Enabling CO₂ Isothermal Compression Using Liquid Piston Within Integrated Gas Cooler



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Objectives

- Develop a novel isothermal compressor that reduces energy consumption by an average of 30% in refrigeration compared to isentropic compression.
- 2. Develop a transient isothermal compressor model that is capable of simulating CO₂ compression and heat rejection within the gas cooler.
- 3. Validate system model based on experimental data and then applied it into the field test.

Team and Partners

University of Maryland Emerson Climate Technologies



Stats

Performance Period: October 2021 ~ September 2024 DOE Budget: \$1,200 k, Cost Share: \$300 k Milestone 1: Develop a proof-of-concept compressor and a simulation tool Milestone 2: Construct a 1-ton capacity testing facility and improved simulation tool Milestone 3: Perform field test in partner's supermarket

Problem

- Total energy consumption projection increases by as much as 15% from 2022 to 2050. Households in the U.S. residential sector reached 5.1 Quads in 2022 and may reach between 5.9 Quads and 6.3 Quads from 2022 to 2050 based on the projections^[1].
- European Parliament adopted the revised F-Gas regulation banning HFCs and HFOs in multiple applications and phasing out HFCs by 2050
- The compressor is the major energy consumption component in an HVAC system. Study and research on compressor benefits in achieving carbon reduction.

Total energy consumption by end-use sector, United States (2010–2050) quadrillion British thermal units



Data source: U.S. Energy Information Administration, Annual Energy Outlook 2023

[1] - U.S. Energy Information Administration, <u>Annual Energy Outlook 2023</u>
[2] - IEA (2022), Energy Efficiency, IEA, Paris https://www.iea.org/reports/energyefficiency, License: CC BY 4.0

Alignment and Impact

Advancement of Compressor Technology

- Realize the near-isothermal compression in air conditioning and refrigeration application
- Integration mechanism of compressor chamber and gas cooler
- 30% less compression work than the isentropic compression

Air Conditioning and Refrigeration Industry Impact

 Replacing HFCs with natural refrigerants like CO₂ has the potential to avoid emissions of up to 5.6 Gt CO₂ eq per year by 2050^[3]

Acceleration of Energy Intensity Improvement toward NZB by 2050

- In 2022, the energy consumption for the residential and commercial sectors was 33 quads^[4]
- Space cooling and refrigeration account for 20% (6.6 quads)
- If deployed the technology to both sectors with 30% energy consumption reduction, the yearly energy savings will be 2.0 quads

[3] - United Nations Environment Programme. (2014). HFCs: A Critical Link in Protecting Climate and the Ozone Layer. Retrieved from <u>https://wedocs.unep.org/handle/20.500.11822/9417</u>
 [4] - U.S. Energy Information Administration, <u>Monthly Energy Review – Table 2.1</u>

Approach - Framework



Approach- Tasks and Milestones

Year-1: Develop the Proof-of-Concept Isothermal Compressor and a Transient Model (completed)

Extensive literature review on nearly isothermal compression technology Develop a transientbased isothermal compressor model Fabricate the proof-ofconcept isothermal compressor Successfully compress CO₂ above critical point and generate enough mass flow rate for a half-ton refrigeration cooling

Year-2: Improve the proof-of-concept compressor; Fabricate and test 1-ton refrigeration system

Collect feedback from Y1 and determine necessary modifications to the proof of concept compressor

Conduct system-level transient simulations to improve the isothermal compressor Select refrigeration cycle components based on the Y1 tests

Conduct system-level refrigeration cycle tests with the isothermal compressor

Year-3: Improve and scale up the compressor and conduct tests in a supermarket environment

Collect feedback on the testing data and modify the compressor to perform field test Calibrate and improve the refrigeration system model Refrigeration system tests in the supermarket test facility

Final documentation delivery

Progress – Compression Chamber

- Applied isothermal compressor using liquid piston integrated with gas cooler
 - Discharge compression work and internal energy simultaneously during compression
 - Up to 30% of work reduction
 - Manufacture and validate the proof-of-concept compression chamber in-house

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Dimension	Unit	Value
Upper header	mm	10.9
Bottom header	mm	16.6
Tube ID	mm	6.2
Tube spacing	mm	19
Wall thickness	mm	1.65
Height	Inch	24
Width	inch	21.6



CO₂ P-h Diagram

Progress – Liquid Level Sensor

- The liquid level sensor determination and experiments
 - Visualization of CO₂ bubble formation when decompressed
 - Create a mind map for potential liquid-level sensor options
 - Test and validate the candidate sensors on the proof-of-concept prototype
- Required characteristics of the candidate sensors
 - Fast response, reliable, high-pressure proof, low-cost
- The optical sensor and thermal sensor fit the application most.



Progress – Proof of Concept Experiment Setup

- A proof-of-concept isothermal compressor prototype was built to examine the actual compression efficiency
 - The cooling capacity was estimated based on the refrigerant mass flow rate
 - Inlet condition: 40°C, 4,000 kPa; Outlet condition:10,000 kPa



Test Facility

Progress – Proof of Concept Testing Results

- 60% of isothermal efficiency was achieved in the 0.5-ton capacity testing condition
 - Successfully compressed CO₂ over the critical point (7,385 kPa)
 - Chamber temperature was suppressed to 55°C compared with 118°C if isentropic compression
- Test results show insufficient heat transfer area to achieve 85% of isothermal efficiency



System Pressure

Chamber Temperature



Compressor Power

Progress – Proof of Concept Testing Results (cont'd)

- Efficiency Improvement Manipulation
 - Pump speed was lowered to one-third in tests #2 and #3 for extending heat transfer time
 - Apply evaporative cooling on the compression chamber surface for heat transfer enhancement



 By slowing the process and combined with evaporative cooling, the efficiency improved to 80% and 94%, respectively



	Test #1	Test #2	Test #3
Pump Speed	900	270	270
Water spray*	No	No	Yes
Efficiency	60%	80%	94%

*Room Conditions: 35°C and 20% in humidity Water Temperature: 35°C

Significant Temperature Reduction

Progress - Modeling and Validation for Isothermal Compressor

$$\begin{bmatrix} V \frac{\partial \rho}{\partial P} h - V & V \frac{\partial \rho}{\partial h} h + V \rho & 0 & \rho h - P \\ V \frac{\partial \rho}{\partial P} & V \frac{\partial \rho}{\partial h} & 0 & \rho \\ 0 & 0 & C_{th,w} & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \dot{P} \\ \dot{h} \\ \dot{T}_{w} \\ \dot{V} \end{bmatrix} = \begin{bmatrix} \dot{m}_{in} h_{in} - \dot{m}_{out} h - \alpha_{r} A_{i} \left(T_{r} - T_{w}\right) \\ \dot{m}_{in} - \dot{m}_{out} \\ \alpha_{r} A_{i} \left(T_{r} - T_{w}\right) - \alpha_{a} A_{a} \left(T_{w} - T_{a}\right) \\ -A_{s} v \end{bmatrix}$$

- The model shows satisfying agreement between modeling and experiment results in terms of overall trends
- Pressure and temperature delays are observed. Improvements should be made by calibrating the thermal capacitance of chamber
- The fluctuation of liquid pump power from experiment could be a result of system noise
- The model didn't consider the make-up refrigerant that dissolved in oil during experiment



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Progress –1-ton System Development

- Compression Chamber Heat Transfer Calculation and Selection
 - CO₂ was the dominant heat transfer resistance
 - Coefficient correlations for combined forced and natural convection were selected from open literature*
 - An EES code was developed to compare different type heat exchanger's performance
 - A dimensionless number was defined to evaluate the heat transfer capability against required discharge heat stored in the CO₂, the higher, the better









Progress - System Level Model Development

Deliver

Suction

- Challenge •
 - The refrigerant mass flow rate is intermittent through the system
 - System can't reach to steady state or quasi-steady state
 - Heat transfer correlations for both air-side and refrigerant-side significantly affect isothermal efficiency
- Strategies
 - Consider the receiver in the system model to provide the continuous mass flow rate
 - Ensure the suction period equals to the time for compression and deliver process
 - Calibrate the heat transfer coefficient based on the experimental data



• Experimental Work

- Select a type of compressor chamber based on the proof-of-concept compressor
- Modify and design the 1-ton refrigeration system
- Conduct system-level refrigeration cycle tests with the isothermal compressor
- Scale up and modify the 1-ton refrigeration system for supermarket environment
- Conduct field test and collect data including cooling capacity, COP of the refrigeration system, isothermal compression efficiency
- Simulation Work
 - Develop system level model and instruct the experiential setup design and prototyping based on the simulation results
 - Conduct system level validation and performance assessments

Thank You

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REFERENCE SLIDES

Mind Map of Liquid Sensor



Approach - Simulation

- Conservation differential equations for the refrigerant (CO2) energy $\dot{U} = m_{in}h_{in} - m_{out}h_{out} - \alpha_r A_i (T_r - T_w) + W$
- Conservation differential equations for the refrigerant mass

$$\dot{m}_e = m_{in} - m_{out}$$

Conservation differential equations for the tube wall energy

$$\dot{E} = C_{th,w}\dot{T}_{w} = \alpha_{r}A_{i}\left(T_{r}-T_{w}\right) - \alpha_{a}A_{a}\left(T_{w}-T_{a}\right)$$

• Reformulation of the above governing equations

$$\begin{bmatrix} V \frac{\partial \rho}{\partial P} h - V & V \frac{\partial \rho}{\partial h} h + V \rho & 0 & \rho h - P \\ V \frac{\partial \rho}{\partial P} & V \frac{\partial \rho}{\partial h} & 0 & \rho \\ 0 & 0 & C_{th,w} & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \dot{P} \\ \dot{h} \\ \dot{T}_{w} \\ \dot{V} \end{bmatrix} = \begin{bmatrix} \dot{m}_{in} h_{in} - \dot{m}_{out} h - \alpha_{r} A_{i} \left(T_{r} - T_{w}\right) \\ \dot{m}_{in} - \dot{m}_{out} \\ \alpha_{r} A_{i} \left(T_{r} - T_{w}\right) - \alpha_{a} A_{a} \left(T_{w} - T_{a}\right) \\ -A_{s} v \end{bmatrix}$$

Isothermal compressor



Project Execution

		FY2022		FY2023			FY2024					
Planned budget												
Spent budget												
	Q1	Q2	Q3	Q4	Q1	Q2	Q3	Q4	Q1	Q2	Q3	Q4
Past Work												
Q1 Milestone: IPMP is completed		•										
Q1 Milestone: A comprehensive summary on literature review is												
completed												
Q1 Milestone: A Matlab-based transient isothermal compressor model		Þ										
Q2 Proof-of-concept compressor test setup is assembled												
Q3 Milestone: The proof-of-concept is fabricated												
Q4 Milestone: The proof-of-concept compressor generate 0.5-ton												
cooling capacity					Í							
Current/Future Work												
Q1 Milestone: Brainstorm on Year 1 design feedback is completed												
Q1 Milestone: The Matlab-based transient model validated and												
improved												
Q2 Milestone: Design of 1-ton refrigeration system is completed												
Q3 The test setup can conduct refrigeration test												
Q4 The isothermal compressor successfully created 1-ton cooling												

Team

- University of Maryland: Expertise: 30+ years of experience in R&D of heat pumps, refrigerant, HVAC&R components and systems, modeling and optimization software development; system and component test facilities; funded by industry and government



 Emerson Climate Technologies
 Expertise: 30 years of experience in airconditioning compressors and system design and development.

