

# Advancement in Fuel Spray and Combustion Modeling for Compression Ignition Engine Applications

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Project ID # ACE075

This presentation does not contain any proprietary, confidential, or otherwise restricted information

# Overview

## Timeline

Past Funding: FY 09, FY 10

Project start: April 1<sup>st</sup> 2012

## Partners

**Project Lead:** Sibendu Som

**Argonne National Laboratory**

Chemical Science and Engineering

Mathematics and Computing Science

**Convergent Science Inc.**

**Lawrence Livermore National Laboratory**

**Caterpillar Inc.**

**Sandia National Laboratory (Engine  
Combustion Network [ECN])**

**University of Connecticut**

**Cummins (Pending)**

## Barriers

- ❑ “Inadequate understanding of stochastics of fuel injection”
- ❑ “Improving the predictive nature of spray and combustion models”
- ❑ “Incorporating more detailed chemical kinetics into fluid dynamics simulations”

## Budget

FY 12: 350 K

Starting April 1<sup>st</sup> 2012

# Objectives/Relevance - 1

- Development of dynamically-coupled nozzle flow and spray simulations through improvements in Kelvin Helmholtz – Aerodynamic Cavitation Turbulence (KH-ACT) model
- Extensive validation of the dynamically coupled KH-ACT model:
  - ❑ X-ray radiography data from Argonne National Laboratory in the near nozzle region
  - ❑ Optical constant volume data from Sandia National Laboratory through the Engine Combustion Network (ECN) under evaporating and combustion conditions
- Fuel spray breakup in the near nozzle region plays a central role in combustion and emission processes, and is governed by primary breakup mechanism caused by:

Cavitation



- Current spray models only account for aerodynamic breakup, hence, are not predictive in nature with changing fuel types and nozzle orifice geometries

# Objectives/Relevance - 2

- “Building a bridge” between fundamental chemical-kinetics (DOE Office of Science) and applied combustion research through computational combustion modeling of more realistic fuel surrogates
  - Implementing and validating reduced mechanisms for diesel fuel surrogates against ECN data
    - ❑ n-dodecane
    - ❑ n-dodecane + m-xylene
  - N-heptane is used as a diesel fuel surrogate. Not an ideal choice due to its high-volatility and low carbon content
  - N-dodecane + m-xylene is a suitable diesel surrogate since it better mimics diesel Cetane characteristics
  - Detailed chemical kinetic models are large, mechanism reduction is necessary
- Computational-time scales with  $N^2 \sim N^3$  where ‘N’ is number of species



# Objectives/Relevance - 3

- High Performance Computing (HPC):
  - ❑ Demonstrate scalability up to 1000 processors
  - ❑ Demonstrate grid-independence of spray and combustion parameters
- Current state-of-the-art for engine simulations in OEMs involve up to 50 processors only
- OEMs prefer quick turn-around times for engine simulations which may not be possible as the resolution, spray, turbulence, and chemical kinetic models become more detailed
- This is possible if scalable simulations are feasible by increasing the number of processors by a factor of 5



# Milestones, FY 12

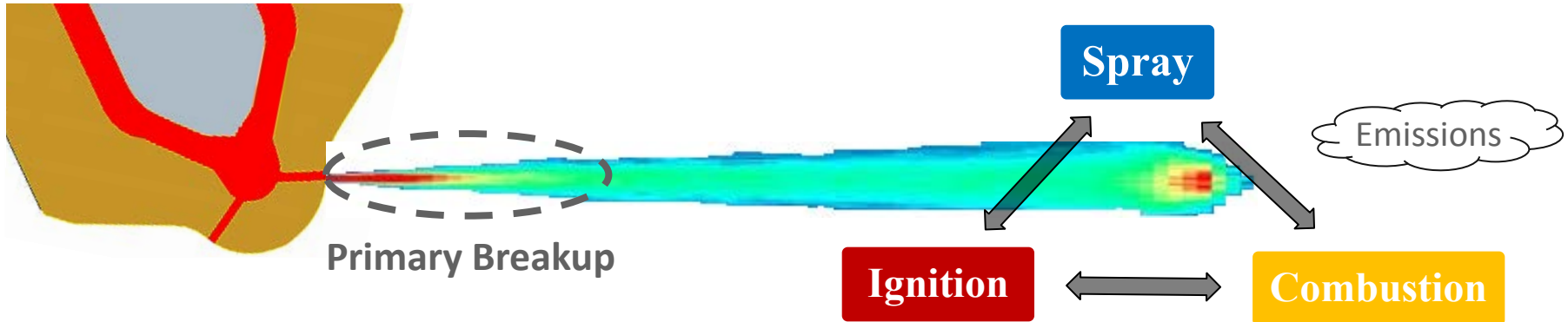
- Task 1: Dynamic – coupling of injector nozzle and spray processes: Extension of KH-ACT model
  - ❑ Improving the predictive capability of KH-ACT primary breakup model  
(September 2012)
  
- Task 2: Develop a surrogate mechanism for diesel fuel for multi-dimensional CFD simulations
  - ❑ Updating the 103-species n-dodecane reduced mechanism (May 2012)
  - ❑ Validation and improvements in combustion modeling based on ECN data  
(August 2012)
  
- Task 3: Simulation of Internal combustion engines with HPC tools
  - ❑ Assess grid independence of spray and combustion parameters (June 2012)
  - ❑ Assess scalability of CONVERGE tool on (up to) 100-500 processors (July 2012)



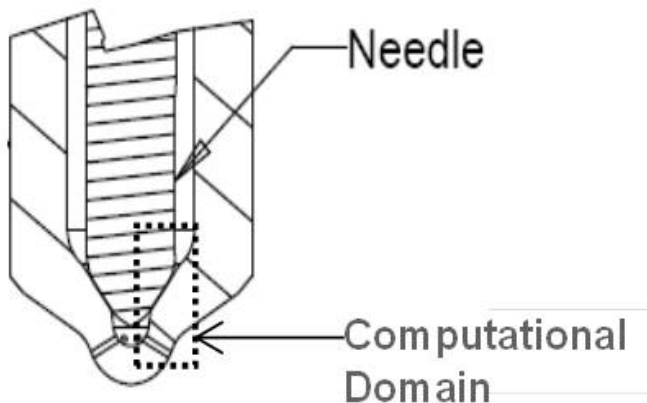
# Integrated Modeling Approach

**Influence of fuel properties and nozzle orifice geometry on nozzle flow, spray, and combustion characteristics!!**

Inner Nozzle Flow



**6-hole production Injector**

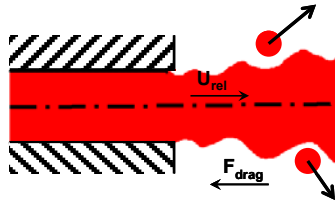


- ☐ **KH-ACT** primary breakup model:  
Aerodynamics, Cavitation, Turbulence
- ☐ Detailed inner-nozzle flow modeling with realistic fuel properties
- ☐ **Dynamic – coupling** of inner nozzle flow and spray simulations
- ☐ Spray Validation:  
**X-ray radiography** data provides information in the near nozzle region

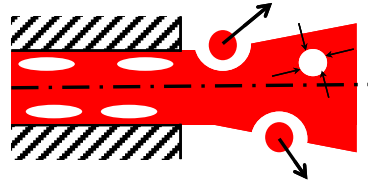
# Primary Breakup Model

## KH-ACT (Kelvin-Helmholtz-Aerodynamics Cavitation Turbulence) Model\*

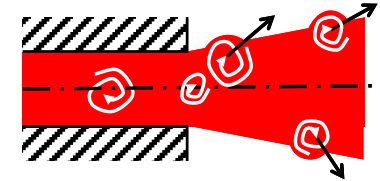
- Length and time scales are calculated



**Aerodynamically**  
induced breakup:  
Based on Kelvin-Helmholtz (KH)  
and Rayleigh Taylor (RT)  
instability



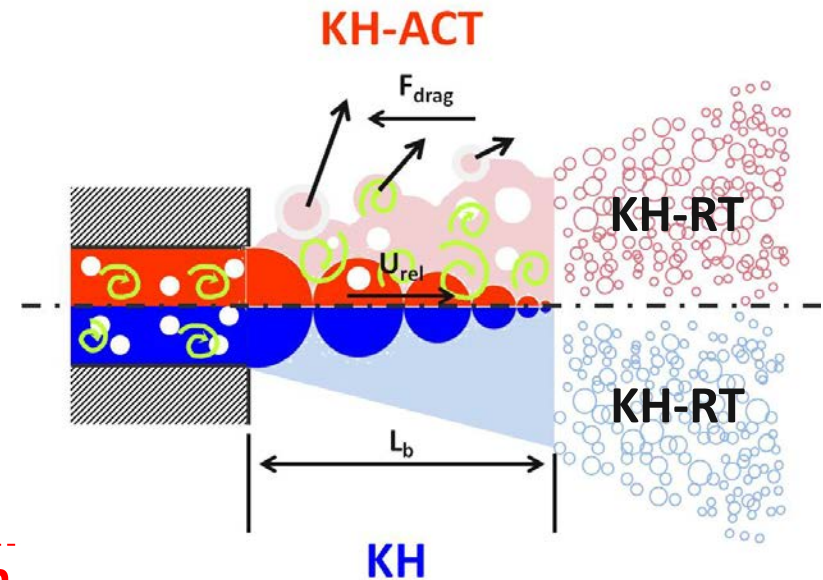
**Cavitation**  
induced breakup:  
Based on bubble  
collapse and burst  
times



**Turbulence**  
induced breakup:  
Based on k- $\epsilon$  model

- Dominant ratio of length/time scale causes breakup
- Different combinations of length and time scales for ACT model will be tested (more information in back-up slides)

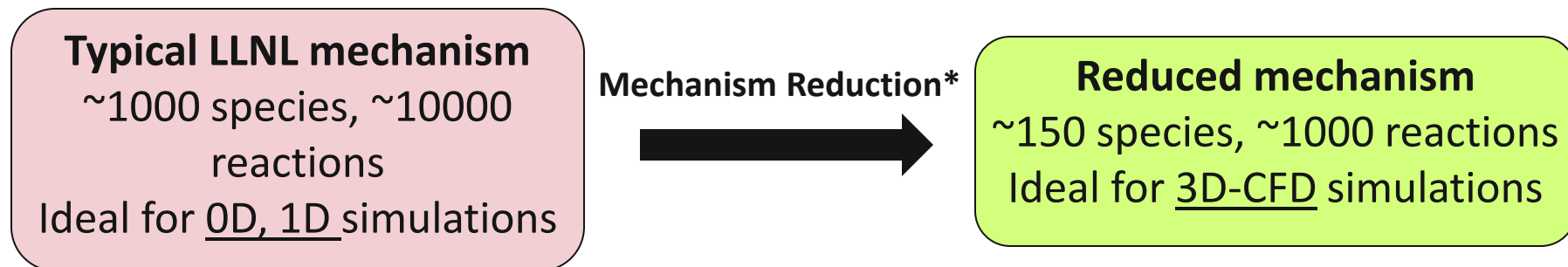
\*Som et al., SAE 2009-01-0838, Combustion and Flame 2010, Fuel 2010, Fuel 2011



**Approach**



# Detailed Chemical Mechanisms in Engine Simulations



Research on mechanism reduction techniques is funded by DOE office of Science at Chemistry group at Argonne, and University of Connecticut

## Our Approach:

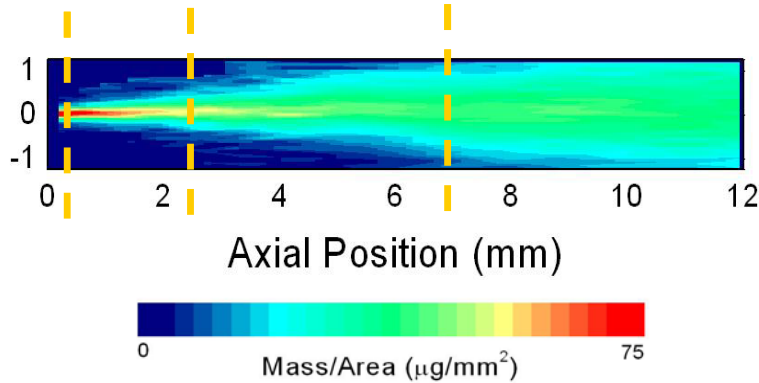
- Provide the mechanism reductionists with fuel surrogates of interest for the transportation sector
- Extensive validation against ECN spray-combustion and engine data
- Provide feedback on the performance of the reduced mechanism to the mechanism developers, based on 3D-CFD simulations



\* Z Luo, M Plomer, T Lu, M Maciaszek, S Som, DE Longman. *Energy and Fuels* (24) 2010

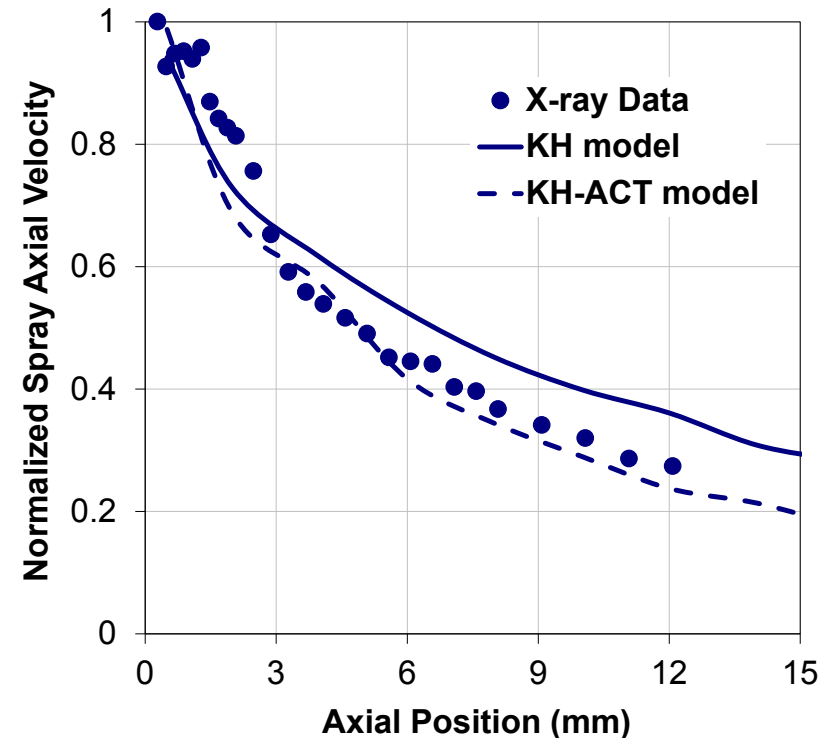
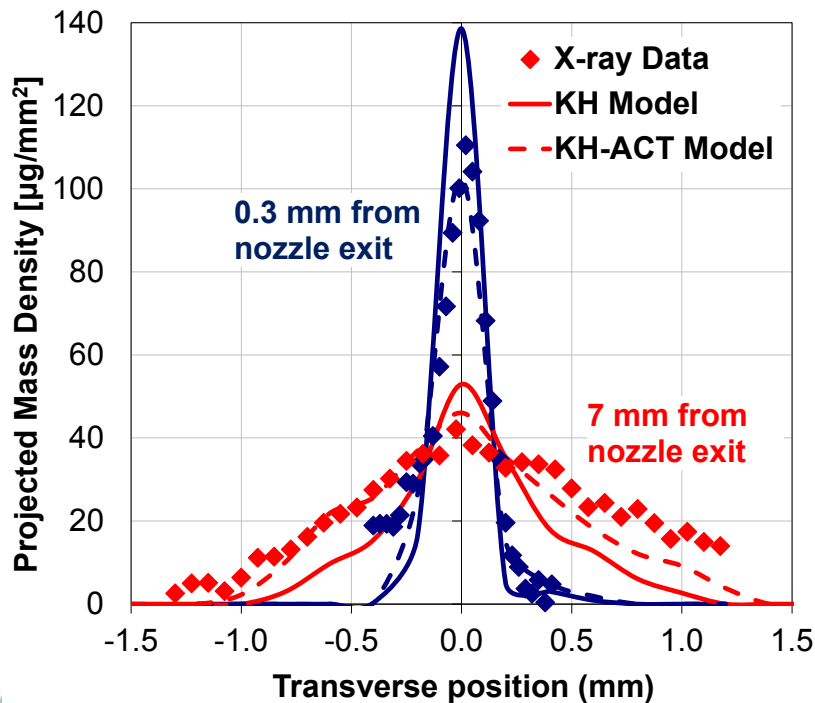
\* T. Lu, M. Plomer, Z. Luo, S.M. Sarathy, W.J. Pitz, S. Som, D.E. Longman,. *US Combustion meeting, 2011*

# KH-ACT Model Validation against X-ray Data\*



\*X-ray radiography Data: Ramirez et al., JEF 2009

(Experimental conditions available in back-up slides)



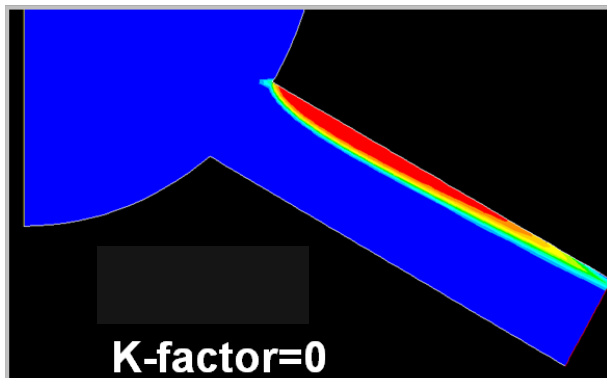
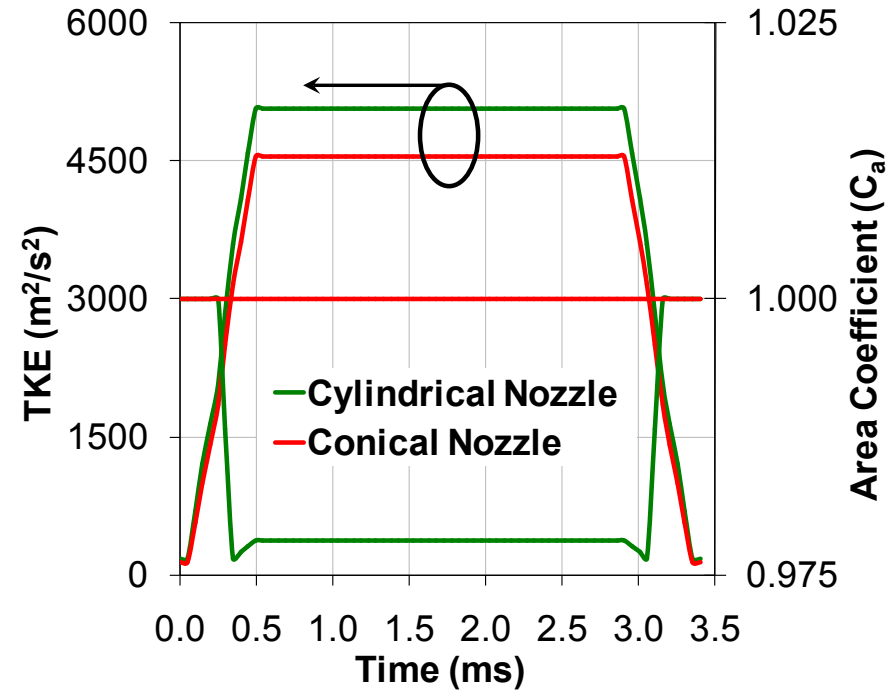
- ❑ The spray loses half of its initial velocity within the first 6 mm
- ❑ Simulation capture the Gaussian mass distributions from x-ray data well
- ❑ Spray Dispersion accurately captured by only the KH-ACT model. KH model under-predicts spray spreading

Technical Accomplishment and Progress

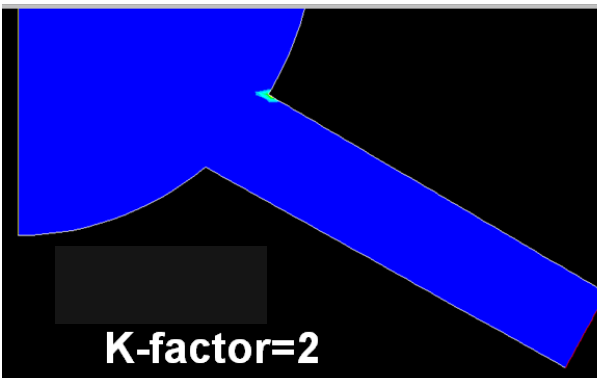
# Effect of Conicity on Inner Nozzle Flow

Geometrical Characteristics	Cylindrical Nozzle	Conical Nozzle
$D_{in}$ ( $\mu m$ )	169	169
$D_{out}$ ( $\mu m$ )	169	149
$K_{factor}$	0	2
$L/D$	4.2	4.7

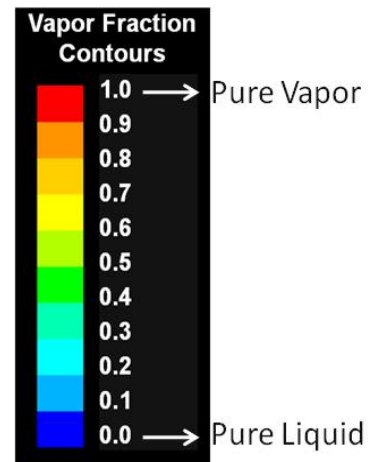
$$K_{factor} = \left( \frac{D_{in} - D_{out}}{10} \right) \mu m$$



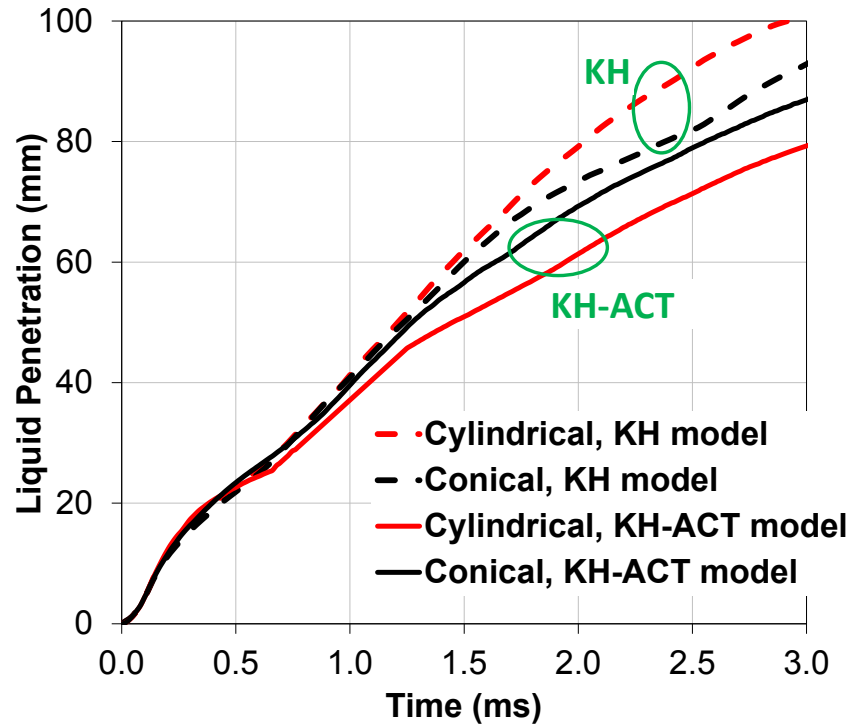
**Cylindrical Nozzle**



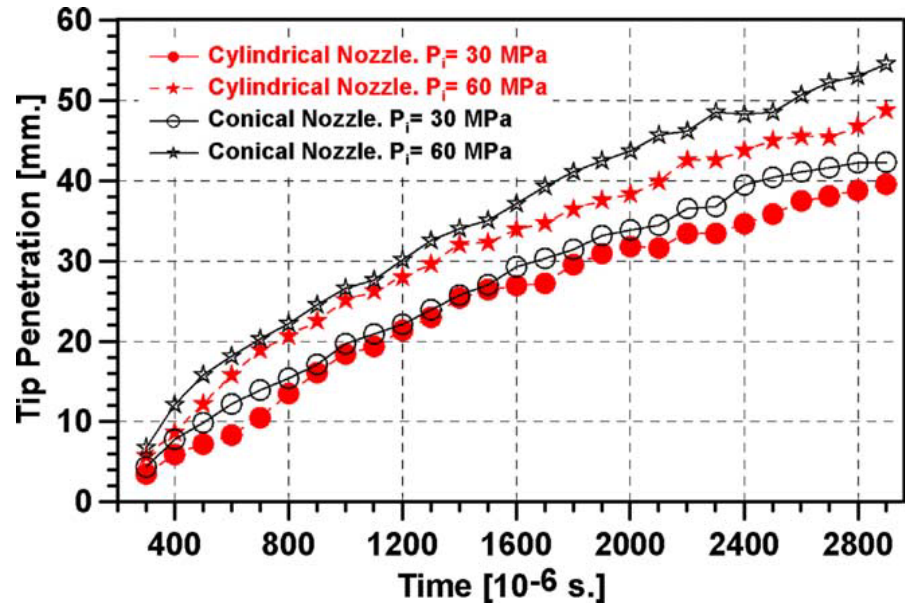
**Conical Nozzle**



# KH-ACT model Accurately Predicting the Influence of Nozzle Geometry



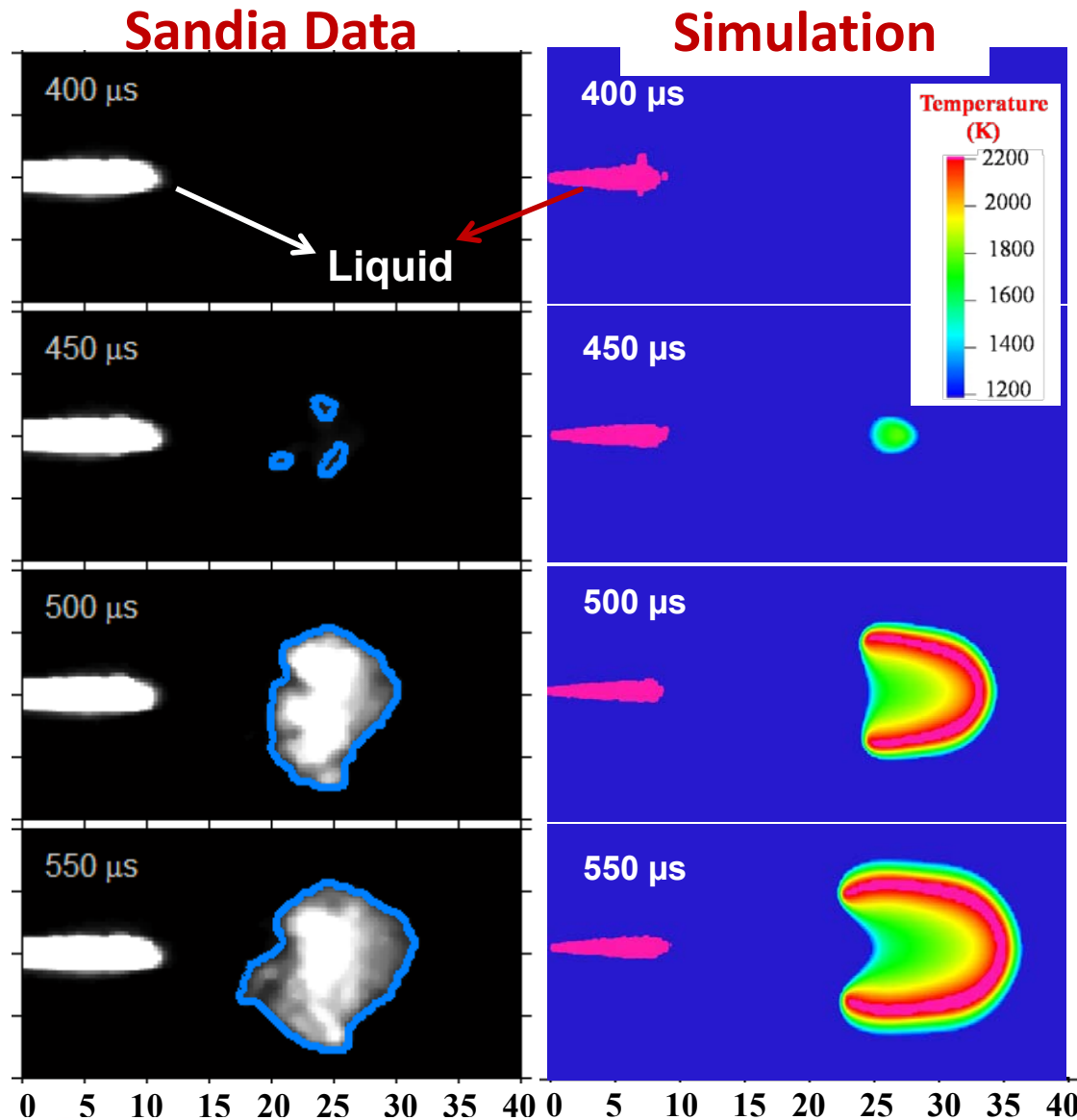
F Payri, V Bermudez, R Payri, FJ Salvador: FUEL (2004)



- ❑ Penetration characteristics of cylindrical and conical nozzles predicted by KH-ACT model (only) are consistent with experimental trends observed by Payri et al.
- ❑ Cylindrical nozzle predicts fastest breakup. This is due to enhanced cavitation and turbulence thus: 1) SMD, 2) Spray penetration are lower

\*S Som, DE Longman, AI Ramirez, SK Aggarwal. FUEL 2011

# Combustion modeling with n-dodecane



## Experiments:

(Conditions available in back-up slides)

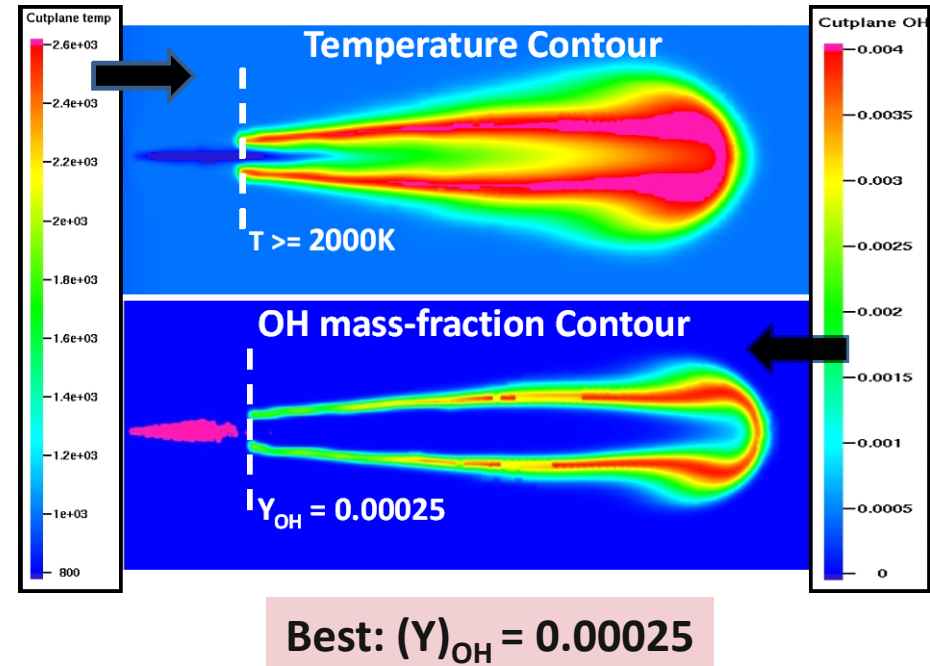
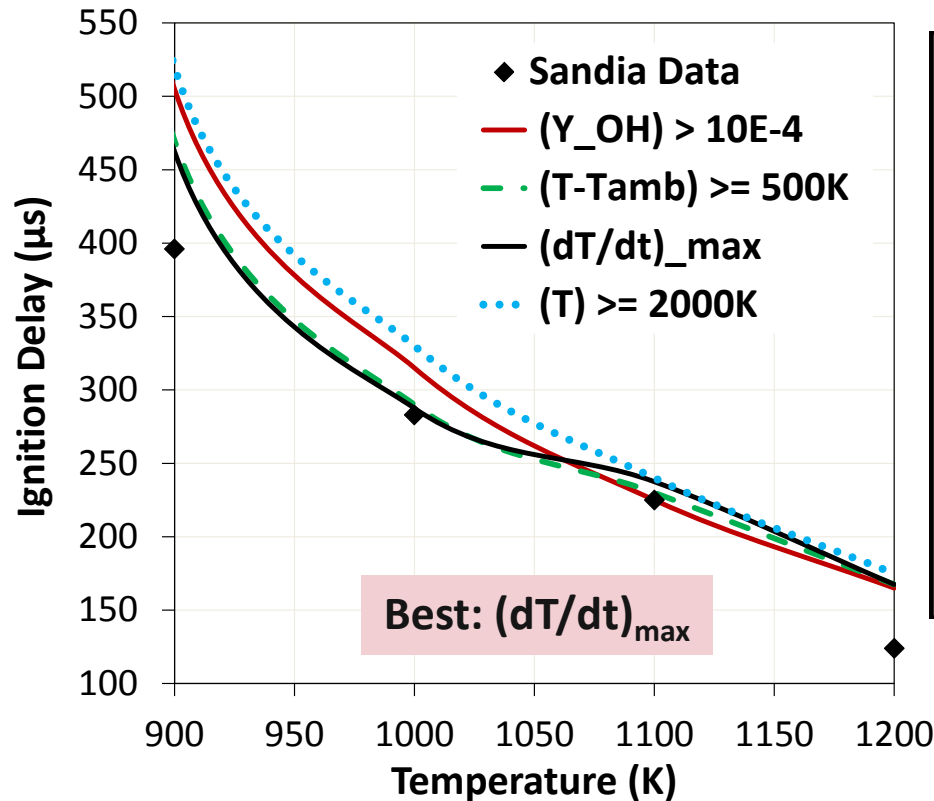
## Simulation:

Temperature contours plotted to capture ignition location and delay simulated using the 103 species n-dodecane reduced mechanism (cf. Slide 9)

**Spray and Combustion modeling able to predict the liquid fuel distribution, ignition location, ignition time, flame shape, etc.**

<http://www.sandia.gov/ecn/>

# Identify Appropriate Definitions for Ignition Delay and Flame Lift-off Length

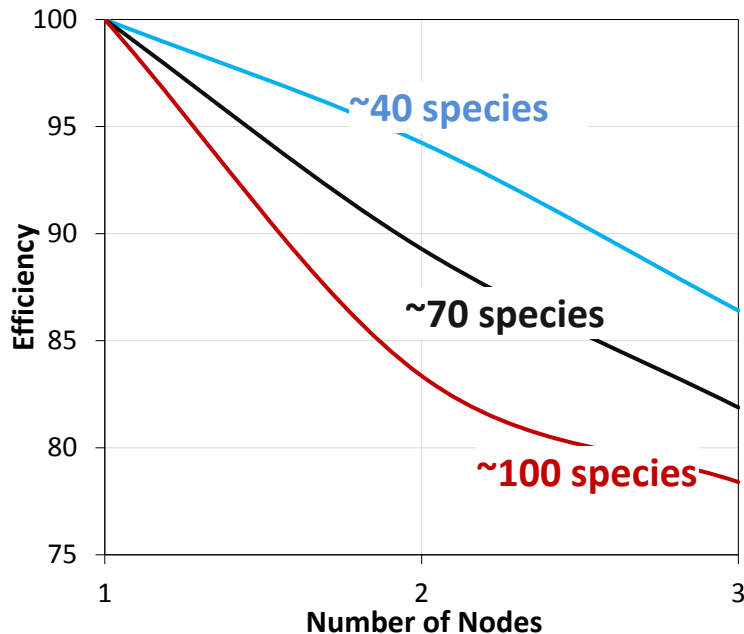


- ❑ Differences are observed in predicted ignition delays and flame lift-off lengths based on definitions chosen
- ❑ Most relevant definitions for ignition delay and flame lift-off lengths identified. This will be proposed to the Engine Combustion Network

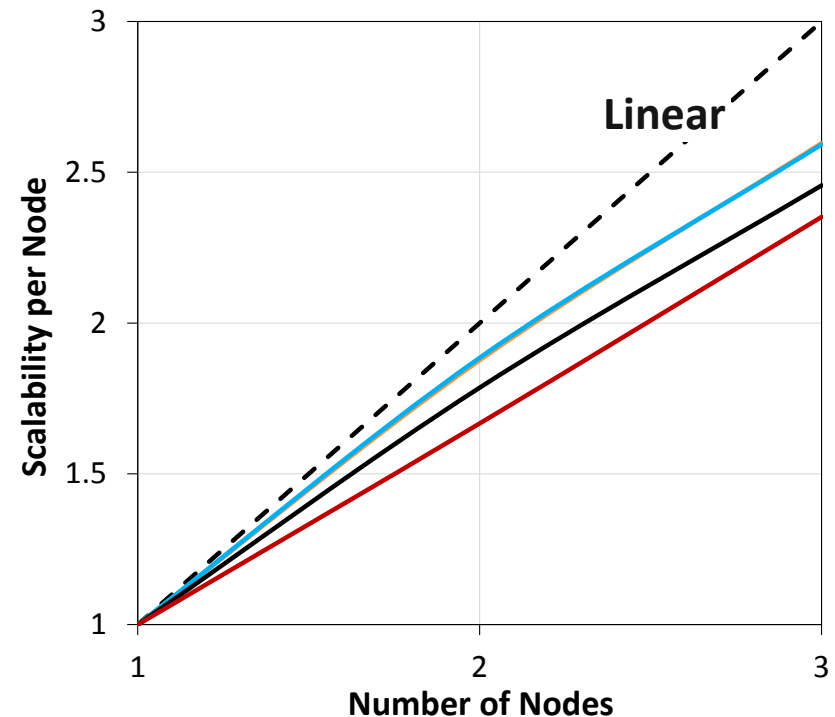
# Computational Cost & Scalability

Increasing the number of species results in rapid decrease in scalability and efficiency.

**Our focus is on improving the load-balancing schemes to obtain better scalability.**



Mechanisms	Wall-clock Time (for node)
~40 species: n-heptane	~ 42 hours
~100 species: n-dodecane	~ 120 hours



Scalability per node =  $T_1/T_n$

Efficiency per node =  $T_1 \times 100 / nT_n$

$n$  = Number of compute nodes

$T_1$  = Wall-clock time on 1 node

$T_n$  = Wall-clock time on  $n$  nodes

Each node has 8 processors

**Technical Accomplishment and Progress**

# Collaborations

## Argonne National Laboratory

Engine and Emissions Group: (Provide data for model validation)

Chemical Science and Engineering Group: (Mechanism development and reduction)

Mathematics and Computing Science: (HPC resources)

Convergent Science Inc. (Algorithm and code development in CONVERGE )

Sandia National Laboratory (Provide experimental data through the ECN)

Lawrence Livermore National Laboratory (Mechanism development)

University of Connecticut (Mechanism Reduction)

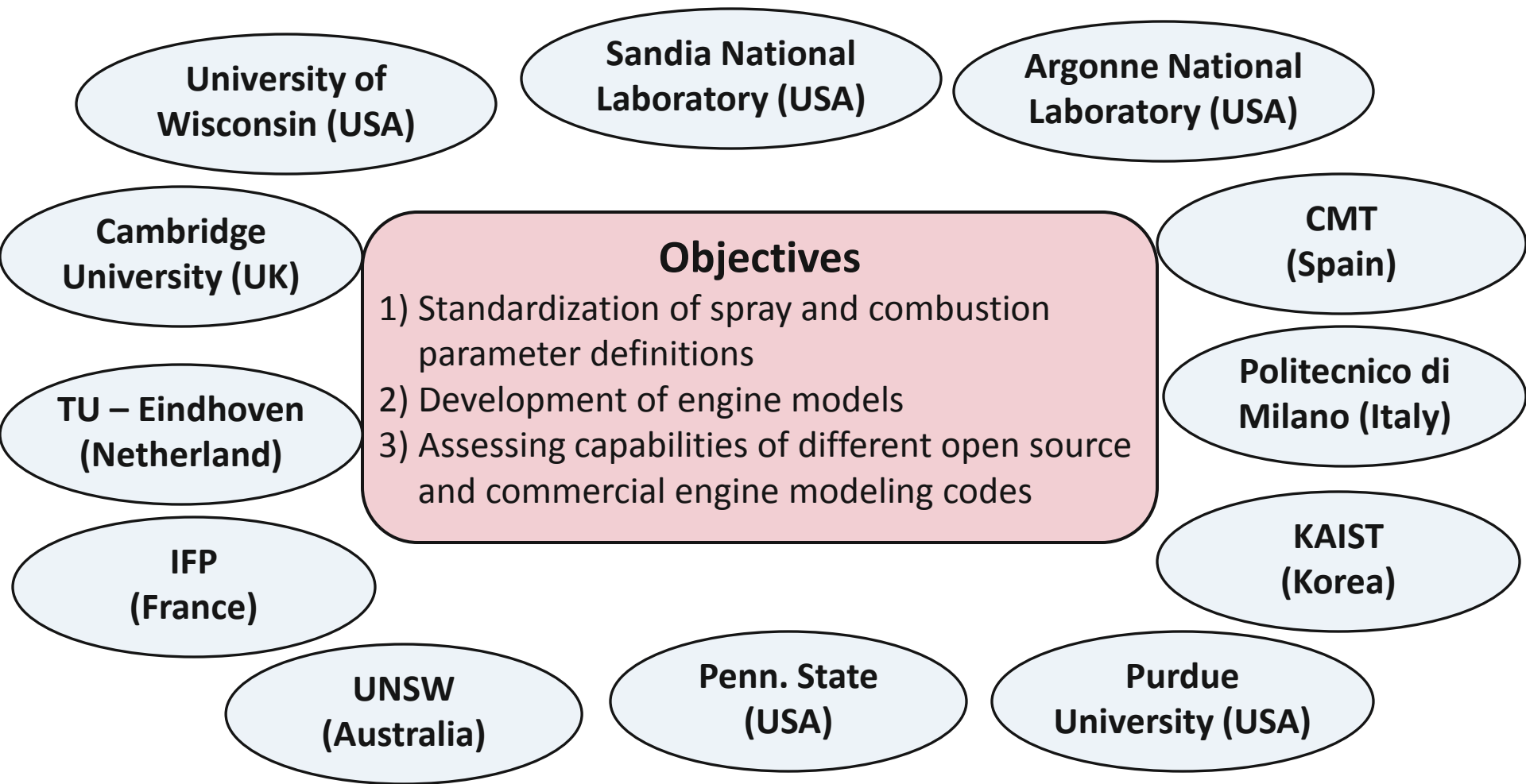
Cummins (Provide experimental data, alpha testing) {Pending}

Caterpillar Inc. (Testing and implementation of HPC tools)





# ECN Modeling Coordination



- ❑ Coordinated Spray A modeling session in ECN 1 (Ventura, May 2011): 9 groups
- ❑ Coordinating modeling sessions in ECN 2 (Heidelberg, September 2012): 12-15 groups expected

# Proposed Future Work in FY13

- **Task 1: Dynamic – coupling of injector nozzle and spray processes: Extension of KH-ACT model**
  - ❑ Implement an improved nozzle flow model with moving needle capability
  - ❑ Implement and test a dynamic-coupling approach
  - ❑ Validation against x-ray radiography data
- **Task 2: Develop a surrogate mechanism for diesel fuel for multi-dimensional CFD simulations - Validation against ECN data**
  - ❑ Further reduction and testing of the 103 species n-dodecane mechanism
  - ❑ Implement and test n-dodecane + m-xylene reduced mechanism for 3D combustion simulations
  - ❑ Capture the influence of ambient temperature and density variations on combustion characteristics such as ignition delay, flame lift-off length etc.
- **Task 3: Simulation of Internal combustion engines with high-performance computing tools**
  - ❑ Demonstrate grid independence for multi-cylinder simulations involving intake and exhaust ports
  - ❑ Assess scalability of CONVERGE tool on (up to) 1000 - 1500 processors

# Summary

## ❑ Objective

- Development of predictive spray and combustion models aided by high-performance computing tools and robust validation

## ❑ Approach

- Coupling expertise from DOE Office of Science on fundamental chemical kinetics and HPC resources for development of robust engine models

## ❑ Technical Accomplishment

- KH-ACT model performs static coupling of nozzle flow and spray simulations
- n-dodecane reduced mechanism captures combustion characteristics well

## ❑ Collaborations and coordination

- with industry, academia, and national laboratories in US
- through ECN with researchers world-wide

## ❑ Future Work - FY13

- Dynamic coupling of nozzle flow and spray
- Development of realistic diesel surrogate model
- Demonstrate scalability of engine models on 1000-1500 processors

# Technical Back-Up Slides

(Note: please include this “separator” slide if you are including back-up technical slides (maximum of five). These back-up technical slides will be available for your presentation and will be included in the DVD and Web PDF files released to the public.)



# 3D Spray-Combustion Modeling Set-up

Modeling Tool	CONVERGE
Dimensionality and type of grid	3D, structured with Adaptive Mesh Resolution
Spatial discretization approach	2 <sup>nd</sup> order finite volume
Smallest and largest characteristic grid size(s)	Base grid size: 2mm Finest grid size: 0.25mm <u>Gradient based AMR</u> on the velocity and temperature fields <u>Fixed embedding</u> in the near nozzle region to ensure the finest grid sizes
Total grid number	550K-650K for 0.25mm – RANS simulations
Parallelizability	Good scalability up to 48 processors

Turbulence and scalar transport model(s)	RNG k- $\epsilon$
Spray models	Breakup: KH-RT with breakup length concept Collision model: NTC, O'Rourke Coalescence model: Post Collision outcomes Drag-law: Dynamic model
Time step	Variable based on spray, evaporation, combustion processes
Turbulence-chemistry interactions model	Direct Integration of detailed chemistry well-mixed (no sub-grid model)
Time discretization scheme	PISO (Pressure Implicit with Splitting of Operators)

\* Senecal et al., SAE 2007-01-0159; Som ,PhD. Thesis 2009

# Primary Breakup Model: KH-ACT Model

Characteristic time scale due to cavitation is assumed to be the smaller of bubble collapse time and bubble burst time:

$$\tau_{CAV} = \min(\tau_{Collapse} : \tau_{Burst})$$

Effective radius of an equivalent bubble from the nozzle calculated as: ( $L_{CAV}$ )

$$R_{CAV} = r_{hole} \sqrt{(1 - C_a)}$$

Length and time scale for turbulence induced breakup :

$$K(t) = \left\{ \frac{(K_0)^{C_\varepsilon}}{K_0 (1 + C_\mu - C_\mu C_\varepsilon) + \varepsilon_0 t (C_\varepsilon - 1)} \right\}^{1/(1-C_\varepsilon)} \quad \varepsilon(t) = \varepsilon_0 \left\{ \frac{K(t)}{K_0} \right\}^{C_\varepsilon}$$

Obtained from KH model

$$\frac{L_A}{\tau_A} = \max \left\{ \frac{L_{KH}}{\tau_{KH}}; \frac{L_{CAV}}{\tau_{CAV}}; \frac{L_T(t)}{\tau_T(t)} \right\}$$

Due to breakup the radius of the parent droplet 'r' decreases continuously with time according to:

$$\frac{dr}{dt} = -C_{T,CAV} \frac{L_A}{\tau_A}$$

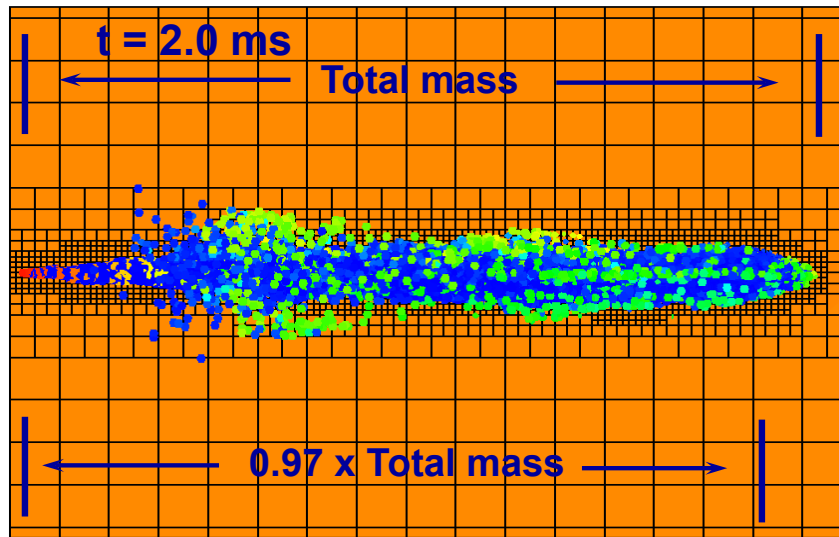
Model constant

Further information available\*:

- 1) S. Som, Ph.D thesis – University of Illinois at Chicago, 2009
- 2) S. Som, et al. *Combustion and Flame* (157), 2010
- 3) S. Som, et al. *Fuel* (90), 2011

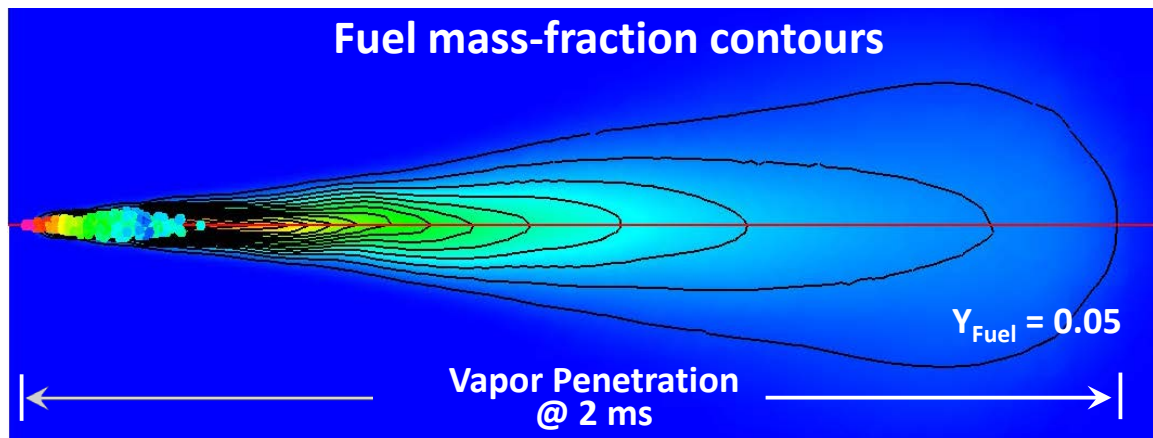
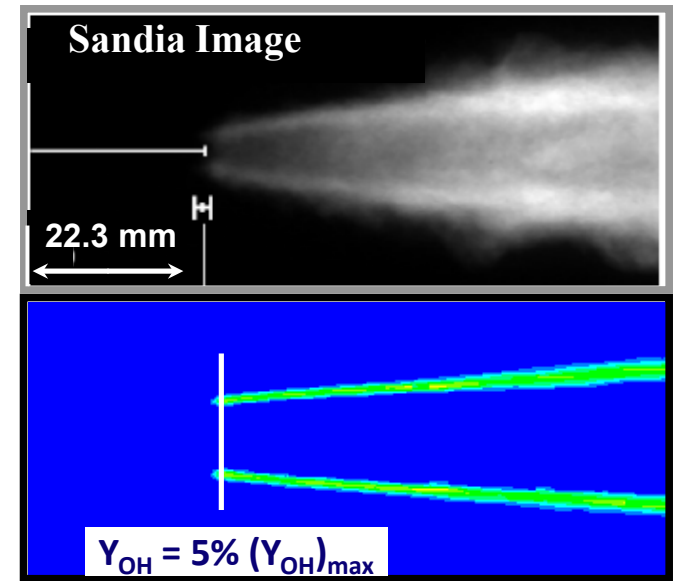
$K_0, \varepsilon_0, C_a$  are obtained from nozzle flow modeling

# 3D Simulations: Standard Definitions Used



**Spray penetration @ 2 ms**

## Lift-off length



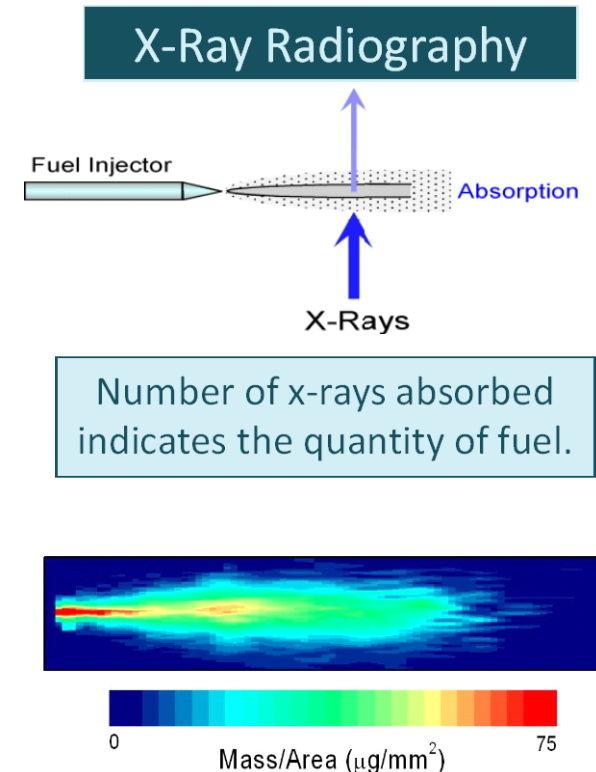
**Ignition delay:** Ignition is said to occur when  $T \geq 2000$  K in a particular cell. Usually, coincides with appearance of OH.

# Experimental Conditions: X-ray radiography data\*

Parameter	Quantity
Injection System	Caterpillar HEUI 315B
Number of Orifices	6
Orifice Diameter	169 $\mu\text{m}$ with $L/D = 4.412$
Oil Rail Pressure	Case 1: 17 MPa
Pressure Intensification ratio	6.6
Fill Gas	Nitrogen ( $\text{N}_2$ )
Chamber Density	34.13 $\text{kg}/\text{m}^3$
Fuel Density	865.4 $\text{kg}/\text{m}^3$
Fuel Temperature	40 $^{\circ}\text{C}$
Fuel Injection Quantity	100 [ $\text{mm}^3/\text{stroke}$ ]

Further information available\*:

- 1) A.I. Ramirez, S. Som, et al. *Experiments in Fluids* 47: 119-134, 2009.
- 2) A.I. Ramirez, S. Som, et al. *SAE Paper No. 2009-01-0846*, 2009.



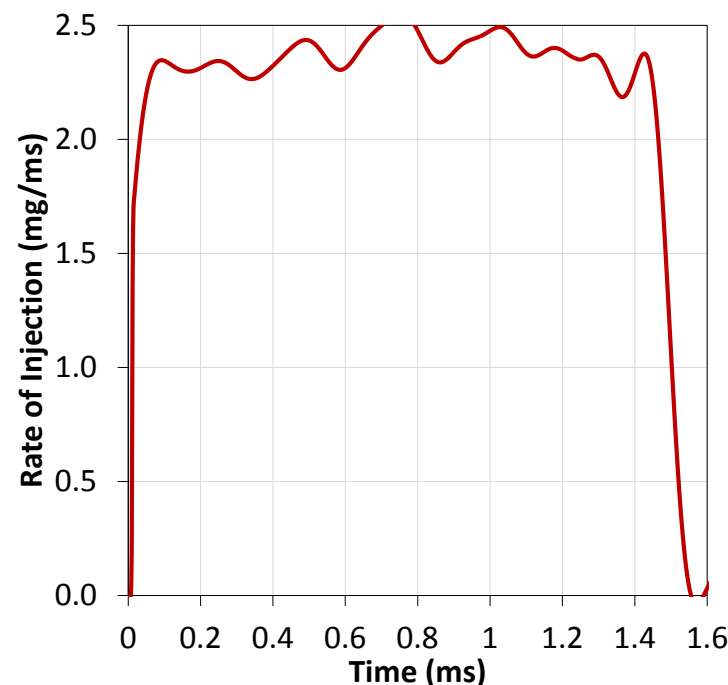
- ❑ Experiments performed under non-evaporating conditions at engine relevant densities
- ❑ Data available for : Spray penetration, cone-angle, fuel mass distribution near nozzle, normalized spray axial velocity, transverse integrated mass



# Experimental Conditions from ECN

Parameter	Quantity
Fuel	n-dodecane
Nozzle outlet diameter	90 $\mu\text{m}$
Nozzle K-factor	1.5
Nozzle shaping	Hydro-eroded
Discharge coefficient	0.86
Fuel injection pressure	150 MPa
Fuel temperature	363 K
Injection duration	1.5 ms
Injected fuel mass	3.5 mg
Injection rate shape	Square
Ambient temperature	800 - 1200 K
Ambient gas density	22.8 Kg/m <sup>3</sup>
Ambient O <sub>2</sub> Concentration	15 %

- ❑ Experiments performed under both evaporating and combusting conditions.
- ❑ Data available for : Spray penetration, liquid length, vapor penetration, mixture fraction, ignition delay, flame lift-off length, soot distribution , high-speed movies



<http://www.sandia.gov/ecn/>

Back-up