

Adaptive Control to Improve Low Temperature Diesel Engine Combustion

**Ming Zheng, Graham T Reader, Raj Kumar, Clarence Mulenga,
Usman Asad, Yuyu Tan, and Meiping Wang**

**Clean Diesel Engine Research Laboratory
University of Windsor**

U.S. Department of Energy
2006 Diesel Engine Emission Reduction Conference
Detroit, Michigan, August 21-24 2006

Contact: mzheng@uwindsor.ca

Acknowledgements: *International Truck and Engine Corporation, Ford Motor Company, Imperial Oil Canada, NI, SIEMENS, BOSCH, DELPHI, NRCan, CRD and:*



Canada Research
Chairs



**NSERC
CRSNG**



Canada Foundation
for Innovation
Fondation canadienne
pour l'innovation



AUTO21

Research Themes

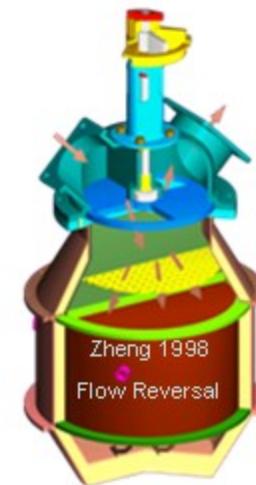
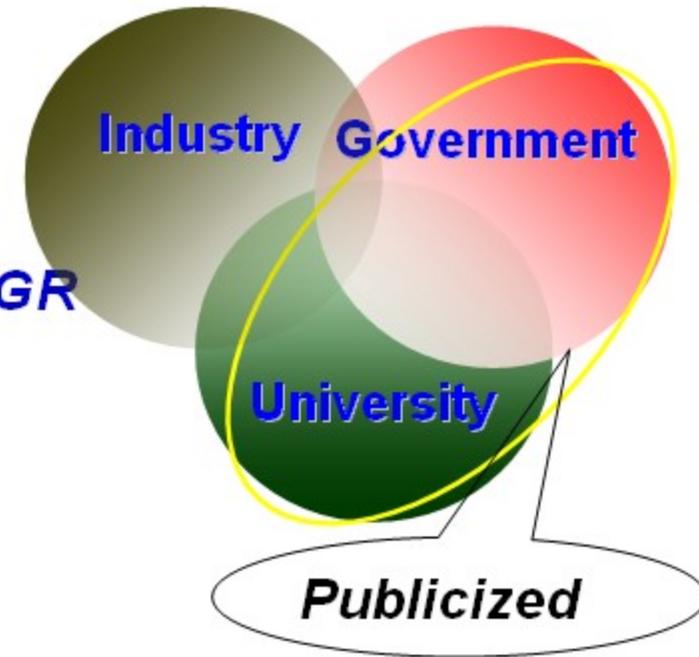
Premixed Diesel Combustion

- *Low temperature combustion with high EGR*
- *Diesel HCCI*
- *On-fly combustion control*

Active Flow Control Aftertreatment

- Flow reversal, parallel flow, flow stagnation
- Active flow control valve innovation
- Real-time control

Modeling, Diagnosis, and Dynamometer Tests



Diesel Low Temperature Combustion

- Heavy EGR
- Late Injection
- Multiple early injection

Ignition delay {
soot

from end of injection > 1.5 ms
or
50% prolonging from conventional

< 15 ppm NOx & near zero

IMEP > 6bar

Diesel HCCI

Cylinder charge:
Locally lean
Halved O₂

Adaptive Control

Navigate through narrow operating corridors

Phasing

Injection scheduling
Boost modulation

Efficiency Emission

Power
Robust
Roughness

Mode Switch

HCCI Diesel

Load

Idle
Low
High
Full

Low EGR
Heavy EGR

High EGR
High EGR

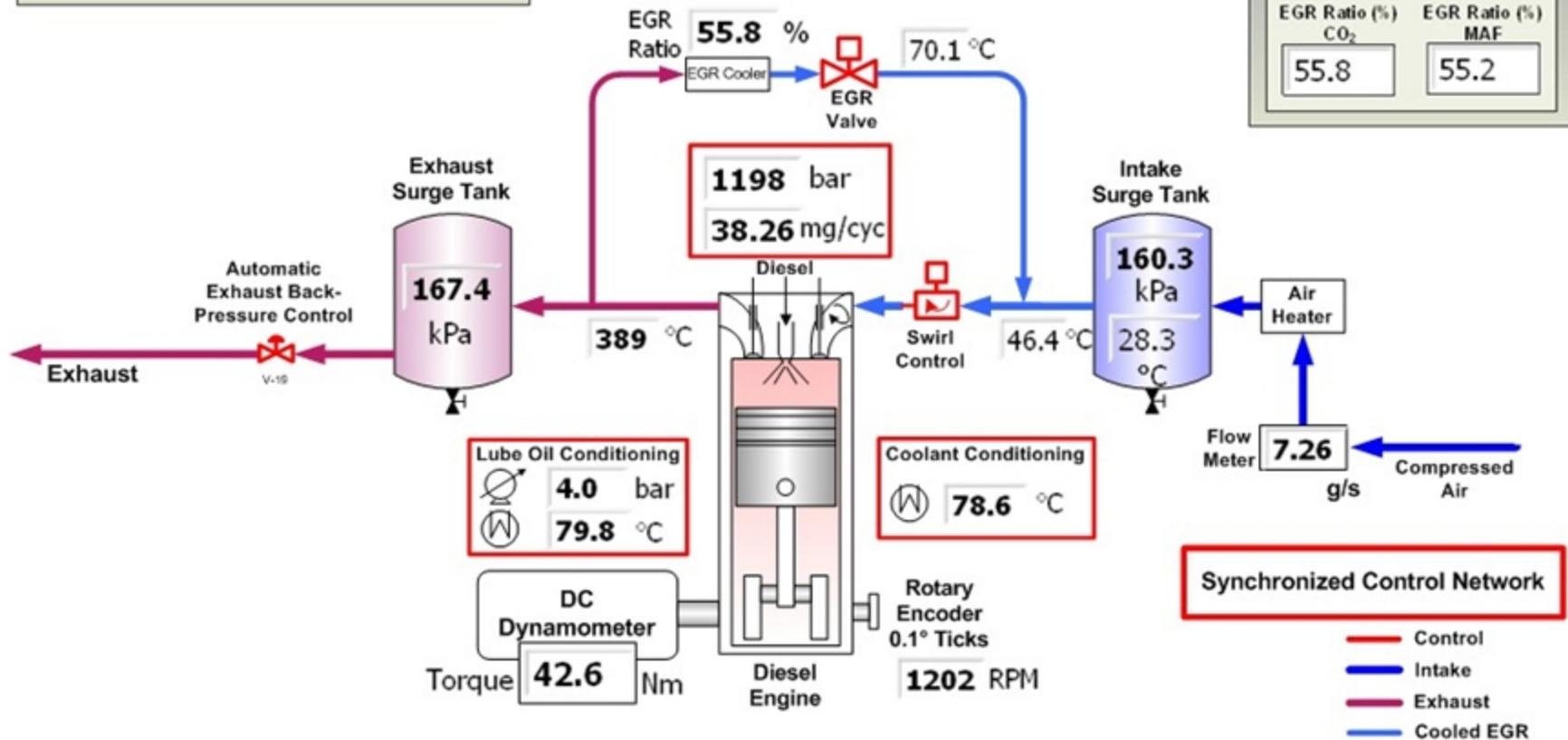
University of Windsor Engine Test System

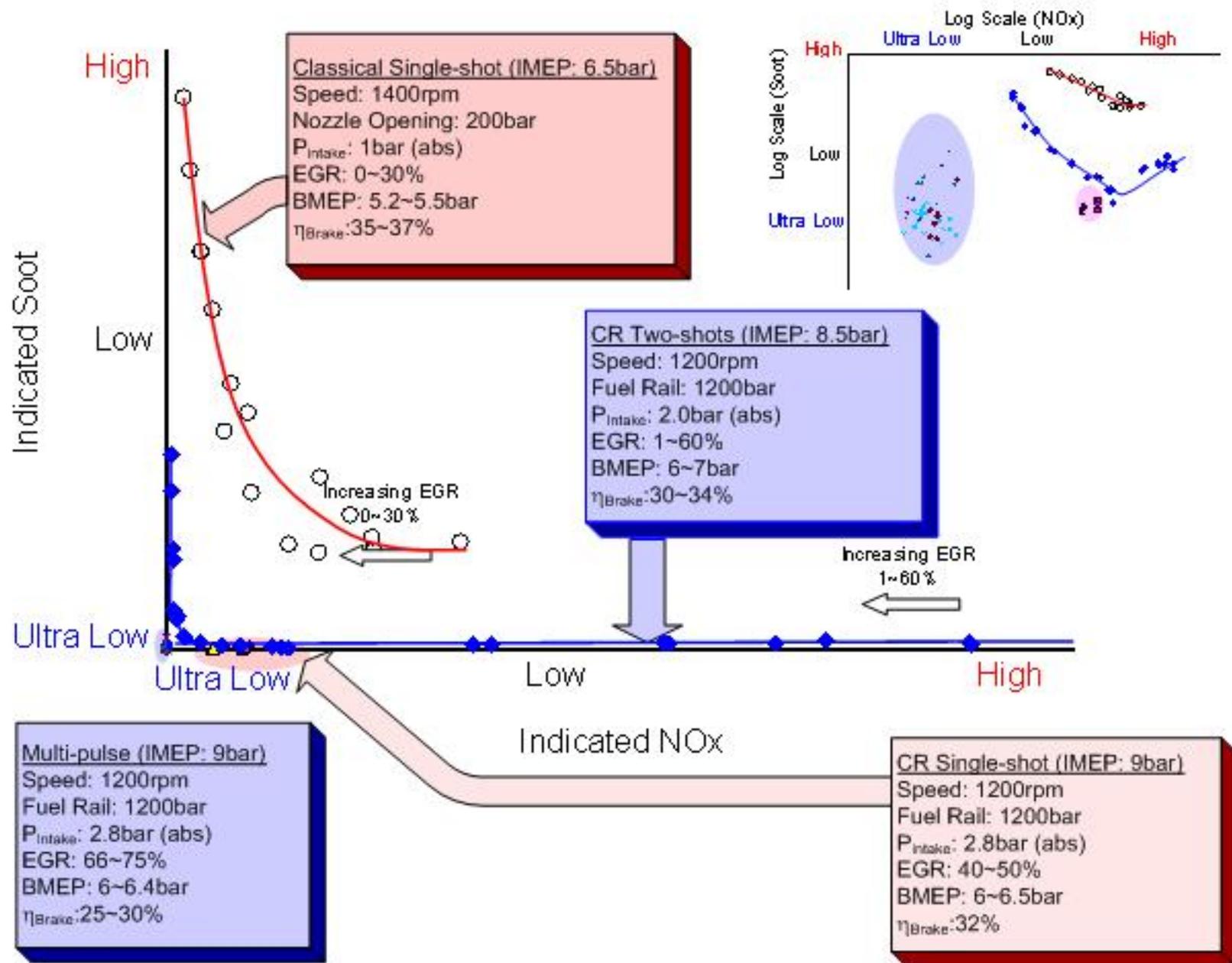
Sync Manager | Online Data Processing | Online Analysis | Network Status

EXHAUST GAS		
CO ₂ (0 - 40.00 %)	CO (0 - 5000 ppm)	O ₂ (0 - 25.00 %)
12.26	1982	4.13
HC (0 - 3000 ppm)	NOx (0 - 3000 ppm)	Smoke (FSN)
305	23	0.017

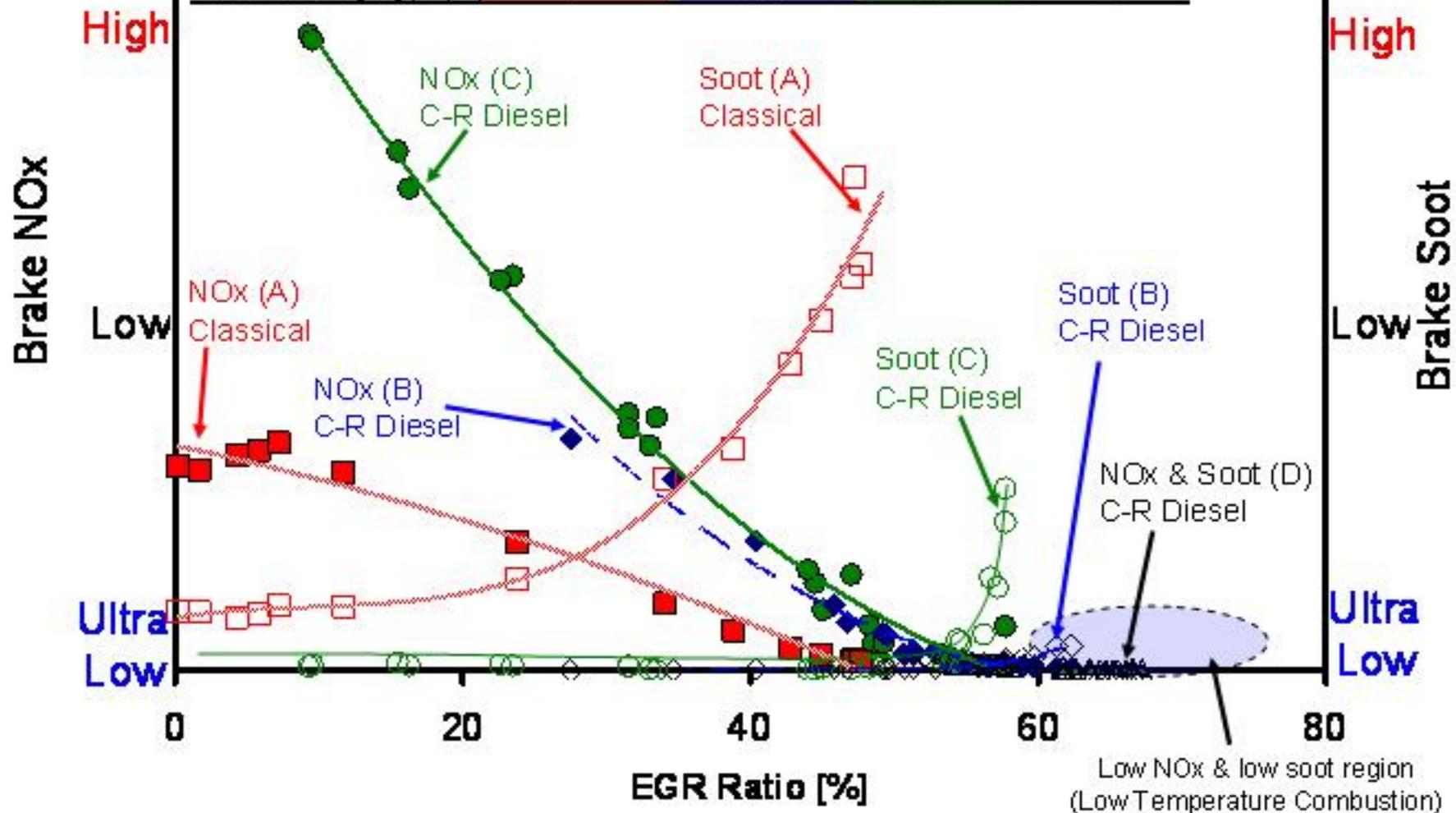
INTAKE CHARGE	
CO ₂ (0 - 10.00 %)	O ₂ (0 - 25.00 %)
6.28	10.88

Engine Parameters	
Oil Pressure (psi)	Coolant Temperature (°C)
46.83	78.6
IMEP (bar) 200 cyc Average	Exhaust Back Pressure (kPa)
9.30	7.08
EGR Ratio (%) CO ₂	EGR Ratio (%) MAF
55.8	55.2





	A	B	C	D
Engine Type	Classical Diesel	Common-Rail	Common-Rail	Common-Rail
No. of Injections	1	1	2	6
BMEP [bar]	3.4	4.4	6.1	3.6
Fuel Pressure [bar]	NOP: 200	Rail: 1200	Rail: 1200	Rail: 1200
Intake Pressure [bar] (abs)	1	2.75	2.0	2.7



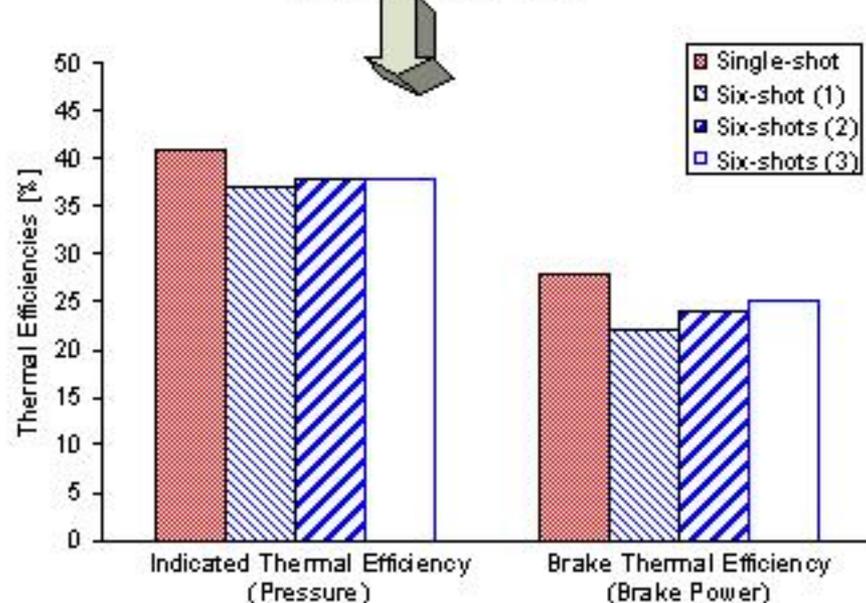
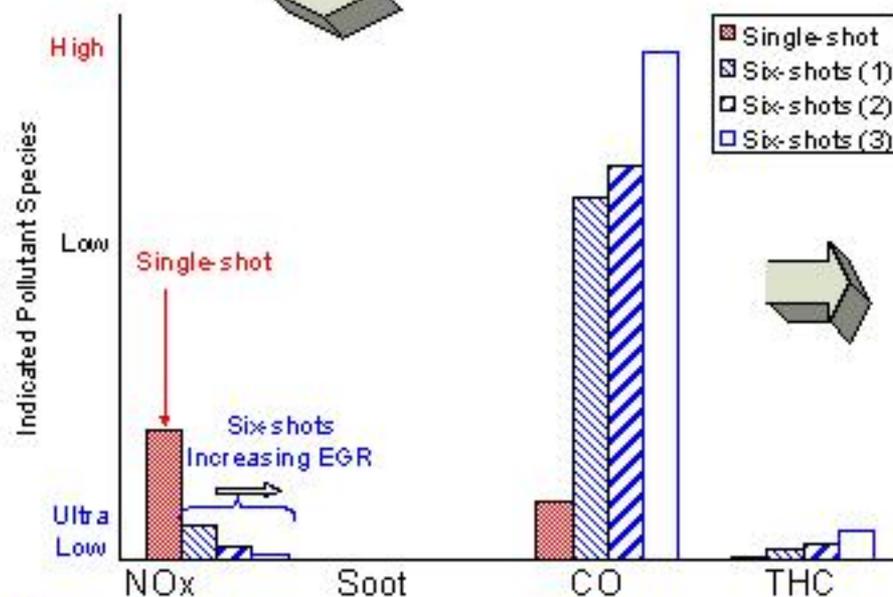
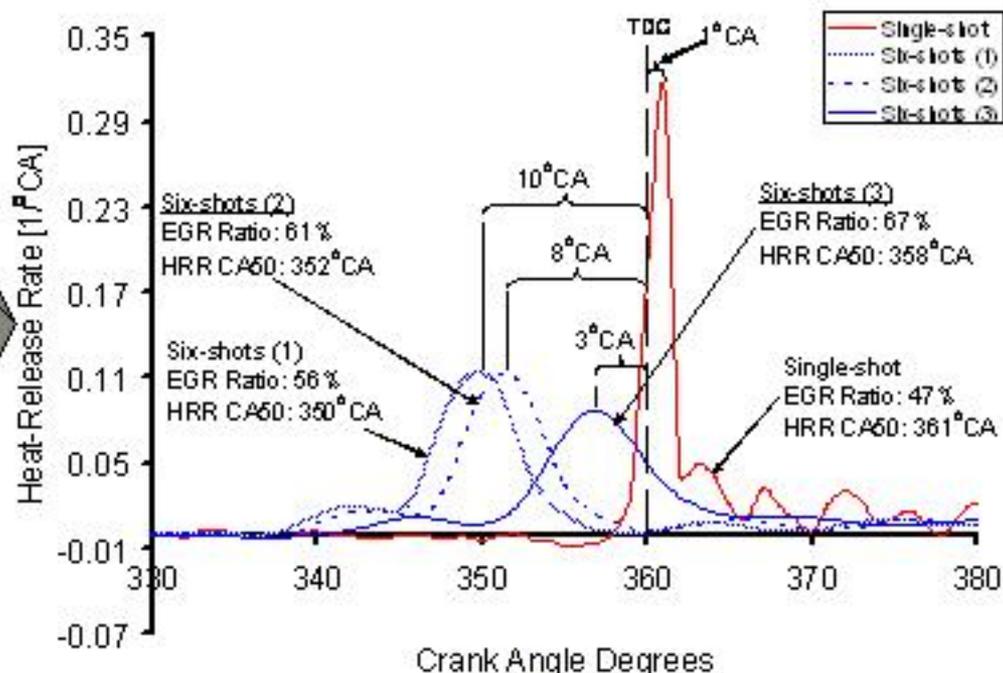
LOW TEMPERATURE COMBUSTION

Test Engine Condition

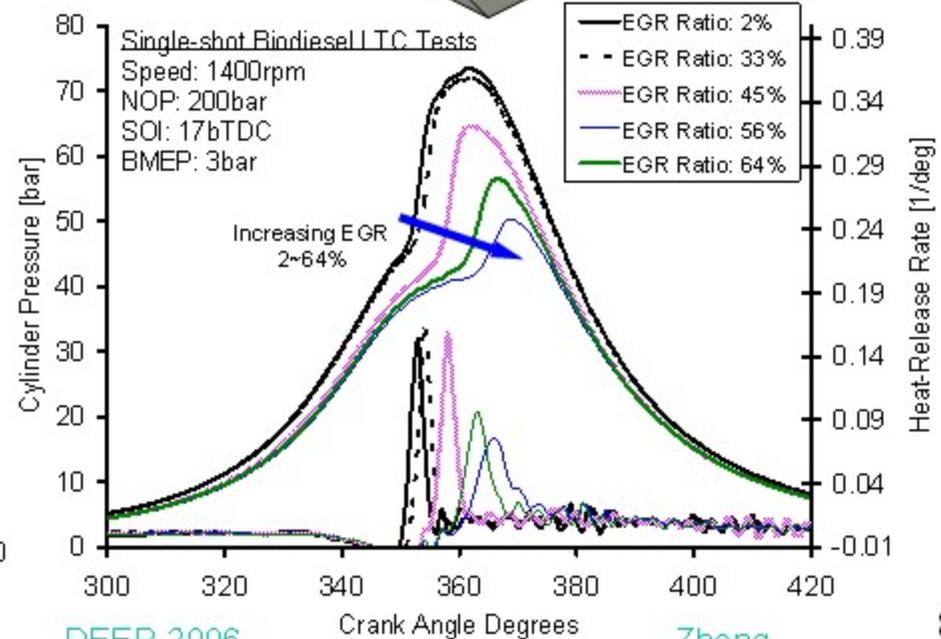
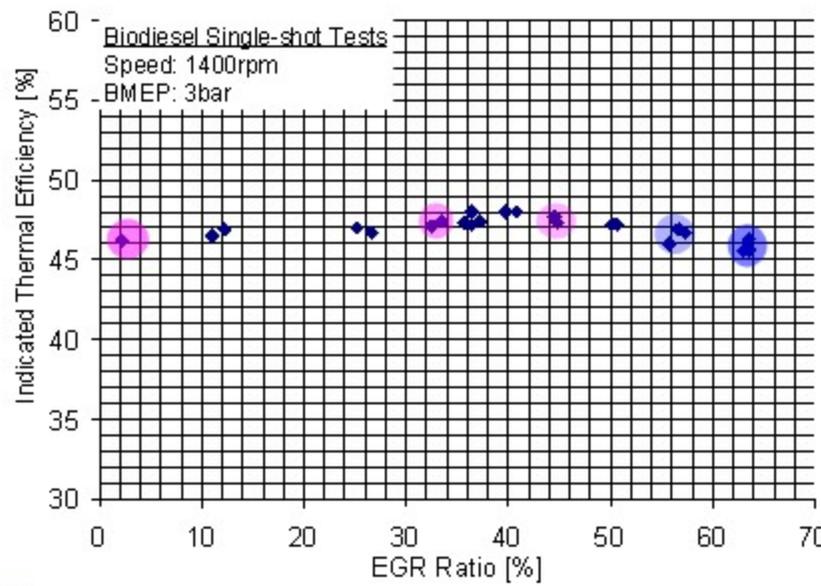
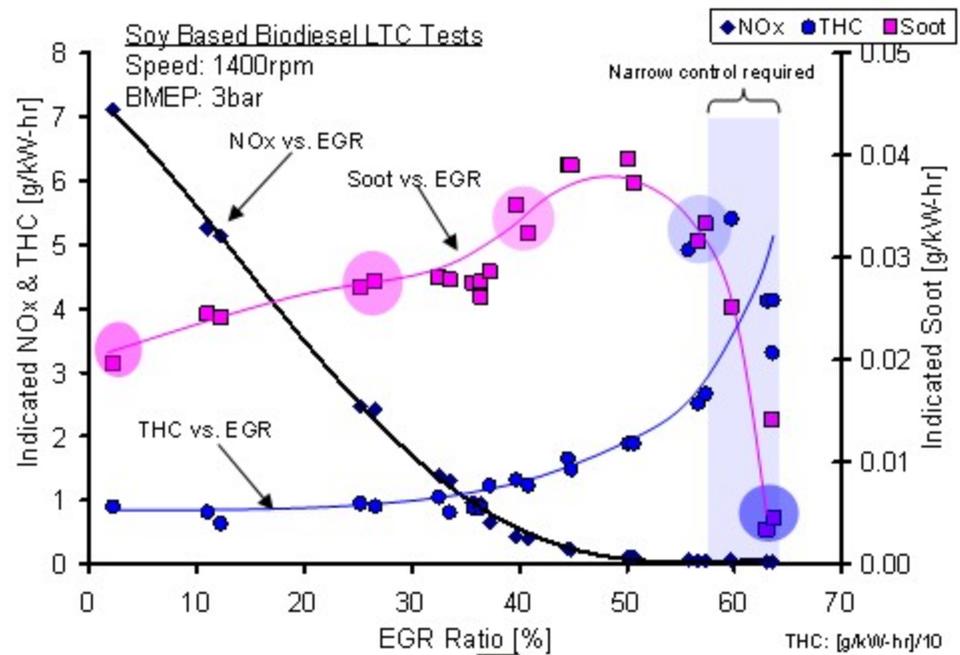
Intake Pressure: ~2bar (abs)
 EGR Ratios: 50~68%
 Fuel Rail Pressure: 1200bar
 Engine Speed: 1200rpm

Injections

Single-shot vs multi-pulse

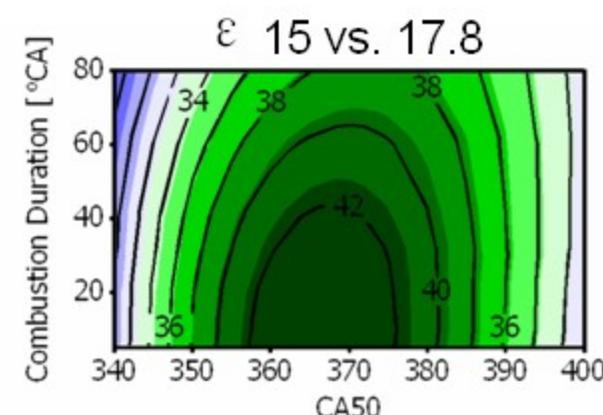
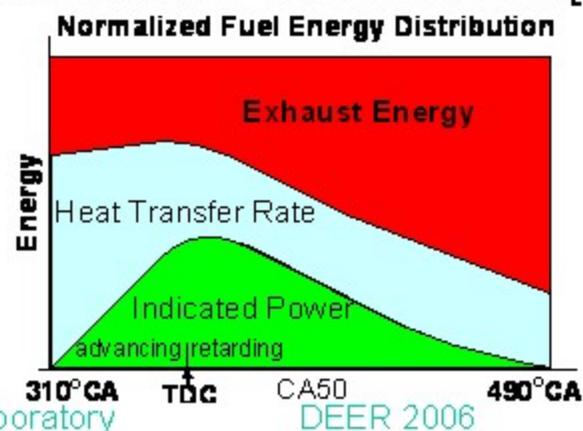
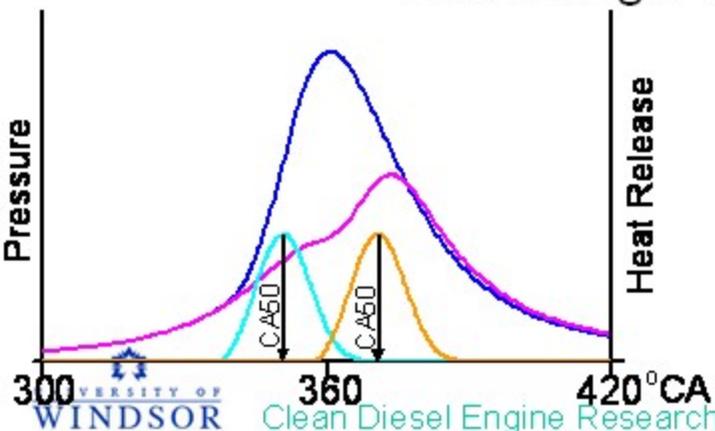
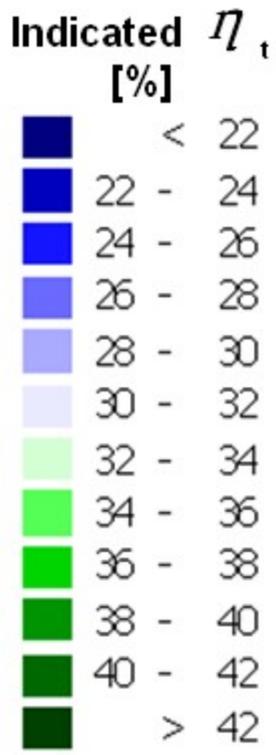
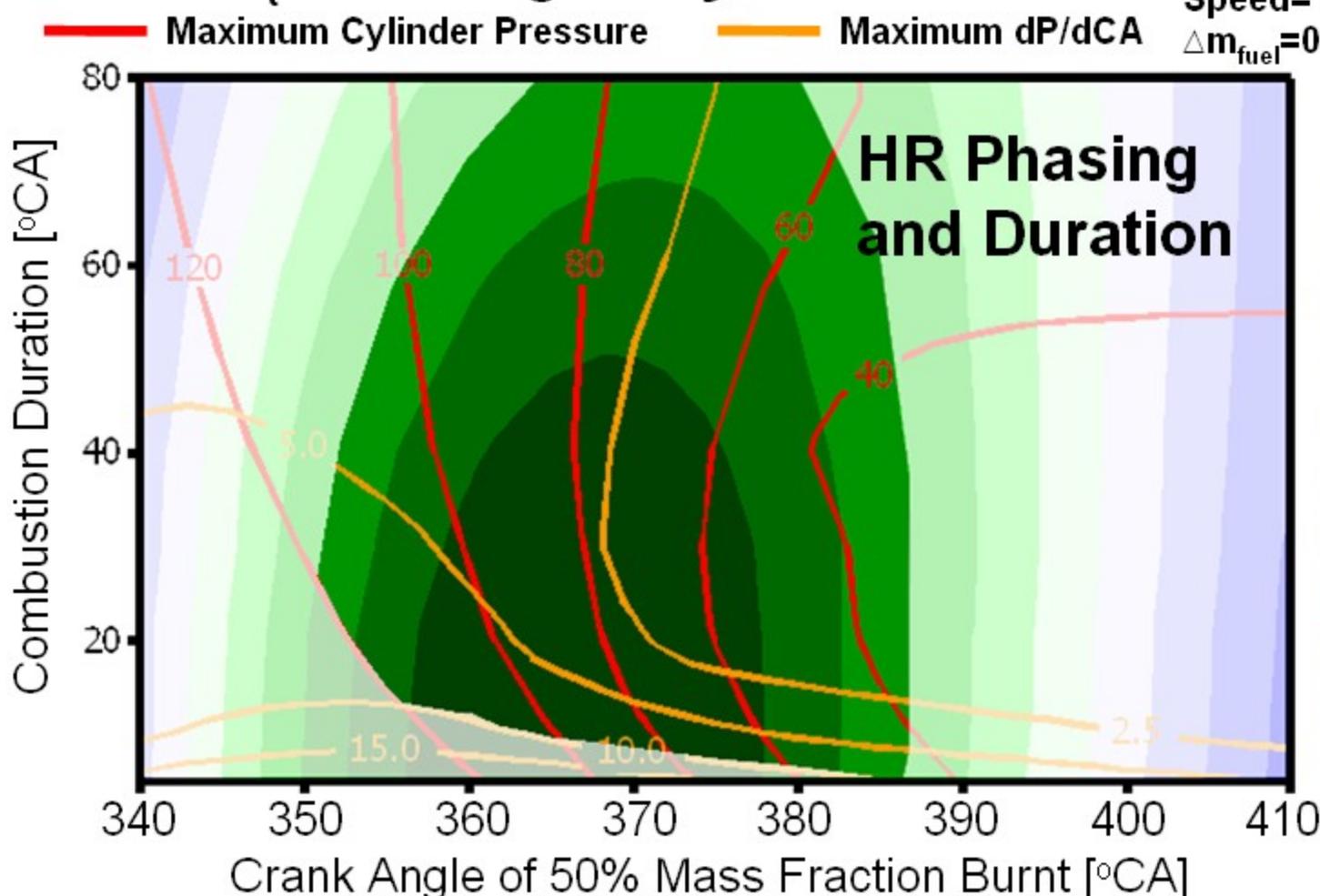


Soy Based Biodiesel LTC
 Speed: 1400rpm
 CR: 17.8
 BMEP: 3bar
 P_{Intake} : 1bar (abs)
 T_{Intake} : 30°C
 Nozzle Opening: 200bar
 SOI: 17°CA bTDC
 EGR: 2~64%

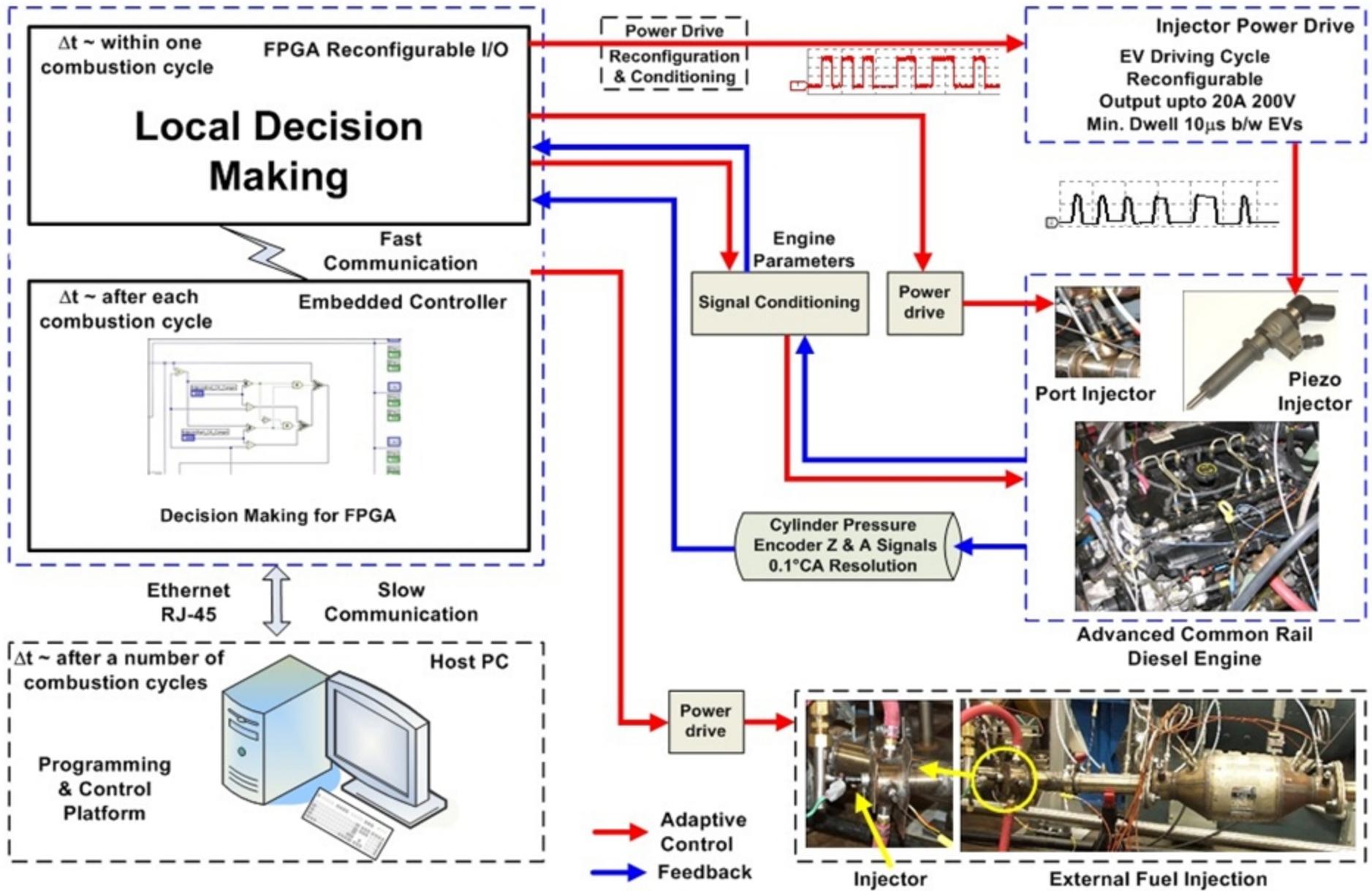


Indicated η_t from engine cycle simulation

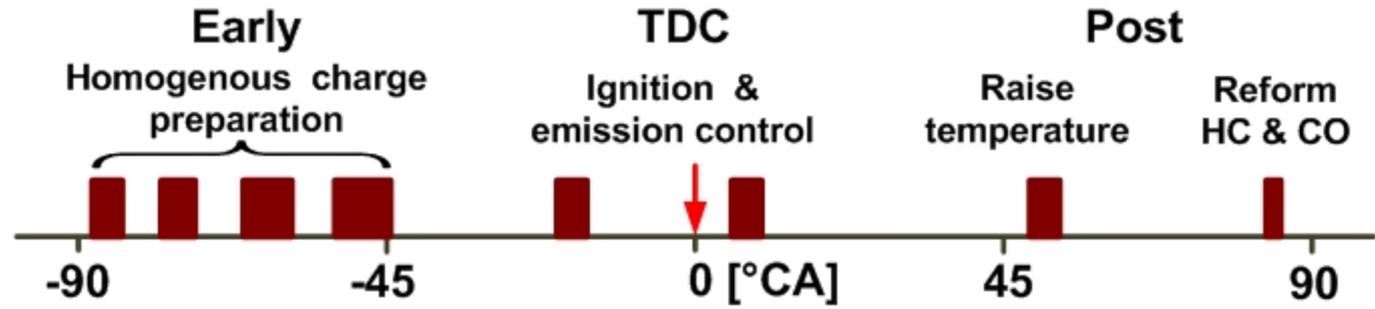
$P_{int}=1\text{bar}$, $T_{int}=300\text{K}$, $\epsilon = 17.8$
 Speed=1400 RPM
 $\Delta m_{fuel}=0.0335\text{ g/cycle}$



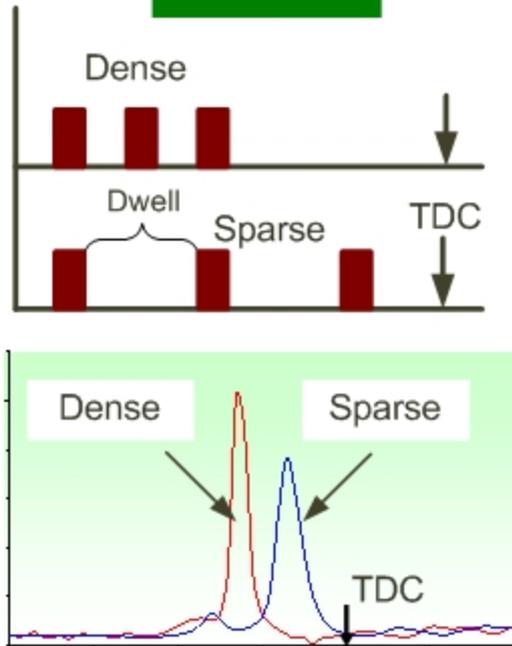
University of Windsor Adaptive Control System



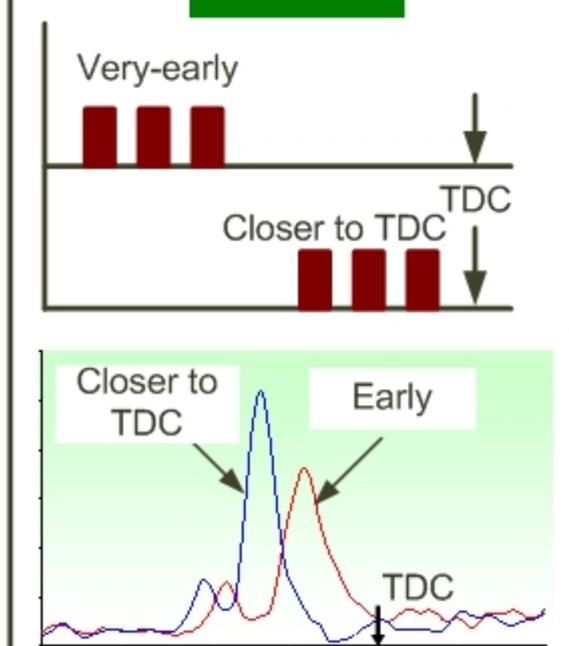
Fuel Injection Strategy Tests



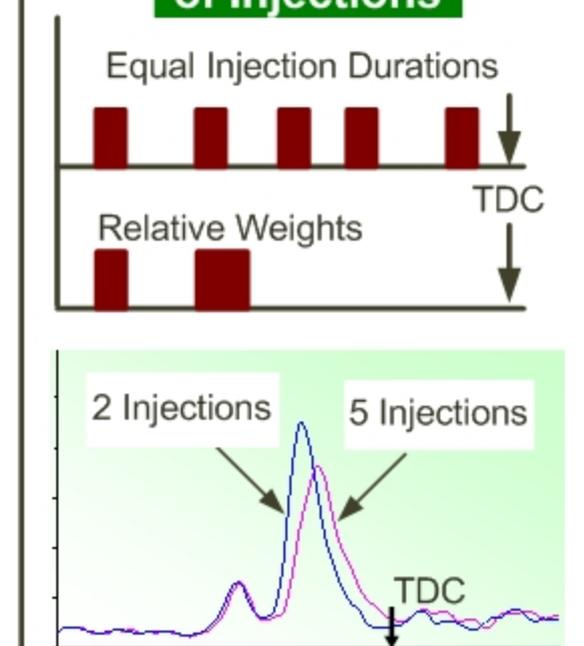
Dwell



Timing

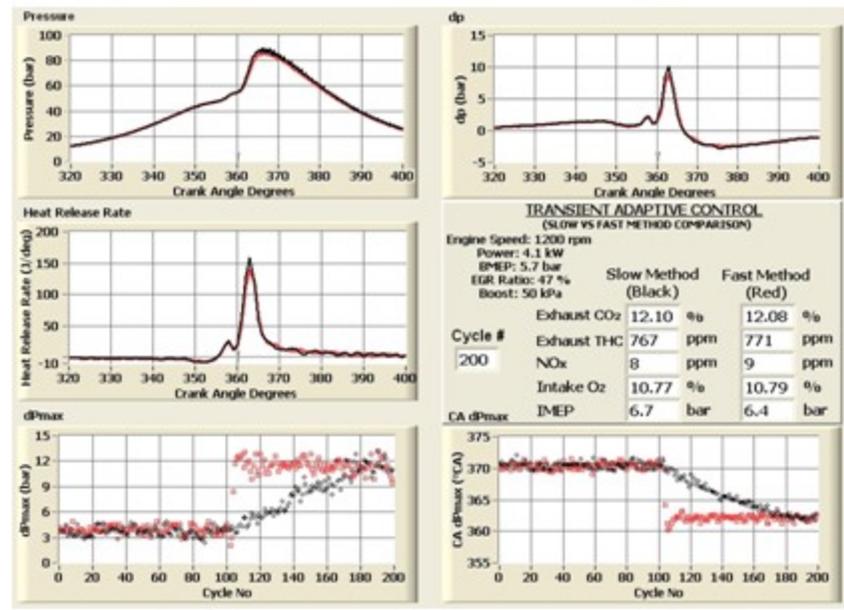
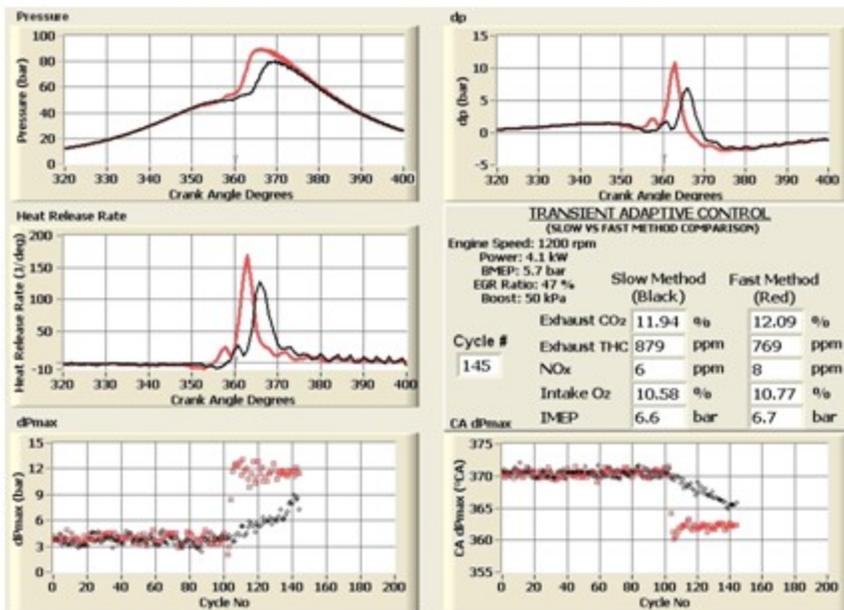
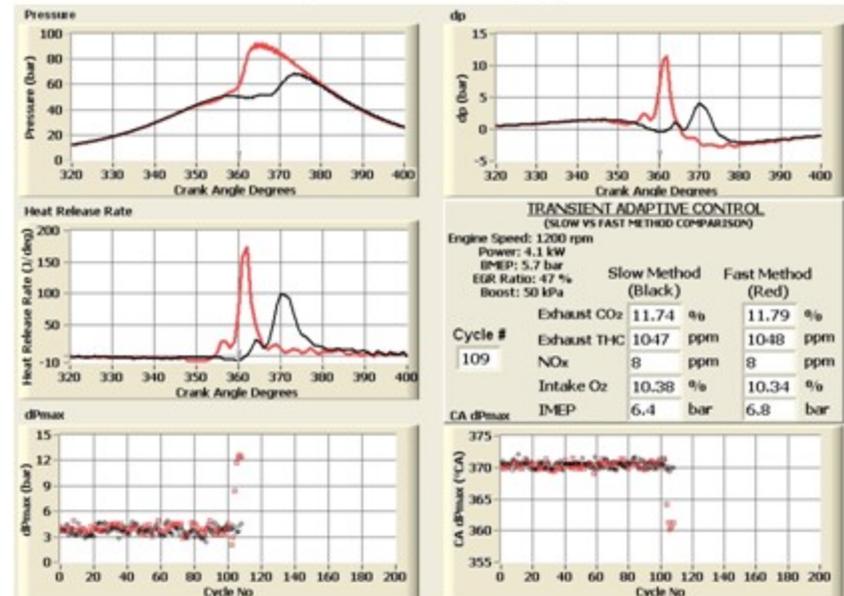
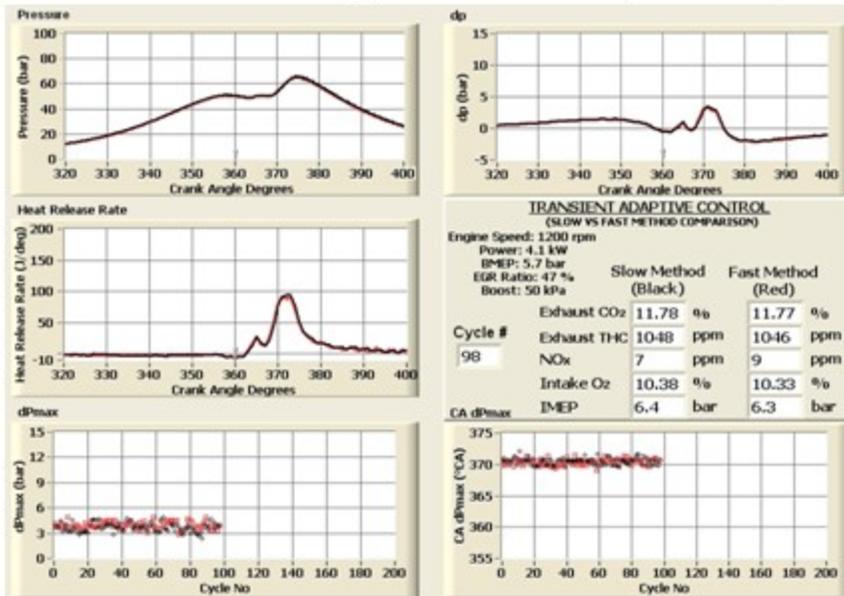


Number of Injections



Example-Adaptive Control Algorithm for Timing of $(dp/d\theta)_{max}$

Other Algorithms (P_{max} , CA50% HR etc) are Applicable

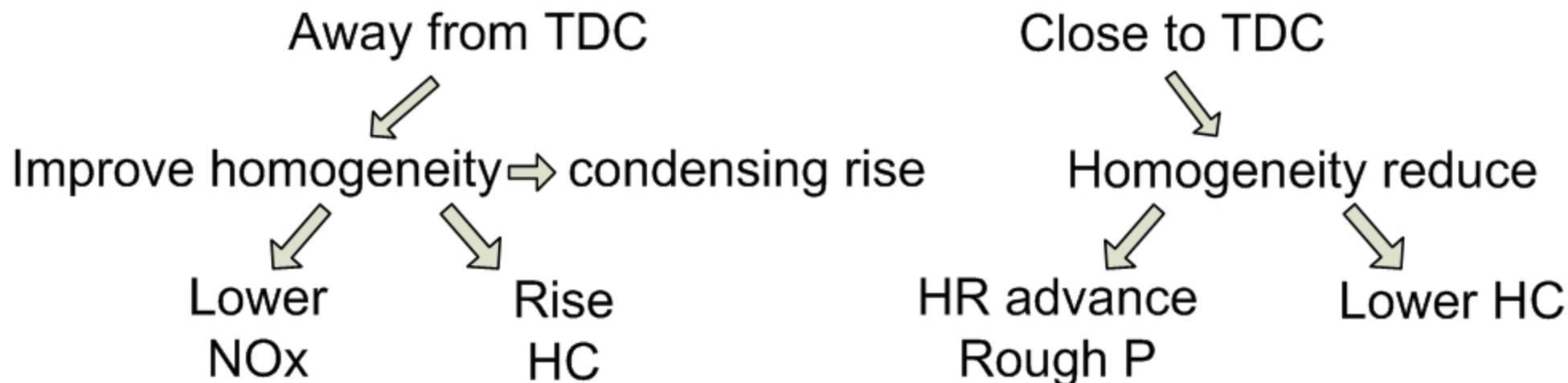


Conclusions:

1. Diesel HCCI Engine Fuel Energy Efficiency

- < Conventional diesel (Diesel+ SCR)
- ≈ Conventional diesel + heavy emission control
- > SI gasoline engine with emission control

2. Early Multi-event Injection



3. Minimum early injection for sufficient mixing but reduced condensing and adaptive combustion control is the key.

Thank you!