



Expanding the Knock/Emissions/Misfire Limits for the Realization of Ultra-Low Emissions, High Efficiency Heavy Duty Natural Gas Engines

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Project ID: FT079

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Project Overview

Timeline

Project Start Date: 5/23/2018

Project End Date: 5/22/2021

Percent Completion: 33%

Budget

Total Project Cost: \$1,572,922

Federal = \$1,257,633

Cost Share = \$315,289

Budget Period 1 Federal: \$463,242

Budget Period 2 Federal: \$405,149

Budget Period 3 Federal: \$389,242

Barriers

- Goal: Increase brake efficiency to 44% for NG engine
 - Reduced kinetic model to predicted end gas autoignition (EGAI)
 - Advanced controls to maintain controlled EGA1 at high BMEP and variable fuel quality.

Partners

Project Lead: Colorado State University

Cummins Inc.

Woodward, Inc.

Milestones

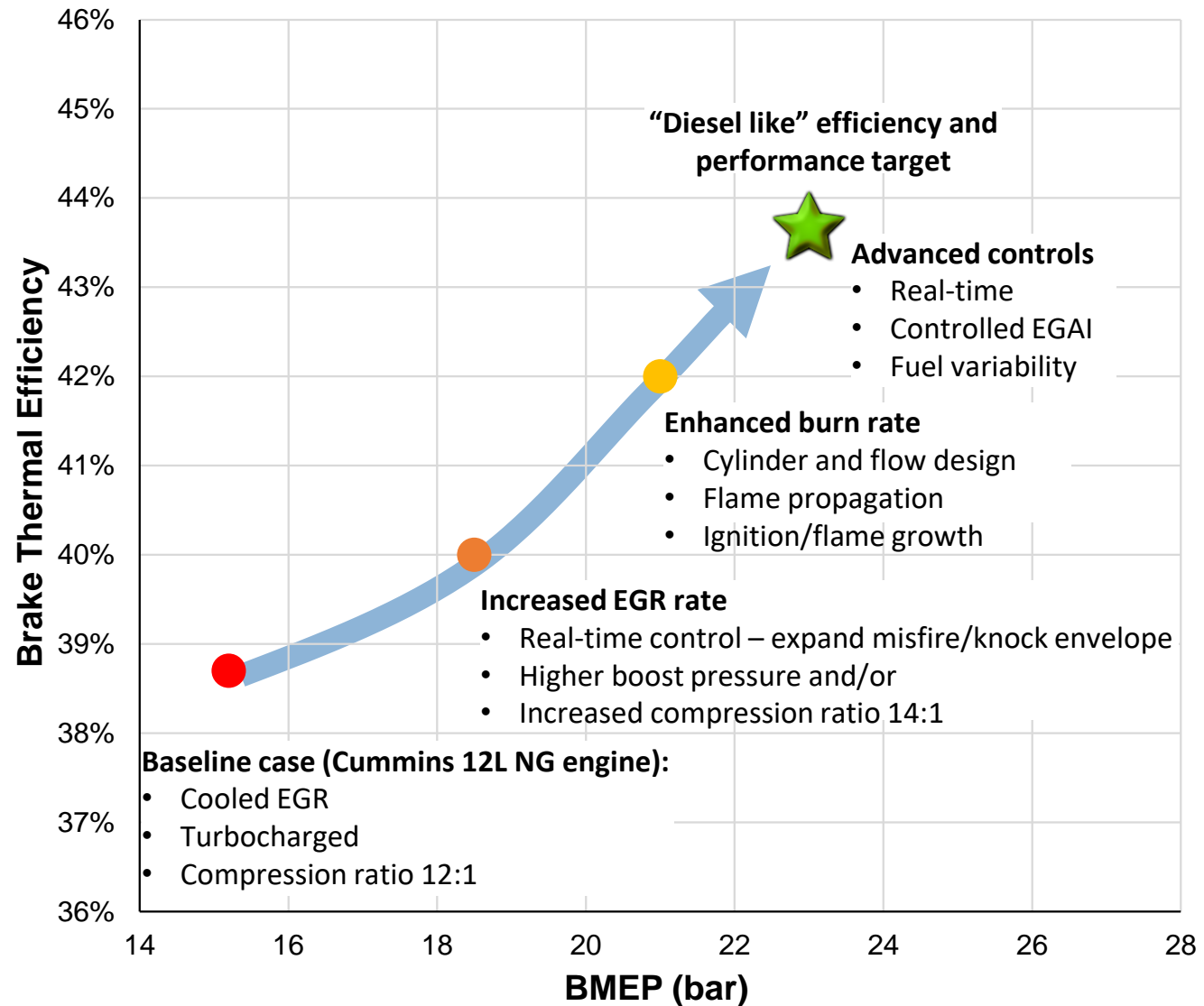
| Project Tasks, Milestones, and Go/No-Go Decisions | Budget Period 1 | | Budget Period 2 | | | | Budget Period 3 | | | | | |
|--|-----------------|------|-----------------|------|-----|------|-----------------|-----|----|------|------|------|
| | 2018 | | 2019 | | | | 2020 | | | | 2021 | |
| | Q3 | Q4 | Q1 | Q2 | Q3 | Q4 | Q1 | Q2 | Q3 | Q4 | Q1 | Q2 |
| 1. Validate and Development Modelling Tools | | | | M1.3 | GN1 | M2.1 | | | | | | |
| 2. CFR Experiments and Modelling | | M1.1 | | M1.2 | | | | | | | | |
| 3. Combustion Chamber Design | | | | | | M2.2 | | | | | | |
| 4. Single Cylinder Engine (SCE) Development | | | | | | | M2.3 | GN2 | | M3.1 | | |
| 5. Control System Development | | | | | | | M3.2 | | | | | |
| 6. System Optimization - Target 44% Efficiency | | | | | | | | | | | | M3.3 |
| <div></div> - Complete <div></div> - To Be Completed | | | | | | | | | | | | |

Approach



Engine Configuration to Meet Goal

- Stoichiometric SI, turbocharged
- High levels of cooled EGR
- Combustion chamber design for high burn rate
- Prechamber spark plugs
- Advanced engine controls



Pathway to “Diesel-like” Efficiency and Performance



Technical Accomplishments and Progress

Detailed Parent Mechanism



| Detailed Mechanism | Origin | Species | Reactions |
|----------------------------|---------------------------------------|---------|-----------|
| Aramco 3.0 | National University Ireland Galway | 581 | 3,034 |
| Aramco 2.0 | National University Ireland Galway | 493 | 2,714 |
| NUIG NGM II | National University Ireland Galway | 229 | 1,359 |
| Ranzi V1412 | Polytechnic University of Milan | 115 | 2,141 |
| GRI Mech 3.0 | University California Berkeley | 53 | 325 |
| San Diego | University California San Diego | 57 | 268 |
| USC Mech Version II | University Southern California | 111 | 784 |

- 7 Detailed parent mechanisms were selected for evaluation
- Mechanisms were designed to predict ignition delay and laminar flame speed of HC species < C5
- Desired performance for Methane, Ethane, Propane natural gas fuels with MN 34-95 and pressured from 5-85 bar
- Reduced mechanism (~50 Species) will be tuned using rapid compression machine (RCM) ignition delay and flame propagation rate data collected at CSU

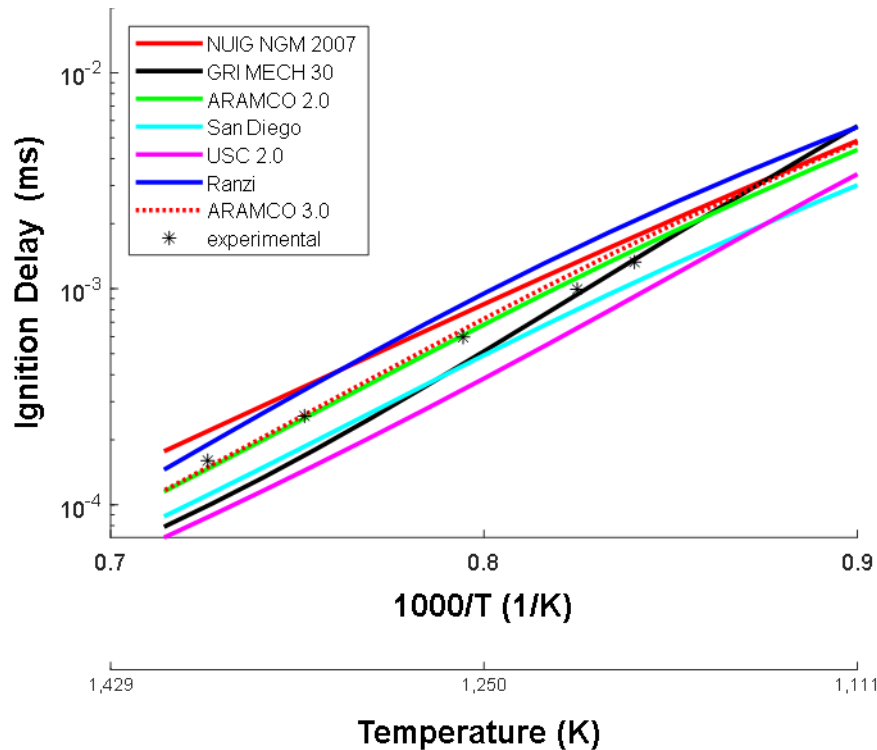


Technical Accomplishments and Progress

Detailed Parent Mechanism Performance

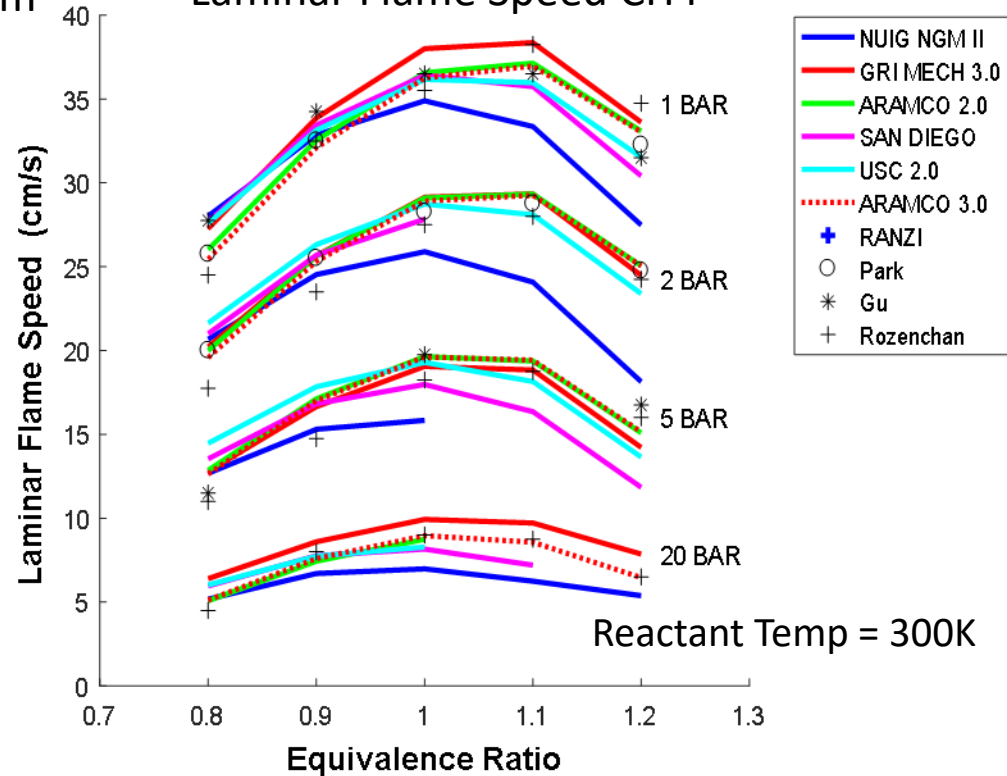


Ignition Delay CH₄+C₂H₆ PHI=1 at 30 atm



- Ignition delay performance at high pressure with methane-ethane blend. Note good agreement of Aramco mechanisms at elevated pressure to experimental (black stars)

Laminar Flame Speed CH₄



- Laminar flame speed performance of methane. Note good agreement of Aramco mechanisms to experimental points (black markers)
- Selected Aramco 3.0 as parent



Technical Accomplishments and Progress

Reduced Mechanism Performance

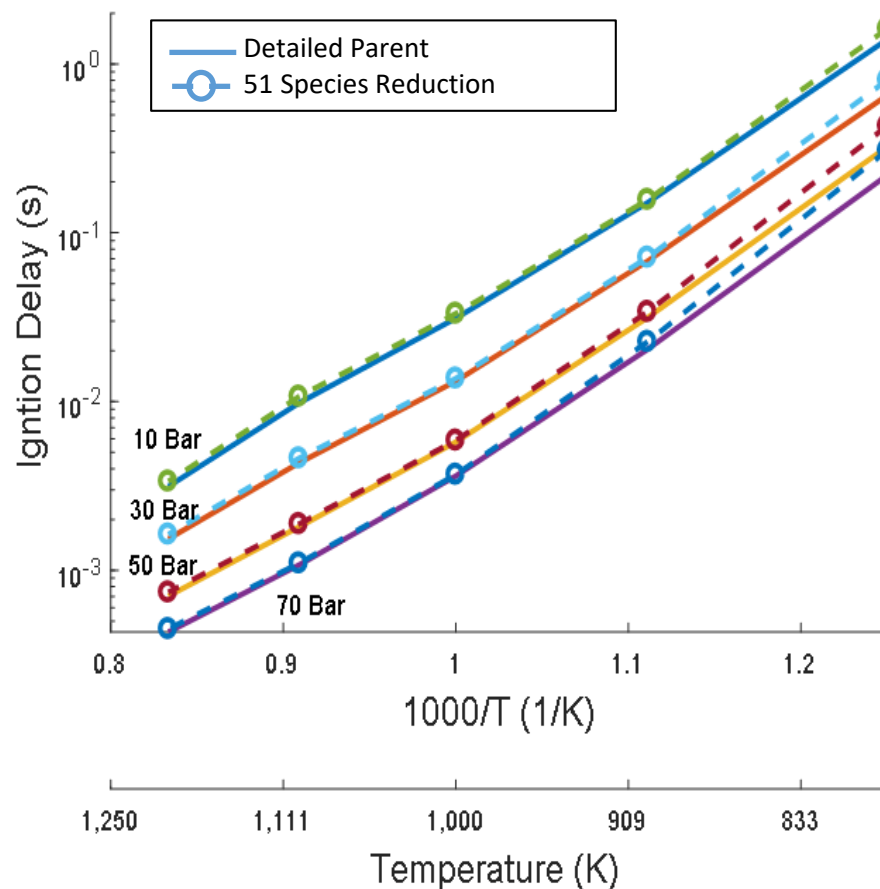


Tested Fuel Blends

| | Dry | Middle | Wet | Propane |
|---------|------|--------|-----|---------|
| Methane | 99% | 95% | 82% | 0% |
| Ethane | 0.5% | 4% | 15% | 0% |
| Propane | 0.5% | 1% | 3% | 100% |
| MN | 95 | 86 | 68 | 34 |

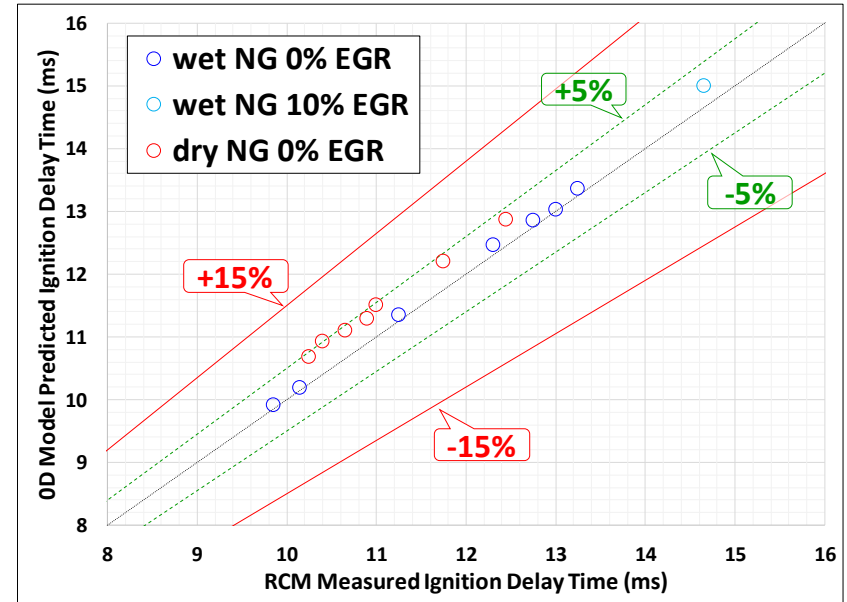
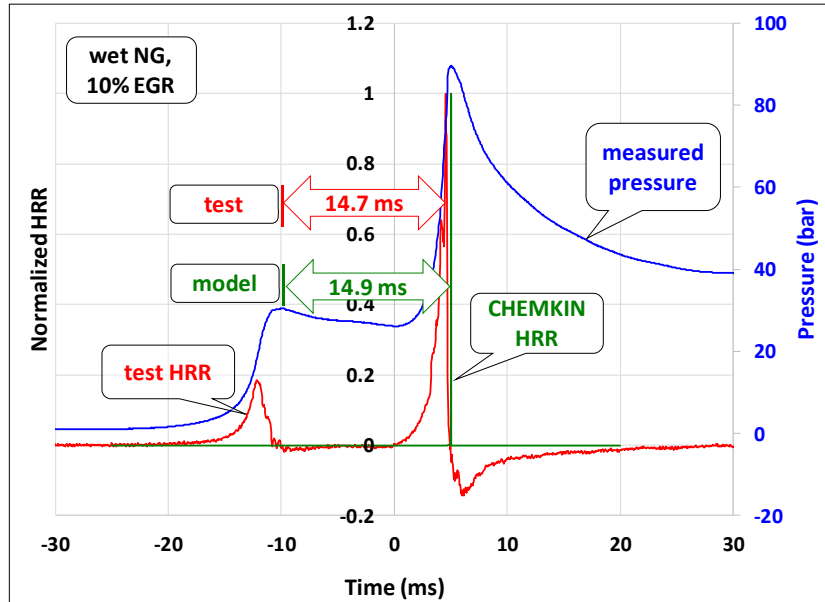
- The detailed mechanism was reduced using Chemkin to create a 51 species mechanism
- Ignition delay was main tuning parameter and benchmarked against detailed mechanism performance (right) for 4 fuel blends shown (above)

Ignition Delay Performance of Middle NG blend PHI=1 51 Species Mechanism



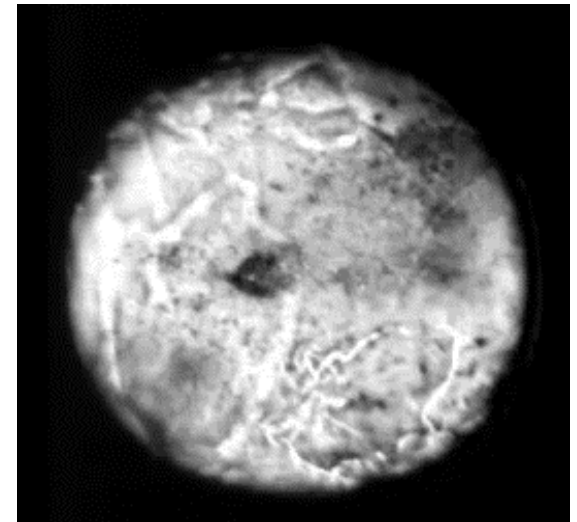
Technical Accomplishments and Progress

Reduced Mechanism Performance



- Reduced mechanism can reliably **predict autoignition time in OD CHEMKIN simulations**
- CHEMKIN OD homogeneous premixed model was used.
- Inputs include measured pressure with time, initial temperature, fuel type, and air/fuel ratio
- Overall autoignition prediction accuracy within 5%**

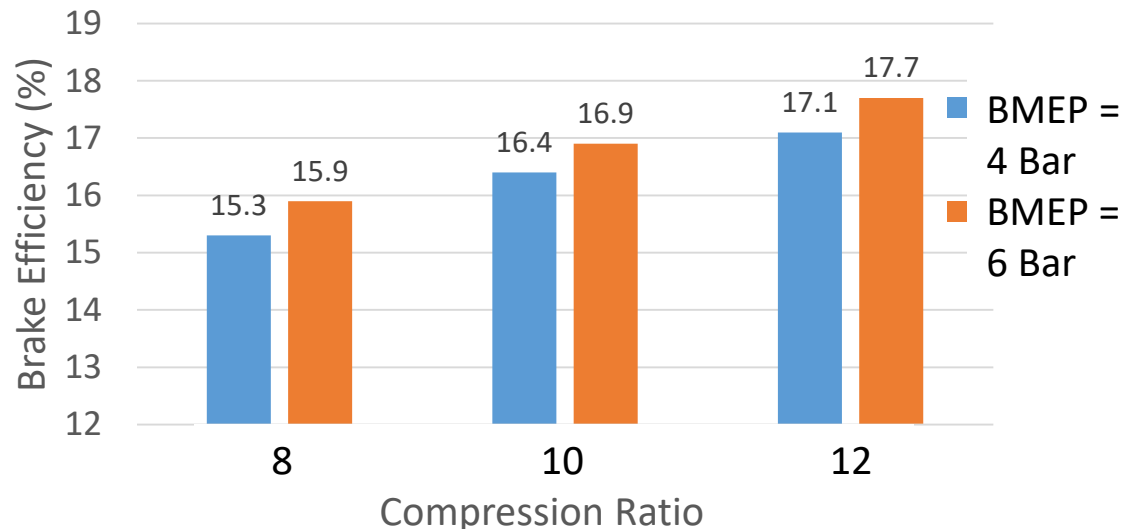
Experiment Specifications: \longrightarrow
Wet Blend Fuel: 82% CH_4 , 15% C_2H_6 , 3% C_3H_8
Oxidizer/Inert: 21% O_2 / 59% Ar / 20% N_2
Stoichiometric - 0% EGR - Laser-ignition: 10 ms



Technical Accomplishments and Progress

Preliminary Engine Experiments

- Cooperative Fuels Research (CFR) engine upgrades:
 - Woodward Large Engine Control Module (LECM)
 - Dynamic Pressure Sensors
 - Exhaust Gas Recirculation (EGR) Test Cart
 - Fuel Blending System
- Established knock intensities for knock detection method comparison
- Provided baseline engine data for OD and CFD engine model development



EGR Cart



*Cooperative Fuels Research
(CFR) Engine*

Technical Accomplishments And Progress

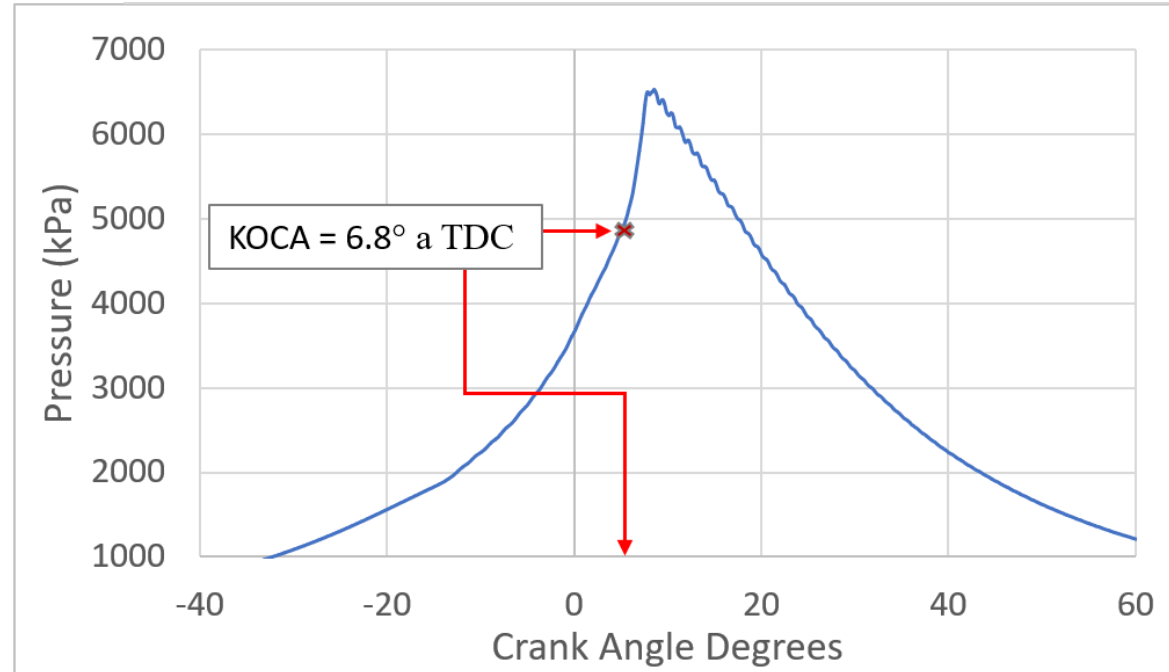
Knock Detection Method Quantification



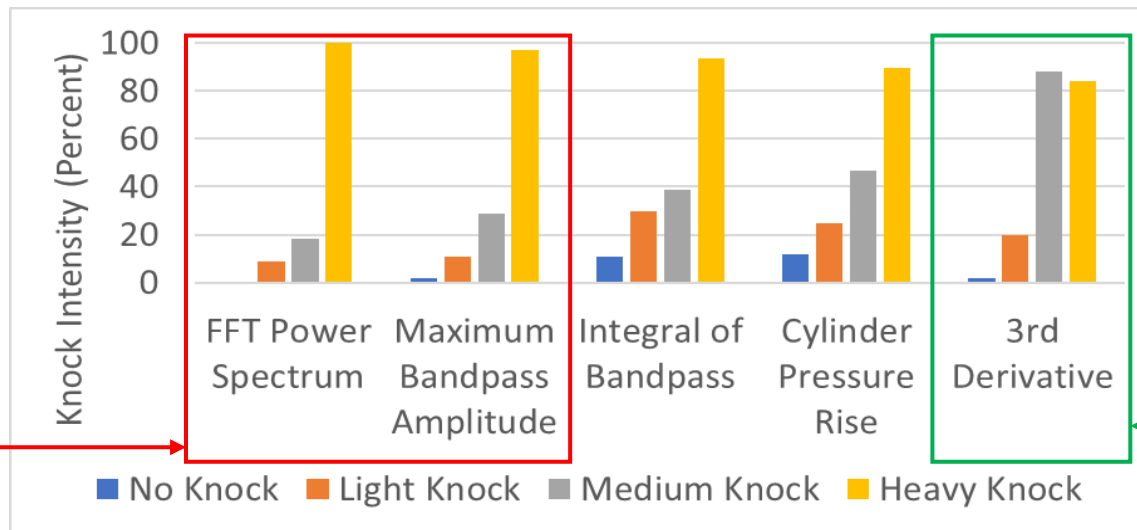
- Explored pressure based knock detection methods

Knock Location and Intensity:

- Necessary for “Controlled End Gas Auto-Ignition”
- Will operate within window of knock onset to light knock



Initial selections for auto-ignition detection

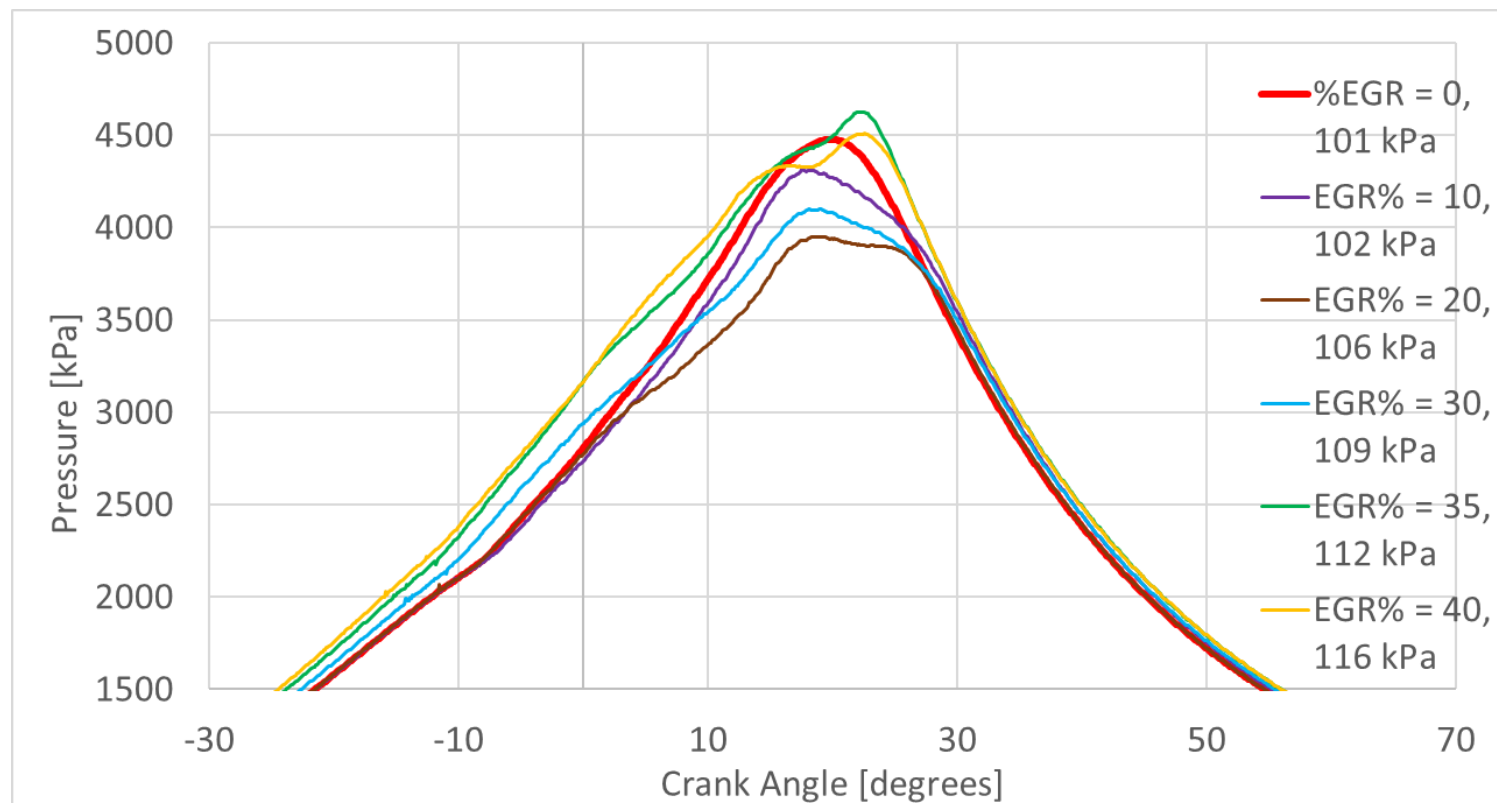


Initial selection for KOCA calculation



Technical Accomplishments and Progress

Exhaust Gas Recirculation Operation on CFR Engine



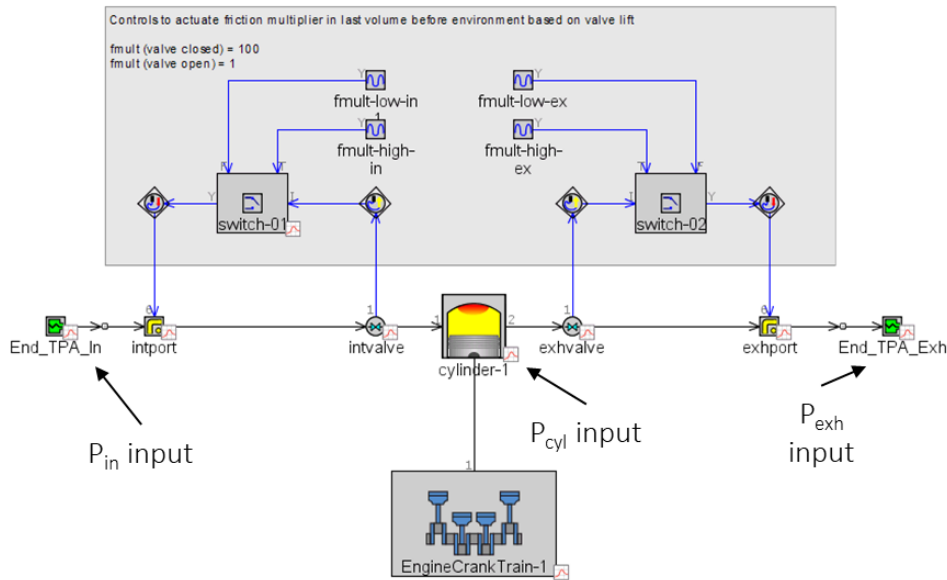
Phi = 1
IMEP = 8 Bar
RPM = 942
Intake Temp. = 65°C
CA50 = 13.8° aTDC
CR = 11.9

- EGR limit = COV Peak Pressure ≥ 10.0
 - Observed EGR Limit $\sim 35\%$
- Subsequent tests will explore increasing compression ratio (CR) and brake mean effective pressure (BMEP) to improve efficiency



Technical Accomplishments and Progress

CFR Engine Modeling

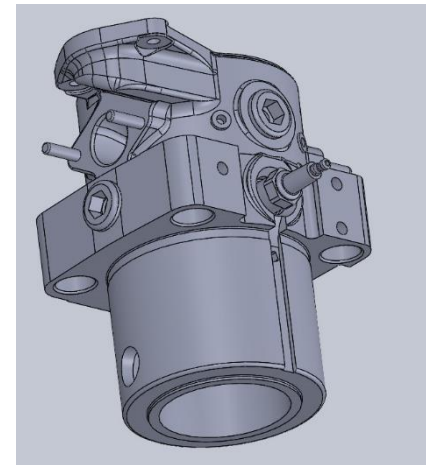
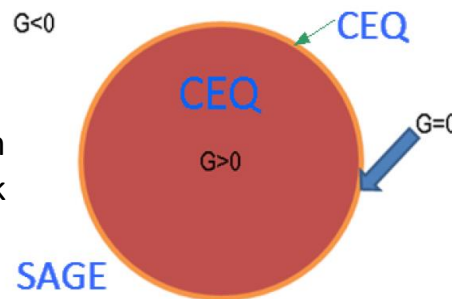


GT-Power Three-Pressure Analysis Model

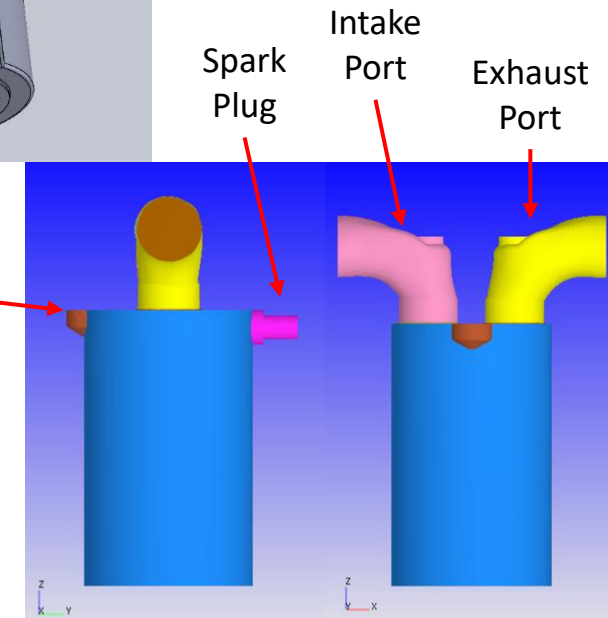
CFD Model using:

- CONVERGE CFD (commercial code)
- RANS RNG k- ϵ Turbulence Model
- Adaptive Mesh Refinement (AMR)
- Fixed Embedding
- Combustion model: G-Equation + SAGE
 - G-Equation: track flame propagation
 - SAGE: chemical kinetics solver (track autoignition)

Combustion Model²



ANL 3D Scanned Model¹ of the CFR Engine



CONVERGE CFD Surface of the CFR Engine



Technical Accomplishments and Progress

CFR Engine Modeling



$$s_{l_ref} = B_m + B_2(\phi - \phi_m)^2$$

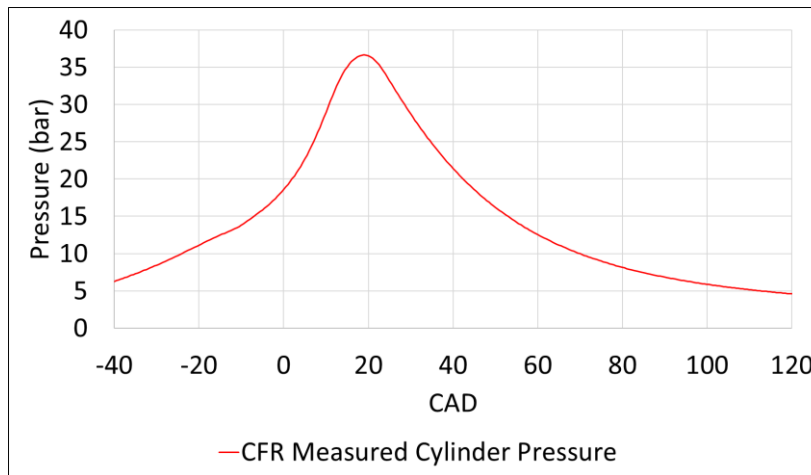
$$s_l = s_{l_ref} \left(\frac{T_u}{T_{u_ref}} \right)^\gamma \left(\frac{P}{P_{ref}} \right)^\beta (1 - 2.1 Y_{dil})$$

$$\gamma = a + m(\phi - 1)$$

$$\beta = a + m(\phi - 1)$$

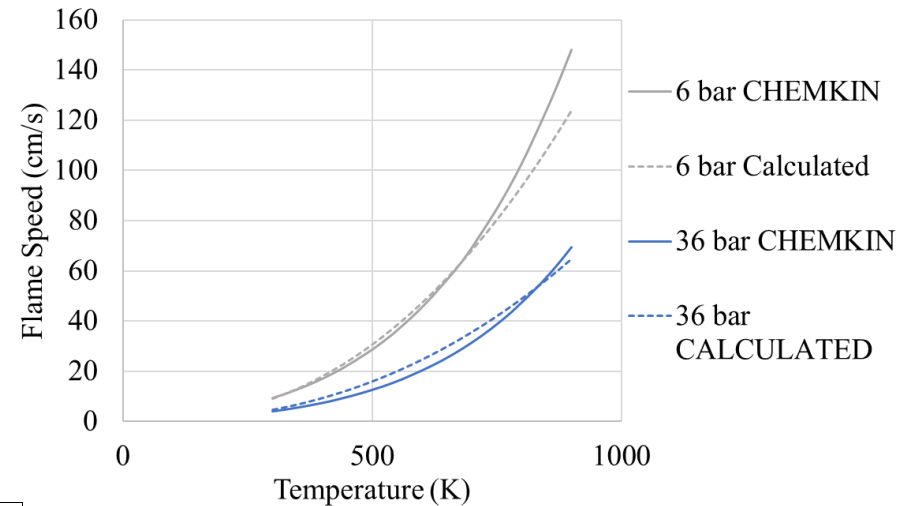
*Metghalchi and Keck 1982

Combustion Model Equations for Laminar Flame Speeds



Engine conditions at which flame speeds are calculated. Combustion model properly calculates flame speeds at these conditions

- Tracks flame speed at engine relevant conditions
- Enable the use of reduced mechanisms to reduce computational cost



Flame Speeds calculated from implemented combustion model agree with flame speeds calculated using a chemical kinetics solver

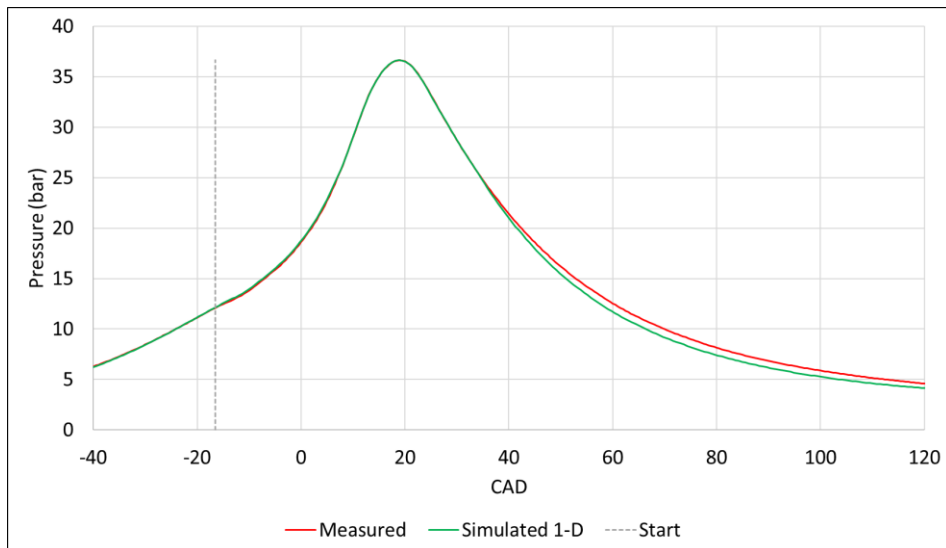
| Species | Mol % |
|-------------------------------|-----------|
| CH ₄ | 81.9 |
| C ₂ H ₆ | 14.28 |
| C ₃ H ₈ | 3.485 |
| N ₂ | 0.31 |
| MN | 68 |

Natural Gas composition used in this work



Technical Accomplishments and Progress

CFR Engine Modeling

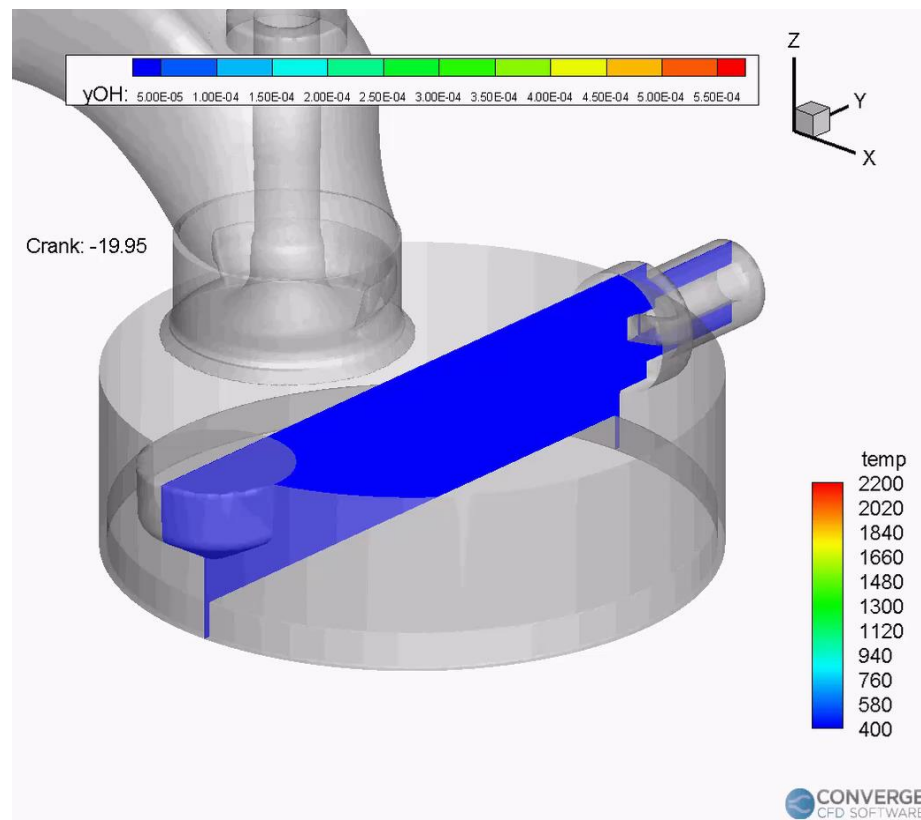


GT-Power TPA Model results closely agree with measured data

| GT-Power Performance | | | |
|----------------------|--------|----------|----------|
| | CFR | GT-Power | Unit |
| CA50 | 10.07 | 10.50 | CAD |
| gIMEP | 8.27 | 8.13 | bar |
| P _{peak} | 39.31 | 38.70 | bar |
| P _{peak} at | 18.83 | 18.60 | CAD |
| m _{trapped} | 456.00 | 468.70 | mg/cycle |

GT-Power Model Performance results closely predict performance of the CFR Engine

CONVERGE CFD Flame Propagation



CONVERGE CFD Flame Propagation. Enables the study of the influence of engine geometry on turbulence and therefore, engine performance



Technical Accomplishments and Progress

Single Cylinder Engine (SCE) Development



X15 Engine

- Bore x Stroke: 137 x 169 mm
- 2.5 L per cylinder

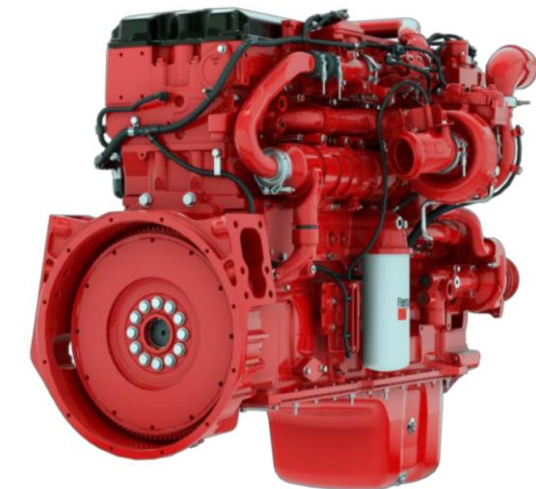
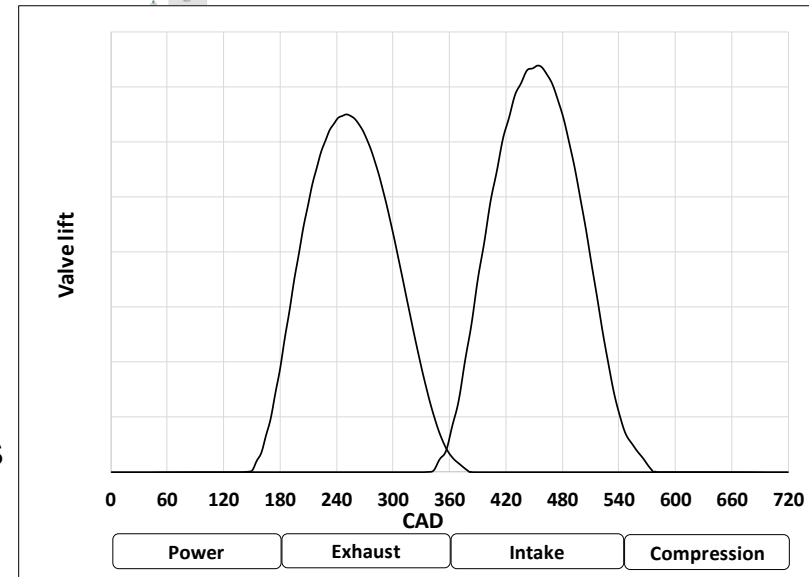
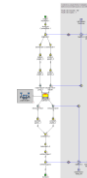
Parts to remove

- Diesel fuel pump and filter
- Injectors and rails
- ECM
- EGR valve and cooler
- EGR crossover
- Turbocharger

Parts to replace or modify

- Pistons
- #6 cylinder liner
- Camshaft
- Bearings and seals
- Exhaust manifold
- Intake manifold

- GT-Power model is under development
- Simulations utilized to guide design and predict performance

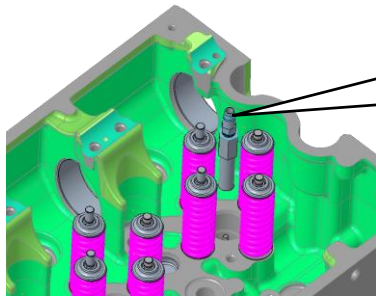


Technical Accomplishments and Progress

Single Cylinder Engine Development

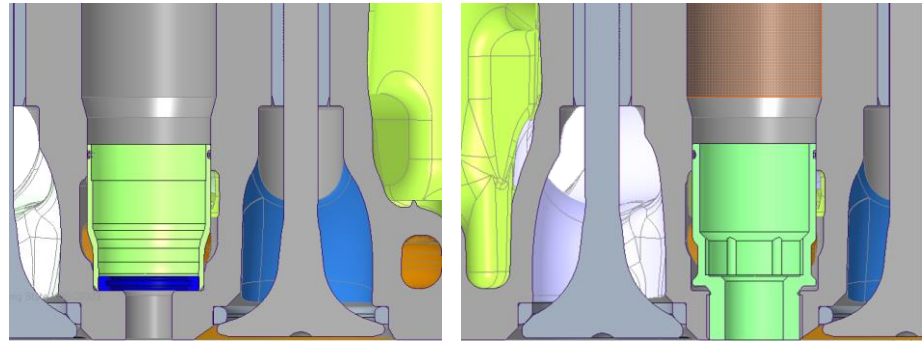


Cylinder pressure transducer
(AVL QC34C)

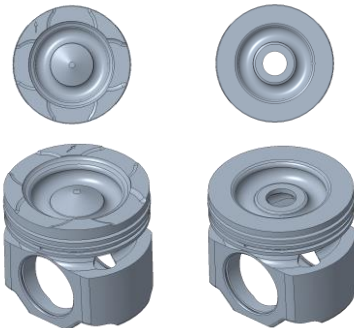


Cylinder pressure
transducer sleeve

Replace diesel injector with spark plug
adaptor



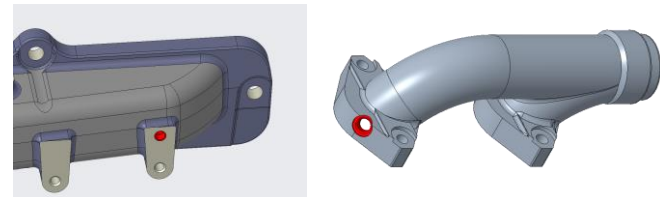
Piston modification



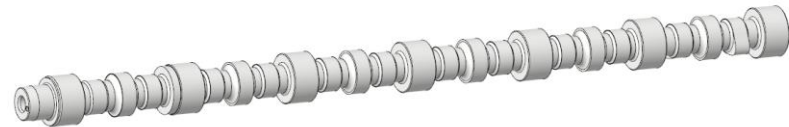
#6

#1-5

IMP and EMP pressure



New camshaft design



Collaboration and Coordination with Other Institutions



Prime Contractor: Colorado State University

PI: Daniel Olsen

Co-PIs: Anthony Marchese, Bret Windom

Students: Jeffrey Mohr, Andrew Zdanowicz, Diego Bestel, Scott Bayliff, Jack MacDonald

Sub-contractor: Cummins Inc.

PI: Hui Xu

Key Contributor: Robin Bremmer

Sub-contractor: Woodward, Inc.

PI: Greg Hampson

Key Contributors: Suraj Nair, Domenico Chiera

- Cummins team responsibilities:
 - Support RCM, CFR, and SCE experiments and modelling technical discussions
 - Build and deliver the SCE, support SCE installation, testing and modelling
- Woodward team responsibilities:
 - Technical guidance for 1-D simulation and CFD modeling and related testing
 - Program, install, and commission Large Engine Control Module (LECM) on CFR and SCE engines

Remaining Challenges and Barriers

Challenges

- Matching of CFR data with CFD, so CFD can be utilized for combustion chamber design for SCE
- Demonstration of controlled EGAI with high compression ratio and high EGR using the Woodward LECM
- Test cell setup for high EGR, advanced controls, and variable fuel composition
- Final fabrication of SCE and commissioning in test cell

Barriers

- No barriers identified at this time

Proposed Future Research

Budget Period 2 (2019-20)

- Complete CFD model validation with CFR and RCM data
- Apply CFD to SCE for combustion chamber design
- Install and commission 2.5 liter SCE at CSU
- Demonstrate baseline NG efficiency of 39% (Go/No-go)

Budget Period 3 (2020-21)

- Complete SCE mapping
- Final programming of LECM algorithm for real-time control
- Selection of final engine configuration and operating parameters
- Demonstration of diesel-like efficiency of 44% on SCE

Any proposed future work is subject to change based on funding levels.

Summary Slide

Approach

- Reduced chemical kinetic mechanism development in support of CFD modeling utilizing CFR engine and RCM
- Develop 2.5 liter SCE configuration: stoichiometric SI, turbocharged, high levels of cooled EGR, combustion chamber design for high burn rate, prechamber spark plugs, advanced engine controls

Technical Accomplishments and Progress

- Production of CFR engine and RCM experimental data for model development
- Development and demonstration of EGR cart on CFR engine
- Development of reduced kinetic mechanism (~50 species)

Next Steps

- Finalize model validation with CFR engine data
- Perform modeling in support of SCE combustion chamber design
- Install and commission SCE at CSU
- Collect baseline performance data