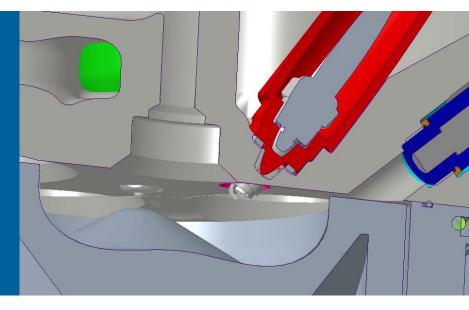


### Project ID# ACE132

# HEAVY-DUTY GASOLINE COMPRESSION IGNITION



CHRISTOPHER KOLODZIEJ, HEE JE SEONG, JORGE PULPEIRO GONZALEZ, MICHAEL PAMMINGER, BUYU WANG, DOUG LONGMAN

2019 DOE Vehicle Technologies Office Annual Merit Review Advanced Combustion Engines 9:30 AM, June 13, 2019

DOE Vehicle Technologies Office Management: Michael Weismiller & Gurpreet Singh

### **OVERVIEW**

### **Timeline**

- Started Oct 2018
- End date Sept 2021
  - Lab Call reset Oct 2018
- 16% Completed

### **Budget**

- Total project funding
  - DOE share 100%
  - Contractor share 0%
- Funding received in:
  - FY19 (new) \$950k

### **Barriers**

- Reduced heat loss
- Lower engine-out emissions
- Expand Low Temperature Combustion (LTC) operation
- Robust cold start and low load for HD GCI

### **Partners**

- Caterpillar
  - Engine head modifications
  - Common rail injection equipment

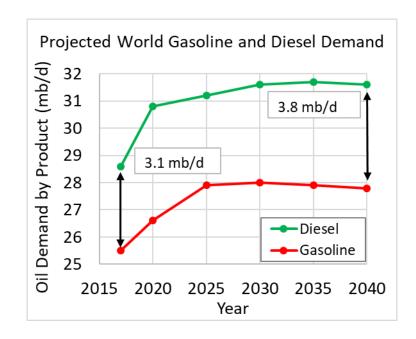


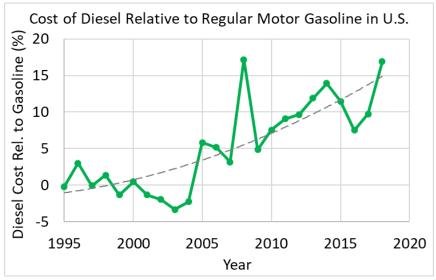
### RELEVANCE/OBJECTIVES

### **Energy Security and Economic Vitality:**

- Diesel is the main fuel of MD/HD engines
- Projections show that world-wide diesel demand will grow faster than gasoline
- In the US, the cost of on-highway diesel fuel has been increasing consistently relative to regular grade gasoline
- However, current gasoline spark-ignition (SI) engines do not satisfy the efficiency and load requirements of the MD/HD fleet

Gasoline Compression Ignition (GCI) engines offer high efficiency fuel flexibility to the MD/HD fleet



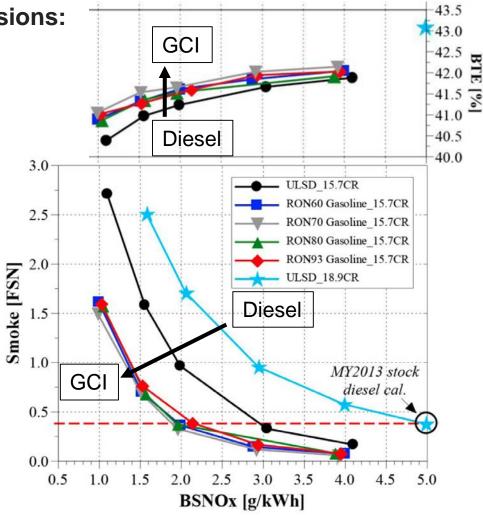


### RELEVANCE/OBJECTIVES

**Increased Efficiency and Lower Emissions:** 

- Diesel-like efficiency
- 50% smoke reduction at same BSNOx

Gasoline Compression Ignition (GCI) has the potential for high diesel-like efficiency and reduced emissions.



14.9 L HD Engine. 1375 RPM, 10 bar BMEP.

Argonne 🕰

CA50 = 6 °aTDC

Ref. Zhang, Y, et al., SAE 2018-01-0226

### RELEVANCE/OBJECTIVES

- Obtain the most efficient use of gasoline in a HD engine through compression ignition
  - a. Reduced heat losses
  - b. Combustion chamber (piston) design
  - c. Injector design and injection parameters
- 2. Reduce engine-out PM and NOx emissions relative to diesel combustion
  - a. Minimize total fluid consumption (fuel and DEF) and total cost of ownership
  - b. Identify and avoid conditions of liquid spray impingement
  - c. Injection parameters to reduce engine-out soot emissions and number of active particulate filter regenerations
    - a. With the wide range of off-road duty cycles, this could have high significance.
- 3. Identify robust cold start and low load approaches
  - Compare spark ignition, spark assisted compression ignition, and compression ignition in a large bore quiescent HD combustion chamber
  - b. Fast aftertreatment warm-up



### RESOURCES

- Caterpillar Single-Cylinder Oil Test Engine (SCOTE)
  - 2.4 L Displacement
  - 16.2:1 Compression Ratio



Currently	Being Installed
SCOTE Head with HEUI, <u>Endoscope</u> , and Cyl. Press. Transducer	SCOTE Head with Common Rail Injector, Spark Plug, and Cyl. Press. Transducer
Hydraulic Electronic Unit Inj. (HEUI) System	Stand-alone Common Rail Pump System
Absorption-only DC Dyno	AVL Motoring AC Dyno
AVL Fuel Balance	Resol Coriolis Meter and Fuel Conditioning
Caterpillar ECU	Vieletech LabVIEW Controls/Data Acq.
	Cylinder Fast-Sampling Valve

- Existing Equipment:
  - Port Fuel Injection (PFI), AVL Visioscope, AVL FT-IR, Dekati FPS-4000 Diluter, TSI SMPS



### RESOURCES

## **Heavy-Duty Gasoline Compression Ignition Engine Research for Off-Road Vehicles**

Task	Name	FY19 Funding
1	HD Off-Road Industry Workshop	\$50k
2	Heavy-Duty Gasoline Compression Ignition	\$600k
3	Ultra-fast sampling system for in-cylinder, crank resolved emissions.	\$300k
Total		\$950k



### MILESTONES

Quarter	Milestone	Status
FY19-Q1	Upgrades to engine and test cell for single-cylinder HD GCI research. Baseline on diesel fuel.	Postponed to Q4
FY19-Q2	Single-cylinder HD GCI combustion and emissions baseline on gasoline fuel.	Postponed to Q1 FY20
F19-Q2	Complete parametric study of fast sampling valve from a high-pressure chamber.	Postponed to Q3
F19-Q3	Identify cylinder densities and injection conditions where fuel impingement occurs.	On Track
FY19-Q4	Install fast sampling valve in engine cylinder head.	On Track
FY20-Q2	Define load ranges for LTC and PP-MCCI with initial CR.	On Track
FY20-Q4	Investigate required cylinder conditions for cold start and examine gaseous and particulate formation processes.	On Track



### **APPROACH**

Use gasoline-like fuels for reduced emissions, increased efficiency, and efficient HD engine fuel flexibility.

### High Load

- Use partially-premixed mixing-controlled compression ignition (PP-MCCI) for robust combustion phasing control
- Minimize heat loss and particulate emissions
- Investigate injection parameters and injector design
- Find the minimum operable load for MCCI combustion

### Moderate Load

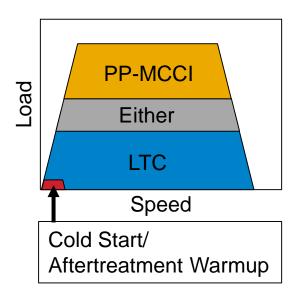
- Area where engine could operate in PP-MCCI or Low Temperature Combustion (LTC)
- Identify trade-offs between these two combustion modes

### Low Load

- Use LTC for highest efficiency and lowest particulate and NOx emissions, but manage combustion noise and HC/CO emissions
- Use spray modeling to avoid any liquid impingement

### Cold-Start/Aftertreatment Warmup

- Compare combustion modes at idle in a quiescent large bore HD engine (Stoichiometric SI, SACI, CI)
- Develop approaches to transition between modes





### APPROACH

Use advanced tools to better understand processes of injection, sprays, combustion, and emissions.

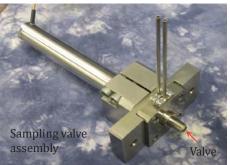
- Use x-ray tomography to monitor injector cavitation erosion with gasoline
- 1D/3D spray simulations to identify conditions of liquid length impingement
  - Test 120°, 130°, and 150° inclusion angle nozzles
- Endoscopic imaging of soot radiation/pool fires
- Cylinder fast sampling valve to characterize soot formation and oxidation processes
- Particle size distribution and TEM morphology



\*X-ray image of new CAT injector tip (Powell et al.)



Endoscopic imaging



Fast Sampling Valve



# **TECHNICAL ACCOMPLISHMENTS & PROGRESS**



### TEST CELL UPGRADES

- AVL motoring dynamometer
  - Arrived April, 2019
  - To be commissioned July, 2019
- Resol low pressure fuel supply and measurement
  - Moved from previous GCI test cell
- Common rail high pressure fuel unit
  - Being converted for safe operation with gasoline
    - New explosion-proof electric motor
    - 480V variable speed drive relocation
- Vieletech (formerly Drivven) LabVIEW same/nextcycle controller and full test cell data acquisition
  - To be commissioned after dynamometer

Full test cell recommissioning expected: September, 2019





Source: https://www.avl.com/load-unit-for-engine-testing

Source: http://www.ni.com/en-us/support/model.dcm-2301.html



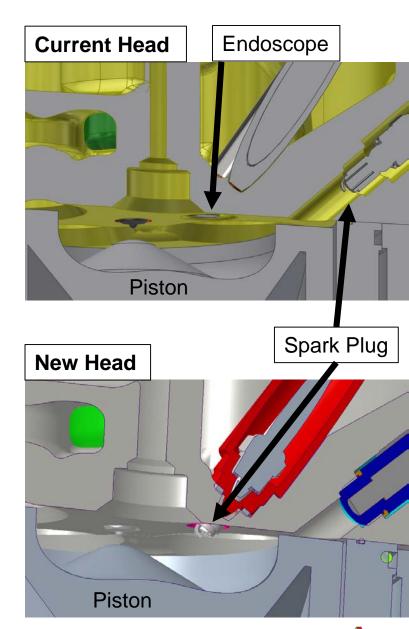
### CATERPILLAR CYLINDER HEAD MODIFICATIONS

### **Current Cylinder Head:**

- Originally modified to work with CAT HEUI
- CAT designed modifications for common rail
- Endoscope port previously existed
- M8 spark plug inserted into non-ideal location (unused pressure relief port)
  - Located in squish region
  - Significant recession

### **New Cylinder Head:**

- Spark plug adapter in place of endoscope
  - Inside the piston bowl
- Cylinder pressure transducer in pressure relief port, common location
- Injector bore modified for common rail injector





INITIAL PORT FUEL INJECTION (PFI) SPARK IGNITION (SI) TESTING

### **Objectives:**

 Identify range of spark timings between knock and combustion stability limits

### **Constraints:**

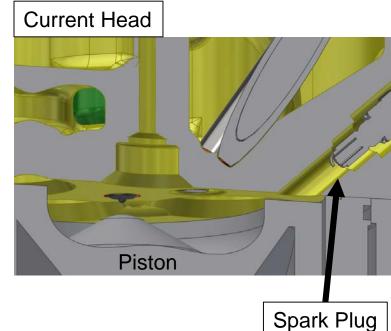
- Low swirl, turbulence, and flame speed
- Large bore for flame to propagate across
- Spark plug recessed above squish
- High 16.2:1 CR

### **Test Limits:**

- 0.75 bar Maximum Amplitude of Pressure Oscillations (MAPO)
- 3% COV of IMEP combustion stability limit

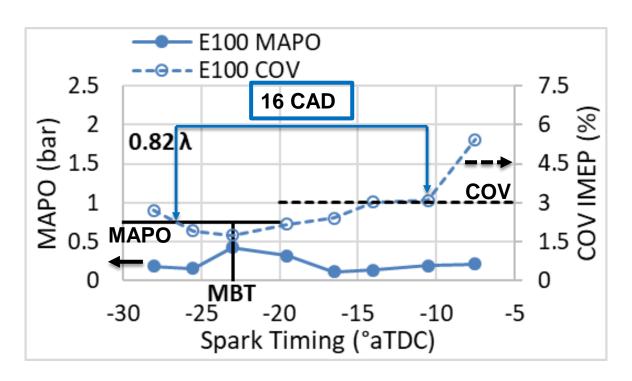
### **Fuels Investigated:**

- E100 (100% Ethanol)
  - High flame speed
  - High knock resistance
- E10 gasoline (87 AKI)



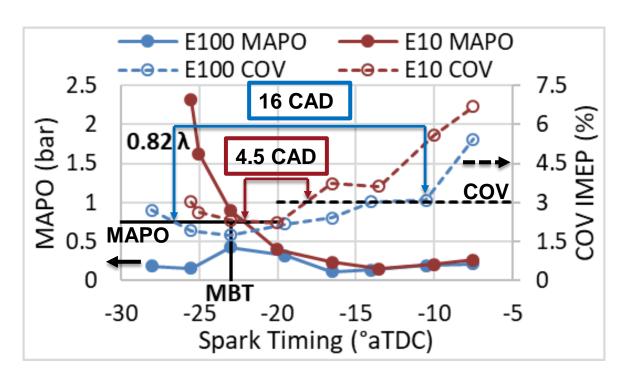


- 750 RPM, 3.2 bar IMEPg
- -23 °aTDC MBT spark timing
  - Without end-gas autoignition
- E100 not knock limited at 0.82 λ



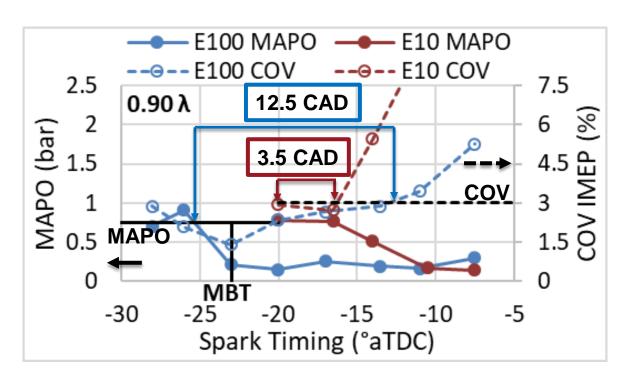


- 750 RPM, 3.2 bar IMEPg
- -23 °aTDC MBT spark timing
  - Without end-gas autoignition
- E100 not knock limited at 0.82 λ



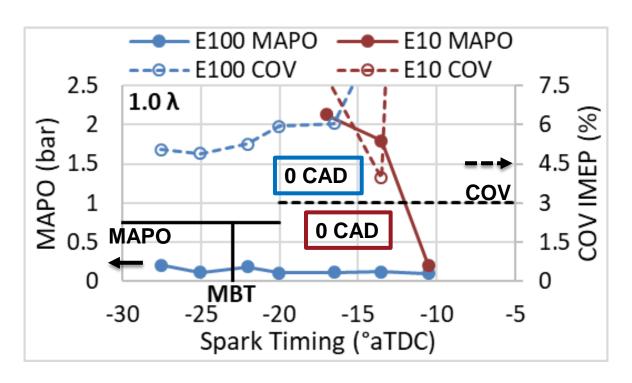


- 750 RPM, 3.2 bar IMEPg
- -23 °aTDC MBT spark timing
  - Without end-gas autoignition
- E100 is knock limited at 0.90 λ





- 750 RPM, 3.2 bar IMEPg
- -23 °aTDC MBT spark timing
  - Without end-gas autoignition
- E100 not knock limited at 1.0 λ

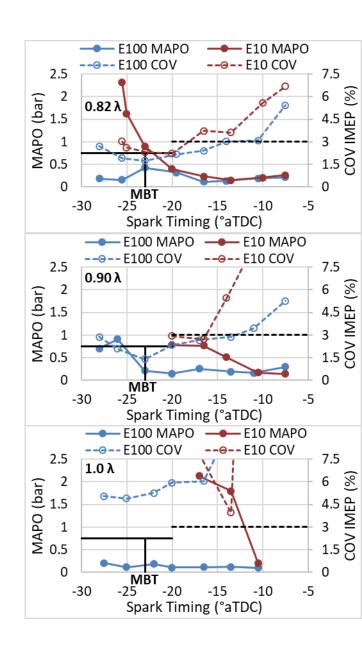




- 750 RPM, 3.2 bar IMEPg
- -23 °aTDC MBT spark timing
  - Without end-gas autoignition

Fuel	λ	SOI Range
E100 E10	0.82	16 CAD 4.5 CAD
E100 E10	0.90	12.5 CAD 3.5 CAD
E100 E10	1.00	0 CAD 0 CAD

E100 at 0.82 λ allowed for the widest spark timing range from a combination of higher knock resistance (earlier ST limit) and laminar flame speed (later ST limit).

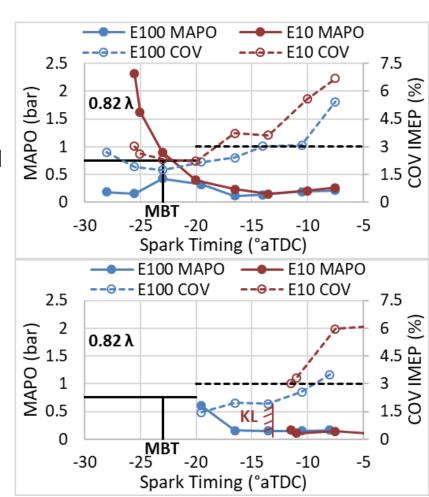




### **EFFECTS OF ENGINE LOAD**

- 3.2 bar vs. 3.8 bar IMEPg
- Operable range shifted later
- MBT was knock limited (E10 & E100)
- Later ST at higher fueling allowed increased exhaust enthalpy

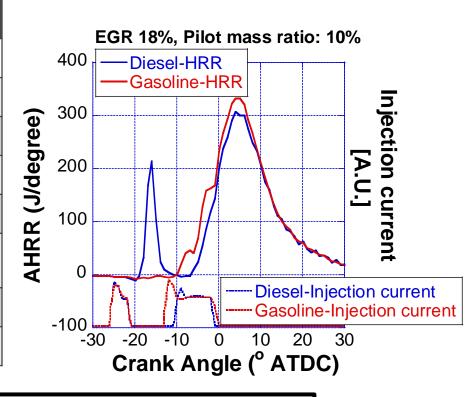
Fuel	λ	IMEPg	SOI Range
E100 E10	0.82	3.2 bar	16 CAD 4.5 CAD
E100 E10	0.82	3.8 bar	11 CAD <3 CAD





# DIESEL VS. GASOLINE PARTICULATE CHARACTERIZATION AT HIGH LOAD

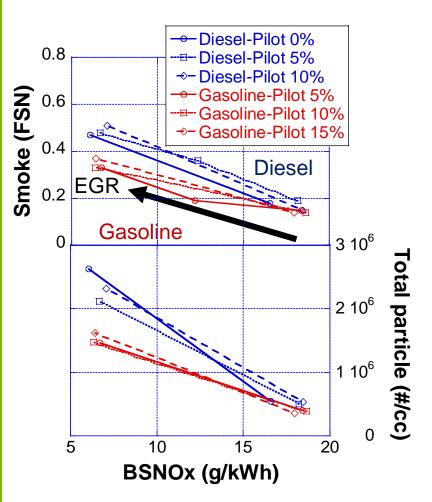
Parameter	Value	
Navistar 12.4L 6-cylinder (CR 17:1)		
Engine Speed [rpm]	1038	
Engine Load [bar BMEP]	14.1	
CA50 [°aTDC]	6.3 ± .4	
Fuel	ULSD & Tier II EEE	
Start of Injection [°aTDC]	-27.5/-14.5~-12 (varied)	
Pilot mass ratio (%)	0, 5, 10 & 15	
EGR [%]	0, 9 & 18	

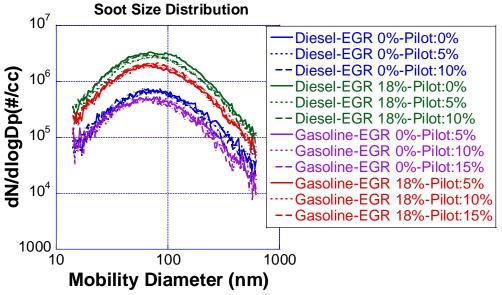


- Methodology: Hold constant CA50 by varying SOI<sub>2</sub>
- Observations:
  - Longer ignition delay of pilot fuel injection with gasoline
  - EGR impact (0-18%) on combustion was minor for the same fuel



### GASOLINE REDUCED THE SOOT-NO<sub>x</sub> TRADE-OFF

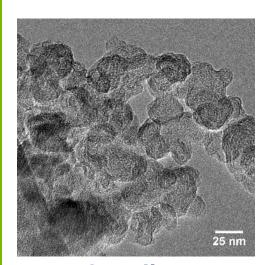




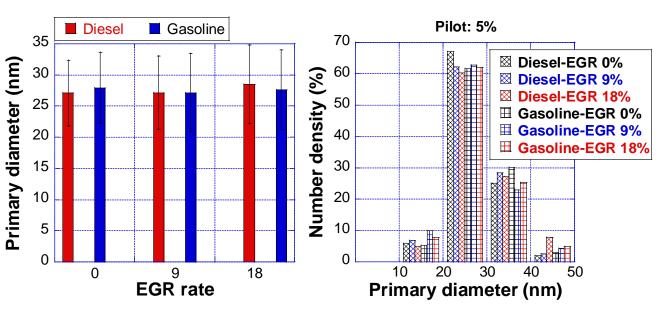
- Lower Soot-NO<sub>x</sub> trade-off with gasoline
- Similar shaped size distributions for both fuels
  - But fewer particles with gasoline



### TEM & RAMAN PARTICULATE STRUCTURE ANALYSIS



Gasoline (EGR 0%, Pilot 5%)



- Transmitting Electron Microscope (TEM) analysis of soot primary particle size and distributions showed strong similarities between diesel and gasoline under high load (14 bar BMEP) MCCI combustion
- No significant effect for 0-18% EGR
- Raman analysis for carbon crystalline structure performed
  - In technical back-up slides
- Similar crystalline structures for both fuels with EGR



### **RESPONSES TO 2018 AMR REVIEWER COMMENTS**

Project# ACE132 "Heavy-Duty Gasoline Compression Ignition" is a new project for FY19 and was not reviewed last year.



### **COLLABORATIONS**

### Caterpillar (POC, Dr. Adam Dempsey)

Engineering and hardware support

### Clear Flame (Dr. Julie Blumreiter)

Collaboration on PFI SI engine testing

### ACE#010 (Powell, ANL)

 X-ray tomography of HD injector nozzle cavitation erosion with gasoline fuel

### **Navistar**

 Diesel vs. GCI particulate characterization performed on Navistar Supertruck engine installed at ANL

### **Aramco Services Co.**

Discussions on HD GCI research

### **DOE VTO AEC Working Group**

 CAT, Cummins, DDC, Mack, John Deere, GE, Navistar, Ford, GM, Chrysler, ExxonMobil, ConocoPhillips, Shell, Chevron, BP, ANL, SNL, LLNL, ORNL



### REMAINING BARRIERS AND CHALLENGES

### **High Load:**

- Reduced heat loss
- Reduced particulate and NOx engine-out emissions
- Increasing the premixed fuel fraction without exceeding combustion noise level limits

### Low Load:

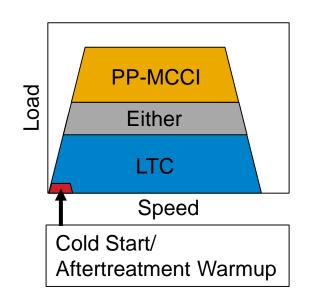
- Achieve robust LTC with higher efficiency and lower emissions than diesel combustion
- Avoid all liquid fuel spray impingement on the combustion chamber surfaces

### **Cold Start:**

 Determine the most robust cold start approach that minimizes aftertreatment heating time

### **All Conditions:**

Characterize and mitigate cavitation erosion



### PROPOSED FUTURE WORK

- Recommission test cell and convert engine from HEUI to common rail injection equipment
- Baseline engine on diesel fuel with guidance from Caterpillar
- Baseline GCI efficiency and emissions with initial injector and piston
- Evaluate improved low load ST range with spark plug location in the piston bowl
  - Vary depth of the spark plug protrusion
- Determine the cold start strategy with the highest exhaust enthalpy
- Use spray simulations to determine the conditions where impingement occurs
- Measure soot formation and oxidation processes using fast-sampling valve
- Record injector cavitation erosion from gasoline operation



### SUMMARY

Relevance: HD GCI has lower emissions and diesel-like efficiency, while expanding the high efficiency fuel flexibility of the MD/HD fleet.

### Approach:

 Operate a HD engine on gasoline with partially-premixed mixing-controlled compression ignition at high load and LTC at low load for lower emissions and higher efficiency

### Technical Accomplishment:

- Modification to a production CAT SCOTE cylinder head to incorporate a spark plug
- Initial PFI SI testing, with a non-ideal spark plug location, shows significant knock limitations for stoichiometric operation at ≈3 bar IMEPg with both E10 and E100
- Despite different chemical composition, gasoline and diesel produced remarkably similar soot aggregate structures at high load with MCCI combustion

### Future Work:

- Re-test low load PFI SI knock and combustion stability limitations with spark plug located in the piston bowl
- Compare low load operation between SI and CI combustion
- Implement a cylinder fast-sampling valve and detailed emissions characterization
- Perform x-ray tomography imaging of injector nozzle to track cavitation erosion



