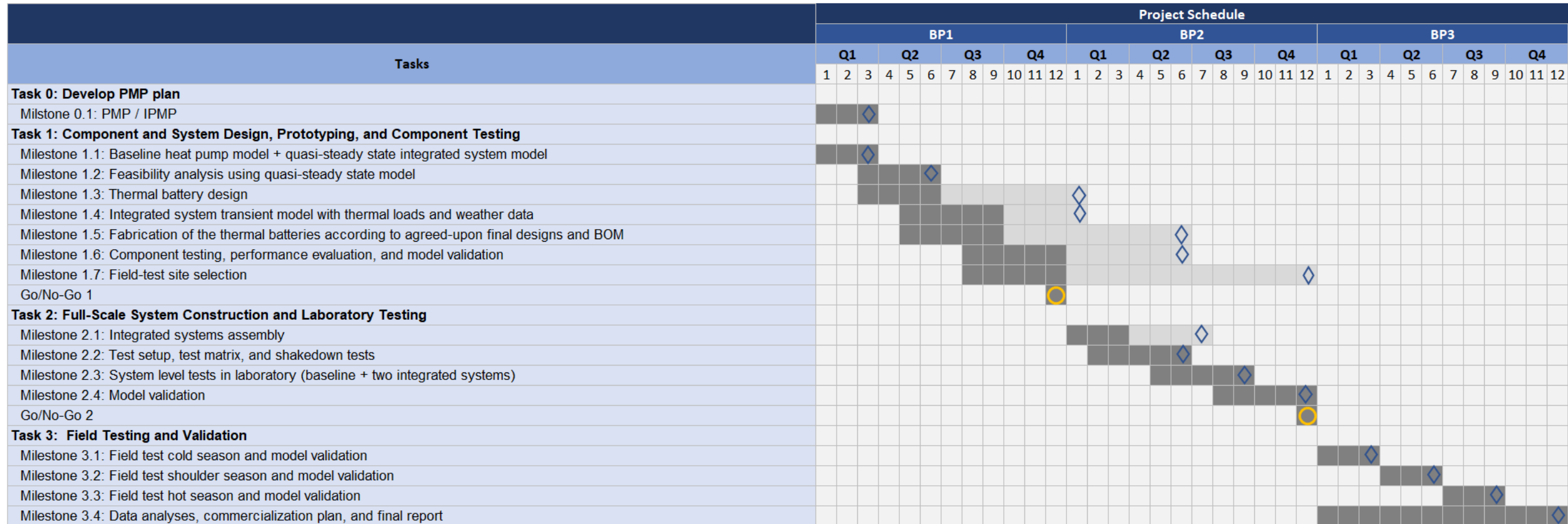
# Thank You

**Performing Organizations: University of Maryland, Oak Ridge National Laboratory, Electric Power Research Institute, Rheem Manufacturing, Insolcorp, Heat Transfer Technologies**  
**Professor Reinhard Radermacher, Dr. Vikrant Aute, Dr. James Tancabel**  
**vikrant@umd.edu**

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# REFERENCE SLIDES

# Project Execution



◆ Milestone ○ Go/No-Go

# Team

- **University of Maryland (Prime recipient)**
  - System modeling/design, data analysis, project management
- **Oak Ridge National Laboratory (Sub-recipient)**
  - System modeling, system assembly, laboratory testing, technical advisor
- **Electric Power Research Institute (Sub-recipient)**
  - System assembly, laboratory testing, technology demonstration lead, technical & code/regulation advisor
- **Rheem Manufacturing (Sub-recipient)**
  - System/component supplier, technical & commercialization advisor
- **Insolcorp (Sub-recipient)**
  - PCM supplier, system assembly, technical & commercialization advisor
- **Heat Transfer Technologies (Sub-recipient)**
  - Thermal battery design, assembly, technical advisor
- **Southern California Edison (EPRI's partner)**
  - Cost share contributor, technology demonstration promoter
- **Rahn Industries (industry partner)**
  - Heat exchanger coating for thermal battery