

# Thermal Strategies for High Efficiency Thermoelectric Power Generation

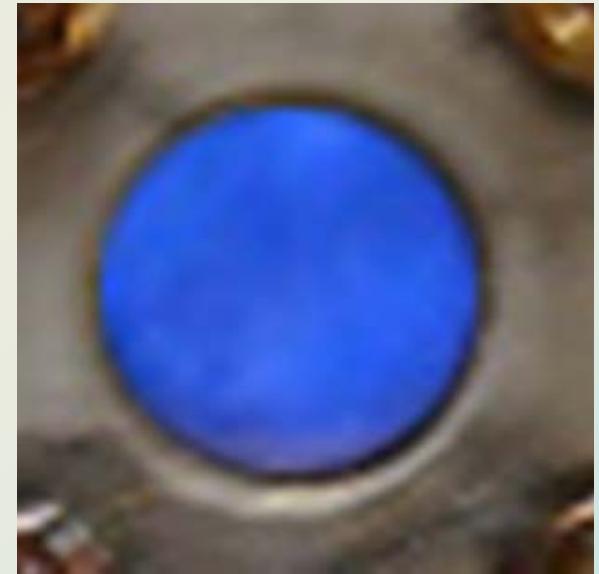
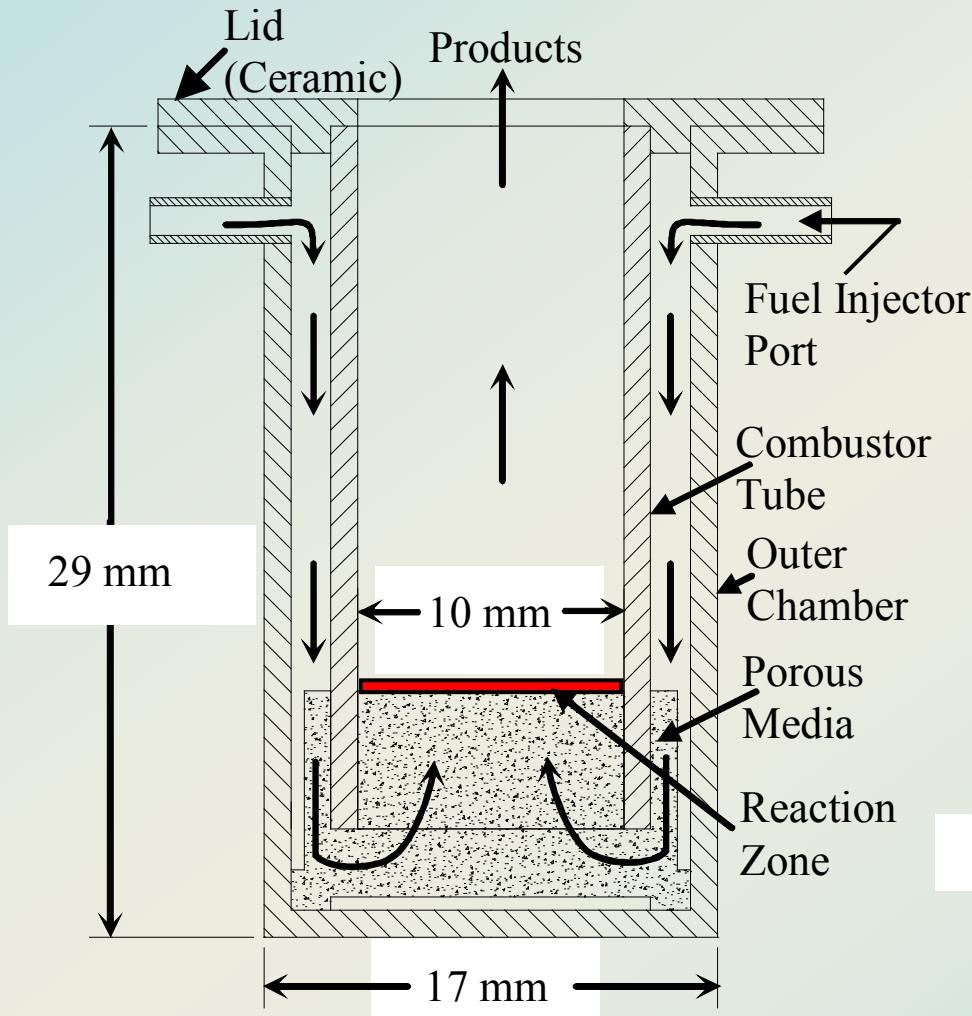
Ajay K. Agrawal

Department of Mechanical Engineering  
The University of Alabama, Tuscaloosa





# Portable Power Generation



**Blue Flame**

**Kerosene flow rate = 40 g/hr,**  
 **$\Phi = 0.65$ ,  $Q_{th} = 460 \text{ W}$**



# Thermoelectric Efficiency

TE Efficiency:

$$\eta_{TE} = \frac{W_e}{Q_h} = \eta_c \cdot \gamma$$

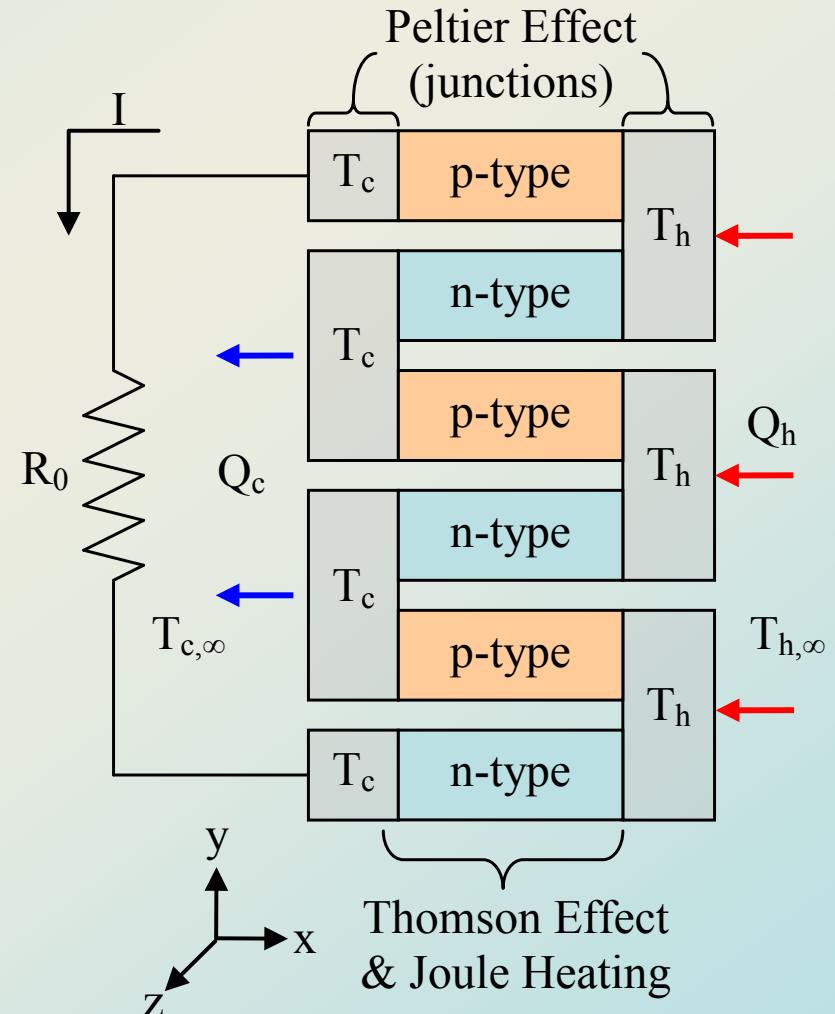
Carnot Efficiency:  $\eta_c = \frac{T_h - T_c}{T_h}$

TE Material Parameter:

$$\gamma = \frac{\sqrt{1 + Z T} - 1}{\sqrt{1 + Z T} + T_c / T_h}$$

TE Figure of Merit:  $Z(T) = \frac{\alpha^2(T)}{\rho_e(T) \cdot k(T)}$

$$T = \frac{T_h + T_c}{2}$$

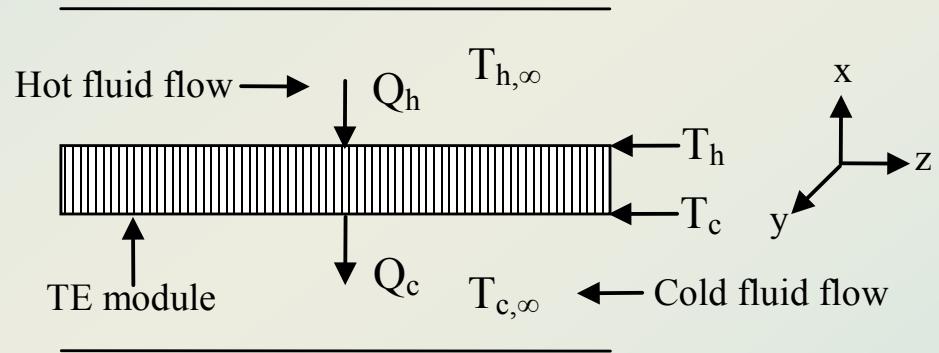




# Thermoelectric Efficiency

- TE Module Efficiency

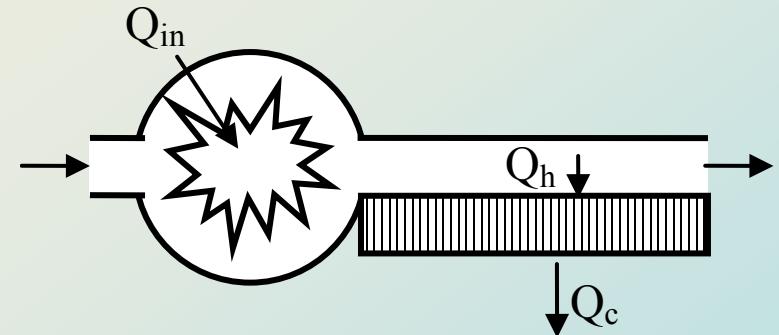
$$\eta_m = \frac{\sum W_e}{\sum Q_h}$$



- TE System Efficiency

$$\eta_s = \frac{\sum W_e}{Q_{in}} = \frac{\sum Q_h}{Q_{in}} \cdot \frac{\sum W_e}{\sum Q_h}$$

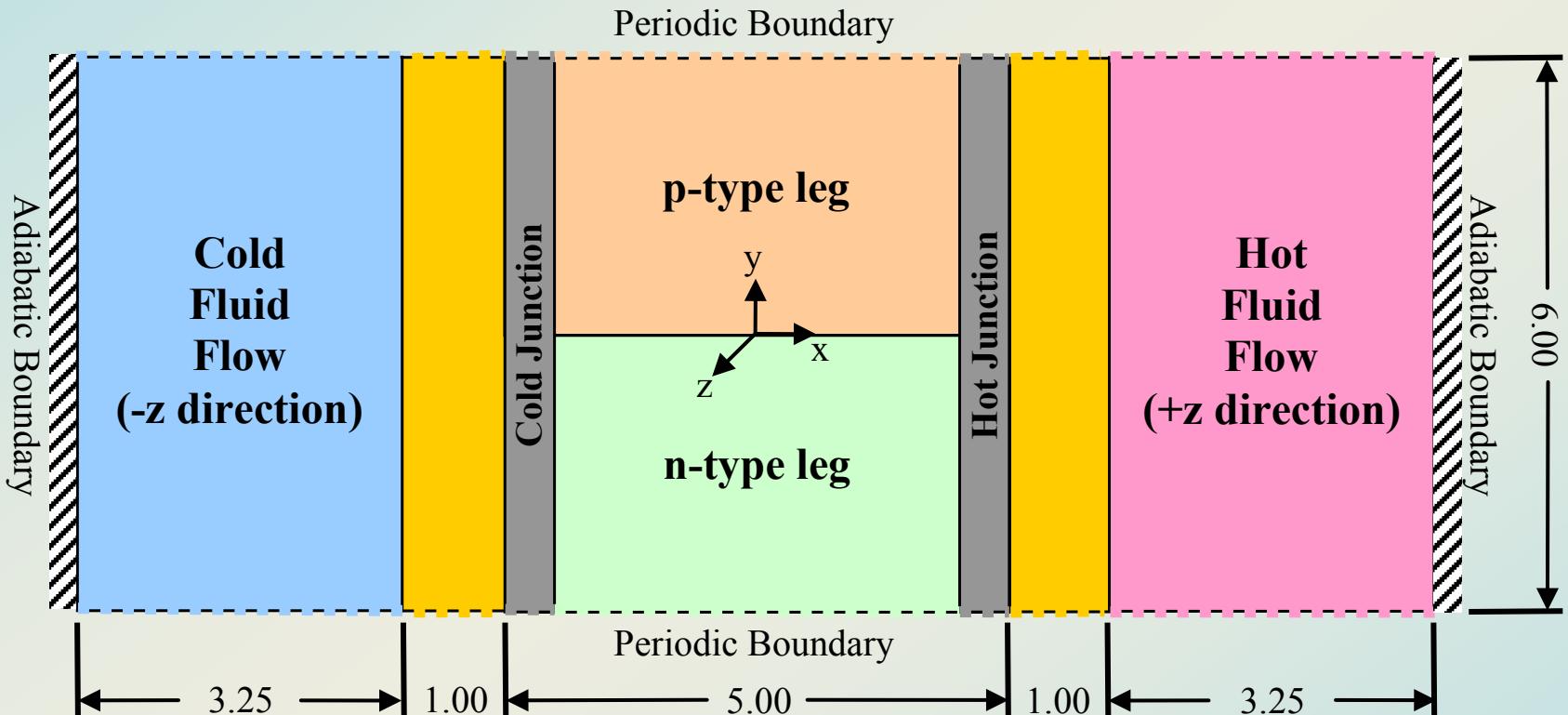
$$\eta_s = Q_R \cdot \eta_m$$



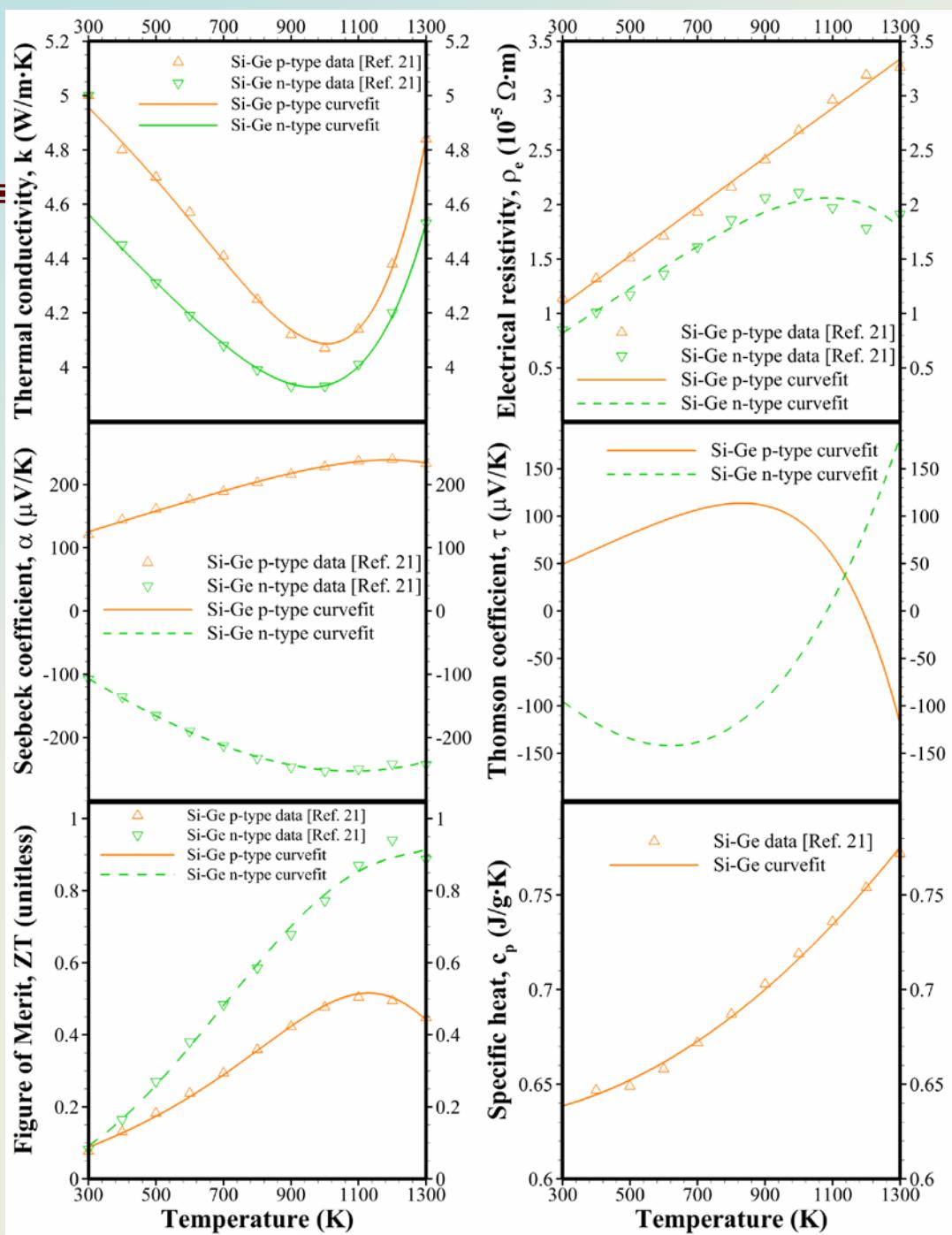
$Q_R$  is the Heat Input Ratio



# 3D CFD-TE Analysis



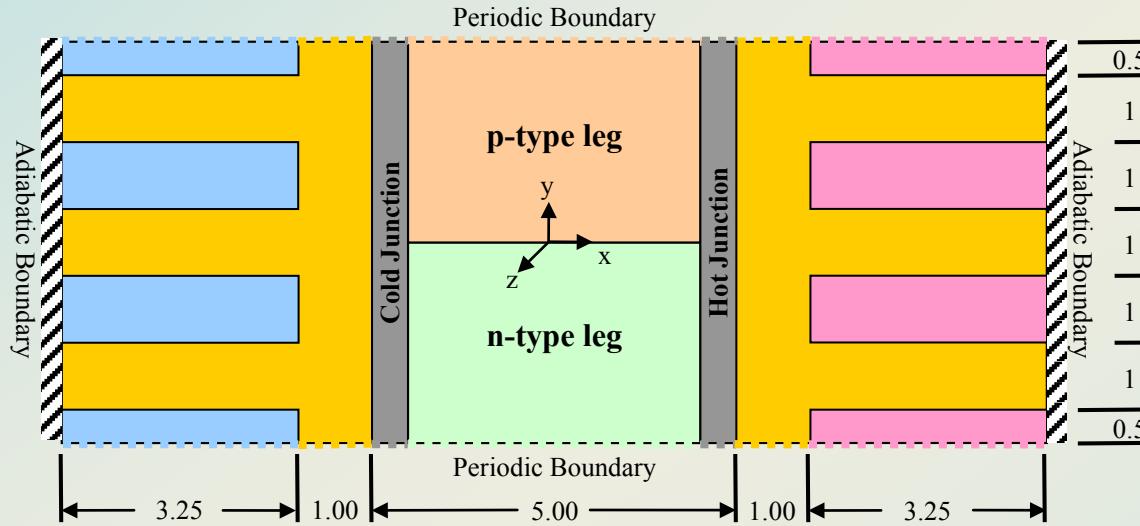
Baseline Configuration



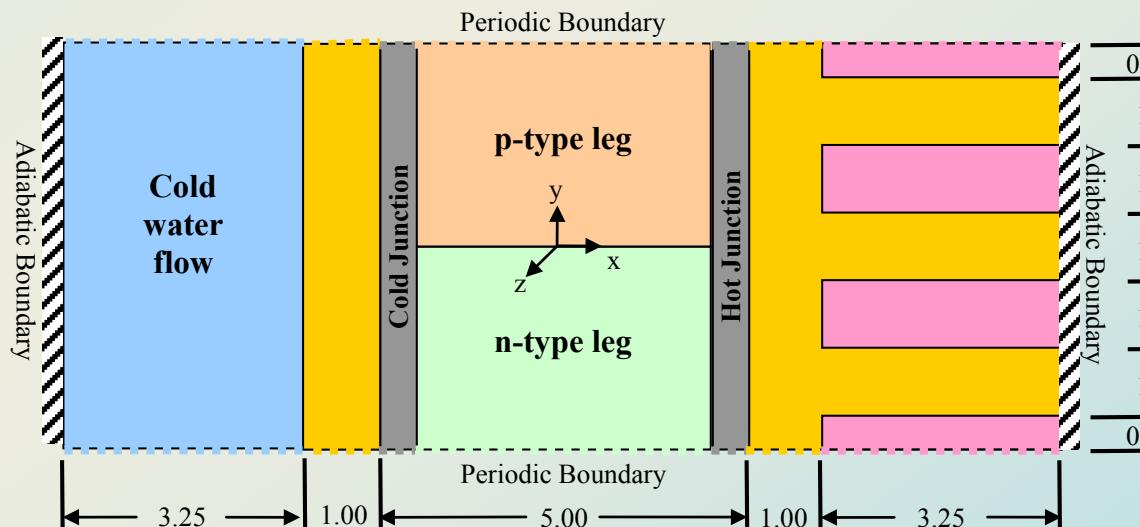
TE Material Properties  
Of Si-Ge taken from  
  
Handbook of  
Thermoelectrics



# Model Layout



Finned



Water-cooled



# CFD Governing Equations

- Conservation of Mass

$$\frac{\partial}{\partial z}(\rho v_z) + \frac{\partial}{\partial r}(\rho v_r) + \frac{\rho v_r}{r} = S_m$$

- Conservation of Momentum

$$\nabla \cdot (\rho v_x \vec{v}) = -\frac{\partial P}{\partial x} + \frac{\partial}{\partial x} \left( 2\mu \frac{\partial v_x}{\partial x} - \frac{2}{3} \mu \nabla \cdot \vec{v} \right) + \frac{\partial}{\partial y} \left( \mu \frac{\partial v_x}{\partial y} + \mu \frac{\partial v_y}{\partial x} \right) + \frac{\partial}{\partial z} \left( \mu \frac{\partial v_x}{\partial z} + \mu \frac{\partial v_z}{\partial x} \right) + S_x$$

$$\nabla \cdot (\rho v_y \vec{v}) = -\frac{\partial P}{\partial y} + \frac{\partial}{\partial x} \left( \mu \frac{\partial v_y}{\partial x} + \mu \frac{\partial v_x}{\partial y} \right) + \frac{\partial}{\partial y} \left( 2\mu \frac{\partial v_y}{\partial y} - \frac{2}{3} \mu \nabla \cdot \vec{v} \right) + \frac{\partial}{\partial z} \left( \mu \frac{\partial v_y}{\partial z} + \mu \frac{\partial v_z}{\partial y} \right) + S_y$$

$$\nabla \cdot (\rho v_z \vec{v}) = -\frac{\partial P}{\partial z} + \frac{\partial}{\partial x} \left( \mu \frac{\partial v_z}{\partial x} + \mu \frac{\partial v_x}{\partial z} \right) + \frac{\partial}{\partial y} \left( \mu \frac{\partial v_z}{\partial y} + \mu \frac{\partial v_y}{\partial z} \right) + \frac{\partial}{\partial z} \left( 2\mu \frac{\partial v_z}{\partial z} - \frac{2}{3} \mu \nabla \cdot \vec{v} \right) + S_z$$

- Conservation of Energy

$$\frac{\partial}{\partial x}(\rho v_x c_p T) + \frac{\partial}{\partial y}(\rho v_y c_p T) = \frac{\partial}{\partial x} \left( k_x \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left( k_y \frac{\partial T}{\partial y} \right) + S_E$$

- Source Terms

- Mass -  $S_m$ ; Momentum -  $S_x$ ,  $S_y$  &  $S_z$ ; Energy -  $S_E$



# Thermoelectric Equations

- Thermoelectric junctions

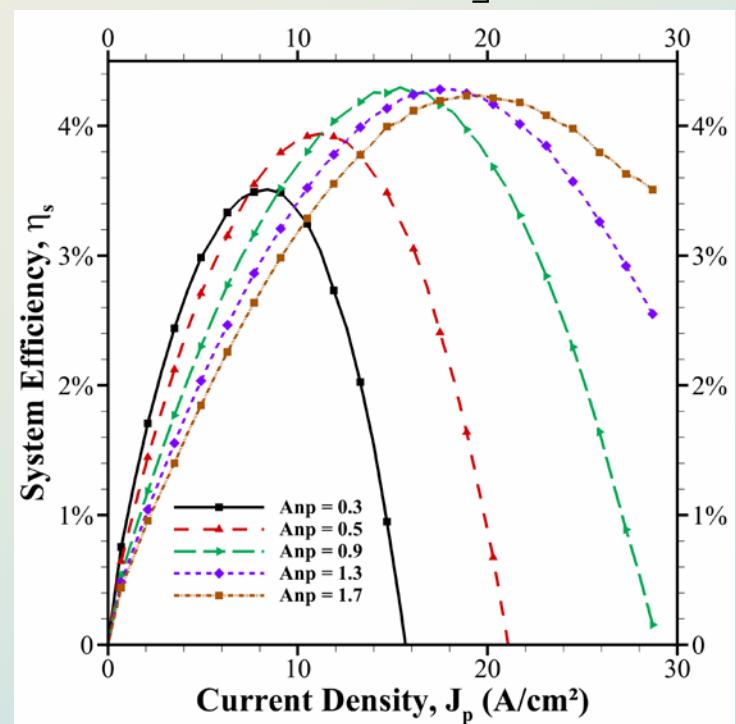
- Hot junction  $S_{E,h} = \frac{1}{1 + A_{np}} \left[ \left( \rho_{e,p} + \frac{\rho_{e,n}}{A_{np}} \right) J_p^2 - \frac{J_p \cdot T \cdot (\alpha_p - \alpha_n)}{t_h} \right]$
- Cold junction  $S_{E,c} = \frac{1}{1 + A_{np}} \left[ \left( \rho_{e,p} + \frac{\rho_{e,n}}{A_{np}} \right) J_p^2 - \frac{J_p \cdot T \cdot (\alpha_n - \alpha_p)}{t_h} \right]$

- Thermoelectric legs

- p-type  $S_{E,p} = \rho_{e,p} J_p^2 - J_p \cdot \tau_p \cdot \frac{\partial T_p}{\partial x}$
- n-type  $S_{E,n} = \rho_{e,n} \frac{J_p^2}{A_{np}^2} + \frac{J_p}{A_{np}} \cdot \tau_n \cdot \frac{\partial T_n}{\partial x}$

- $J_p = I/A_p$ ;  $A_{np} = A_n/A_p$

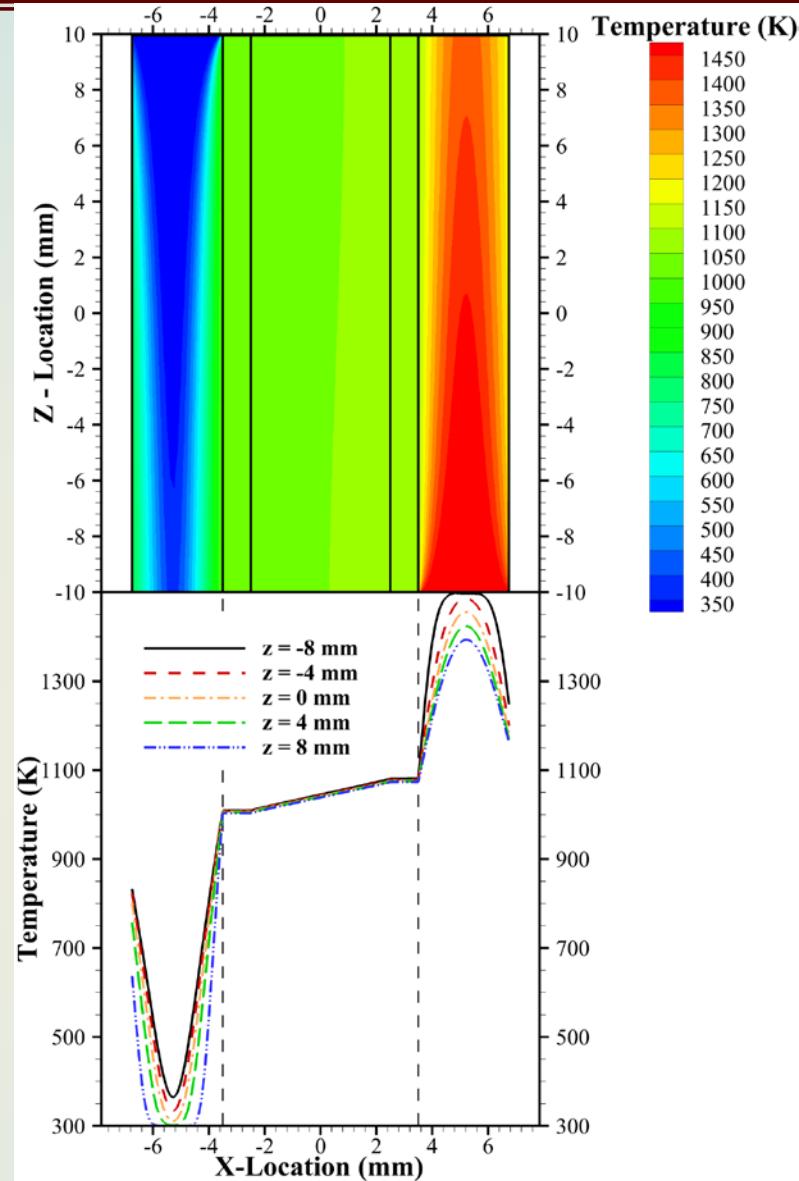
- $J_p$  and  $A_{np}$  optimized for  $\eta_s$





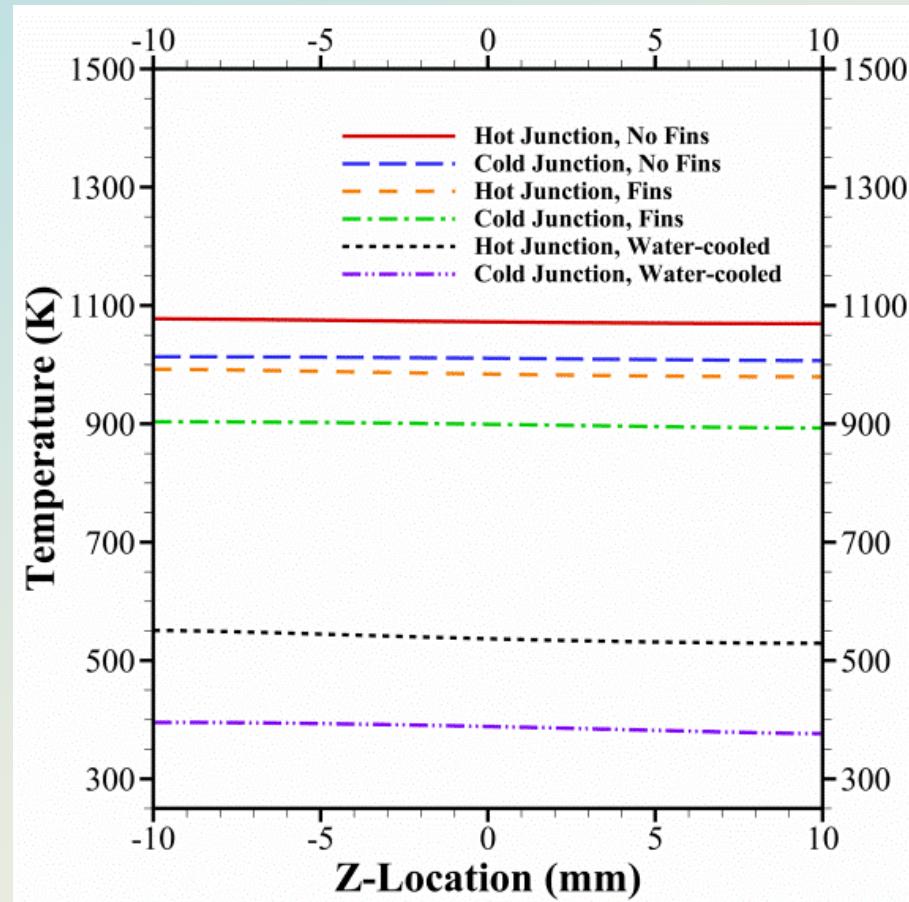
# Temperature Contours

No fins

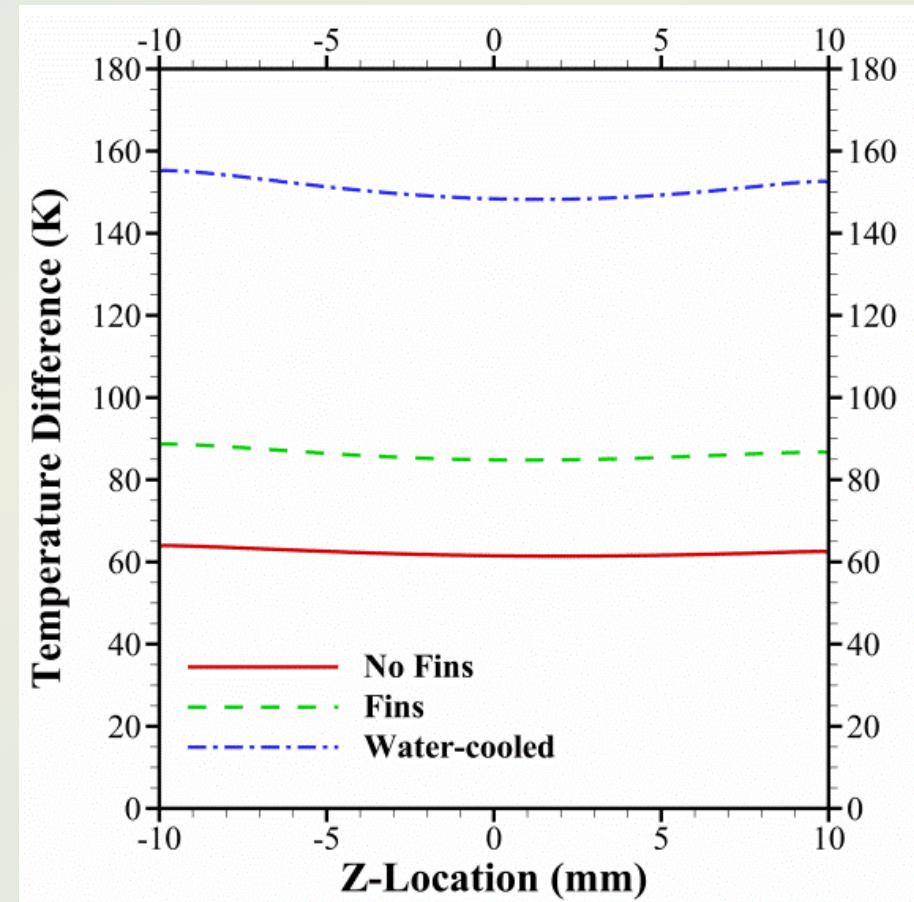




# Axial Mean Temperature Profile



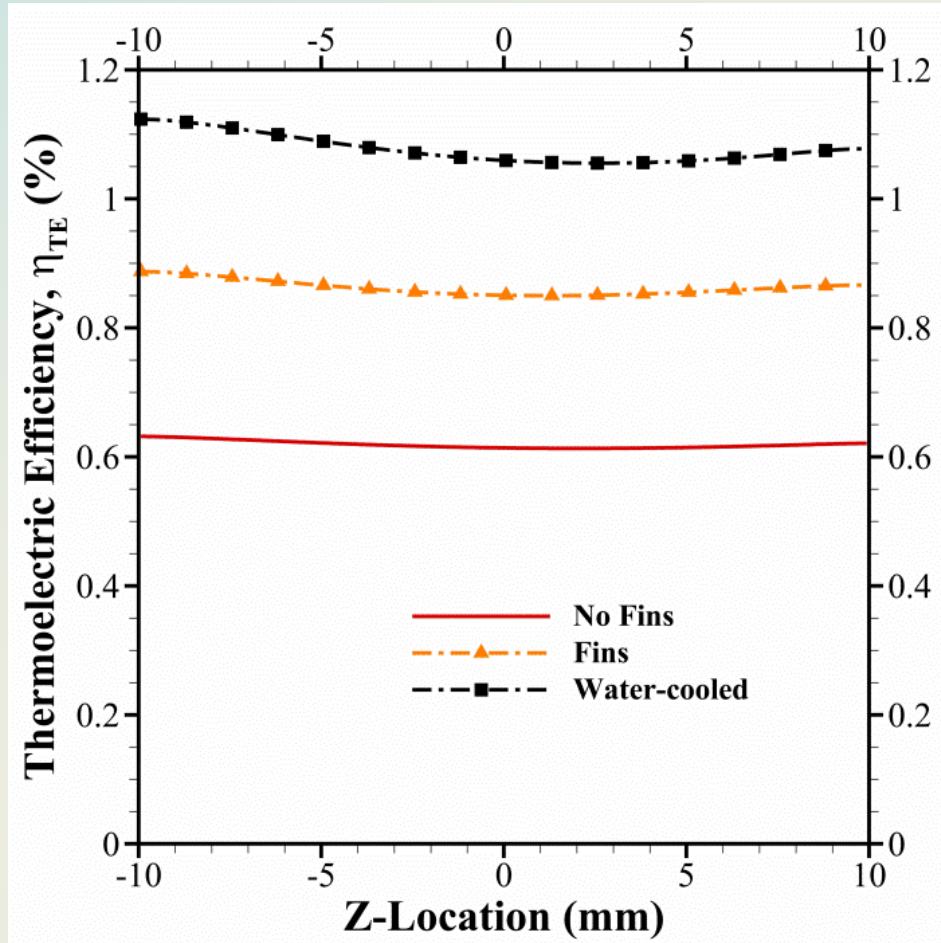
Mean Junction Temperature



Hot Junction - Cold Junction



# TE Module Performance

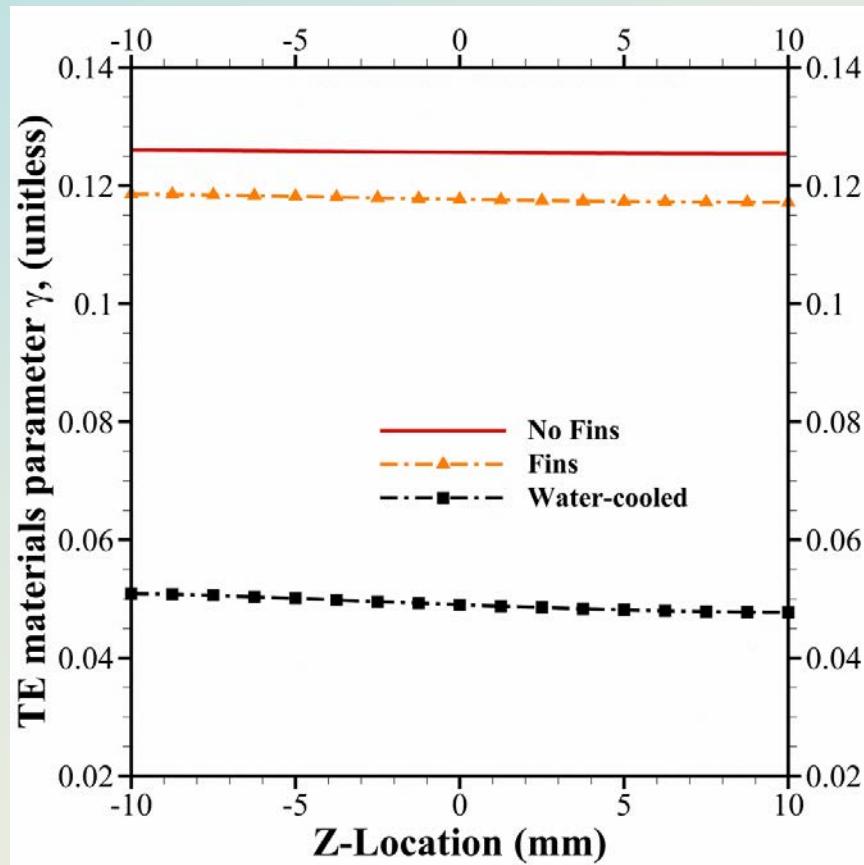


$$\eta_{TE} = \frac{W_e}{Q_h} = \eta_c \cdot \gamma$$

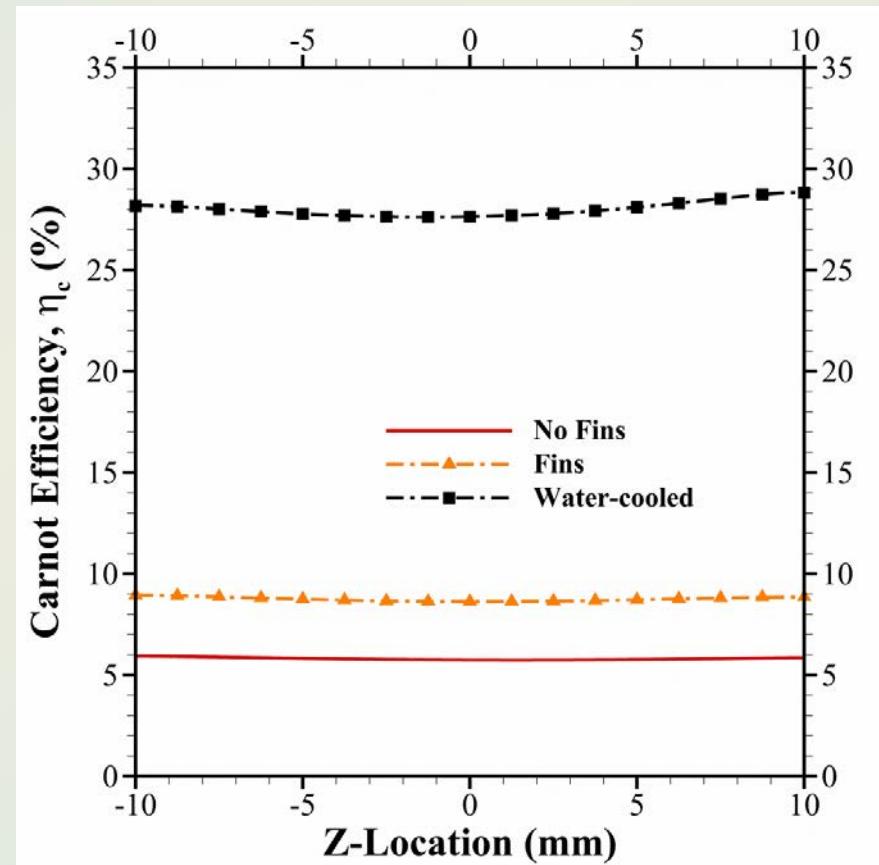
Thermoelectric Efficiency,  $\eta_{TE}$



# TE Module Performance



TE material parameter,  $\gamma$

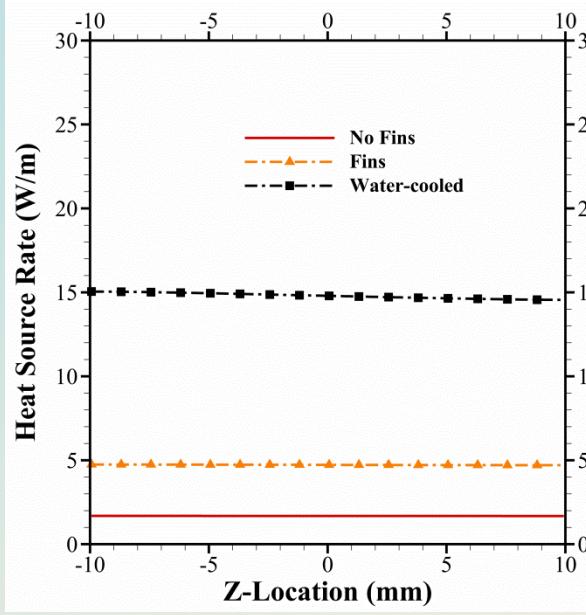


Carnot Efficiency

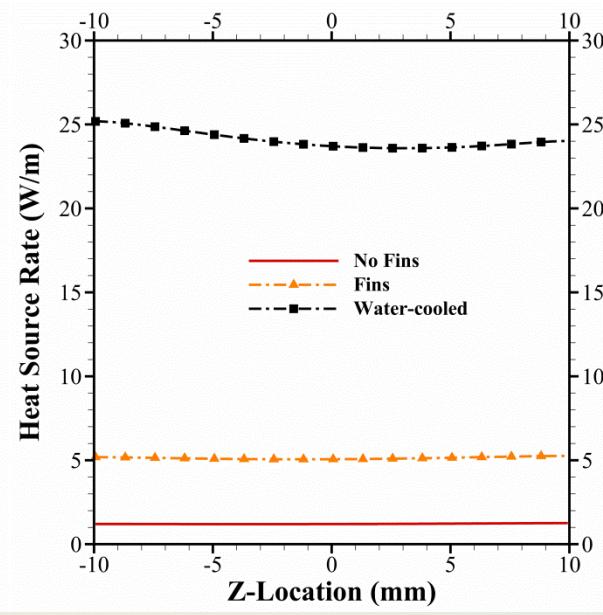
$$\eta_{TE} = \frac{W_e}{Q_h} = \eta_c \cdot \gamma$$



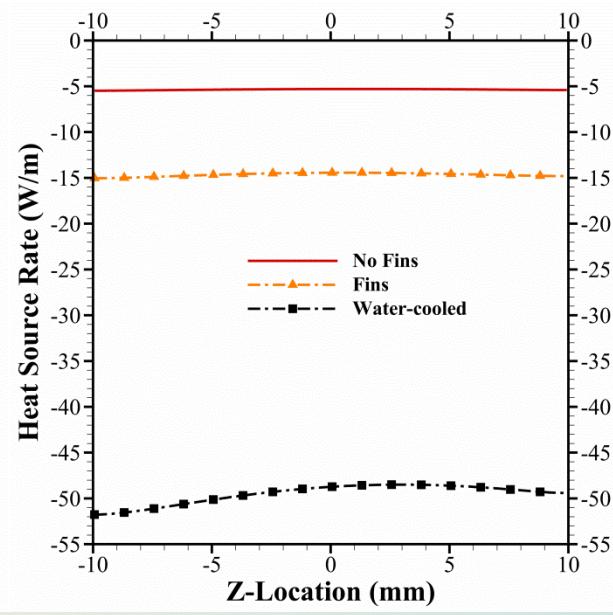
# Individual Thermoelectric Effects



Joule Heating



Thomson Effect



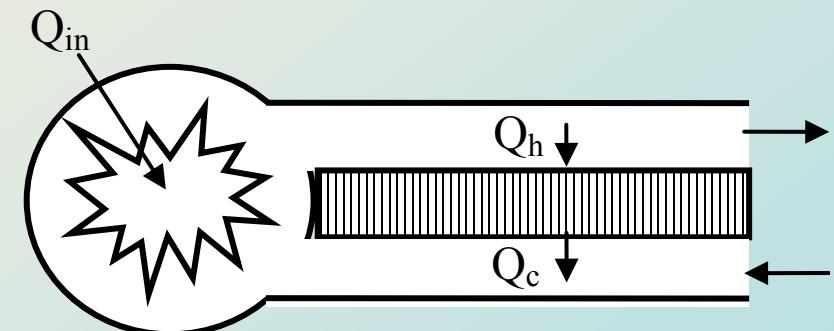
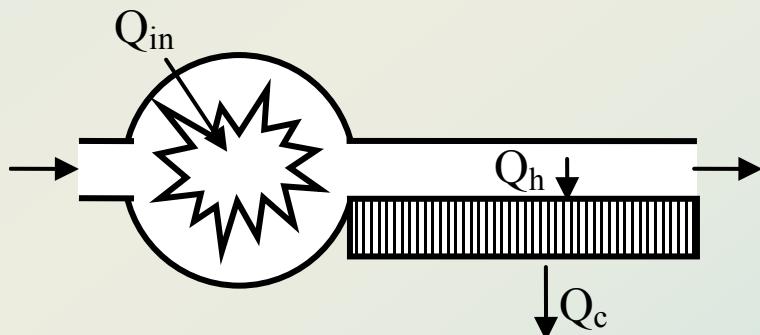
Peltier Effect

- Heat source rate of individual TE effects
  - Joule heating always causes power loss
  - Thomson effect is also causing power loss in this case
  - Peltier effect absorbs heat to generate power



# TE Module Performance

	No Fins	Finned	Water-cooled
Thermoelectric efficiency, $\eta_m$	0.62 %	0.86 %	1.08 %
Heat Input Ratio, $Q_R$	0.17	0.46	0.81
System efficiency, $\eta_s$	0.10 %	0.40 %	0.87 %
<b>With Heat Recirculation</b>			
Heat Input Ratio, $Q_R$	0.20	0.84	<b>4.05</b>
System efficiency, $\eta_s$	0.13 %	0.72 %	<b>4.36 %</b>





# Summary Remarks

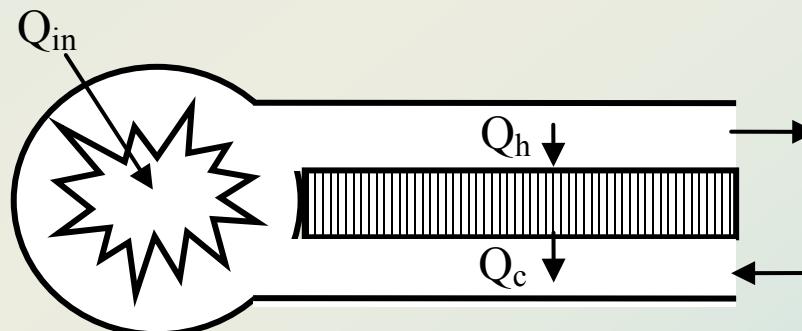
- The system efficiency increases by an order of magnitude even though the TE material and fluid inlet temperatures are the same.
- High ZT does not translate into high TE system efficiency without the thermal considerations.
- Heat recirculation offers an opportunity to greatly increase TE system efficiency (factor of 44 in present study).
- Thermal strategy is extremely important.



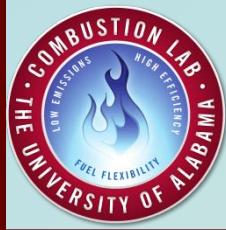
# Heat Exchange Effectiveness

- Define heat exchange effectiveness of the TE module with heat recirculation:

$$\varepsilon = \frac{\dot{m}(h_{h,in} - h_{h,out})}{\dot{m}(h_{h,in} - h_{c,in})} = \frac{\sum Q_h}{Q_{in} + \sum Q_c}$$



- Heat exchange effectiveness of 1.0 will require infinite surface area, but a value of 0.8 is plausible in modern heat exchange systems.



# Heat Recirculation

$$\eta_s = \frac{\sum W_e}{Q_{in}} = \frac{\sum Q_h}{Q_{in}} \cdot \frac{\sum W_e}{\sum Q_h}$$

$$\eta_s = Q_R \cdot \eta_m$$

$$Q_R = \frac{1}{\frac{1-\varepsilon}{\varepsilon} + \eta_m}$$

$\eta_m$	$\varepsilon=0.6$		$\varepsilon=0.7$		$\varepsilon=0.8$	
	$Q_R$	$\eta_s$	$Q_R$	$\eta_s$	$Q_R$	$\eta_s$
5%	1.4	7.0	2.1	10.5	3.33	16.0
10%	1.3	13.0	1.9	19.0	2.86	28.6
20%	1.2	24.0	1.6	32.0	2.22	44.4



# Summary

- Thermal resistance between TE module and fluids can significantly reduce the TE system efficiency.
- Thermal strategies can lead to high TE system efficiency, even with poor TE material performance.
- Improved understanding of the interaction of heat transfer, fluid flow, TE materials, and TE power generation is feasible with our 3D CFD-TE model.
- TE system efficiency can be increased significantly by employing heat recirculation.
- We are developing integrated TE system configurations that can achieve high heat exchange effectiveness and thus, high TE system efficiency.



# Acknowledgments

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# Putting the heat on Thermoelectrics

Questions ?