# **High Efficiency Cold Climate Heat Pump**

2014 Building Technologies Office Peer Review





Development









Energy Efficiency & Renewable Energy

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### **Project Summary**

# High Efficiency Cold Climate Heat Pump -(CCHP) CRADA

#### <u>Timeline</u>:

Start date: **01-Oct-2010** 

Planned end date: 30-Sep-2015

Key Milestones (single-stage)

1. Equipment modeling and EnergyPlus simulation

report - March/2013

Lab prototype fabricated and installed -Dec/2013

3. Meet 77% capacity at-13°F vs. 47°F; COP=4.1 at 47°F - March/2014

#### **Budget**:

Total DOE \$ to date: \$2,299k

Total future DOE \$: \$300k

#### **Target Market/Audience**:

The principal target market is 14.4 M electric-heated dwellings using 0.16 quad/year for heating in cold regions.

#### **Key Partners**:

CRADAs with both Emerson/Copeland and Unico:



**Equipment:** Two-stage compression system development



Solution: Single-stage compression system development and assessment

#### **Project Goal**:

Develop a 3-ton, high efficiency CCHP to minimize resistance heating

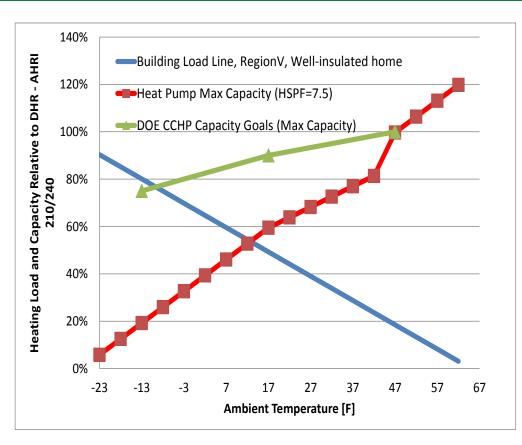
- Achieve COP@47°F > 4.0; achieve capacity@-13°F > 75%, vs. rated capacity@47°F.
- Maximize COP at 17°F and -13°F with acceptable payback period.



### **Purpose & Objectives**

#### **Problem Statement:**

- Typical HPs don't work well at low ambient temps due to very high discharge temp and pressure ratio
- HP heating capacity not sufficient to match building load
- COP degrades significantly with ambient temperature



From March 2012 Buildings Energy Data Book

**Target market/audience:** The principal target market is 14.4 M electricheated dwellings using 0.16 quad/year for heating in cold regions.



### **Purpose & Objectives**

### **Impact of Project:**

- 1. High-performance CCHPs result in significant savings (> 60% compared to electric resistance heating); result in an annual primary energy savings of 0.1 quads when fully deployed, equivalent to 5.9 million tons of annual CO<sub>2</sub> emissions reduction Long Term
- 2. Analytical tools development to facilitate best practice of CRADA partners Near Term
- 3. Project outcomes (single-stage) transferred to US OEMs by Emerson Climate Technologies Mid Term

### Technical goals (project outputs):

Develop a 3-ton, CCHP to minimize resistance heating

- COP@47°F > 4.0; HP capacity@-13°F > 75%, vs. rated capacity@47°F.
- Maximize COPs at 17°F and -13°F with acceptable payback period.



### **Approach**

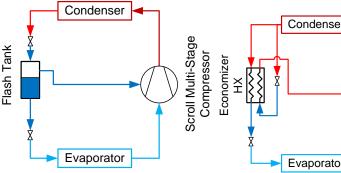
### Approach:

Market assessment and extensive vapor compression system modeling

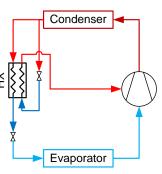
EnergyPlus building energy simulation

#### Single-stage compression concepts:

Vapor injection with flash tank



Vapor injection with economizer

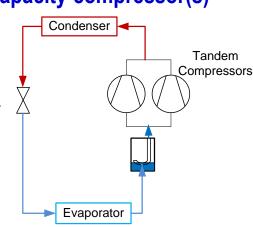


**Ejector cycle** 

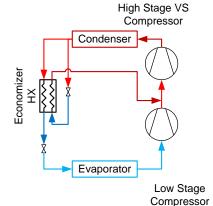
Condenser Compressor Scroll Multi-Stage Compressor **Ejector Evaporator** 

Tandem (parallel) and multicapacity compressor(s)

peland Scroll



Two-stage (series) compression w/ inter-stage economizing



The Unico System<sup>®</sup>

Small-Duct Central Heating & Air Conditioning

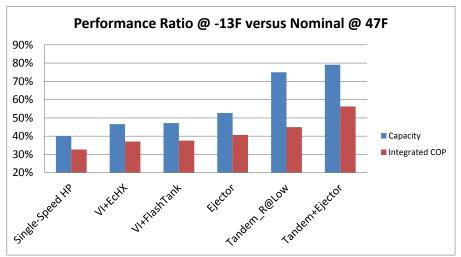
ORNL to assist in modeling, analysis, component sizing, and performance verification testing



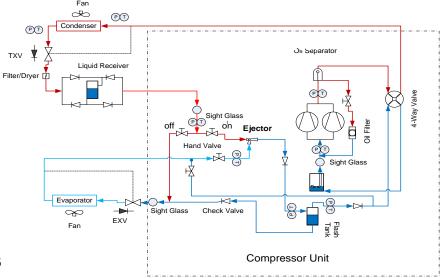
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### **Approach**

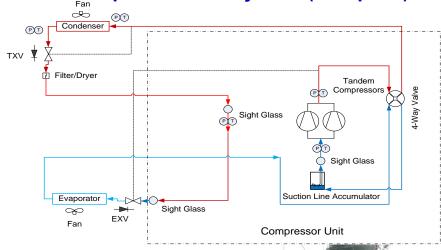
### Approach: Single-stage lab prototype system development and testing

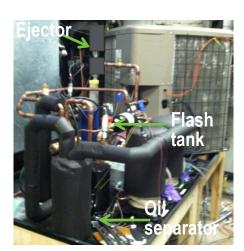


Established test rig to test multiple configurations (EXV vs. ejector, vapor injection system, etc.)



#### **Equal Tandem System (Complete)**









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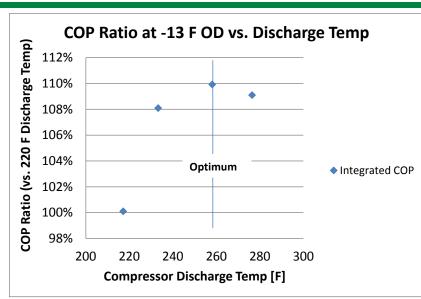
### **Approach**

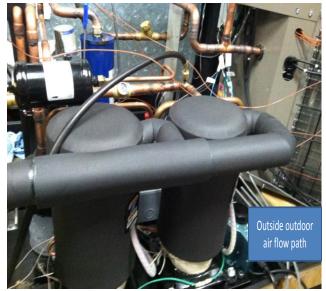
#### **Key Issues – Equal Tandem System:**

- Compressor(s) capable of high discharge temperature (up to 280°F)
- Tandem compressors provide overcapacity at low ambient temperatures, relative to 1 compressor running @47°F, rated point
- EXV controlling discharge temperature to optimize efficiency over a wide range
- Minimize heat losses from compressor shell and discharge line

#### **Distinctive Characteristics:**

- HP operates down to -20°F
- Significant reduction in supplemental resistance heat use, i.e. near zero in wellinsulated homes
- Tandem+EXV is a simple option with good reliability







### **Progress and Accomplishments**

Market Assessment

**Concept Design** 

Lab Prototyping

**Optimization** 

Field Verification

#### **Accomplishments:**

- Lab prototype with tandem + EXV developed and verified
- Established test rig to test multiple configurations
- Achieved capacity goals, i.e. >75% at -13°F (77%) → Go/No Go
   Milestone, 2014
- Achieved COP goal, >4.0 at  $47^{\circ}$ F (4.1)

#### **Progress on Goals:**

- Emerson Climate Technologies to optimize a compressor design for heating at low ambient temps (6% higher COP at low ambient temperatures)
- ORNL to test a tandem of vapor injection compressors
- Evaluate ejector cycle over an extensive operation range

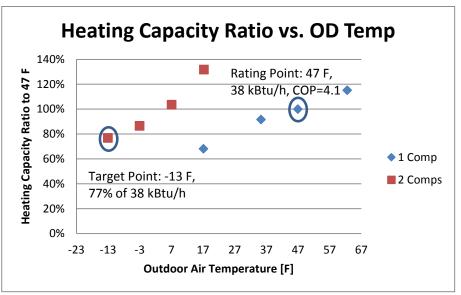
#### **Market Impact:**

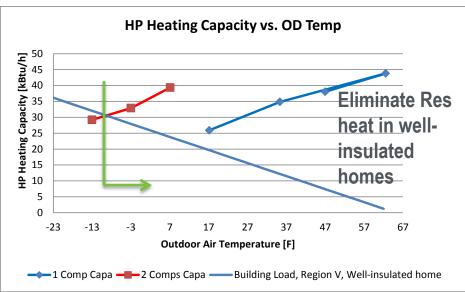
Emerson Climate Technologies to provide compressor and system solutions to US OEMs in the next three years

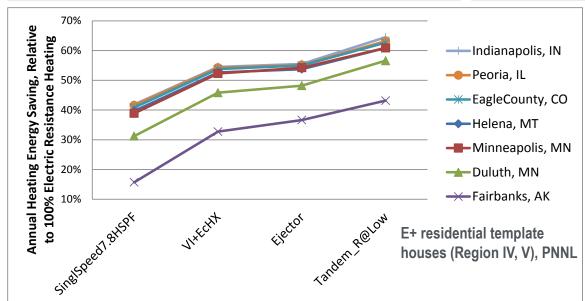


### **Progress and Accomplishments**

### Met Go/No Go Milestone (tandem + EXV):









### **Project Integration and Collaboration**

### **Project Integration:**

**Single-stage solutions:** Collaborative R&D agreement (CRADA) with Emerson Climate Technologies (US component and control). World leader of compressor technologies. Outcomes of the project will be transferred to US OEMs.

**Two-stage compression equipment:** Provide design, analysis, laboratory and field testing support to Unico INC. (CRADA).

**Partners, Subcontractors, and Collaborators:** CRADA partners Emerson Climate Technologies and Unico Inc.

#### **Communications:**

- Market assessment report (ORNL research report, http://info.ornl.gov)
- 1 paper published in IEA Heat Pump Conference 2011
- 1 paper published in ASHRAE Winter Conference 2012
- 1 paper submitted to IEA Heat Pump Conference 2014
- International collaboration via IEA Heat Pump Program Annex 41 enables team to learn from recent developments in EU and Far East



### **Next Steps and Future Plans**

### **Next Steps and Future Plans:**

- Develop control strategy and design control board June 2014
- Complete fabrication of field test units and initiate field installation- Sept 2014
- Complete field testing report for heating and cooling seasons Sept 2015
- Emerson Climate Technologies to provide compressors and system solutions to US OEMs - 2015



## REFERENCE SLIDES



### **Project Budget**

**Project Budget**: DOE total \$2,599k - FY11-15

Cost to Date: ~\$1,680k through February 2014 (FY11-\$206k; FY12-\$304K; FY13-

\$678k; FY14-\$492k)

Additional Funding: None expected

Budget History											
FY2011 — FY2013 (past)			014 rent)	FY2015 (planned)							
DOE	Cost-share	DOE	Cost-share	DOE	Cost-share						
\$1849k	*	\$450k	*	\$300k	*						

 In-kind contribution from CRADA partner –exceeds DOE funding level; exact total is confidential information



### **Project Plan and Schedule**

Original initiation date: 01-Oct-2010 -- Planned completion date: 30-Sept-2015 (minor delays in lab prototype fabrication and testing schedules) Go/no-go decision points

- \* March '13 Equipment Modeling and Building Energy Simulation *Passed*
- \* March '14 Achieve 75% capacity at -13°F *Passed*
- \* late CY14/early CY15 proceed to field testing (ORNL)

	Milestone/Deliverable (Originally Planned) use for missed										
•	Milestone/Deliverable (Actual) use when met on time										
	FY2013			FY2014			FY2015				
Q1 (Oct-Dec)	Q2 (Jan-Mar)	Q3 (Apr-Jun)	Q4 (Jul-Sep)	Q1 (Oct-Dec)	Q2 (Jan-Mar)	Q3 (Apr-Jun)	Q4 (Jul-Sep)	Q1 (Oct-Dec)	Q2 (Jan-Mar)	Q3 (Apr-Jun)	Q4 (Jul-Sep)
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